



제 CG-2014-0853 호

안 전 인 증 서

(사업장명) KITO CORP.

(소재지) 2000, TSUI IARAI SHOWA-CHO, NAKAKOMAGUN YAMANASHI,
409-3853 JAPAN

위 사업장에서 제조하는 아래의 품목이 「산업안전보건법」 제34조 및 같은 법 시행규칙 제58조의4제4항에 따른 안전인증 심사결과 안전·보건기준에 적합하므로 안전인증표시의 사용을 인증합니다.

_____	품 목 : 호이스트	_____
_____	형식(용량): KDL-ER2-010(1 ton)	_____
_____	인증번호 : 14-CG2AC-0853	_____
_____	인증기준 : 위험기계·기구 의무안전인증기준 (고용노동부고시 제2012-33호)	_____
_____	인증조건 : 산업안전보건법 "제34조 준수"	_____

2014년 05월 16일

한국승강기안전기술원 이사장



【별지 제4호서식】

동 일 형 식 일 람 표

사업장명	KITO CORP.		개정일자 및 번호	2014.04.14	인증번호	
형식 및 모델번호			동일형식 항목 및 내역			비고
형식번호	모델번호	동일형식 항목1	동일형식 항목2	동일형식 항목3		
KDL-ER2-010	KITO-ER2D010L-S	Lift max 30m 권상모타 0.9kW .L : 4.2m/min .IL: 4.2/0.7m/min Inverter control	횡행모터 0.4kW .S : 24m/min .L: 12m/min .IS:24/4m/min .IL:12/2m/min	전기Trolley 결합 type		
	KITO-ER2D010L-L					
	KITO-ER2D010L-IS					
	KITO-ER2D010L-IL					
	KITO-ER2D010IL-S					
	KITO-ER2D010IL-L					
	KITO-ER2D010IL-IS					
	KITO-ER2D010IL-IL					
	KITO-C-ER2D010L-S					
	KITO-C-ER2D010L-L					
	KITO-C-ER2D010L-IS					
	KITO-C-ER2D010L-IL					
	KITO-C-ER2D010IL-S					
	KITO-C-ER2D010IL-L					
	KITO-C-ER2D010IL-IS					
	KITO-C-ER2D010IL-IL					
				전기Trolley 결합 Clean type		



14/16

제 2012-BJ-0009 호



안 전 인 증 서

정호엔지니어링

경기도 광명시 노온사동 440-5

위 사업장에서 제조하는 아래의 품목이 산업안전보건법 제34조 및 같은 법 시행규칙 제58조의4제4항에 따른 안전인증 심사 결과 안전·보건기준에 적합하므로 안전인증표시의 사용을 인증합니다.

품 목

양중기용 과부하방지장치

형식·모델/용량·등급/인증번호

형식·모델	용량·등급	인증번호
JDL-100	J-2	12-AV2BJ-0009

인 증 기 준

방호장치 의무안전인증 고시(고용노동부고시 제2010-36호)

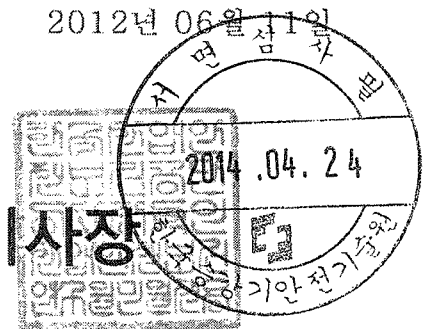
인 증 조 건

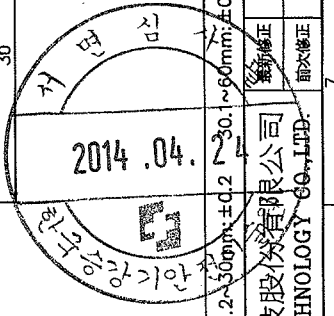
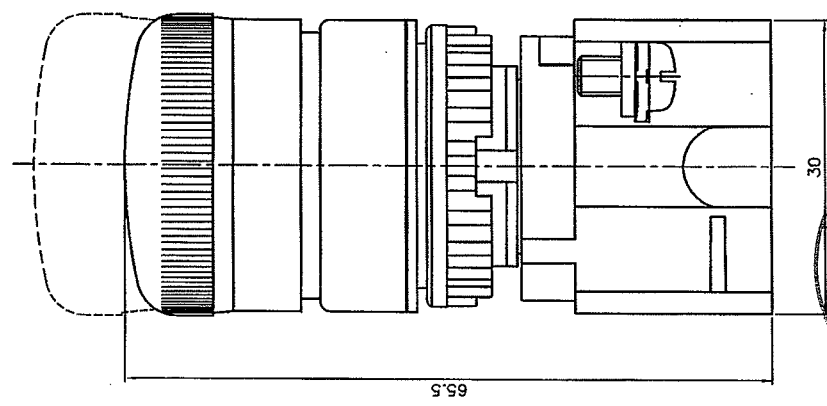
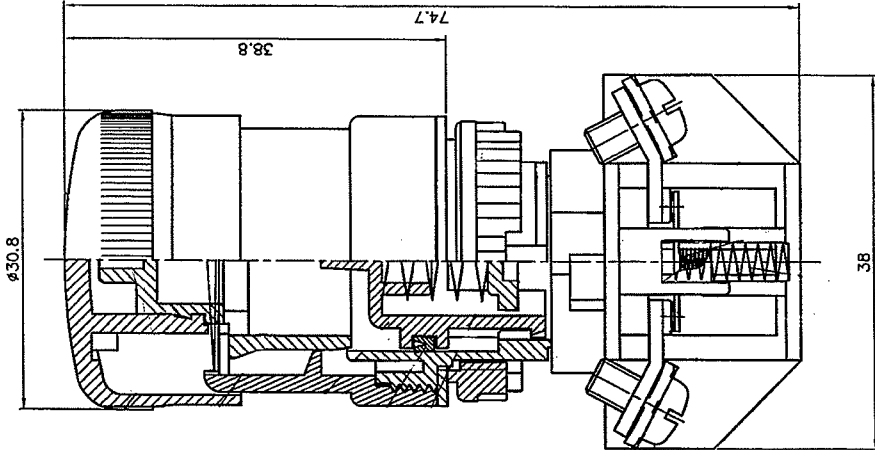
아래 주소에서 생산되는 제품에 한함.

정호엔지니어링, 경기도 광명시 노온사동 440-5

2012년 06월 11일

한국산업안전보건공단 이사장

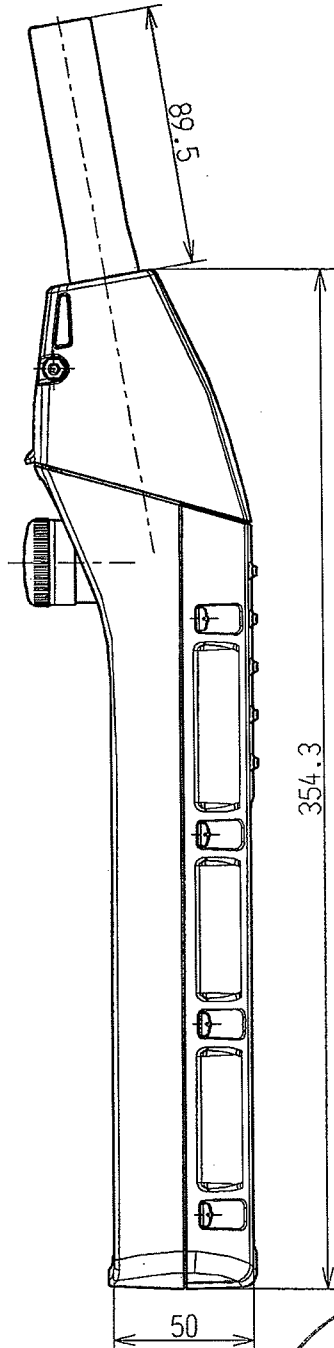
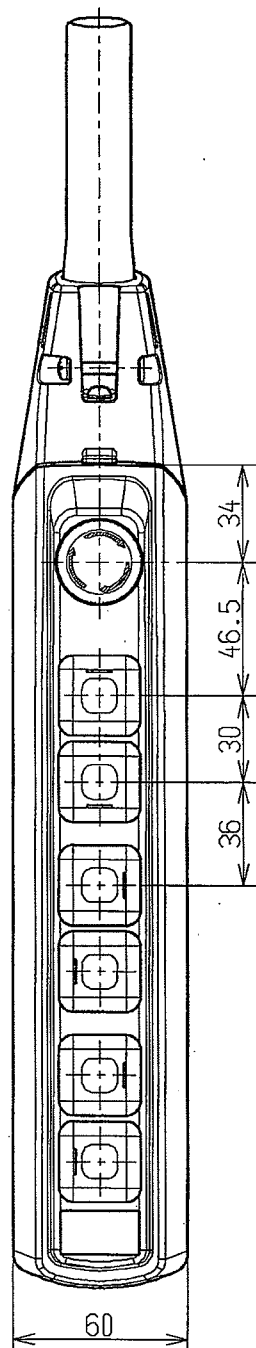




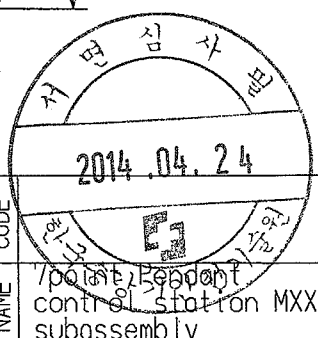
一般公差	0.2~0.50mm: ±0.2	30.~60mm: ±0.3	60.1~300mm: ±0.5	模具材質	研發部 95.05.24 韓健廷	模具有效	研發部 95.05.24 周家祥	模具孔數	設計部 95.05.24 吳宗達	單位	mm	材質	圖號	T2-BKH
品保	品保部 95.05.24 林建宏	核准	研發部 95.05.24 韓健廷	核對	研發部 95.05.24 周家祥	繪圖	設計部 95.05.24 吳宗達	比例	2:1	表面處理	顏色	品名	T2 BKH 連鎖開關	
品保	品保部 95.05.24 林建宏	核准	研發部 95.05.24 韓健廷	核對	研發部 95.05.24 周家祥	繪圖	設計部 95.05.24 吳宗達	投影法	第一角	顏色				

2/12/BKH/12-BKH.dwg

Revision	Incidence	Description	Date	Charge	Approved



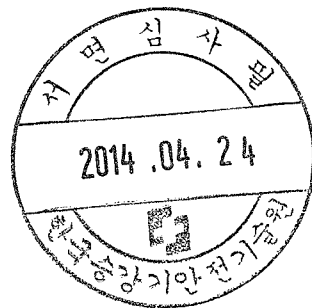
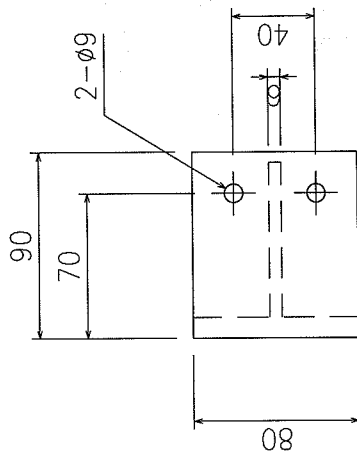
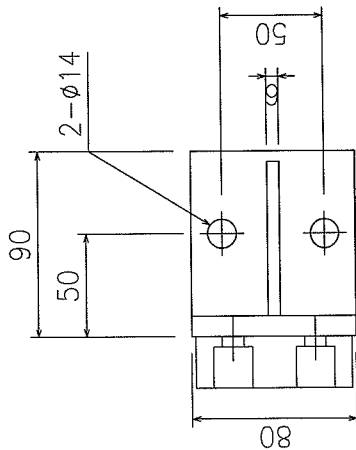
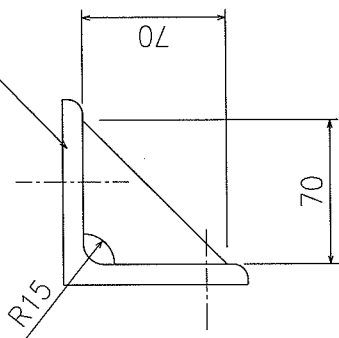
E
W
S
N



⑥
⑤
④
③
②
①

NOTE							DWG. NO.	NGS./UNIT MATERIAL	NAME	CODE
APPROVED	ISHIKAWA	CHECKED	FURIYA	DESIGNED	KOBAYASHI	DRAWN	KOBAYASHI	SCALE	-	SWD2XXXAA1
Date issued	08.02.08		08.02.08		08.02.08		08.02.08			

L - 90x10t

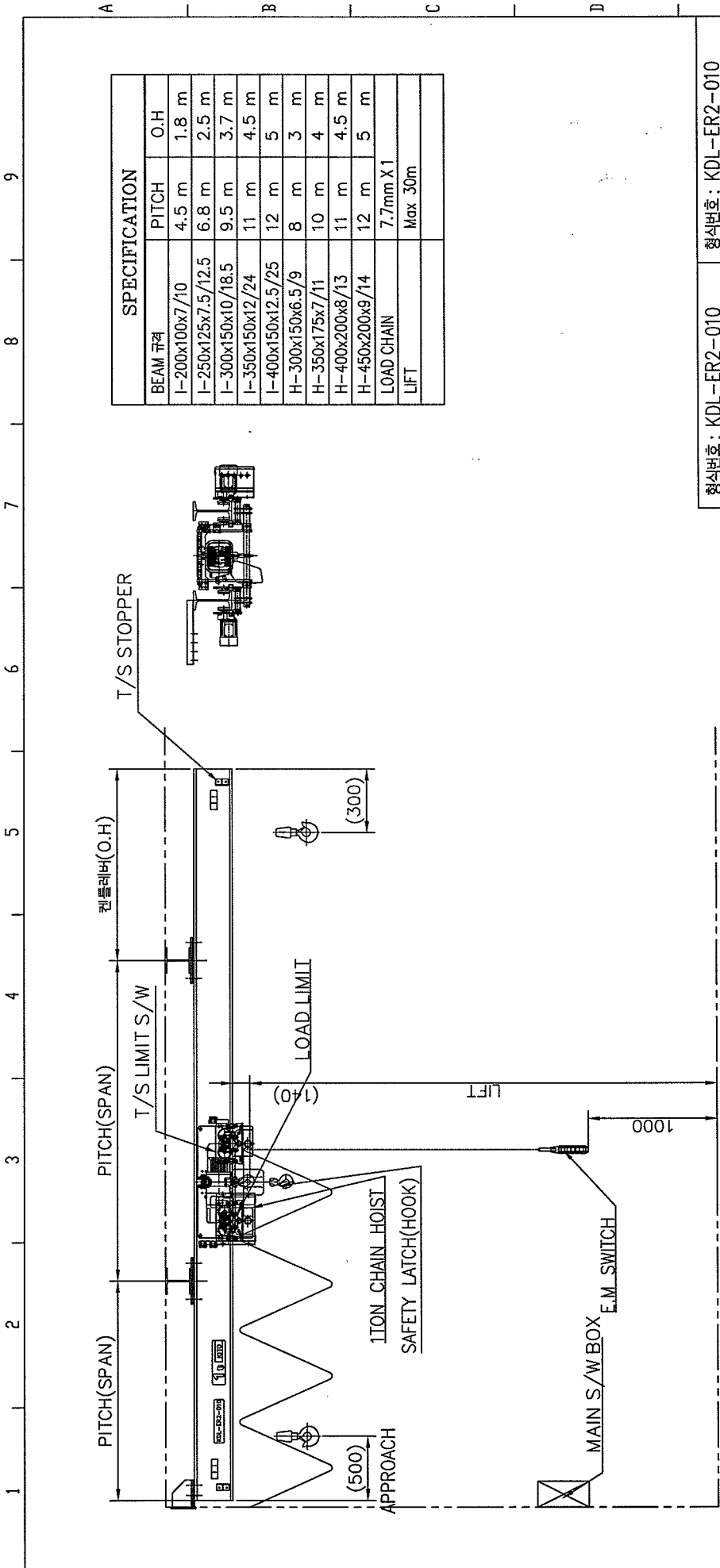


No.	Part Name	Description	Mat'l	Unit	Q'ty	Weight (KG)	Remark
1	STOPPER	STOPPER - traversing	SS400		4		차량용기
TITLE							
Part No.			STOPPER				
DWG No.			SCALE				
REV.			REV.				

APPROVED	CHECKED	DESIGNED	DRAWN
J. S. CHO	J. S. CHO	W.H.EUN	W.H.EUN

REV.	DATE	APPROVED	DRAWN

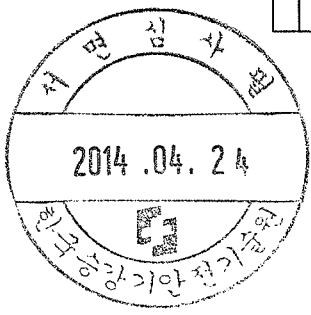
CONTENTS	



SPECIFICATION		
BEAM 桁架	PITCH	O.H
I-200x100x7/10	4.5 m	1.8 m
I-250x125x7.5/12.5	6.8 m	2.5 m
I-300x150x10/18.5	9.5 m	3.7 m
I-350x150x12/24	11 m	4.5 m
I-400x150x12.5/25	12 m	5 m
H-300x150x6.5/9	8 m	3 m
H-350x175x7/11	10 m	4 m
H-400x200x8/13	11 m	4.5 m
H-450x200x9/14	12 m	5 m
LOAD CHAIN	7.7mm X 1	
LIFT	Max 30m	

NAME PLATE

정격 하중	1 ton
전기 용량	V
제조자	
제조년월	20 . .
안전인증 표시	표s
형식 번호	KDL-ER2-010
제조번호	

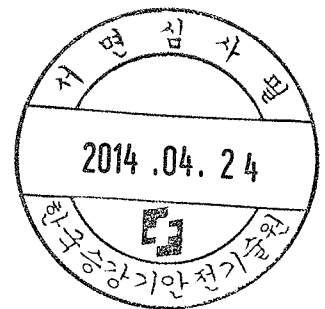


형식번호: KDL-ER2-010	형식번호: KDL-ER2-010
Model number. KITO-C-ER2D010L-S KITO-C-ER2D010L-L KITO-C-ER2D010L-IS KITO-C-ER2D010L-IL KITO-C-ER2D010L-S KITO-C-ER2D010L-L KITO-C-ER2D010L-IS KITO-C-ER2D010L-IL	Model number. KITO-ER2D010L-S KITO-ER2D010L-L KITO-ER2D010L-IS KITO-ER2D010L-IL KITO-ER2D010L-S KITO-ER2D010L-L KITO-ER2D010L-IS KITO-ER2D010L-IL

사명 KTO CORP	대표이사 EDUNWON HEE	설계 EDUNWON HEE	검核 J.S. CHO	承認 J.S. CHO	年.月.日 DATE	年.月.日 DATE	REV.	REV.	
1t MONO RAIL HOIST GENERAL ASSEMBLY-1		R 尺度 SCALE		R 尺度 SCALE		R 尺度 SCALE		R 尺度 SCALE	
MONO RAIL 1		MONO RAIL 1		MONO RAIL 1		MONO RAIL 1		MONO RAIL 1	

4. 전 기 도 면

- 1) ELECTRICAL SPECIFICATION
- 2) SYMBOL LIST
- 3) 배선배관도 & 접지계통도
- 4) 전기회로도
- 5) PANEL 관련도



LOAD SUMMARY 1 (ER2-010IL-IL/IS)

*POWER SOURCE : AC 3Φ 220(208)V

OBJECT	HOISTING	TRAVERSING	CONTROL CIRCUIT
MOTOR OUTPUT	0.9KW x 4P	0.4KW x 4P x 2SET	
FULL LOAD CURRENT	5.7 (A)	6 (A)	0.5 (A)

*크레인 하중상태를 HOIST의 정격 LOAD의 100(%)를 사용했을때를 기준으로 작성하였음.

*** NOMAL 전류값 ***

권상과 횡행시 : HOISTING + TRAVERSING + CONTROL CIRCUIT = 12.2 A

*** PEAK 전류값 ***

K= NAMAL 전류치가 50A미만일때 1.25, 50A이상일때 1.1적용

NOMAL 전류값 * K = 12.2 * 1.25 = 15.2 A

*POWER SOURCE : AC 3Φ 380(440)V

OBJECT	HOISTING	TRAVERSING	CONTROL CIRCUIT
MOTOR OUTPUT	0.9KW x 4P	0.4KW x 4P x 2SET	
FULL LOAD CURRENT	3.6 (A)	5 (A)	0.5 (A)

*크레인 하중상태를 HOIST의 정격 LOAD의 100(%)를 사용했을때를 기준으로 작성하였음.

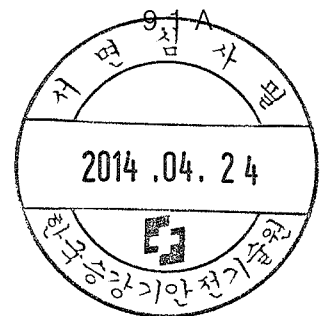
*** NOMAL 전류값 ***

권상과 횡행시 : HOISTING + TRAVERSING + CONTROL CIRCUIT = 9.1 A

*** PEAK 전류값 ***

K= NAMAL 전류치가 50A미만일때 1.25, 50A이상일때 1.1적용

NOMAL 전류값 * K = 9.1 * 1.25 = 11.3 A



LOAD SUMMARY 2 (ER2-010IL-L/S)

*POWER SOURCE : AC 3Φ 220(208)V

OBJECT	HOISTING	TRAVERSING	CONTROL CIRCUIT
MOTOR OUTPUT	0.9KW x 4P	0.4KW x 4P x 2SET	
FULL LOAD CURRENT	5.7 (A)	6 (A)	0.5 (A)

*크레인 하중상태를 HOIST의 정격 LOAD의 100(%)를 사용했을때를 기준으로 작성하였음.

*** NOMAL 전류값 ***

권상과 횡행시 : HOISTING + TRAVERSING + CONTROL CIRCUIT = 12.2 A

*** PEAK 전류값 ***

K= NAMAL 전류치가 50A미만일때 1.25, 50A이상일때 1.1적용

NOMAL 전류값 * K = 12.2 * 1.25 = 15.2 A

*POWER SOURCE : AC 3Φ 380(440)V

OBJECT	HOISTING	TRAVERSING	CONTROL CIRCUIT
MOTOR OUTPUT	0.9KW x 4P	0.4KW x 4P x 2SET	
FULL LOAD CURRENT	3.6 (A)	4.4 (A)	0.5 (A)

*크레인 하중상태를 HOIST의 정격 LOAD의 100(%)를 사용했을때를 기준으로 작성하였음.

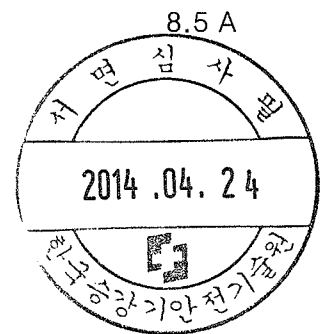
*** NOMAL 전류값 ***

권상과 횡행시 : HOISTING + TRAVERSING + CONTROL CIRCUIT = 8.5 A

*** PEAK 전류값 ***

K= NAMAL 전류치가 50A미만일때 1.25, 50A이상일때 1.1적용

NOMAL 전류값 * K = 8.5 * 1.25 = 10.6 A



LOAD SUMMARY 3 (ER2-010L-IL/IS)

*POWER SOURCE : AC 3Φ 220(208)V

OBJECT	HOISTING	TRAVERSING	CONTROL CIRCUIT
MOTOR OUTPUT	0.9KW x 4P	0.4KW x 4P x 2SET	
FULL LOAD CURRENT	4.7 (A)	6 (A)	0.5 (A)

*크레인 하중상태를 HOIST의 정격 LOAD의 100(%)를 사용했을때를 기준으로 작성하였음.

*** NOMAL 전류값 ***

권상과 횡행시 : HOISTING + TRAVERSING + CONTROL CIRCUIT = 11.2 A

*** PEAK 전류값 ***

K= NAMAL 전류치가 50A미만일때 1.25, 50A이상일때 1.1적용

NOMAL 전류값 * K = 11.2 * 1.25 = 14 A

*POWER SOURCE : AC 3Φ 380(440)V

OBJECT	HOISTING	TRAVERSING	CONTROL CIRCUIT
MOTOR OUTPUT	0.9KW x 4P	0.4KW x 4P x 2SET	
FULL LOAD CURRENT	2.6 (A)	5 (A)	0.5 (A)

*크레인 하중상태를 HOIST의 정격 LOAD의 100(%)를 사용했을때를 기준으로 작성하였음.

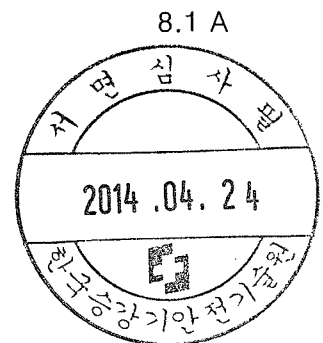
*** NOMAL 전류값 ***

권상과 횡행시 : HOISTING + TRAVERSING + CONTROL CIRCUIT = 8.1 A

*** PEAK 전류값 ***

K= NAMAL 전류치가 50A미만일때 1.25, 50A이상일때 1.1적용

NOMAL 전류값 * K = 8.1 * 1.25 = 10.1 A



LOAD SUMMARY 4 (ER2-010L-L/S)

*POWER SOURCE : AC 3Φ 220(208)V

OBJECT	HOISTING	TRAVERSING	CONTROL CIRCUIT
MOTOR OUTPUT	0.9KW x 4P	0.4KW x 4P x 2SET	
FULL LOAD CURRENT	4.7 (A)	6 (A)	0.5 (A)

*크레인 하중상태를 HOIST의 정격 LOAD의 100(%)를 사용했을때를 기준으로 작성하였음.

*** NOMAL 전류값 ***

권상과 횡행시 : HOISTING + TRAVERSING + CONTROL CIRCUIT = 11.2 A

*** PEAK 전류값 ***

K= NAMAL 전류치가 50A미만일때 1.25, 50A이상일때 1.1적용

NOMAL 전류값 * K = 11.2 * 1.25 = 14 A

*POWER SOURCE : AC 3Φ 380(440)V

OBJECT	HOISTING	TRAVERSING	CONTROL CIRCUIT
MOTOR OUTPUT	0.9KW x 4P	0.4KW x 4P x 2SET	
FULL LOAD CURRENT	2.6 (A)	4.4 (A)	0.5 (A)

*크레인 하중상태를 HOIST의 정격 LOAD의 100(%)를 사용했을때를 기준으로 작성하였음.

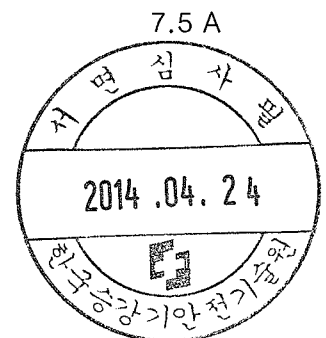
*** NOMAL 전류값 ***

권상과 횡행시 : HOISTING + TRAVERSING + CONTROL CIRCUIT = 7.5 A




*** PEAK 전류값 ***

K= NAMAL 전류치가 50A미만일때 1.25, 50A이상일때 1.1적용





NOMAL 전류값 * K = 7.5 * 1.25 = 9.3 A






ROTATING MACHINE

-  SYNCHRONOUS GENERATOR, 3-PHASE
-  AC INDUCTION MOTOR, 3-PHASE
- * N : NORMAL DUTY
- S : STAND-BY
-  DC MOTOR






LIGHTNING ARRESTERS

-  LA : LIGHTNING ARRESTER
-  SA : SURGE ARRESTER
-  SS : SURGE SUPPRESSOR
-  DISCHARGE COUNTER















INSTRUMENT TRANSFORMERS










-  CURRENT TRANSFORMER
-  ZERO PHASE CURRENT TRANSFORMER
-  POTENTIAL TRANSFORMER

CIRCUIT BREAKERS






-  POWER CIRCUIT BREAKER, FIXED TYPE
-  GCB : SF6 GAS CIRCUIT BREAKER
-  VCB : VACUUM CIRCUIT BREAKER
-  ACB : AIR CIRCUIT BREAKER
-  POWER CIRCUIT BREAKER, DRAWOUT TYPE

SWITCHES























-  DISCONNECTOR SWITCH, SINGLE THROW MANUALLY OPERATED
-  LOAD BREAK SWITCH, SINGLE THROW MANUALLY OPERATED
-  EARTHING SWITCH, SINGLE THROW MANUALLY OPERATED
-  DISCONNECTOR SWITCH, SINGLE THROW MOTOR OPERATED
-  EARTHING SWITCH, SINGLE THROW MOTOR OPERATED
-  VACUUM CIRCUIT SWITCH
-  FUSED DISCONNECTOR SWITCH
-  FUSE-SWITCH
-  LIMIT SWITCH (MAKE CONTACT)
-  LIMIT SWITCH (BREAK CONTACT)
-  PUSH BUTTON, NORMALLY OPEN MOMENTARY CONTACT
-  PUSH BUTTON, NORMALLY CLOSED MOMENTARY CONTACT
-  PUSH BUTTON, NORMALLY OPEN PUSH TO LOCK, RELEASED BY KEY
-  MANUAL SELECTOR SWITCH (LOCKED)

-  CIRCUIT BREAKER, FIXED TYPE
-  MCCB : MOLDED CASE CIRCUIT BREAKER
-  MCB : MINIATURE CIRCUIT BREAKER
-  CIRCUIT BREAKER, DRAWOUT TYPE
-  WITHDRAWABLE INTERCONNECTOR
-  CIRCUIT BREAKER, MANUALLY OPERATED FIXED TYPE WITH THERMAL & MAGNETIC TRIP
-  CIRCUIT BREAKER, MANUALLY OPERATED FIXED TYPE WITH MAGNETIC TRIP ONLY
-  CIRCUIT BREAKER, MANUALLY OPERATED DRAWOUT TYPE WITH THERMAL & MAGNETIC TRIP
-  CIRCUIT BREAKER, MANUALLY OPERATED FIXED TYPE WITH THERMAL & MAGNETIC TRIP AND RESIDUAL CURRENT RELEASE

CONTACTORS AND STARTERS

-  AUX. CONTACT, NORMALLY OPEN WHEN MAIN SWITCHING DEVICE IS DE-ENERGIZED
-  AUX. CONTACT, NORMALLY CLOSED WHEN MAIN SWITCHING DEVICE IS DE-ENERGIZED
-  MAGNETIC CONTACTOR, ELECTRICALLY OPERATED
-  COMBINATION STARTER, FULL VOLTAGE, NON-REVERSING, DRAWOUT TYPE, WITH ELECTRICALLY OPERATED CONTACTORS, WITH MAGNETIC MOTOR CIRCUIT BREAKER, BUILT IN ELECTRONIC OVER-CURRENT RELAY WITH ADJUSTABLE TRIP RATING
-  COMBINATION STARTER, FULL VOLTAGE, NON-REVERSING, FIXED TYPE, WITH ELECTRICALLY OPERATED CONTACTORS, WITH MAGNETIC MOTOR CIRCUIT BREAKER, BUILT IN THERMAL OVER-CURRENT RELAY WITH ADJUSTABLE TRIP RATING

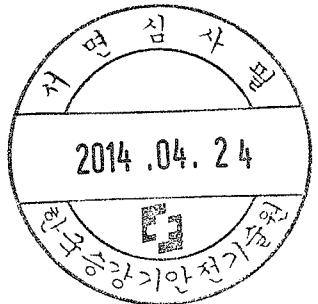
GRAPHIC SYMBOLS

-  GENERAL OPERATING COIL
-  CAPACITOR
-  CAPACITOR VOLTAGE TRANSFORMER (CVT)
-  RESISTOR
-  DIODE
-  CONTROLLED RECTIFIER
-  DC-DC CONVERTER
-  RECTIFIER, BATTERY CHARGER
-  DC-AC INVERTER
-  BATTERY BANK
-  ELECTRIC HEATER, INDICATE 1A OR 3A AND kW RATING, UNLESS OTHERWISE SPECIFIED, TO BE REGARDED AS 1A.
-  EARTHING CONNECTION
-  DISCONNECTION LINK
-  CROSSING OF CONDUCTORS NOT CONNECTED
-  JUNCTION OF CONDUCTORS OR WIRES
-  BUS DUCT
-  SPB : SEGREGATED PHASE BUS DUCT
-  IPB : ISOLATED PHASE BUS DUCT
-  CABLE HEAD AND CABLE CONNECTION
-  AMMETER SWITCH
-  VOLTMETER SWITCH
-  SIGNAL LAMP

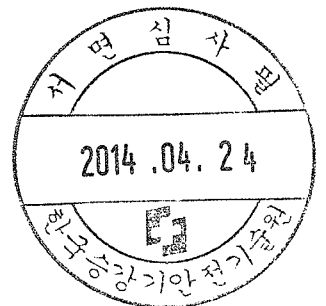
- * R = RED
- G = GREEN
- W = WHITE
- C = CYAN
- Y = YELLOW
- B = BLUE
- A = AMBER

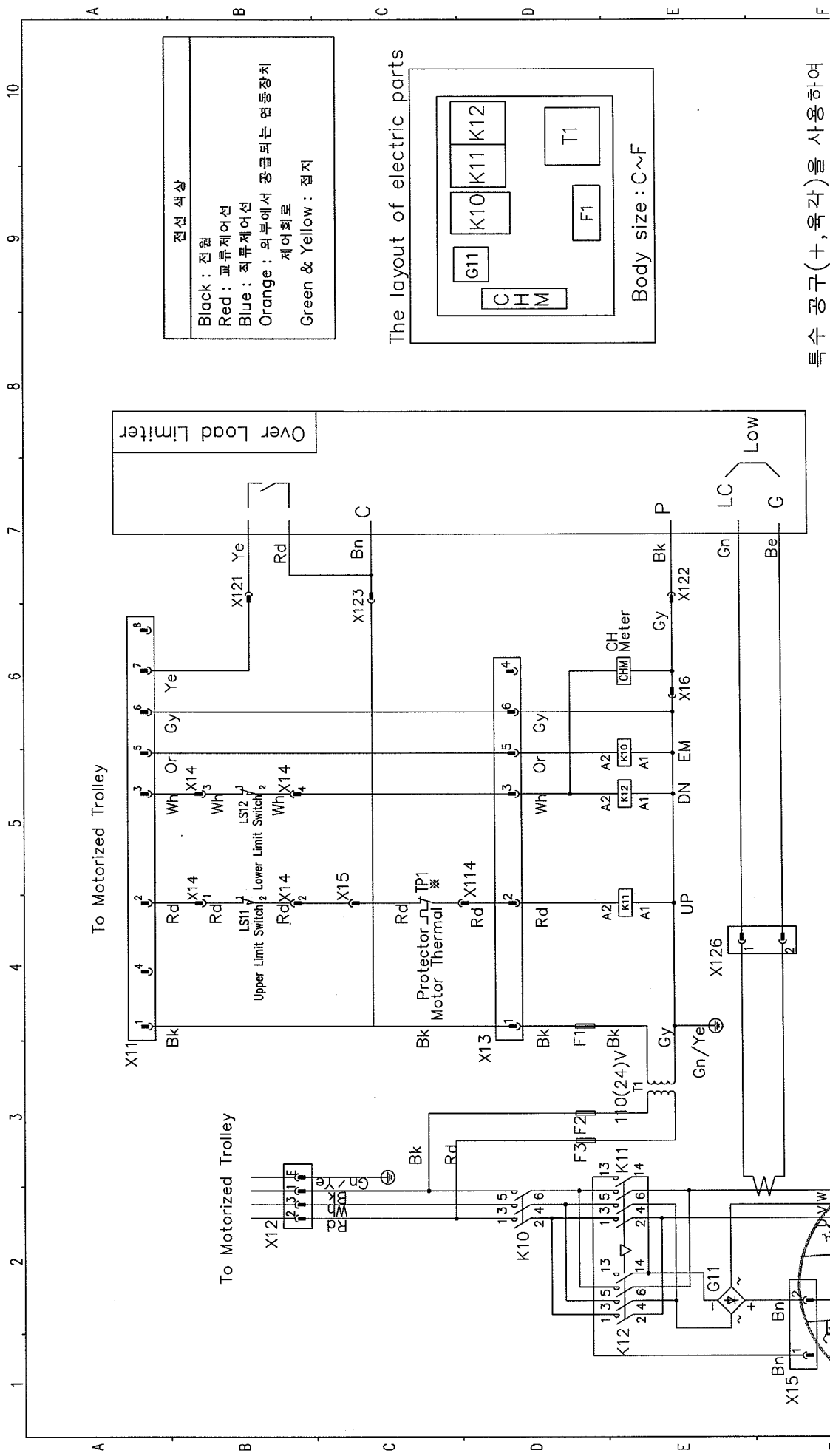
SYMBOL LIST

APPROVED	CHECKED	DESIGNED	CODE	SCALE
KOTO CORP.				
			SYMBOL LIST	DATE

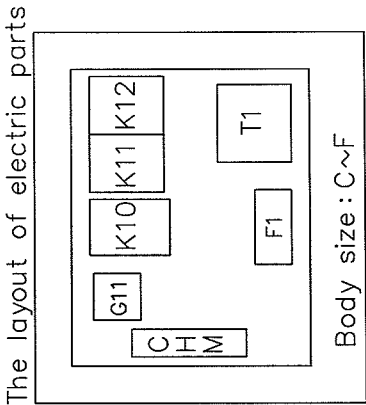


. 1속형 hoisting/. 1속형 traversing





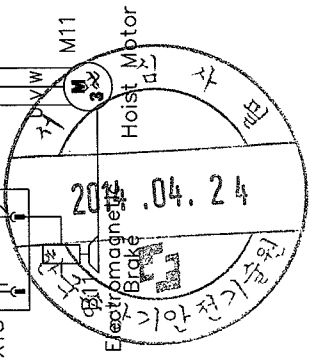
전선 색상
 Black : 전원
 Red : 교류제어선
 Blue : 직류제어선
 Orange : 외부에서 공급되는 연동장치 제어회로
 Green & Yellow : 접지



특수 공구 (+, 육각)을 사용하여
 개방구 쪽은 제외
 For MR connection

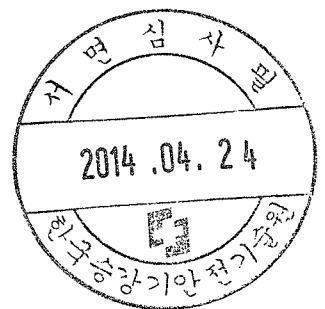
APPROVED		CHECKED	DESIGNED
H.Furiya 09 / 04 / 21		T.Hatano 09 / 04 / 21	K.Miwa 09 / 04 / 21
KATO CORP.			
CODE	SCALE	DATE	
DWG.NO: SEWG3100L01_MR	—	—	

*TP1 is attached only to 380V and 440V.

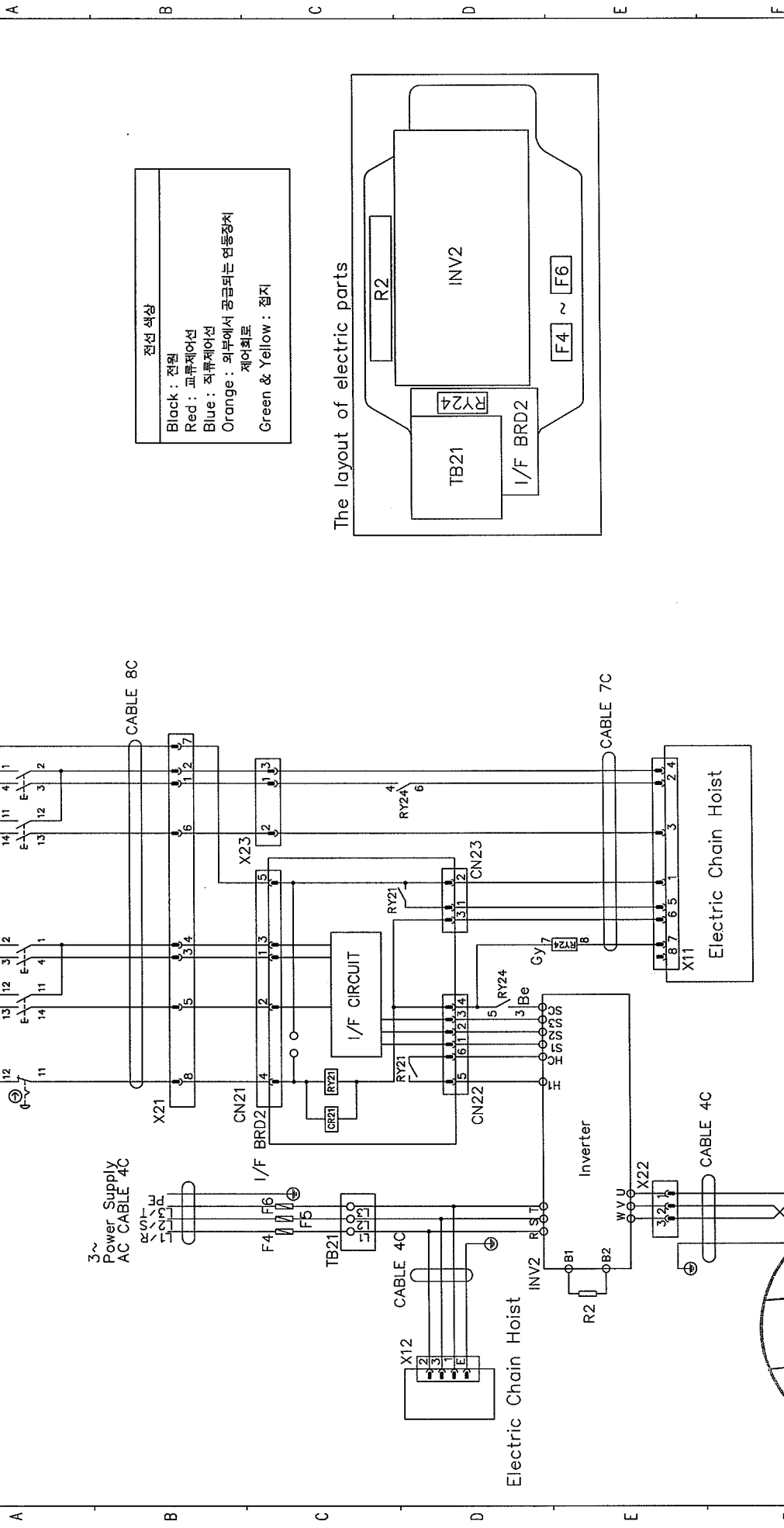


UNIT : mm

. 1속형 hoisting/. 2속형 traversing



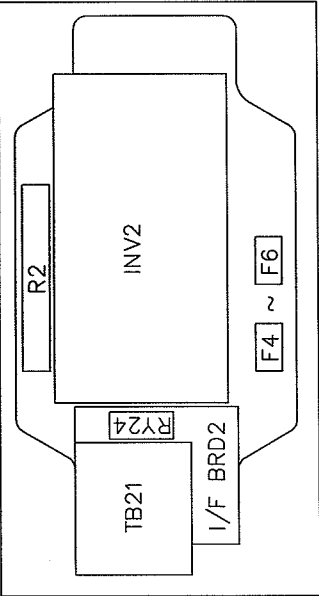
10 9 8 7 6 5 4 3 2 1



전선 색상

Black	: 전원
Red	: 교류제어선
Blue	: 직류제어선
Orange	: 외부에서 공급되는 역동장치 제어회로
Green & Yellow	: 접지

The layout of electric parts



특수 공구(+, 육각)을 사용하여
개방구 쪽은 제외

Traversing 2speed
Wiring Diagram

APPROVED	CHECKED	DESIGNED
H.Furiya	H.Hatano	K.Miwa
09 / 04 / 21	09 / 04 / 21	09 / 04 / 21

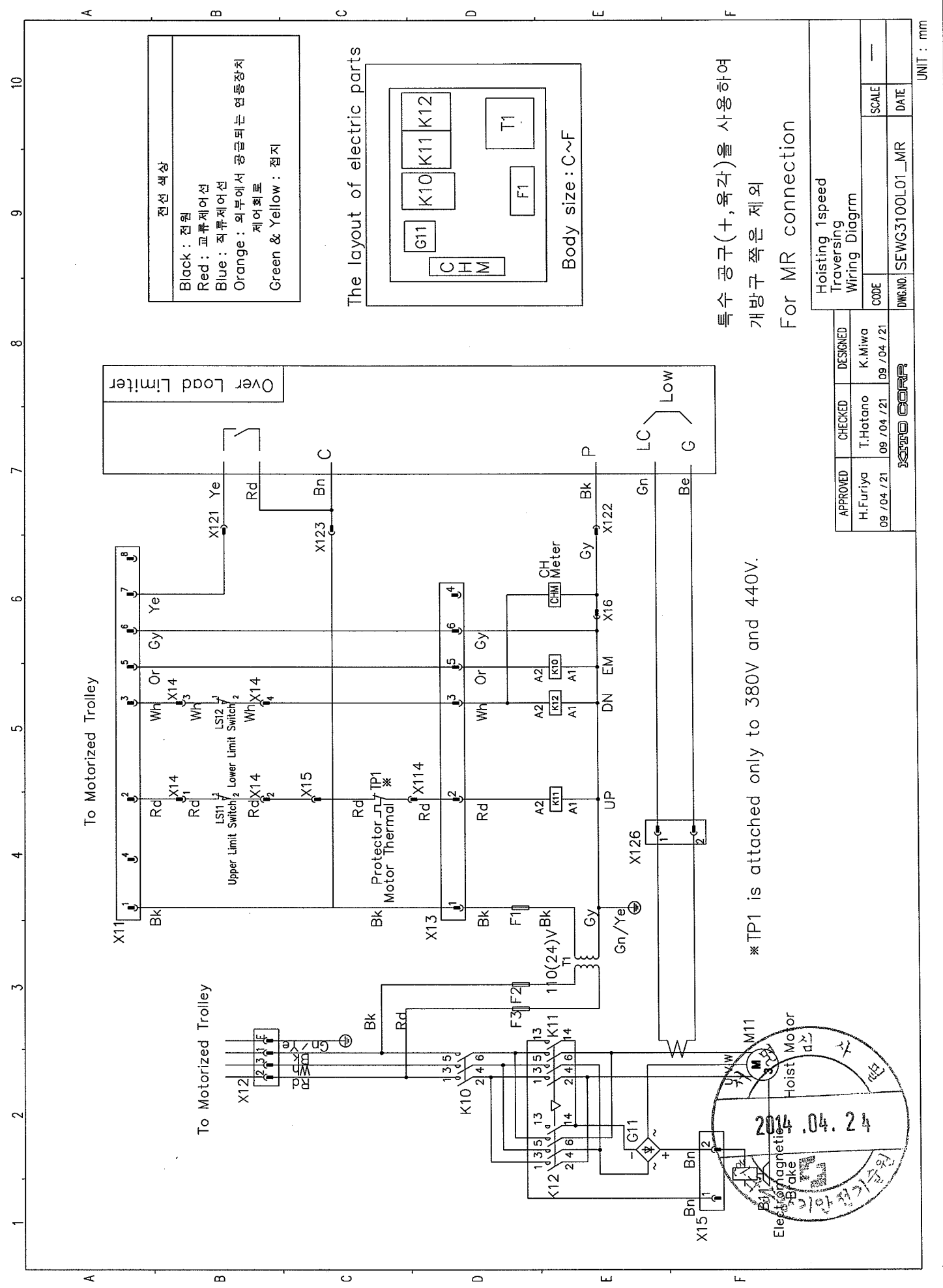
CODE	SCALE
DWG.NO. SEWG3DD0L01	DATE

UNIT : mm

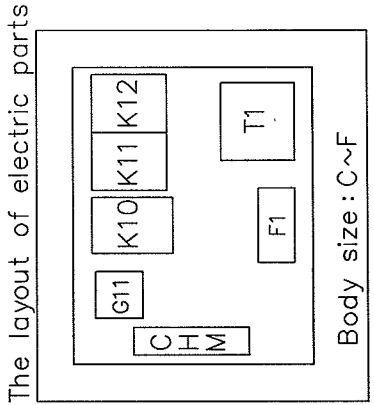
0.4KWx2SET
M21
Traversing

2014 .04. 24

Motor Assy



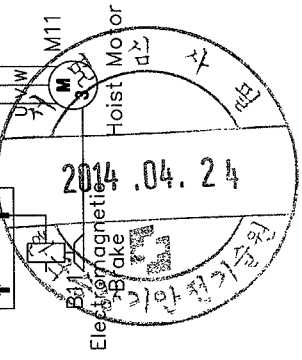
전선 색상
 Black : 전원
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 Green & Yellow : 접지



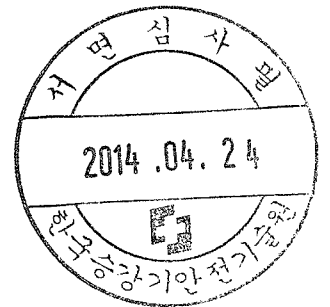
특수 공구(+, 육각)을 사용하여
 개방구 쪽은 제외
 For MR connection

APPROVED		CHECKED	DESIGNED
H.Furiya		T.Hatano	K.Miwa
09/04/21		09/04/21	09/04/21
KOTO CORP			
CODE	SCALE	DATE	
DWG.NO. SEWG3100L01_MR			

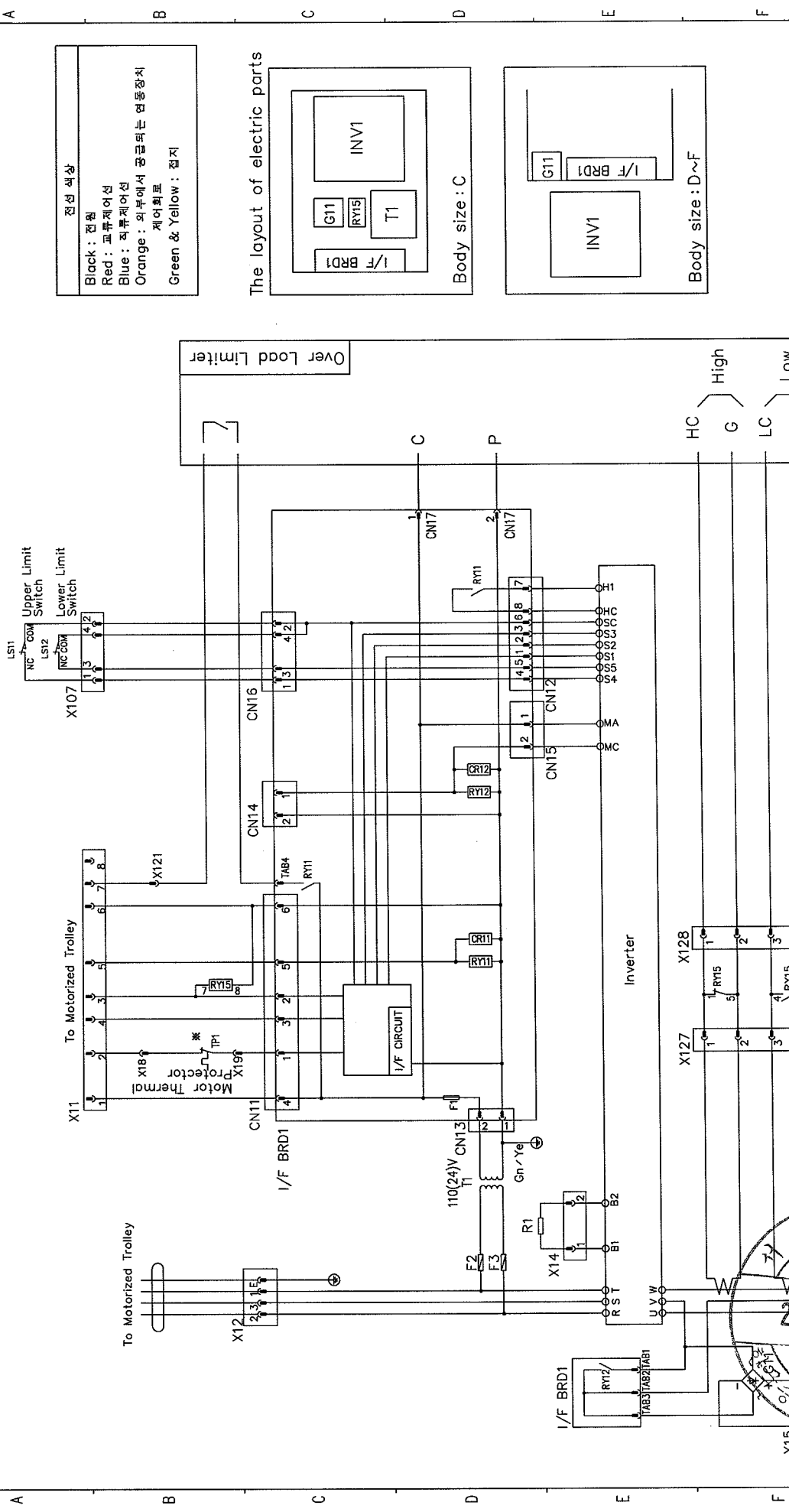
*TP1 is attached only to 380V and 440V.



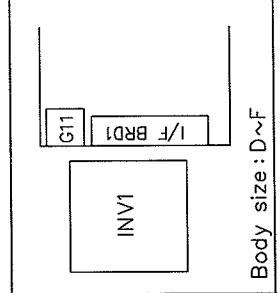
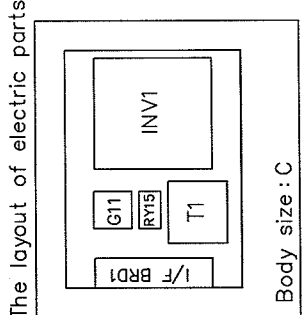
. 2속형 hoisting/. 1속형 traversing



10 9 8 7 6 5 4 3 2 1



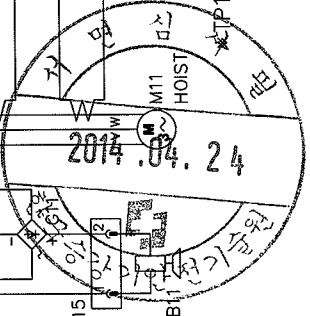
전선 색상
 Black : 전원
 Red : 교류제어선
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 Orange : 외부에서 공급되는 역동장치 제어회로
 Green & Yellow : 접지



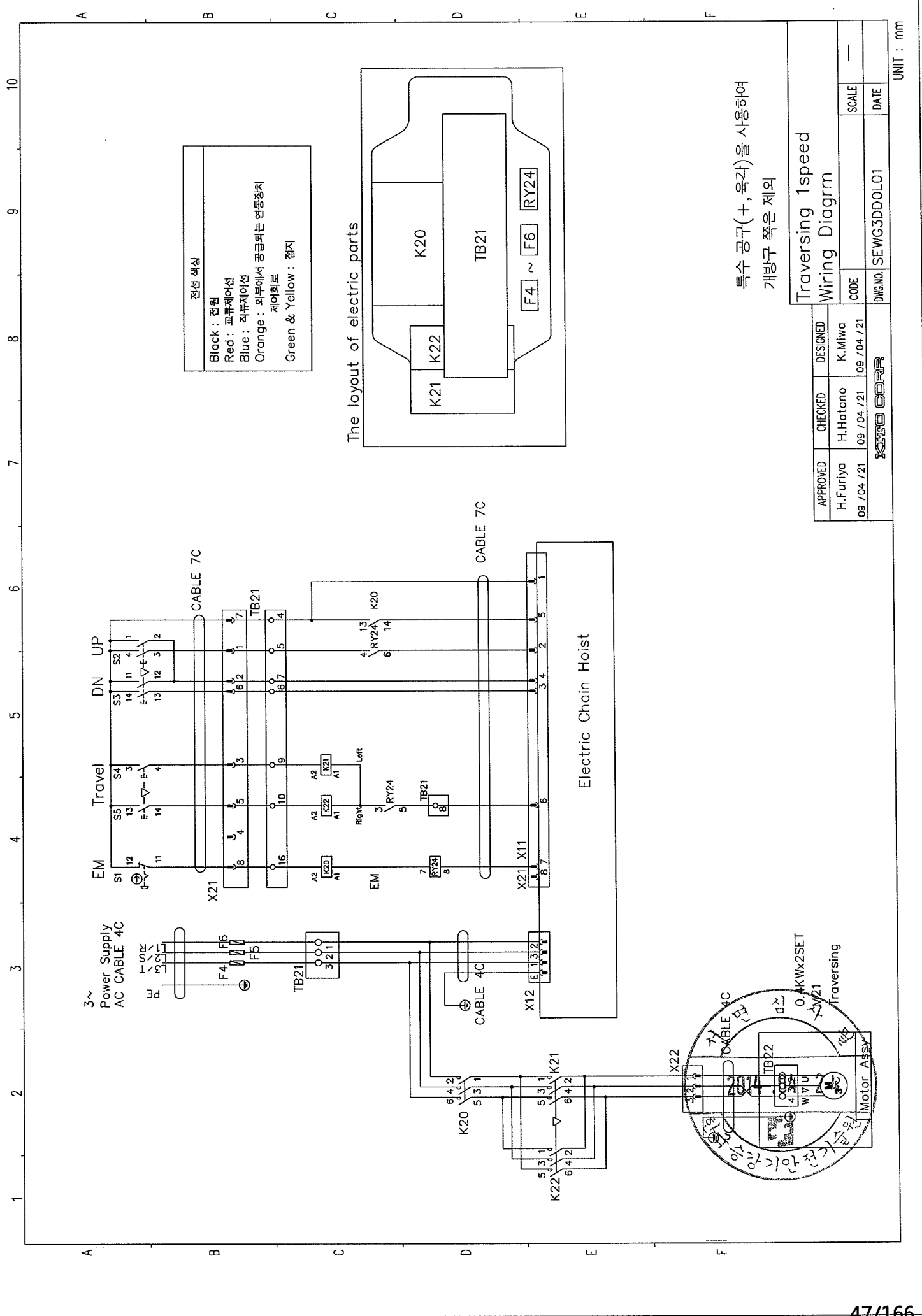
Hoisting 2speed Traversing Wiring Diagram

APPROVED	CHECKED	DESIGNED
H.Furiya 09 / 04 / 21	H.Hatano 09 / 04 / 21	K.Miwa 09 / 04 / 21
CODE	SCALE	DATE
DWG.NO. SEWG3D00L01_MR	---	---

UNIT : mm



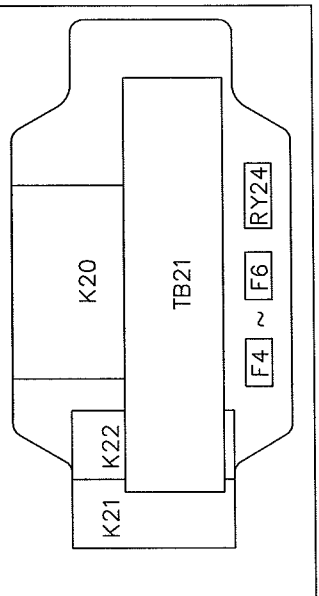
*TP1 is attached only to 380V and 440V.



전선 색상

Black : 전선
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Green & Yellow : 접지

The layout of electric parts



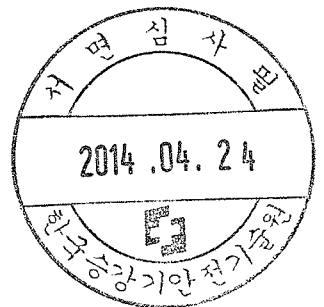
특수 공구(+, 육각)를 사용하여
개방구 쪽은 제외

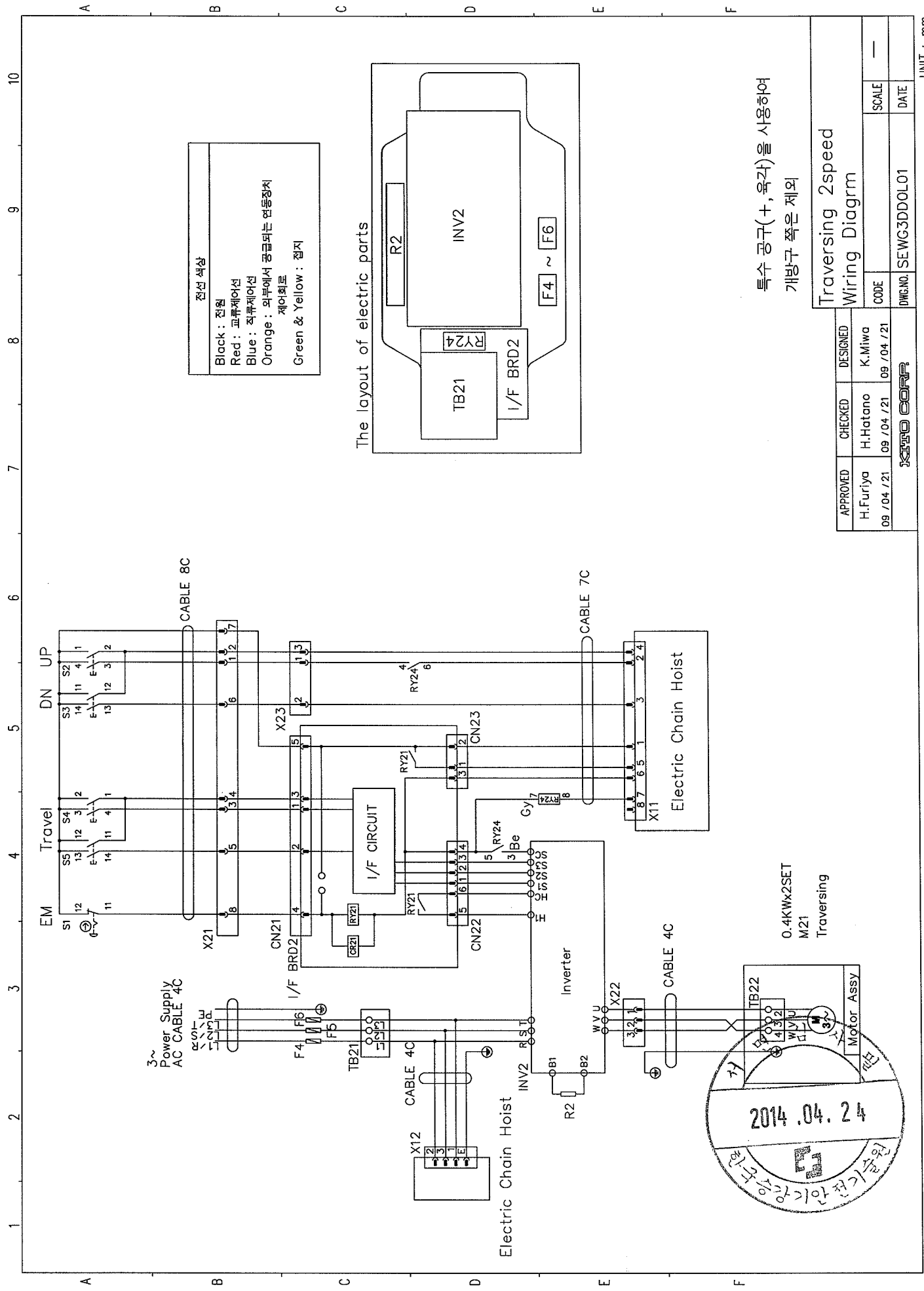
Traversing 1speed
Wiring Diagram

APPROVED	CHECKED	DESIGNED
H.Furiya	H.Hatano	K.Miwa
09 /04 /21	09 /04 /21	09 /04 /21
KATO CORP		
DWG.NO. SEW3DD0L01		
CODE	SCALE	DATE
—	—	—

UNIT : mm

. 2속형 hoisting/. 2속형 traversing

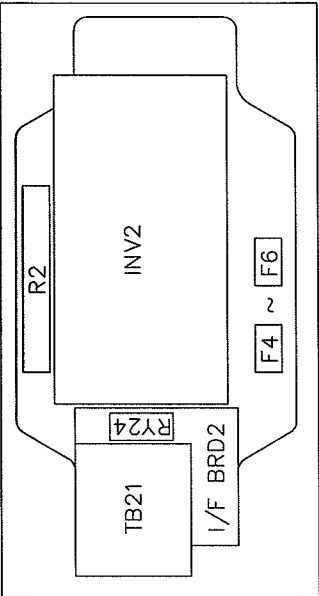




전선 색상

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The layout of electric parts



특수 공구(+, 육각)를 사용하여
 개방구 쪽은 제외

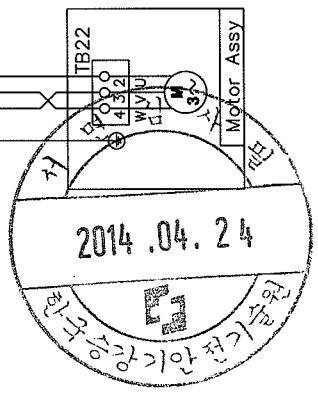
Traversing 2speed
 Wiring Diagram

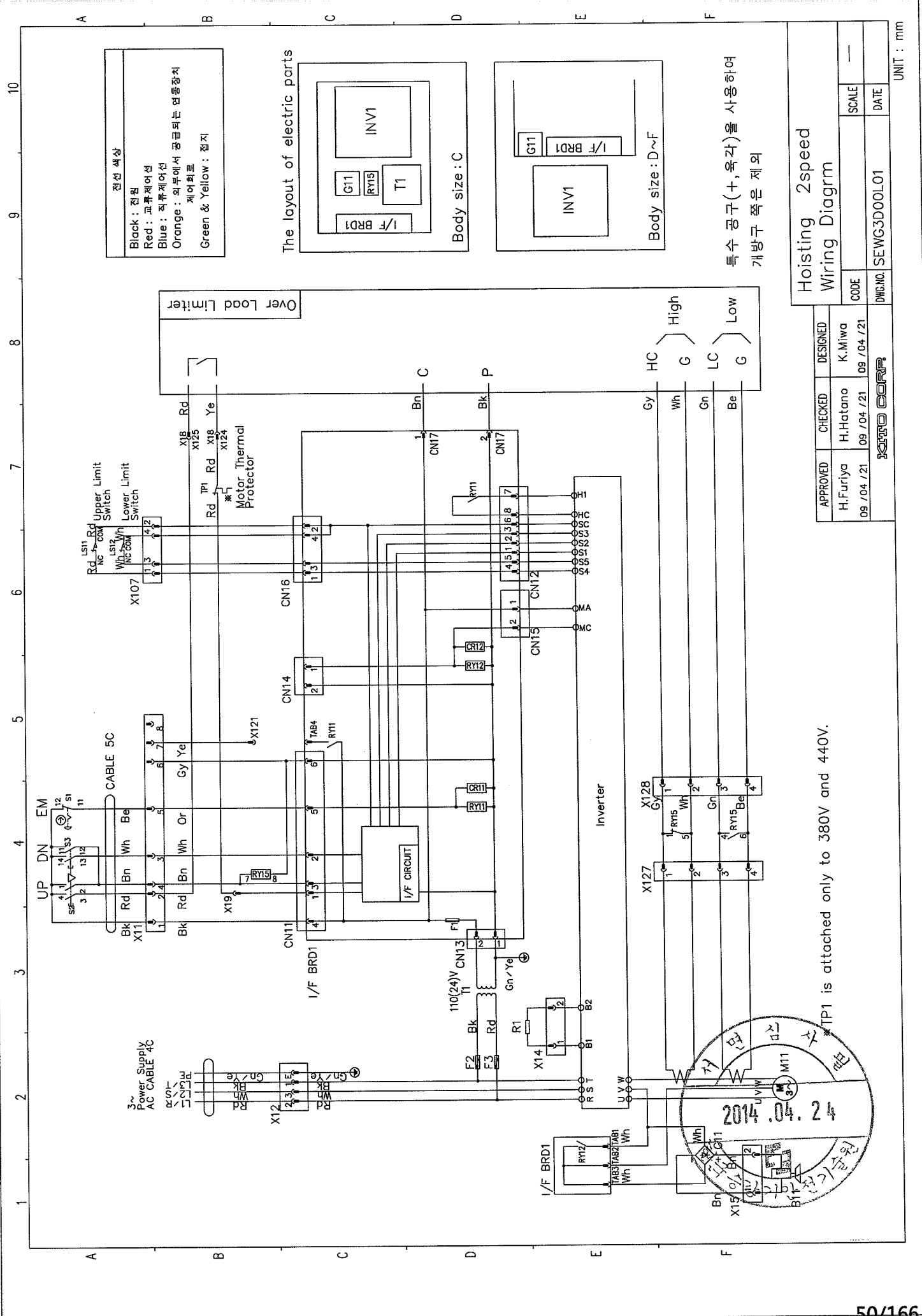
APPROVED	CHECKED	DESIGNED
H.Furiya 09 /04 /21	H.Hatano 09 /04 /21	K.Miwa 09 /04 /21

CODE	SCALE
—	—

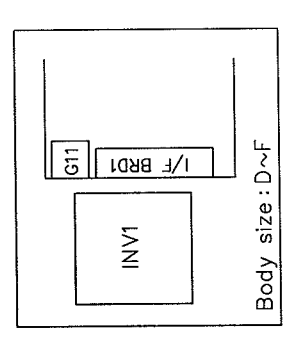
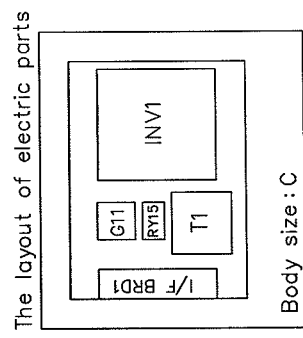
DWG.NO. SEWC3DD0L01

UNIT : mm





전선 색상
 Black : 전선
 Red : 교류제어선
 Blue : 직류제어선
 Orange : 외부에서 공급되는 연동장치 제어회로
 Green & Yellow : 접지



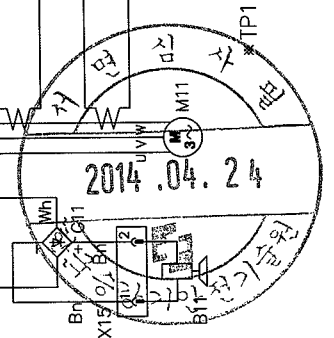
특수 공구 (+, 육각)을 사용하여
 개방구 쪽은 제외

Hoisting 2speed
 Wiring Diagram

APPROVED	CHECKED	DESIGNED
H.Furiya	H.Hatano	K.Miwa
09 / 04 / 21	09 / 04 / 21	09 / 04 / 21
KATO CORP		
CODE	SCALE	DATE
DWG.NO. SEWG3D00L01		

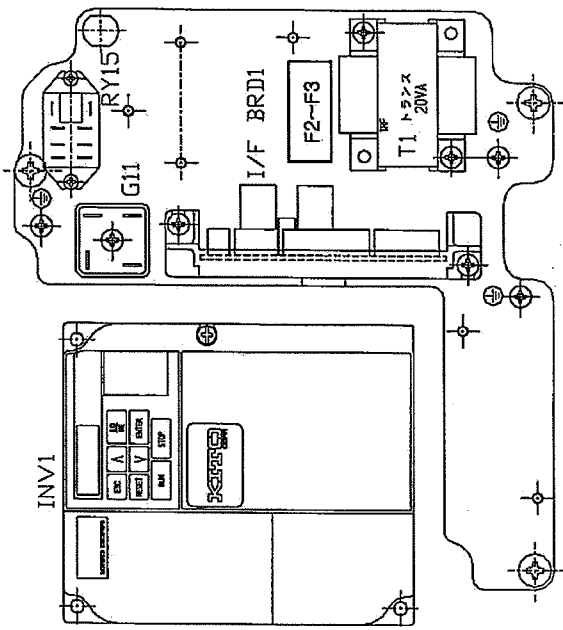
UNIT : mm

*TP1 is attached only to 380V and 440V.

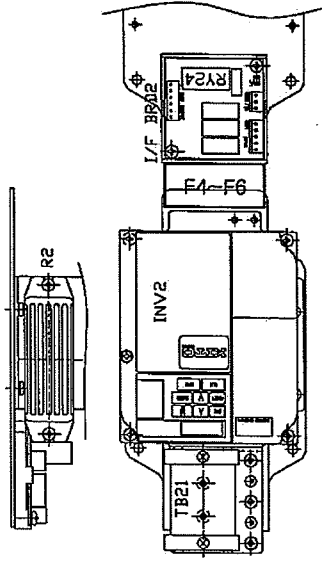


. 호이스트 CONTROL BOX 배치도 (ER2 010 IL-IS/IL)

HOISTING CONTROL BOX



TRAVERSING CONTROL BOX



ENCLOSURE : HOIST BODY - IP55
PUSH BUTTON - IP65

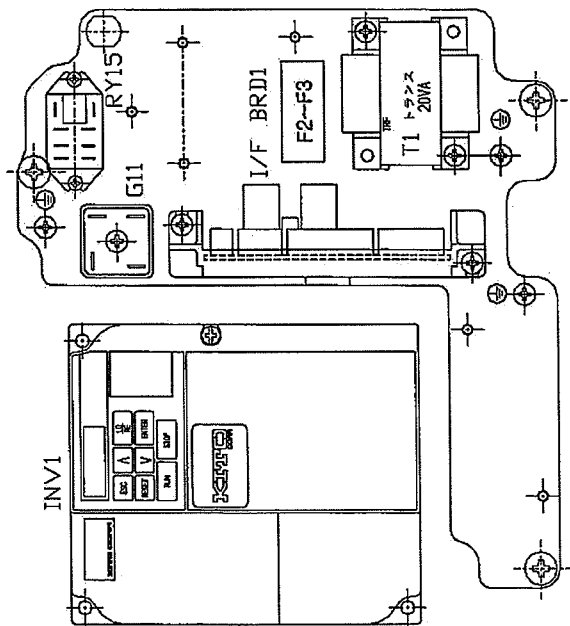
MARK	DESCRIPTION	TYPE OF MODEL		Q'TY	MAKER	REMARKS
		220V	380V			
INV1	INVERTER	V1000	V1000	1	YASKAWA	UP/DOWN
T1	TRANSFORMER	220V/24V 20VA	380V/24V 20VA	1	KITO	CONTROL CIRCUIT
G11	BRIDGE DIODE	S15VB60	S15VB60	1	SHINDENGEN	
I/F BRD1	INTERFACE BOARD	10~15A	10~15A	1	KITO	
F2-F3	GLASS FUSE	10A	10A	2	FUJI	
F4-F6	GLASS FUSE	20A	20A	3	FUJI	
RY15	RELAY	24V	24V	1	OMRON	HIGH/LOW
INV2	INVERTER	V1000	V1000	1	YASKAWA	RIGHT/LEFT
I/F BRD2	INTERFACE BOARD	10~15A	10~15A	1	KITO	
RY24	RELAY	24V	24V	1	OMRON	EMERGENCY STOP
TB21	TERMINAL BOARD 21	10~15A	10~15A	1	KITO	



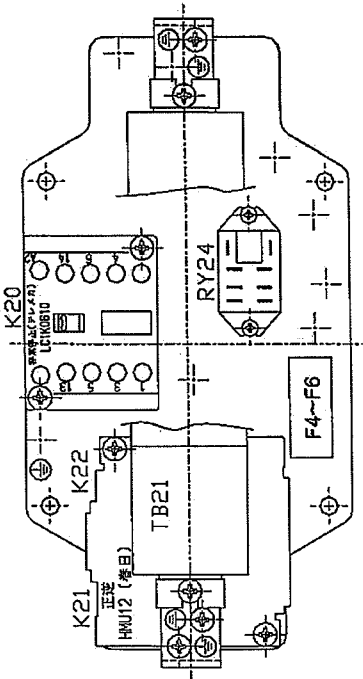
호이스트 CONTROL BOX 배치도

(ER2 010 IL-S/L)

HOISTING CONTROL BOX

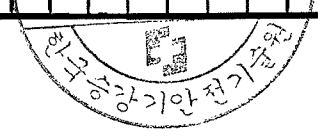


TRAVERSING CONTROL BOX



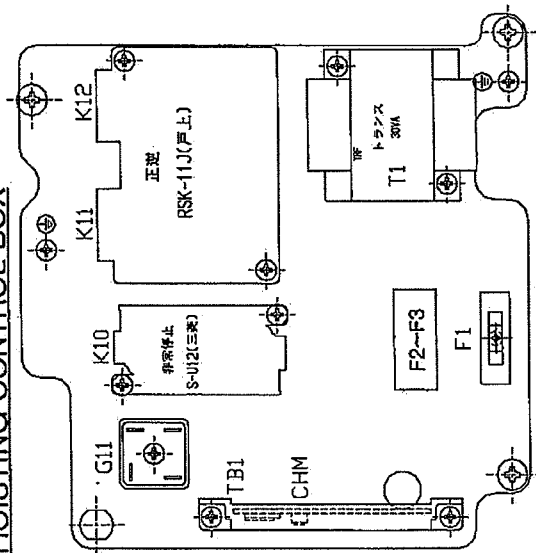
ENCLOSURE : HOIST BODY - IP55
PUSH BUTTON - IP65

MARK	DESCRIPTION	TYPE OF MODEL			Q'TY	MAKER	REMARKS
		220V	380V	440V			
INV1	INVERTER	V1000	V1000	V1000	1	YASKAWA	UP/DOWN
T1	TRANSFORMER	220V/24V 20VA	380V/24V 20VA	440V/24V 20VA	1	KITO	CONTROL CIRCUIT
G11	BRIDGE DIODE	S15VB60	S15VB60	S15VB60	1	SHINDENGEN	
I/F BRD1	INTERFACE BOARD	10~15A	10~15A	10~15A	1	KITO	
F2-F3	GLASS FUSE	10A	10A	10A	2	FUJI	
F4-F6	GLASS FUSE	20A	20A	20A	3	FUJI	
RY15	RELAY	24V	24V	24V	1	OMRON	HIGH/LOW
K20	MAGNET CONTACTOR	LC1K0610B7	LC1K0610B7	LC1K0610B7	1	TELEMECANIQUE	EMERGENCY STOP
K21, K22	MAGNET CONTACTOR	HMU12	HMU12	HMU12	1	KASUGA	RIGHT/LEFT
RY24	RELAY	24V	24V	24V	1	OMRON	EMERGENCY STOP
T B21	TERMINAL BOARD 21	10~15A	10~15A	10~15A	1	KITO	

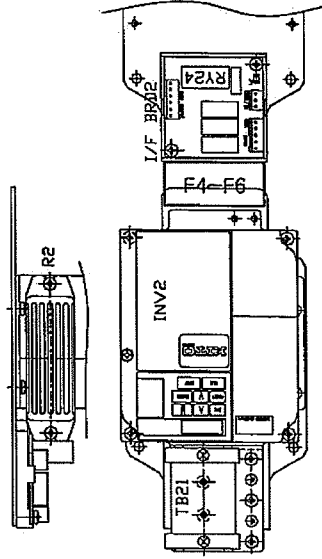


호이스트 CONTROL BOX 배치도 (ER2 010 L-IS/IL)

HOISTING CONTROL BOX



TRaversing CONTROL BOX



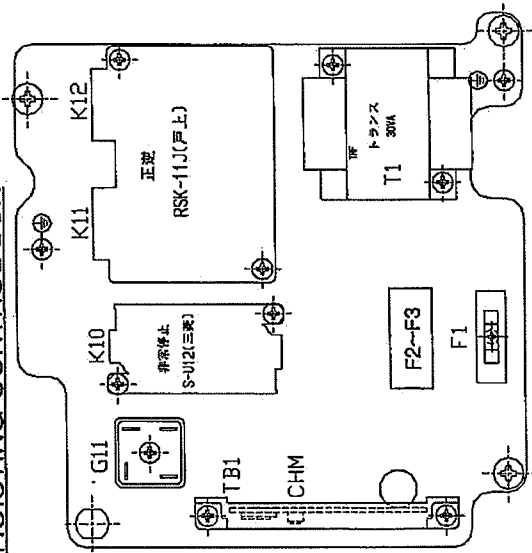
ENCLOSURE : HOIST BODY - IP55
PUSH BUTTON - IP65

MARK	DESCRIPTION	TYPE OF MODEL			Q'TY	MAKER	REMARKS
		220V	380V	440V			
K10	MAGNET CONTACTOR	S-U12	S-U12	S-U12	1	MITSUBISHI	EMERGENCY STOP
K11, K12	MAGNET CONTACTOR	RSK-11J-S95	RSK-11J-S95	RSK-11J-S95	1	TOGAMI	UP/DOWN
T1	TRANSFORMER	220V/24V 30VA	380V/24V 30VA	440V/24V 30VA	1	KITO	CONTROL CIRCUIT
F1	GLASS FUSE	2A	2A	2A	1	FUJI	
F2-F3	GLASS FUSE	10A	10A	10A	2	FUJI	
F4-F6	GLASS FUSE	20A	20A	20A	3	FUJI	
G11	BRIDGE DIODE	ST5VB60	ST5VB60	ST5VB60	1	SHINDENGEN	
TB	TERMINAL BOARD	10-15A	10-15A	10-15A	1	KITO	
CHM	COUNTER HOUR METER	ECP91CHAAT-3	ECP91CHAAT-3	ECP91CHAAT-3	1	OTEC	사용회수, 시간 기록
INV2	INVERTER	V1000	V1000	V1000	1	YASKAWA	RIGHT/LEFT
UF BRD2	INTERFACE BOARD	10-15A	10-15A	10-15A	1	KITO	
RY24	RELAY	24V	24V	24V	1	OMRON	
TB21	TERMINAL BOARD 21	10-15A	10-15A	10-15A	1	KITO	EMERGENCY STOP

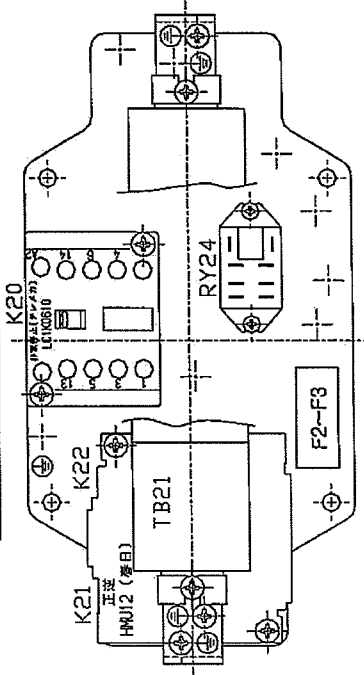


. 호이스트 CONTROL BOX 배치도 (ER2 010 L-S/L)

HOISTING CONTROL BOX



TRAVERSING CONTROL BOX



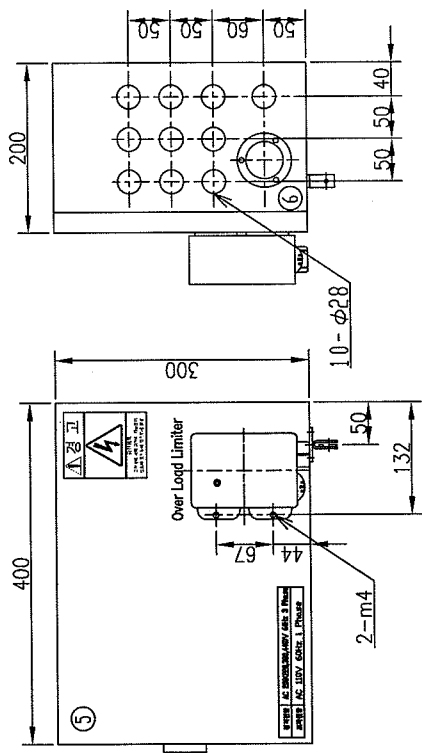
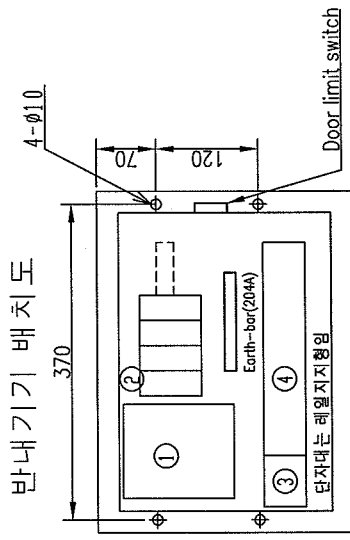
ENCLOSURE : HOIST BODY - IP55
PUSH BUTTON - IP65

MARK	DESCRIPTION	TYPE OF MODEL		Q'TY	MAKER	REMARKS
		220V	380V			
K10	MAGNET CONTACTOR	S-U12	S-U12	1	mitsubishi	EMERGENCY STOP
K11, K12	MAGNET CONTACTOR	RSK-11J-S95	RSK-11J-S95	1	TOGAMI	UP/DOWN
T1	TRANSFORMER	220V/24V 30VA	380V/24V 30VA	1	KITO	CONTROL CIRCUIT
F1	GLASS FUSE	2A	2A	1	FUJI	
F2-F3	GLASS FUSE	10A	10A	2	FUJI	
F4-F5	GLASS FUSE	20A	20A	3	FUJI	
F4-F5	BRIDGE DIODE	S15VB60	S15VB60	1	SHINDENGEN	
G11	TERMINAL BOARD	10-15A	10-15A	1	KITO	
TB	TERMINAL BOARD	ECP91CHAA1-3	ECP91CHAA1-3	1	OTEC	사용회수, 시간 기록
CHM	COUNTER HOUR METER	LC1K0610B7	LC1K0610B7	1	TELEMECANIQUE	EMERGENCY STOP
K20	MAGNET CONTACTOR	HMU12	HMU12	1	KASUGA	RIGHT/LEFT
K21, K22	MAGNET CONTACTOR	24V	24V	1	OMRON	EMERGENCY STOP
RY24	RELAY	10-15A	10-15A	1	KITO	
TB21	TERMINAL BOARD 21					



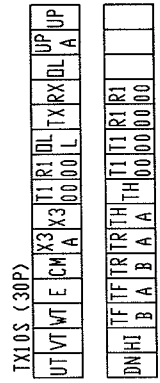
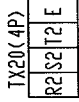
1	2	3	4	5	6	7	8	9	10
압착단자는 전부 절연피복 부착타입을 사용할것				공		작		제	
번호				량		수		기	
JIS B0405 中級				30		120		±0.3	
0.5BL±0.1				400		1000		±0.8	
6%±0.2				1000		2000		±1.2	
30				120		400		±0.5	
120				400		1000		±0.8	
400				1000		2000		±1.2	
1000				2000		4000		±2.0	

塗裝色 : 민색(번호 5Y7/1 (메이커 표준색))
 設定機器 : 인버터



번호	機器 番号	名 称	形 式	メーカ	個數	備 考
1	INV2	인버터	FRN1.5C1S-2J21	富士	1	
2		릴레이	HHS4P-L (AC24V)	富士	4	
3		쇼켓	TP514X1	富士	4	
4		단자대	TX20 (4P)	春日	1式	커버부착형
5		단자대	TX10S (30P)	春日	1式	커버부착형
6		합	CH20-43A	日東	1	
7		Door limits switch	KH-9015-HL	KONDO		
8						
9						
10						

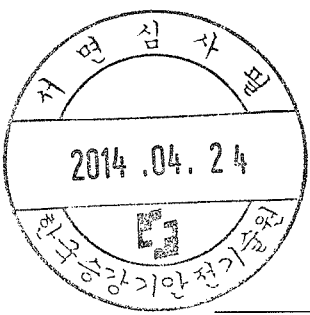
단자대 배열



Note

1) 외함 개방 시 충전 부분이 차단되도록 한다.

製造番号	303910	
名 称	セツゾクハコ	
材 質		
材 質		
尺 寸	尺 度	NOT
製 国	製 国	
設 計	設 計	10.10.8
検 査	検 査	10.10.8
承 認	承 認	10.10.8
年 月 日	年 月 日	承認
訂 数	内	番
樣式 025C-06 三角法 單位: mm		



CABLE 구성도 및 사양 - 권상 용량 0.9kw

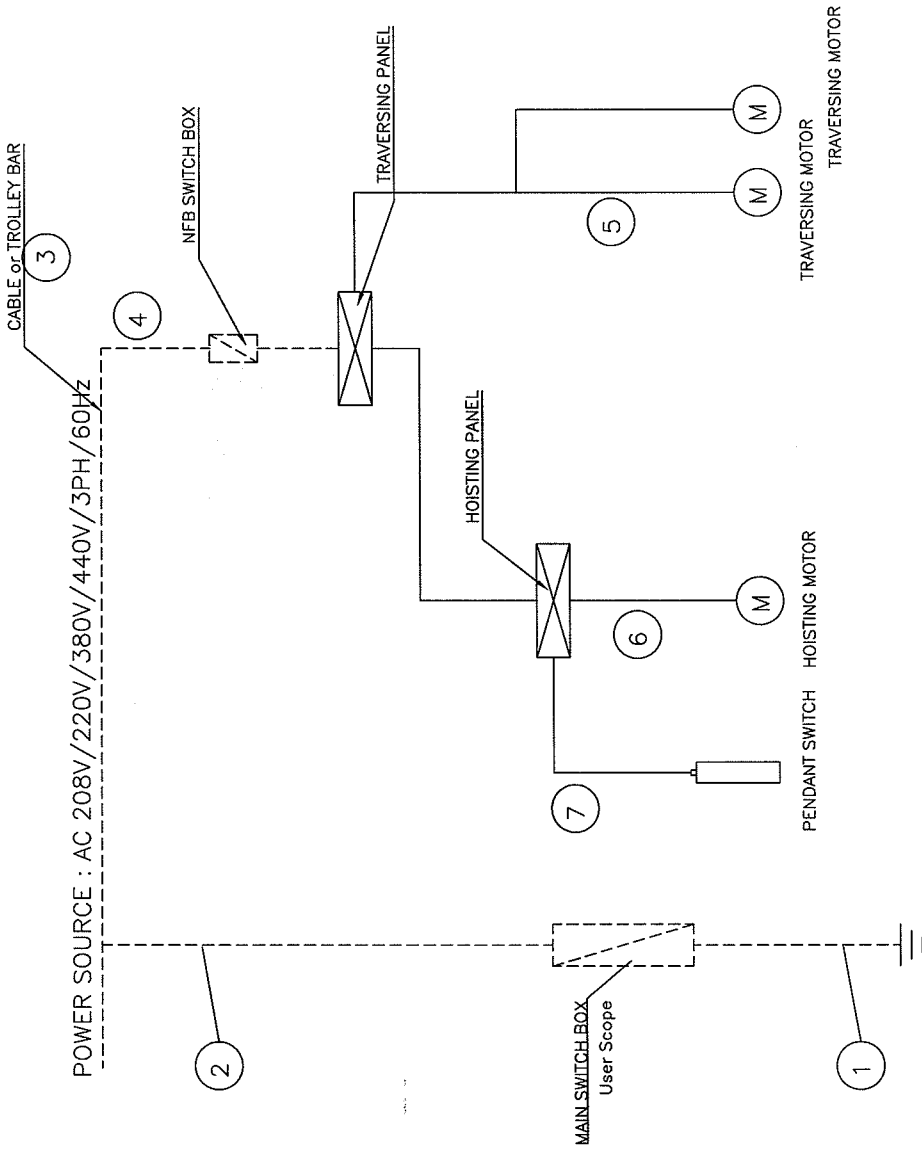
CABLE SPECIFICATION FOR ER2

NO	ITEM	TYPE	ER2	
			SIZE	
①	Power Line	VCT	3.5sq x 4C	
②	Push Button Switch	VCT	1.25sq x 8C	
③	Loas Limit	VCT	0.75sq x 8C	
④	Power Line for ER	VCT	1.5sq x 4C	
⑤	Control Line for ER	VCT	1.25sq x 6C	
⑥	Traversing Motor With Earth	VCT	1.25sq x 4C	

(3Φ 220(208)V / 380V / 440V 60HZ)



1 2 3 4 5 6 7 8 9



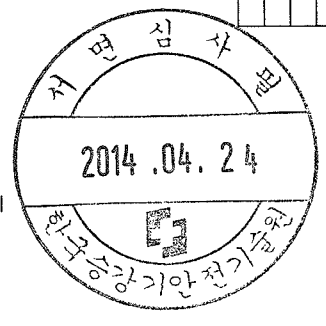
- 점지설비 시공방법
- 전동기의 외형, 제어반 등은 점지를 해야 하며 그 점지사항은 다음의 규정을 따른다.
 - 점지전용 Trolley Duct 및 전선은 당해 전기기기, 기구에 대하여 충분한 용량과 전기적, 기계적인 강도를 가져야 함.
 - 점지선이 외상을 받을 우려가 있는 경우는 전선관, 합성수지관 등과 함께 사용한다.
 - 점지공사는 지표면에서 최저 75cm이상의 길이에 점지봉을 막고 점지봉에는 점지동판을 연결한다.

점지공사	
3중 점지공사	400V 이하 100r이하
특 3중 점지공사	400V 이상 10r이하

단, 반복지역은 전압강하율이 10 이하일것

CABLE 종류 및 굵기

NO	FROM	CABLE SPEC.	Earth Cable
1	Main Earth Line	VCT 4SQx1C	User Scope(1Core)
2	Main Power Sorce	VCT 4SQ x 4C	User Scope(1Core)
3	Trolley bar or Festoon cable	60A x 4P or	User Scope(1Core)
4	Main Power Sorce	VCT 4SQ x 4C	User Scope(1Core)
5	Traversing Motor	VCT1.25SQx6C	(G-1Core)
6	Hoist Motor	VCT1.25SQ x 4C	(G-1Core)
7	Push Button Switch	VCT1.25SQx8C	(G-1Core)
8			



REV.	QTY	CONTENTS	DATE	DRAWN	APPROVED	CHECKED	DESIGNED	DRAWN	TITLE
									1t MOTORIZED-4점식 케이블 구성도 및 점지계통도
									MDL. 942513
									DWG. NO. 3NNU942513
									SCALE 1:1
									REV. 0

허용 최대 SPAN 적용표 (I-BEAM, H-BEAM)

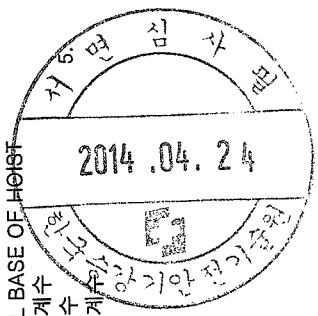
PROJECT NAME : KDL-ER2-010
 RATED LOAD : 1 ton
 DESCRIPTION : LIFT(max) 30 m

NO.	I-BEAM-SIZE (B*H*t1*t2)	Ix cm ⁴	Iy cm ⁴	Zx cm ³	Zy cm ³	A cm ²	Wb kg/m	Wh ton	Wg ton	Wg1 ton	L cm	L1 cm	L2 cm	b cm	E kg/cm ²	φ	ψ	TON/cm ²				결과				
																		Σσ1	Σσ2	Σσ3	Σσ4		δ1	L/800	82	L1/500
I - BEAM																										
1	200x100x7/10t	2170	138	217	27.7	33.06	26	0.16	0.117	0.047	450	180	30	13.2	2100000	1.11	1.1	0.498	0.664	0.863	1.226	0.514	0.563	0.2939	0.36	O.K
2	250x125x7.5/12.5	5180	337	414	53.9	48.79	38.3	0.16	0.26	0.096	680	250	30	13.2	2100000	1.11	1.1	0.455	0.531	0.877	0.991	0.796	0.850	0.3957	0.50	O.K
3	300x150x10/18.5t	12700	886	849	118	83.47	65.5	0.16	0.622	0.242	950	370	30	13.2	2100000	1.11	1.1	0.370	0.441	0.632	0.786	1.037	1.188	0.6273	0.74	O.K
4	350x150x12/24t	22400	1180	1280	158	111.1	87.2	0.16	0.959	0.392	1100	450	30	13.2	2100000	1.11	1.1	0.334	0.407	0.559	0.784	1.037	1.375	0.7039	0.90	O.K
5	400x150x12.5/28t	31700	1240	1580	165	122.1	95.8	0.16	1.15	0.479	1200	500	30	13.2	2100000	1.11	1.1	0.329	0.405	0.597	0.880	1.016	1.500	0.7154	1.00	O.K
H - BEAM																										
6	300x150x6.5/9t	7210	508	481	67.7	40.8	32	0.16	0.256	0.096	800	300	30	13.2	2100000	1.11	1.1	0.455	0.555	0.855	0.966	0.990	1.000	0.5241	0.80	O.K
7	350x175x7/11t	13500	984	771	112	62.91	49.4	0.16	0.494	0.198	1000	400	30	13.2	2100000	1.11	1.1	0.403	0.513	0.698	0.929	1.079	1.250	0.7468	0.80	O.K
8	400x200x8/13t	23500	1740	1170	174	83.37	65.4	0.16	0.719	0.294	1100	450	30	13.2	2100000	1.11	1.1	0.317	0.406	0.470	0.711	0.904	1.375	0.6484	0.90	O.K
9	450x200x9/14t	32900	1870	1460	187	95.43	74.9	0.16	0.899	0.375	1200	500	30	13.2	2100000	1.11	1.1	0.307	0.397	0.485	0.777	0.897	1.500	0.6659	1.00	O.K

X축의 단면 2차모멘트 $\Sigma\sigma_1$ = (PITCH내 계산응력) <math>< 1.279 \text{ TON/CM}^2</math> 이하일 경우 "O.K" (SS400, 응집효율 80% 적용, 풍하중 115% 적용)
 Y축의 단면 2차모멘트 $\Sigma\sigma_2$ = (PITCH내 계산응력) <math>< 1.391 * 80\% (\text{응집효율}) * 115\% (\text{작업시 풍하중}) = 1391 * 80\% (\text{응집효율}) * 115\% (\text{작업시 풍하중}) = 1279 \text{ KG/CM}^2</math>
 X축의 단면계수 $\Sigma\sigma_2$ = (켄틀레버 계산응력) <math>< 1.600 \text{ TON/CM}^2</math> 이하일 경우 "O.K" (SS400, 풍하중 115% 적용)
 Y축의 단면계수 $\Sigma\sigma_2$ = (켄틀레버 계산응력) <math>< 1.600 \text{ TON/CM}^2</math> 이하일 경우 "O.K" (SS400, 풍하중 115% 적용)
 BEAM의 단면적 $\Sigma\sigma_1$ = (PITCH내 계산응력) <math>< 1.447 \text{ TON/CM}^2</math> 이하일 경우 "O.K" (SS400, 풍하중 130% 적용)
 BEAM의 단위중량 $\Sigma\sigma_1$ = (PITCH내 계산응력) <math>< 1.447 \text{ TON/CM}^2</math> 이하일 경우 "O.K" (SS400, 풍하중 130% 적용)
 HOIST 자중 $\Sigma\sigma_3$ = (PITCH내 계산응력) <math>< 1.447 \text{ TON/CM}^2</math> 이하일 경우 "O.K" (SS400, 풍하중 130% 적용)
 GIRDER 자중 $\Sigma\sigma_3$ = (PITCH내 계산응력) <math>< 1.447 \text{ TON/CM}^2</math> 이하일 경우 "O.K" (SS400, 풍하중 130% 적용)
 켄틀레버 GIRDER 자중 $\Sigma\sigma_4$ = (켄틀레버 계산응력) <math>< 1.808 \text{ TON/CM}^2</math> 이하일 경우 "O.K" (SS400, 풍하중 130% 적용)
 SPAN-PITCH내 LENGTH $\Sigma\sigma_4$ = (켄틀레버 계산응력) <math>< 1.808 \text{ TON/CM}^2</math> 이하일 경우 "O.K" (SS400, 풍하중 130% 적용)
 켄틀레버 LENGTH $\Sigma\sigma_4$ = (켄틀레버 계산응력) <math>< 1.808 \text{ TON/CM}^2</math> 이하일 경우 "O.K" (SS400, 풍하중 130% 적용)
 HOOK APPROACH $\Sigma\sigma_4$ = (켄틀레버 계산응력) <math>< 1.808 \text{ TON/CM}^2</math> 이하일 경우 "O.K" (SS400, 풍하중 130% 적용)
 WHEEL BASE OF HOIST $\Sigma\sigma_4$ = (켄틀레버 계산응력) <math>< 1.808 \text{ TON/CM}^2</math> 이하일 경우 "O.K" (SS400, 풍하중 130% 적용)
 종단성계수 δ_1 = (PITCH내 처짐량)
 작업계수 δ_2 = (켄틀레버 처짐량)
 정하중계수 ψ = F

<math>< L / 800</math> 이하일 경우 "O.K"
 <math>< L1 / 500</math> 이하일 경우 "O.K"

AREA CLASSIFICATION : IN DOOR or OUT DOOR

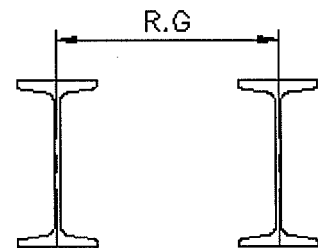
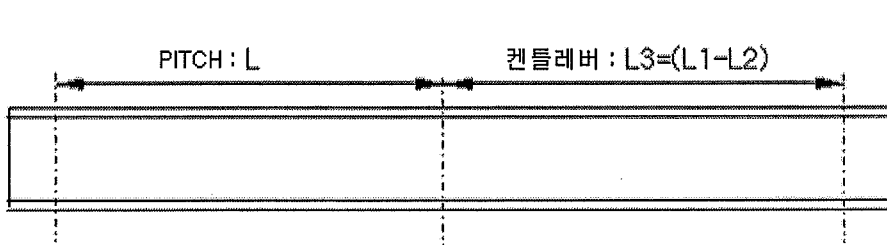


1. I-BEAM 최대허용 가능 SPAN 계산

1. SPECIFICATION

.정격하중	-----	Q =	1	ton
.SPAN (PITCH)	-----	L =	450	cm
.켄틸리버	-----	L1 =	180	cm
.TROLLEY WHEEL BASE	-----	B =	13.2	cm
.WEIGHT OF HOIST	-----	Wh =	0.16	ton
.PITCH내의 GIRDER 무게	-----	Wg =	0.117	ton
.켄틸리버의 GIRDER 무게	-----	Wg1 =	0.047	ton
.I-BEAM SIZE	-----		200x100x7/10t	
		Ix =	2170	cm ⁴
		Iy =	138	cm ⁴
		Zx =	217	cm ³
		Zy =	27.7	cm ³
		A =	33.06	cm ²
		Wb =	26	kg/m
.HOOK APPROACH	-----	L2 =	30	cm
탄성계수	-----	E =	2100000	kg/cm ²
동하중 계수(작업계수)	-----	M(Φ) =	1.11	
정하중 계수(충격계수)	-----	F(Ψ) =	1.10	

1. DESIGN



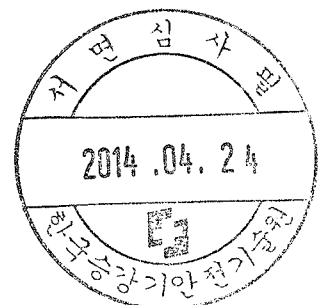
2. ONE SIDE I-BEAM에 작용하는 하중

$$P = (Q + Wh)/2 = (1 + 0.16)/2 = 0.58 \text{ ton}$$

3. 수직하중에 의한 BENDING MOMENT

1) PITCH 지점내 BENDING MOMENT

$$\begin{aligned} M_{h1} &= F \times M \times P \times (L-B/2)^2 / (4 \times L) \\ &= 1.11 \times 1.1 \times 0.58 \times (450-13.2/2)^2 / (4 \times 450) \end{aligned}$$



$$= 77.4 \quad \text{ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$Mh2 = F \times M \times P \times L3$$

$$= 1.11 \times 1.1 \times 0.58 \times (180-30) = 106.23 \quad \text{ton.cm}$$

4. 자중에 의한 BENDING MOMENT(I-BEAM)

1) PITCH 지점내 BENDING MOMENT

$$Mt1 = F \times Wg \times L / 8 = 1.11 \times 0.117 \times 450 / 8 = 7.305 \quad \text{ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$Mt2 = F \times Wg1 \times L1 / 2 = 1.11 \times 0.047 \times 180 / 2 = 4.7 \quad \text{ton.cm}$$

5. 수직하중에 의한 최대 BENDING MOMENT

$$Mmax1 = Mh1 + Mt1 = 84.7 \quad \text{ton.cm}$$

$$Mmax2 = Mh2 + Mt2 = 110.9 \quad \text{ton.cm}$$

6. 풍하중에 의한 수평하중의 BENDING MOMENT

(1) GIRDER에 의한 수직 풍하중

$$\text{작업시 } V=16\text{m/s}, q = 19.9 \text{ kg/m}^2 \quad h(\text{최고양정}) = 30 \text{ M}$$

$$\text{작업시 } q = 8.5 \times 10^{-4} \sqrt{h} = 8.5 \times 10^{-4} \sqrt{42} = 19.9 \text{ kg/m}^2$$

$$H(\text{GIRDER높이}) = 0.25 \text{ m}$$

$$\text{PITCH내 풍하중} = F \times L \times H \times q \times 1.6 = 1.11 \times 4.5 \times 0.25 \times 19.9 \times 1.6 = 40 \text{ kg}$$

$$\text{캔틀레버 풍하중} = F \times L1 \times H \times q \times 1.3 = 1.11 \times 1.8 \times 0.25 \times 19.9 \times 1.3 = 13 \text{ kg}$$

$$M_{FGG} = \frac{0.04 \times 450}{8} - \frac{0.013 \times 180}{2} = 1.08 \quad \text{ton.cm}$$

$$M_{FG1} = \frac{0.013 \times 180}{2} = 1.17 \quad \text{ton.cm}$$

(2) HOIST에 의한 풍하중

* 작업시

$$\text{풍하중} = F \times HB \times HH \times q \times 1.2 = 1.11 \times 0.65 \times 1 \times 19.9 \times 1.2 = 17 \text{ kg}$$

*PITCH내 풍하중

$$M_{FHG} = 17 \times 450 / 4 = 1912.5 \text{ kg.cm} = 1.9125 \text{ ton.cm}$$

*캔틀레버 풍하중

$$M_{FH1} = 17 \times 180 = 3060 \text{ kg.cm} = 3.06 \text{ ton.cm}$$

7. COMBINED MOMENT

*PITCH내

$$M_{HCG} = M_{FGG} + M_{FHG} = 1.08 + 1.9125 = 2.993 \text{ ton.cm}$$

*켄틀레버

$$M_{HC1} = M_{FG1} + M_{FH1} = 1.17 + 3.06 = 4.230 \text{ ton.cm}$$

8. BENDING STRESS

A. VERTICAL BENDING STRESS

1. PITCH 내

$$\sigma v1 = M_{max1} / Zx = 84.7 / 217 = 0.390 \text{ ton/cm}^2$$

2. 켄틀레버

$$\sigma v2 = M_{max2} / Zx = 110.9 / 217 = 0.511 \text{ ton/cm}^2$$

B. 수평 최대 응력

1. PITCH 내

$$\sigma v3 = M_{HCG} / Zy = 2.9925 / 27.7 = 0.108 \text{ ton/cm}^2$$

2. 켄틀레버

$$\sigma v4 = M_{HC1} / Zy = 4.23 / 27.7 = 0.153 \text{ ton/cm}^2$$

9. 합성 응력

$$\Sigma \sigma 1 = \sigma v1 + \sigma v3 = 0.498$$

$$0.498 < 1.279 \text{ ton/cm}^2 \quad \text{O.K}$$

$$\Sigma \sigma 2 = \sigma v2 + \sigma v4 = 0.664$$

$$0.664 < 1.6 \text{ ton/cm}^2 \quad \text{O.K}$$

용접효율 : 80% 응력 $1391 \times 80\% \times 115\% = 1.279 \text{ ton/cm}^2$
 켄틀레버는 용접부 없음 $1391 \times \text{작업시} 1.15 (\text{풍하중포함}) = 1.600 \text{ ton/cm}^2$

10. 휴지시 풍하중에 의한 BENDING MOMENT

휴지시 $V=45\text{m/s}$, $q = 158.0 \text{ kg/m}^2$ $h(\text{최고양정}) = 30 \text{ m}$
 $q = M \times \sqrt[4]{h} = 67.5 \times \sqrt[4]{30} = 158.0 \text{ kg/m}^2$
 $M = V^2 / 30 = 67.5$

*휴지시 풍하중에 의한 수평하중

PITCH내 풍하중 = $F \times L \times H \times q \times 1.6 = 1.11 \times 4.5 \times 0.25 \times 158 \times 1.6 = 316 \text{ kg}$

켄틀레버에 대한 풍하중 = $F \times L_1 \times H \times q \times 1.3 = 1.11 \times 1.8 \times 0.25 \times 158 \times 1.3 = 103 \text{ kg}$

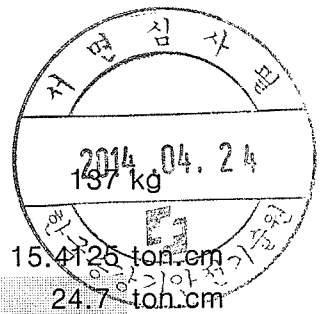
$$MM_{G1} = \frac{0.316 \times 450}{8} - \frac{0.103 \times 180}{2} = 8.505 \text{ ton.cm}$$

$$MM_1 = \frac{0.103 \times 180}{2} = 9.27 \text{ ton.cm}$$

HOIST에 대한 풍하중 = $F \times H_B \times H_H \times q \times 1.2 = 1.11 \times 0.65 \times 1 \times 158 \times 1.2 =$

*PITCH내 $MM_{H0} = 137 \times 450 / 4 = 61650 \text{ KG.CM} = 15.4125 \text{ ton.cm}$

*켄틀레버 $MM_{H1} = 137 \times 180 = 24660 \text{ KG.CM} = 24.7 \text{ ton.cm}$



* COMBINED MOMENT

$$MM_2 = MM_{G1} + MM_{H0} = 8.505 + 15.4125 = 23.9175 \text{ ton.cm}$$

$$MM_4 = MM_1 + MM_{H1} = 9.27 + 24.7 = 33.97 \text{ ton.cm}$$

* BENDING STRESS

$$\Sigma\sigma_3 = MM_2 / Z_y = 23.9175 / 27.7 = 0.863 \text{ ton/cm}^2 < 1.447 \text{ ton/cm}^2 \text{---O.K}$$

$$\Sigma\sigma_4 = MM_4 / Z_y = 33.97 / 27.7 = 1.226 \text{ ton/cm}^2 < 1.808 \text{ ton/cm}^2 \text{---O.K}$$

PITCH내 휴지시 응력 1391 x 80% x 130% = 1447 ton/cm²
 켄틀레버는 용접부 없음 휴지시 응력 1391 x 130% = 1808 ton/cm²

11. DEFLECTION OF GIRDER

* PITCH 내

1) DUE TO RATED & TROLLEY LOAD

$$.D1 = \frac{(Q+Q2)x(L)^3}{48xEx Ix} = 0.4833 \text{ cm}$$

2) DUE TO DEAD LOAD

$$.D2 = \frac{5 x Wg x L^3}{384 x E x Ix} = 0.0305 \text{ cm}$$

3) TOTAL DEFLECTION

$$.s1 = D1 + D2 = 0.514 \text{ cm}$$

$$\text{RATIO : } D3/L = 1/876 < 800 \text{ ---O.K}$$

* 켄틀레버

1) DUE TO RATED & TROLLEY LOAD

$$.D1 = \frac{(Q+Q2)x(L1-L2)^3}{3xEx Ix} = 0.2864 \text{ cm}$$

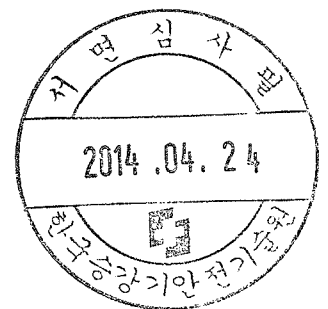
2) DUE TO DEAD LOAD

$$.D2 = \frac{Wg1x L1^3}{8 x E x Ix} = 0.0075 \text{ cm}$$

3) TOTAL DEFLECTION

$$.s2 = D1 + D2 = 0.2939 \text{ cm}$$

$$\text{RATIO : } D3/L = 1/612 < 500 \text{ ---O.K}$$



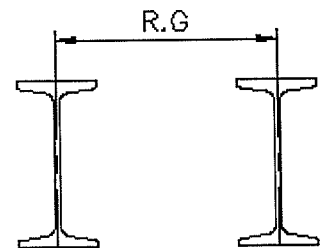
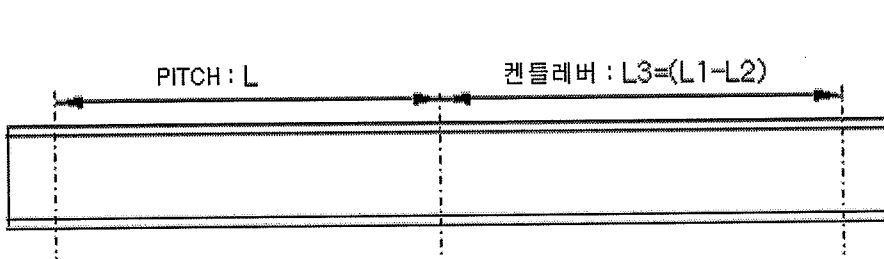
90-4

2. I-BEAM 최대허용 가능 SPAN 계산

1. SPECIFICATION

.정격하중	-----	Q =	1	ton
.SPAN (PITCH)	-----	L=	680	cm
.켄틸리버	-----	L1=	250	cm
.TROLLEY WHEEL BASE	-----	B=	13.2	cm
.WEIGHT OF HOIST	-----	Wh=	0.16	ton
.PITCH내의 GIRDER 무게	-----	Wg=	0.26	ton
.켄틸리버의 GIRDER 무게	-----	Wg1=	0.096	ton
.I-BEAM SIZE	-----		250x125x7.5/12.5t	
		Ix =	5180	cm ⁴
		Iy =	337	cm ⁴
		Zx =	414	cm ³
		Zy =	53.9	cm ³
		A =	48.79	cm ²
		Wb =	38.3	kg/m
.HOOK APPROACH	-----	L2=	30	cm
탄성계수	-----	E=	2100000	kg/cm ²
동하중 계수(작업계수)	-----	M(Φ)=	1.11	
정하중 계수(충격계수)	-----	F(Ψ)=	1.10	

1. DESIGN



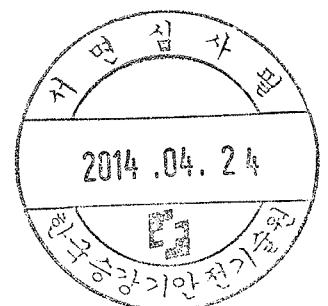
2. ONE SIDE I-BEAM에 작용하는 하중

$$P = (Q + Wh)/2 = (1 + 0.16)/2 = 0.58 \text{ ton}$$

3. 수직하중에 의한 BENDING MOMENT

1) PITCH 지점내 BENDING MOMENT

$$\begin{aligned} M_{h1} &= F \times M \times P \times (L-B/2)^2 / (4 \times L) \\ &= 1.11 \times 1.1 \times 0.58 \times (680-13.2/2)^2 / (4 \times 680) \end{aligned}$$



$$= 118.1 \quad \text{ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$Mh2 = F \times M \times P \times L3$$

$$= 1.11 \times 1.1 \times 0.58 \times (250 - 30) = 155.8 \quad \text{ton.cm}$$

4. 자중에 의한 BENDING MOMENT(I-BEAM)

1) PITCH 지점내 BENDING MOMENT

$$Mt1 = F \times Wg \times L / 8 = 1.11 \times 0.26 \times 680 / 8 = 24.531 \quad \text{ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$Mt2 = F \times Wg1 \times L1 / 2 = 1.11 \times 0.096 \times 250 / 2 = 13.32 \quad \text{ton.cm}$$

5. 수직하중에 의한 최대 BENDING MOMENT

$$Mmax1 = Mh1 + Mt1 = 142.6 \quad \text{ton.cm}$$

$$Mmax2 = Mh2 + Mt2 = 169.1 \quad \text{ton.cm}$$

6. 풍하중에 의한 수평하중의 BENDING MOMENT

(1) GIRDER에 의한 수직 풍하중

$$\text{작업시 } V=16\text{m/s}, q = 19.9 \text{ kg/m}^2 \quad h(\text{최고양정}) = 30 \text{ M}$$

$$\text{작업시 } q = 8.5 \times 4 \sqrt{h} = 8.5 \times 4 \sqrt{42} = 19.9 \text{ kg/m}^2$$

$$H(\text{GIRDER높이}) = 0.25 \text{ m}$$

$$\text{PITCH내 풍하중} = F \times L \times H \times q \times 1.7 = 1.11 \times 6.8 \times 0.25 \times 19.9 \times 1.7 = 64 \text{ kg}$$

$$\text{캔틀레버 풍하중} = F \times L1 \times H \times q \times 1.4 = 1.11 \times 2.5 \times 0.25 \times 19.9 \times 1.4 = 19 \text{ kg}$$

$$M_{FGG} = \frac{0.064 \times 680}{8} - \frac{0.019 \times 250}{2} = 3.065 \quad \text{ton.cm}$$

$$M_{FG1} = \frac{0.019 \times 250}{2} = 2.375 \quad \text{ton.cm}$$

(2) HOIST에 의한 풍하중

* 작업시

$$\text{풍하중} = F \times HB \times HH \times q \times 1.2 = 1.11 \times 0.65 \times 1 \times 19.9 \times 1.2 =$$

$$0.65 \text{ m}$$

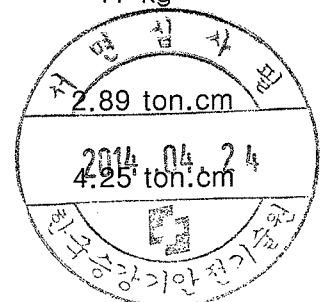
$$17 \text{ kg}$$

*PITCH내 풍하중

$$M_{FHG} = 17 \times 680 / 4 = 2890 \text{ kg.cm} =$$

*캔틀레버 풍하중

$$M_{FH1} = 17 \times 250 = 4250 \text{ kg.cm} =$$



7. COMBINED MOMENT

*PITCH내

$$M_{HCG} = M_{FGG} + M_{FHG} = 3.065 + 2.89 = 5.955 \text{ ton.cm}$$

*컨트레버

$$M_{HC1} = M_{FG1} + M_{FH1} = 2.375 + 4.25 = 6.625 \text{ ton.cm}$$

8. BENDING STRESS

A. VERTICAL BENDING STRESS

1. PITCH 내

$$\sigma v1 = M_{max1} / Z_x = 142.6 / 414 = 0.344 \text{ ton/cm}^2$$

2. 컨트레버

$$\sigma v2 = M_{max2} / Z_x = 169.1 / 414 = 0.408 \text{ ton/cm}^2$$

B. 수평 최대 응력

1. PITCH 내

$$\sigma v3 = M_{HCG} / Z_y = 5.955 / 53.9 = 0.110 \text{ ton/cm}^2$$

2. 컨트레버

$$\sigma v4 = M_{HC1} / Z_y = 6.625 / 53.9 = 0.123 \text{ ton/cm}^2$$

9. 합성 응력

$$\Sigma \sigma 1 = \sigma v1 + \sigma v3 = 0.455$$

$$0.455 < 1.279 \text{ ton/cm}^2 \quad \text{O.K}$$

$$\Sigma \sigma 2 = \sigma v2 + \sigma v4 = 0.531$$

$$0.531 < 1.6 \text{ ton/cm}^2 \quad \text{O.K}$$

용접효율 : 80% 응력 1391x80% x 115% = 1.279 ton/cm²

컨트레버는 용접부 없음 1391*작업시1.15(풍하중포함) = 1.600 ton/cm²

10. 휴지시 풍하중에 의한 BENDING MOMENT

휴지시 V=45m/s, q = 158.0 kg/m² h(최고양정) = 30 m

휴지시 q = M x ⁴√h = 67.5 x ⁴√30 = 158.0 kg/m²

M = V² / 30 = 67.5

*휴지시 풍하중에 의한 수평하중

PITCH내 풍하중 = F x L x H x q x 1.7 = 1.11x6.8x0.25x158x1.7 = 507 kg

컨트레버에 대한 풍하중 = FxL1xHxqx1.4 = 1.11x2.5x0.25x158x1.4 = 153 kg

$$MM_{G1} = \frac{0.507 \times 680}{8} - \frac{0.153 \times 250}{2} = 23.97 \text{ ton.cm}$$

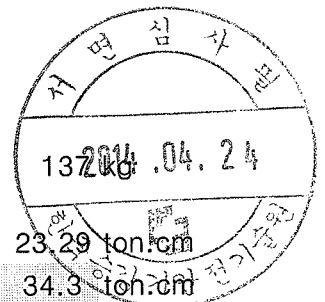
$$MM_1 = \frac{0.153 \times 250}{2} = 19.125 \text{ ton.cm}$$

HOIST에 대한 풍하중 = FxHBxHHxqx1.2 = 1.11x0.65x1x158x1.2 =

*PITCH내 MM_{H0} = 137 x 680 / 4 = 93160 KG.CM = 23.29 ton.cm

*컨트레버 MM_{H1} = 137 x 250 = 34250 KG.CM = 34.3 ton.cm

* COMBINED MOMENT



$$MM_2 = MM_{G1} + MM_{H0} = 23.97 + 23.29 = 47.26 \text{ ton.cm}$$

$$MM_4 = MM_1 + MM_{H1} = 19.125 + 34.3 = 53.43 \text{ ton.cm}$$

* BENDING STRESS

$$\Sigma\sigma_3 = MM_2 / Z_y = 47.26 / 53.9 = 0.877 \text{ ton/cm}^2 < 1.447 \text{ ton/cm}^2 \text{---O.K}$$

$$\Sigma\sigma_4 = MM_4 / Z_y = 53.425 / 53.9 = 0.991 \text{ ton/cm}^2 < 1.808 \text{ ton/cm}^2 \text{---O.K}$$

PITCH내 휴지시 응력 1391 x 80% x 130% = 1447 ton/cm²
 켄틀레버는 용접부 없음 휴지시 응력 1391 x 130% = 1808 ton/cm²

11. DEFLECTION OF GIRDER

* PITCH 내

1) DUE TO RATED & TROLLEY LOAD

$$.D1 = \frac{(Q+Q2) \times (L)^3}{48 \times E \times I_x} = 0.6985 \text{ cm}$$

2) DUE TO DEAD LOAD

$$.D2 = \frac{5 \times W_{gx} \times L^3}{384 \times E \times I_x} = 0.0979 \text{ cm}$$

3) TOTAL DEFLECTION

$$.d1 = D1 + D2 = 0.796 \text{ cm}$$

$$\text{RATIO : } D3/L = 1/854 < 800 \text{ --- O.K}$$

* 켄틀레버

1) DUE TO RATED & TROLLEY LOAD

$$.D1 = \frac{(Q+Q2) \times (L1-L2)^3}{3 \times E \times I_x} = 0.3785 \text{ cm}$$

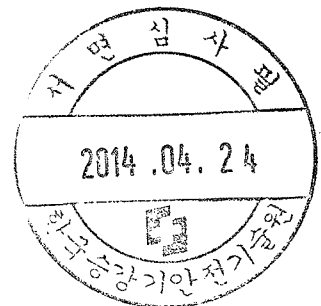
2) DUE TO DEAD LOAD

$$.D2 = \frac{W_{g1} \times L1^3}{8 \times E \times I_x} = 0.0172 \text{ cm}$$

3) TOTAL DEFLECTION

$$.d2 = D1 + D2 = 0.3957 \text{ cm}$$

$$\text{RATIO : } D3/L = 1/632 < 500 \text{ --- O.K}$$

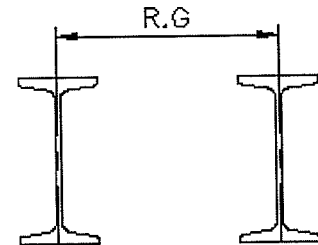
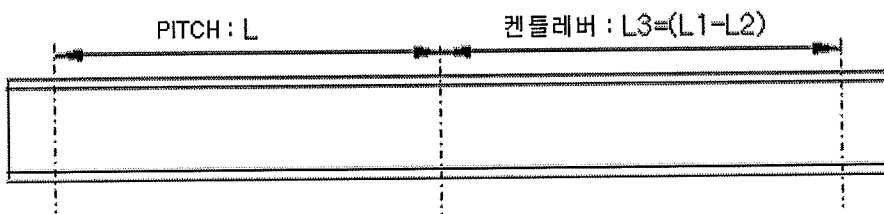


3. I-BEAM 최대허용 가능 SPAN 계산

1. SPECIFICATION

.정격하중	-----	Q =	1	ton
.SPAN (PITCH)	-----	L=	950	cm
.켄틸리버	-----	L1=	370	cm
.TROLLEY WHEEL BASE	-----	B=	13.2	cm
.WEIGHT OF HOIST	-----	Wh=	0.16	ton
.PITCH내의 GIRDER 무게	-----	Wg=	0.622	ton
.켄틸리버의 GIRDER 무게	-----	Wg1=	0.242	ton
.I-BEAM SIZE	-----		300x150x10/18.5t	
		Ix =	12700	cm ⁴
		Iy =	886	cm ⁴
		Zx =	849	cm ³
		Zy =	118	cm ³
		A =	83.47	cm ²
		Wb =	65.5	kg/m
.HOOK APPROACH	-----	L2=	30	cm
탄성계수	-----	E=	2100000	kg/cm ²
동하중 계수(작업계수)	-----	M(Φ)=	1.11	
정하중 계수(충격계수)	-----	F(Ψ)=	1.10	

1. DESIGN



2. ONE SIDE I-BEAM에 작용하는 하중

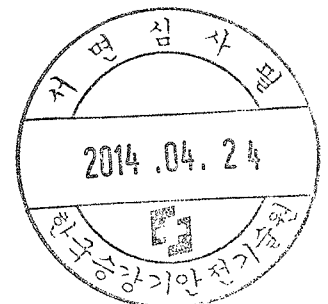
$$P = (Q + Wh)/2 = (1 + 0.16)/2 = 0.58 \text{ ton}$$

3. 수직하중에 의한 BENDING MOMENT

1) PITCH 지점내 BENDING MOMENT

$$M_{h1} = F \times M \times P \times (L-B/2)^2 / (4 \times L)$$

$$= 1.11 \times 1.1 \times 0.58 \times (950-13.2/2)^2 / (4 \times 950)$$



$$= 165.9 \text{ ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$Mh2 = F \times M \times P \times L3$$

$$= 1.11 \times 1.1 \times 0.58 \times (370-30) = 240.78 \text{ ton.cm}$$

4. 자중에 의한 BENDING MOMENT(I-BEAM)

1) PITCH 지점내 BENDING MOMENT

$$Mt1 = F \times Wg \times L / 8 = 1.11 \times 0.622 \times 950 / 8 = 81.987 \text{ ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$Mt2 = F \times Wg1 \times L1 / 2 = 1.11 \times 0.242 \times 370 / 2 = 49.69 \text{ ton.cm}$$

5. 수직하중에 의한 최대 BENDING MOMENT

$$Mmax1 = Mh1 + Mt1 = 247.8 \text{ ton.cm}$$

$$Mmax2 = Mh2 + Mt2 = 290.5 \text{ ton.cm}$$

6. 풍하중에 의한 수평하중의 BENDING MOMENT

(1) GIRDER에 의한 수직 풍하중

$$\text{작업시 } V=16\text{m/s}, q = 19.9 \text{ kg/m}^2 \quad h(\text{최고양정}) = 30 \text{ M}$$

$$\text{작업시 } q = 8.5 \times 10^{-4} \sqrt{h} = 8.5 \times 10^{-4} \sqrt{42} = 19.9 \text{ kg/m}^2$$

$$H(\text{GIRDER높이}) = 0.25 \text{ m}$$

$$\text{PITCH내 풍하중} = F \times L \times H \times q \times 1.7 = 1.11 \times 9.5 \times 0.25 \times 19.9 \times 1.7 = 89 \text{ kg}$$

$$\text{캔틀레버 풍하중} = F \times L1 \times H \times q \times 1.4 = 1.11 \times 3.7 \times 0.25 \times 19.9 \times 1.4 = 29 \text{ kg}$$

$$M_{FGG} = \frac{0.089 \times 950}{8} - \frac{0.029 \times 370}{2} = 5.204 \text{ ton.cm}$$

$$M_{FG1} = \frac{0.029 \times 370}{2} = 5.365 \text{ ton.cm}$$

(2) HOIST에 의한 풍하중

* 작업시

$$\text{풍하중} = F \times HB \times HH \times q \times 1.2 = 1.11 \times 0.65 \times 1 \times 19.9 \times 1.2 = 17 \text{ kg}$$

*PITCH내 풍하중

$$M_{FHG} = 17 \times 950 / 4 = 4037.5 \text{ kg.cm} = 4.0375 \text{ ton.cm}$$

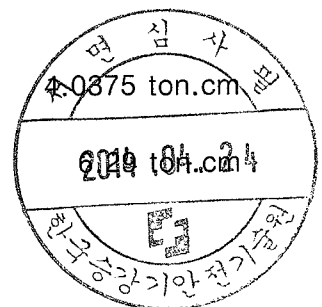
*캔틀레버 풍하중

$$M_{FH1} = 17 \times 370 = 6290 \text{ kg.cm} = 6.29 \text{ ton.cm}$$

7. COMBINED MOMENT

*PITCH내

$$M_{HCG} = M_{FGG} + M_{FHG} = 5.204 + 4.0375 = 9.242 \text{ ton.cm}$$



*컨트레버

$$M_{HC1} = M_{FG1} + M_{FH1} = 5.365 + 6.29 = 11.655 \text{ ton.cm}$$

8. BENDING STRESS

A. VERTICAL BENDING STRESS

1. PITCH 내

$$\sigma v1 = M_{max1} / Z_x = 247.8 / 849 = 0.292 \text{ ton/cm}^2$$

2. 컨트레버

$$\sigma v2 = M_{max2} / Z_x = 290.5 / 849 = 0.342 \text{ ton/cm}^2$$

B. 수평 최대 응력

1. PITCH 내

$$\sigma v3 = M_{HCG} / Z_y = 9.2415 / 118 = 0.078 \text{ ton/cm}^2$$

2. 컨트레버

$$\sigma v4 = M_{HC1} / Z_y = 11.655 / 118 = 0.099 \text{ ton/cm}^2$$

9. 합성 응력

$$\Sigma \sigma 1 = \sigma v1 + \sigma v3 = 0.370$$

$$0.370 < 1.279 \text{ ton/cm}^2 \quad \text{O.K}$$

$$\Sigma \sigma 2 = \sigma v2 + \sigma v4 = 0.441$$

$$0.441 < 1.6 \text{ ton/cm}^2 \quad \text{O.K}$$

용접 효율 : 80% 응력 $1391 \times 80\% \times 115\% = 1.279 \text{ ton/cm}^2$
 컨트레버는 용접부 없음 $1391 \times \text{작업시} 1.15 (\text{풍하중포함}) = 1.600 \text{ ton/cm}^2$

10. 휴지시 풍하중에 의한 BENDING MOMENT

휴지시 $V=45\text{m/s}$, $q = 158.0 \text{ kg/m}^2$ $h(\text{최고양정}) = 30 \text{ m}$
 $q = M \times \sqrt[4]{h} = 67.5 \times \sqrt[4]{30} = 158.0 \text{ kg/m}^2$
 $M = V^2 / 30 = 67.5$

*휴지시 풍하중에 의한 수평하중

PITCH내 풍하중 = $F \times L \times H \times q \times 1.7 = 1.11 \times 9.5 \times 0.25 \times 158 \times 1.7 = 708 \text{ kg}$

컨트레버에 대한 풍하중 = $F \times L_1 \times H \times q \times 1.4 = 1.11 \times 3.7 \times 0.25 \times 158 \times 1.4 = 227 \text{ kg}$

$$MM_{G1} = \frac{0.708 \times 950}{8} - \frac{0.227 \times 370}{2} = 42.08 \text{ ton.cm}$$

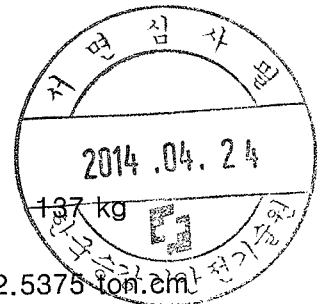
$$MM_1 = \frac{0.227 \times 370}{2} = 41.995 \text{ ton.cm}$$

HOIST에 대한 풍하중 = $F \times H_B \times H_H \times q \times 1.2 = 1.11 \times 0.65 \times 1 \times 158 \times 1.2 = 137 \text{ kg}$

*PITCH내 $MM_{H0} = 137 \times 950 / 4 = 130150 \text{ KG.CM} = 32.5375 \text{ ton.cm}$

*컨트레버 $MM_{H1} = 137 \times 370 = 50690 \text{ KG.CM} = 50.7 \text{ ton.cm}$

* COMBINED MOMENT



$$MM_2 = MM_{G1} + MM_{H0} = 42.08 + 32.5375 = 74.6175 \text{ ton.cm}$$

$$MM_4 = MM_1 + MM_{H1} = 41.995 + 50.7 = 92.70 \text{ ton.cm}$$

* BENDING STRESS

$$\Sigma\sigma_3 = MM_2 / Z_y = 74.6175 / 118 = 0.632 \text{ ton/cm}^2 < 1.447 \text{ ton/cm}^2 \text{---O.K}$$

$$\Sigma\sigma_4 = MM_4 / Z_y = 92.695 / 118 = 0.786 \text{ ton/cm}^2 < 1.808 \text{ ton/cm}^2 \text{---O.K}$$

PITCH내
켄틀레버는 용접부 없음

휴지시 응력 1391 x 80% x 130% =
휴지시 응력 1391 x 130% =

1447 ton/cm²
1808 ton/cm²

11. DEFLECTION OF GIRDER

* PITCH 내

1) DUE TO RATED & TROLLEY LOAD

$$.D1 = \frac{(Q+Q2)x(L)^3}{48xEx Ix} = 0.7769 \text{ cm}$$

2) DUE TO DEAD LOAD

$$.D2 = \frac{5 x Wgx L^3}{384 x E x Ix} = 0.2604 \text{ cm}$$

3) TOTAL DEFLECTION

.δ1 = D1+ D2 = 1.037 cm

RATIO : D3/L = 1/ 916 < 800 ----- O.K

* 켄틀레버

1) DUE TO RATED & TROLLEY LOAD

$$.D1 = \frac{(Q+Q2)x(L1-L2)^3}{3xEx Ix} = 0.5698 \text{ cm}$$

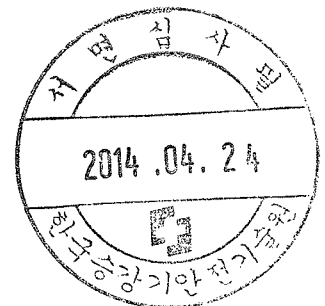
2) DUE TO DEAD LOAD

$$.D2 = \frac{Wg1x L1^3}{8 x E x Ix} = 0.0575 \text{ cm}$$

3) TOTAL DEFLECTION

.δ2 = D1+ D2 = 0.6273 cm

RATIO : D3/L = 1/ 590 < 500 ----- O.K

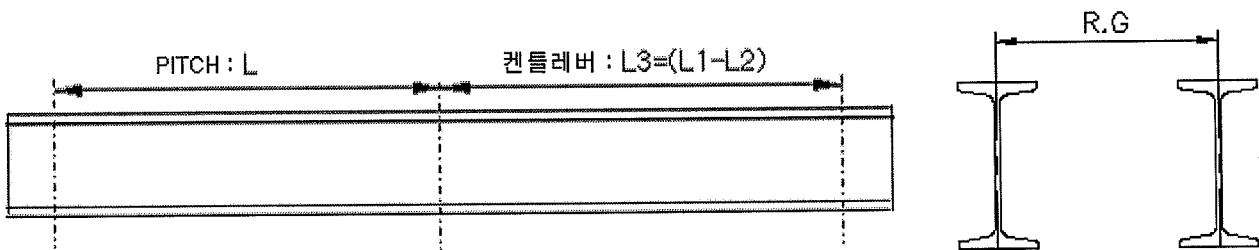


4. I-BEAM 최대허용 가능 SPAN 계산

1. SPECIFICATION

.정격하중	-----	Q =	1	ton
.SPAN (PITCH)	-----	L =	1100	cm
.켄틸리버	-----	L1 =	450	cm
.TROLLEY WHEEL BASE	-----	B =	13.2	cm
.WEIGHT OF HOIST	-----	Wh =	0.16	ton
.PITCH내의 GIRDER 무게	-----	Wg =	0.959	ton
.켄틸리버의 GIRDER 무게	-----	Wg1 =	0.392	ton
.I-BEAM SIZE	-----		350x150x12/24t	
		Ix =	22400	cm ⁴
		Iy =	1180	cm ⁴
		Zx =	1280	cm ³
		Zy =	158	cm ³
		A =	111.1	cm ²
		Wb =	87.2	kg/m
.HOOK APPROACH	-----	L2 =	30	cm
탄성계수	-----	E =	2100000	kg/cm ²
동하중 계수(작업계수)	-----	M(Φ) =	1.11	
정하중 계수(충격계수)	-----	F(Ψ) =	1.10	

1. DESIGN



2. ONE SIDE I-BEAM에 작용하는 하중

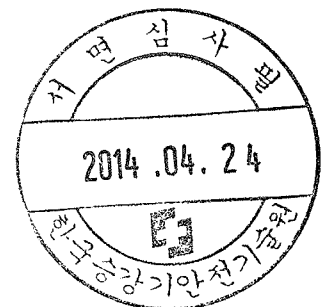
$$P = (Q + Wh)/2 = (1 + 0.16)/2 = 0.58 \text{ ton}$$

3. 수직하중에 의한 BENDING MOMENT

1) PITCH 지점내 BENDING MOMENT

$$Mh1 = F \times M \times P \times (L-B/2)^2 / (4 \times L)$$

$$= 1.11 \times 1.1 \times 0.58 \times (1100 - 13.2/2)^2 / (4 \times 1100)$$



$$= 192.4 \quad \text{ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$Mh2 = F \times M \times P \times L3$$

$$= 1.11 \times 1.1 \times 0.58 \times (450-30) = 297.44 \quad \text{ton.cm}$$

4. 자중에 의한 BENDING MOMENT(I-BEAM)

1) PITCH 지점내 BENDING MOMENT

$$Mt1 = F \times Wg \times L / 8 = 1.11 \times 0.959 \times 1100 / 8 = 146.367 \quad \text{ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$Mt2 = F \times Wg1 \times L1 / 2 = 1.11 \times 0.392 \times 450 / 2 = 97.9 \quad \text{ton.cm}$$

5. 수직하중에 의한 최대 BENDING MOMENT

$$Mmax1 = Mh1 + Mt1 = 338.8 \quad \text{ton.cm}$$

$$Mmax2 = Mh2 + Mt2 = 395.3 \quad \text{ton.cm}$$

6. 풍하중에 의한 수평하중의 BENDING MOMENT

(1) GIRDER에 의한 수직 풍하중

$$\text{작업시 } V=16\text{m/s}, q = 19.9 \text{ kg/m}^2 \quad h(\text{최고양정}) = 30 \text{ M}$$

$$\text{작업시 } q = 8.5 \times 10^{-4} \sqrt{h} = 8.5 \times 10^{-4} \sqrt{42} = 19.9 \text{ kg/m}^2$$

$$H(\text{GIRDER높이}) = 0.25 \text{ m}$$

$$\text{PITCH내 풍하중} = F \times L \times H \times q \times 1.7 = 1.11 \times 11 \times 0.25 \times 19.9 \times 1.7 = 103 \text{ kg}$$

$$\text{컨트레버 풍하중} = F \times L1 \times H \times q \times 1.4 = 1.11 \times 4.5 \times 0.25 \times 19.9 \times 1.4 = 35 \text{ kg}$$

$$M_{FGG} = \frac{0.103 \times 1100}{8} - \frac{0.035 \times 450}{2} = 6.288 \quad \text{ton.cm}$$

$$M_{FG1} = \frac{0.035 \times 450}{2} = 7.875 \quad \text{ton.cm}$$

(2) HOIST에 의한 풍하중

* 작업시

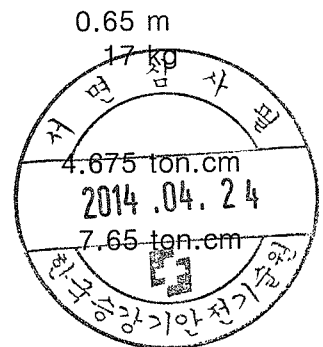
$$\text{풍하중} = F \times HB \times HH \times q \times 1.2 = 1.11 \times 0.65 \times 1 \times 19.9 \times 1.2 =$$

*PITCH내 풍하중

$$M_{FHG} = 17 \times 1100 / 4 = 4675 \text{ kg.cm} =$$

*컨트레버 풍하중

$$M_{FH1} = 17 \times 450 = 7650 \text{ kg.cm} =$$



7. COMBINED MOMENT

*PITCH내

$$M_{HCG} = M_{FGG} + M_{FHG} = 6.288 + 4.675 = 10.963 \text{ ton.cm}$$

20-14

*켄틀레버

$$M_{HC1} = M_{FG1} + M_{FH1} = 7.875 + 7.65 = 15.525 \text{ ton.cm}$$

8. BENDING STRESS

A. VERTICAL BENDING STRESS

1. PITCH 내

$$\sigma v1 = M_{max1} / Zx = 338.8 / 1280 = 0.265 \text{ ton/cm}^2$$

2. 켄틀레버

$$\sigma v2 = M_{max2} / Zx = 395.3 / 1280 = 0.309 \text{ ton/cm}^2$$

B. 수평 최대 응력

1. PITCH 내

$$\sigma v3 = M_{HCG} / Zy = 10.963 / 158 = 0.069 \text{ ton/cm}^2$$

2. 켄틀레버

$$\sigma v4 = M_{HC1} / Zy = 15.525 / 158 = 0.098 \text{ ton/cm}^2$$

9. 합성 응력

$$\Sigma \sigma 1 = \sigma v1 + \sigma v3 = 0.334$$

0.334	<	1.279	ton/cm ²	O.K
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$$\Sigma \sigma 2 = \sigma v2 + \sigma v4 = 0.407$$

0.407	<	1.6	ton/cm ²	O.K
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용접효율 : 80% 응력 1391x80% x 115% = 1.279 ton/cm²

켄틀레버는 용접부 없음 1391*작업시1.15(풍하중포함) = 1.600 ton/cm²

10. 휴지시 풍하중에 의한 BENDING MOMENT

휴지시 V=45m/s , q = 158.0 kg/m² h(최고양정) = 30 m

휴지시 q = M x ⁴√h = 67.5 x ⁴√30 = 158.0 kg/m²

M = V² / 30 = 67.5

*휴지시 풍하중에 의한 수평하중

PITCH내 풍하중 = F x L x H x q x 1.7 = 1.11x11x0.25x158x1.7 = 820 kg

켄틀레버에 대한 풍하중 = FxL1xHxqx1.4 = 1.11x4.5x0.25x158x1.4 = 276 kg

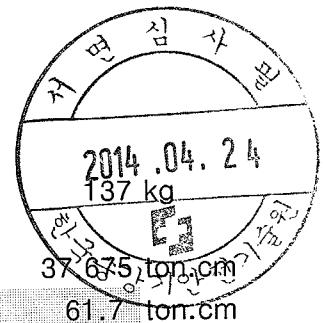
$$MM_{G1} = \frac{0.82 \times 1100}{8} + \frac{0.276 \times 450}{2} = 50.65 \text{ ton.cm}$$

$$MM_1 = \frac{0.276 \times 450}{2} = 62.1 \text{ ton.cm}$$

HOIST에 대한 풍하중 = FxHBxHHxqx1.2 = 1.11x0.65x1x158x1.2 =

*PITCH내 MM_{H0} = 137 x 1100 / 4 = 150700 KG.CM =

*켄틀레버 MM_{H1} = 137 x 450 = 61650 KG.CM =



37.675 ton.cm

61.7 ton.cm

* COMBINED MOMENT

$$MM_2 = MM_{G1} + MM_{H0} = 50.65 + 37.675 = 88.325 \text{ ton.cm}$$

$$MM_4 = MM_1 + MM_{H1} = 62.1 + 61.7 = 123.80 \text{ ton.cm}$$

* BENDING STRESS

$$\Sigma \sigma_3 = MM_2 / Z_y = 88.325 / 158 = 0.559 \text{ ton/cm}^2 < 1.447 \text{ ton/cm}^2 \text{---O.K}$$

$$\Sigma \sigma_4 = MM_4 / Z_y = 123.8 / 158 = 0.784 \text{ ton/cm}^2 < 1.808 \text{ ton/cm}^2 \text{---O.K}$$

PITCH내 휴지시 응력 1391 x 80% x 130% = 1447 ton/cm²
 캔틀레버는 용접부 없음 휴지시 응력 1391 x 130% = 1808 ton/cm²

11. DEFLECTION OF GIRDER

* PITCH 내

1) DUE TO RATED & TROLLEY LOAD

$$.D1 = \frac{(Q+Q2) \times (L)^3}{48 \times E \times I_x} = 0.6838 \text{ cm}$$

2) DUE TO DEAD LOAD

$$.D2 = \frac{5 \times W_g \times L^3}{384 \times E \times I_x} = 0.3533 \text{ cm}$$

3) TOTAL DEFLECTION

$$.d1 = D1 + D2 = 1.037 \text{ cm}$$

$$\text{RATIO : } D3/L = 1/1061 < 800 \text{ ---O.K}$$

* 캔틀레버

1) DUE TO RATED & TROLLEY LOAD

$$.D1 = \frac{(Q+Q2) \times (L1-L2)^3}{3 \times E \times I_x} = 0.6090 \text{ cm}$$

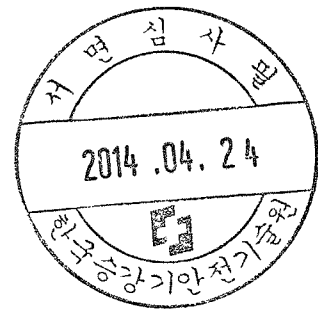
2) DUE TO DEAD LOAD

$$.D2 = \frac{W_{g1} \times L1^3}{8 \times E \times I_x} = 0.0949 \text{ cm}$$

3) TOTAL DEFLECTION

$$.d2 = D1 + D2 = 0.7039 \text{ cm}$$

$$\text{RATIO : } D3/L = 1/639 < 500 \text{ ---O.K}$$

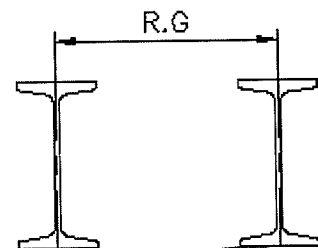
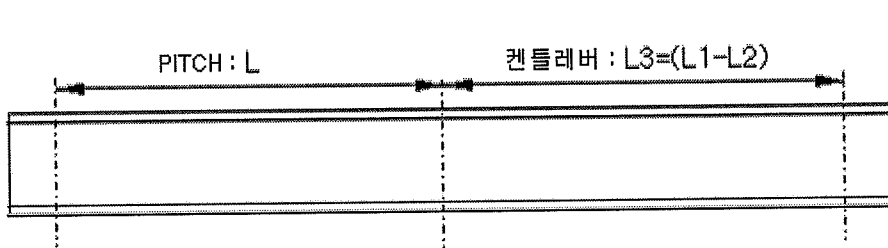


5. I-BEAM 최대허용 가능 SPAN 계산

1. SPECIFICATION

.정격하중	-----	Q =	1	ton
.SPAN (PITCH)	-----	L=	1200	cm
.켄틸리버	-----	L1=	500	cm
.TROLLEY WHEEL BASE	-----	B=	13.2	cm
.WEIGHT OF HOIST	-----	Wh=	0.16	ton
.PITCH내의 GIRDER 무게	-----	Wg=	1.15	ton
.켄틸리버의 GIRDER 무게	-----	Wg1=	0.479	ton
.I-BEAM SIZE	-----		400x150x12.5/25t	
		Ix =	31700	cm ⁴
		Iy =	1240	cm ⁴
		Zx =	1580	cm ³
		Zy =	165	cm ³
		A =	122.1	cm ²
		Wb =	95.8	kg/m
.HOOK APPROACH	-----	L2=	30	cm
탄성계수	-----	E=	2100000	kg/cm ²
동하중 계수(작업계수)	-----	M(Φ)=	1.11	
정하중 계수(충격계수)	-----	F(Ψ)=	1.10	

1. DESIGN



2. ONE SIDE I-BEAM에 작용하는 하중

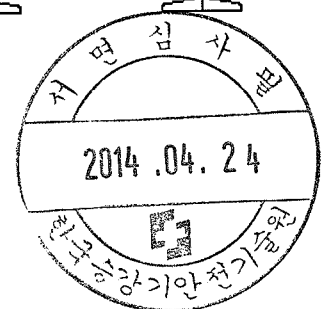
$$P = (Q + Wh)/2 = (1 + 0.16)/2 = 0.58 \text{ ton}$$

3. 수직하중에 의한 BENDING MOMENT

1) PITCH 지점내 BENDING MOMENT

$$M_{h1} = F \times M \times P \times (L-B/2)^2 / (4 \times L)$$

$$= 1.11 \times 1.1 \times 0.58 \times (1200 - 13.2/2)^2 / (4 \times 1200)$$



$$= 210.1 \quad \text{ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$Mh2 = F \times M \times P \times L3$$

$$= 1.11 \times 1.1 \times 0.58 \times (500-30) = 332.84 \quad \text{ton.cm}$$

4. 자중에 의한 BENDING MOMENT(I-BEAM)

1) PITCH 지점내 BENDING MOMENT

$$Mt1 = F \times Wg \times L / 8 = 1.11 \times 1.15 \times 1200 / 8 = 191.475 \quad \text{ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$Mt2 = F \times Wg1 \times L1 / 2 = 1.11 \times 0.479 \times 500 / 2 = 132.92 \quad \text{ton.cm}$$

5. 수직하중에 의한 최대 BENDING MOMENT

$$Mmax1 = Mh1 + Mt1 = 401.6 \quad \text{ton.cm}$$

$$Mmax2 = Mh2 + Mt2 = 465.8 \quad \text{ton.cm}$$

6. 풍하중에 의한 수평하중의 BENDING MOMENT

(1) GIRDER에 의한 수직 풍하중

$$\text{작업시 } V=16\text{m/s}, q = 19.9 \text{ kg/m}^2 \quad h(\text{최고양정}) = 30 \text{ m}$$

$$\text{작업시 } q = 8.5 \times 10^{-4} \sqrt{h} = 8.5 \times 10^{-4} \sqrt{42} = 19.9 \text{ kg/m}^2$$

$$H(\text{GIRDER높이}) = 0.25 \text{ m}$$

$$\text{PITCH내 풍하중} = F \times L \times H \times q \times 1.7 = 1.11 \times 12 \times 0.25 \times 19.9 \times 1.7 = 113 \text{ kg}$$

$$\text{캔틀레버 풍하중} = F \times L1 \times H \times q \times 1.4 = 1.11 \times 5 \times 0.25 \times 19.9 \times 1.4 = 39 \text{ kg}$$

$$M_{FGG} = \frac{0.113 \times 1200}{8} - \frac{0.039 \times 500}{2} = 7.2 \quad \text{ton.cm}$$

$$M_{FG1} = \frac{0.039 \times 500}{2} = 9.75 \quad \text{ton.cm}$$

(2) HOIST에 의한 풍하중

* 작업시

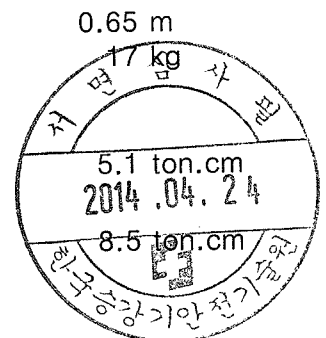
$$\text{풍하중} = F \times HB \times HH \times q \times 1.2 = 1.11 \times 0.65 \times 1 \times 19.9 \times 1.2 =$$

*PITCH내 풍하중

$$M_{FHG} = 17 \times 1200 / 4 = 5100 \text{ kg.cm} =$$

*캔틀레버 풍하중

$$M_{FH1} = 17 \times 500 = 8500 \text{ kg.cm} =$$



7. COMBINED MOMENT

*PITCH내

$$M_{HCG} = M_{FGG} + M_{FHG} = 7.2 + 5.1 = 12.300 \text{ ton.cm}$$

*컨틀레버

$$M_{HC1} = M_{FG1} + M_{FH1} = 9.75 + 8.5 = 18.250 \text{ ton.cm}$$

8. BENDING STRESS

A. VERTICAL BENDING STRESS

1. PITCH 내

$$\sigma_1 = M_{max1} / Z_x = 401.6 / 1580 = 0.254 \text{ ton/cm}^2$$

2. 컨틀레버

$$\sigma_2 = M_{max2} / Z_x = 465.8 / 1580 = 0.295 \text{ ton/cm}^2$$

B. 수평 최대 응력

1. PITCH 내

$$\sigma_3 = M_{HCG} / Z_y = 12.3 / 165 = 0.075 \text{ ton/cm}^2$$

2. 컨틀레버

$$\sigma_4 = M_{HC1} / Z_y = 18.25 / 165 = 0.111 \text{ ton/cm}^2$$

9. 합성 응력

$$\Sigma\sigma_1 = \sigma_1 + \sigma_3 = 0.329$$

0.329	<	1.279	ton/cm ²	O.K
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$$\Sigma\sigma_2 = \sigma_2 + \sigma_4 = 0.405$$

0.405	<	1.6	ton/cm ²	O.K
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용접효율 : 80% 응력 1391x80% x 115% = 1.279 ton/cm²

컨틀레버는 용접부 없음 1391*작업시1.15(풍하중포함) = 1.600 ton/cm²

10. 휴지시 풍하중에 의한 BENDING MOMENT

휴지시 V=45m/s, q = 158.0 kg/m² h(최고양정) = 30 m

휴지시 q = M x ⁴√h = 67.5 x ⁴√30 = 158.0 kg/m²

M = V² / 30 = 67.5

*휴지시 풍하중에 의한 수평하중

PITCH내 풍하중 = F x L x H x q x 1.7 = 1.11x12x0.25x158x1.7 = 894 kg

컨틀레버에 대한 풍하중 = FxL1xHxqx1.4 = 1.11x5x0.25x158x1.4 = 307 kg

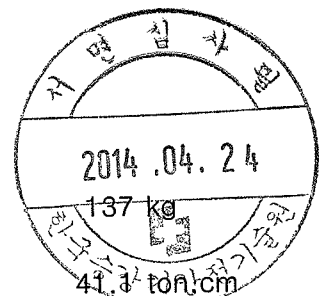
$$MM_{G1} = \frac{0.894 \times 1200}{8} - \frac{0.307 \times 500}{2} = 57.35 \text{ ton.cm}$$

$$MM_1 = \frac{0.307 \times 500}{2} = 76.75 \text{ ton.cm}$$

HOIST에 대한 풍하중 = FxHBxHHxqx1.2 = 1.11x0.65x1x158x1.2 =

*PITCH내 MM_{H0} = 137 x 1200 / 4 = 164400 KG.CM =

*컨틀레버 MM_{H1} = 137 x 500 = 68500 KG.CM =



68.5 ton.cm

* COMBINED MOMENT

$$MM_2 = MM_{G1} + MM_{H0} = 57.35 + 41.1 = 98.45 \text{ ton.cm}$$

$$MM_4 = MM_1 + MM_{H1} = 76.75 + 68.5 = 145.25 \text{ ton.cm}$$

* BENDING STRESS

$$\Sigma\sigma_3 = MM_2 / Z_y = 98.45 / 165 = 0.597 \text{ ton/cm}^2 < 1.447 \text{ ton/cm}^2 \text{---O.K}$$

$$\Sigma\sigma_4 = MM_4 / Z_y = 145.25 / 165 = 0.880 \text{ ton/cm}^2 < 1.808 \text{ ton/cm}^2 \text{---O.K}$$

PITCH내 휴지시 응력 $1391 \times 80\% \times 130\% = 1447 \text{ ton/cm}^2$
 캔틀레버는 용접부 없음 휴지시 응력 $1391 \times 130\% = 1808 \text{ ton/cm}^2$

11. DEFLECTION OF GIRDER

* PITCH 내

1) DUE TO RATED & TROLLEY LOAD

$$.D1 = \frac{(Q+Q2) \times (L)^3}{48 \times E \times I_x} = 0.6273 \text{ cm}$$

2) DUE TO DEAD LOAD

$$.D2 = \frac{5 \times W_g \times L^3}{384 \times E \times I_x} = 0.3887 \text{ cm}$$

3) TOTAL DEFLECTION

$$.s1 = D1 + D2 = 1.016 \text{ cm}$$

$$\text{RATIO : } D3/L = 1/1181 < 800 \text{ --- O.K}$$

* 캔틀레버

1) DUE TO RATED & TROLLEY LOAD

$$.D1 = \frac{(Q+Q2) \times (L1-L2)^3}{3 \times E \times I_x} = 0.6030 \text{ cm}$$

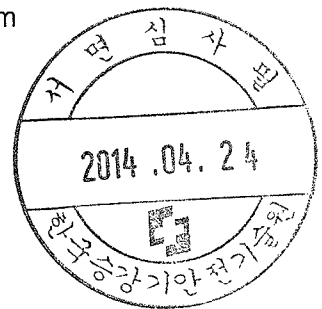
2) DUE TO DEAD LOAD

$$.D2 = \frac{W_g \times L1^3}{8 \times E \times I_x} = 0.1124 \text{ cm}$$

3) TOTAL DEFLECTION

$$.s2 = D1 + D2 = 0.7154 \text{ cm}$$

$$\text{RATIO : } D3/L = 1/699 < 500 \text{ --- O.K}$$

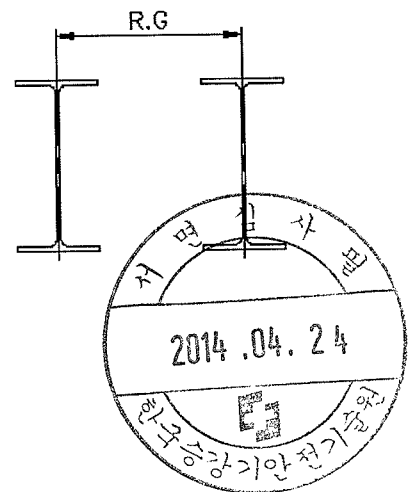
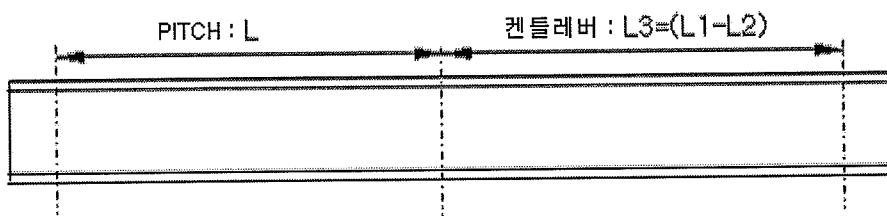


6. H-BEAM 최대허용 가능 SPAN 계산

1. SPECIFICATION

.정격하중	-----	Q =	1	ton
.SPAN (PITCH)	-----	L =	800	cm
.켄틸리버	-----	L1 =	300	cm
.TROLLEY WHEEL BASE	-----	B =	13.2	cm
.WEIGHT OF HOIST	-----	Wh =	0.16	ton
.PITCH내의 GIRDER 무게	-----	Wg =	0.256	ton
.켄틸리버의 GIRDER 무게	-----	Wg1 =	0.096	ton
.H-BEAM SIZE	-----		300x150x6.5/9t	
		Ix =	7210	cm ⁴
		Iy =	508	cm ⁴
		Zx =	481	cm ³
		Zy =	67.7	cm ³
		A =	40.8	cm ²
		Wb =	32	kg/m
.HOOK APPROACH	-----	L2 =	30	cm
탄성계수	-----	E =	2100000	kg/cm ²
동하중 계수(작업계수)	-----	M(Φ) =	1.11	
정하중 계수(충격계수)	-----	F(Ψ) =	1.10	

1. DESIGN



2. ONE SIDE H-BEAM에 작용하는 하중

$$P = (Q + Wh)/2 = (1 + 0.16)/2 = 0.58 \text{ ton}$$

3. 수직하중에 의한 BENDING MOMENT

1) PITCH 지점내 BENDING MOMENT

$$Mh1 = F \times M \times P \times (L-B/2)^2 / (4 \times L)$$

$$= 1.11 \times 1.1 \times 0.58 \times (800 - 13.2/2)^2 / (4 \times 800)$$

$$= 139.3 \quad \text{ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$M_{h2} = F \times M \times P \times L3$$

$$= 1.11 \times 1.1 \times 0.58 \times (300-30) = 191.21 \quad \text{ton.cm}$$

4. 자중에 의한 BENDING MOMENT(I-BEAM)

1) PITCH 지점내 BENDING MOMENT

$$M_{t1} = F \times Wg \times L / 8 = 1.11 \times 0.256 \times 800 / 8 = 28.416 \quad \text{ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$M_{t2} = F \times Wg1 \times L1 / 2 = 1.11 \times 0.096 \times 300 / 2 = 15.98 \quad \text{ton.cm}$$

5. 수직하중에 의한 최대 BENDING MOMENT

$$M_{max1} = M_{h1} + M_{t1} = 167.7 \quad \text{ton.cm}$$

$$M_{max2} = M_{h2} + M_{t2} = 207.2 \quad \text{ton.cm}$$

6. 풍하중에 의한 수평하중의 BENDING MOMENT

(1) GIRDER에 의한 수직 풍하중

$$\text{작업시 } V=16\text{m/s}, q = 19.9 \text{ kg/m}^2 \quad h(\text{최고양정}) = 30 \text{ m}$$

$$\text{작업시 } q = 8.5 \times \sqrt{h} = 8.5 \times \sqrt{42} = 19.9 \text{ kg/m}^2$$

$$H(\text{GIRDER높이}) = 0.25 \text{ m}$$

$$\text{PITCH내 풍하중} = F \times L \times H \times q \times 1.6 = 1.11 \times 8 \times 0.25 \times 19.9 \times 1.6 = 71 \text{ kg}$$

$$\text{컨트레버 풍하중} = F \times L1 \times H \times q \times 1.3 = 1.11 \times 3 \times 0.25 \times 19.9 \times 1.3 = 22 \text{ kg}$$

$$M_{FGG} = \frac{0.071 \times 800}{8} - \frac{0.022 \times 300}{2} = 3.8 \quad \text{ton.cm}$$

$$M_{FG1} = \frac{0.022 \times 300}{2} = 3.3 \quad \text{ton.cm}$$

(2) HOIST에 의한 풍하중

* 작업시

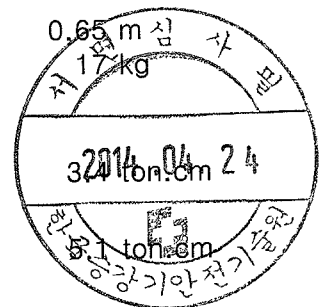
$$\text{풍하중} = F \times HB \times HH \times q \times 1.2 = 1.11 \times 0.65 \times 1 \times 19.9 \times 1.2 =$$

*PITCH내 풍하중

$$M_{FHG} = 17 \times 800 / 4 = 3400 \text{ kg.cm} =$$

*컨트레버 풍하중

$$M_{FH1} = 17 \times 300 = 5100 \text{ kg.cm} =$$



7. COMBINED MOMENT

*PITCH내

$$M_{HCG} = M_{FGG} + M_{FHG} = 3.8 + 3.4 = 7.200 \text{ ton.cm}$$

*켄틀레버

$$M_{HC1} = M_{FG1} + M_{FH1} = 3.3 + 5.1 = 8.400 \text{ ton.cm}$$

8. BENDING STRESS

A. VERTICAL BENDING STRESS

1. PITCH 내

$$\sigma v1 = M_{max1} / Z_x = 167.7 / 481 = 0.349 \text{ ton/cm}^2$$

2. 켄틀레버

$$\sigma v2 = M_{max2} / Z_x = 207.2 / 481 = 0.431 \text{ ton/cm}^2$$

B. 수평 최대 응력

1. PITCH 내

$$\sigma v3 = M_{HCG} / Z_y = 7.2 / 67.7 = 0.106 \text{ ton/cm}^2$$

2. 켄틀레버

$$\sigma v4 = M_{HC1} / Z_y = 8.4 / 67.7 = 0.124 \text{ ton/cm}^2$$

9. 합성 응력

$$\Sigma \sigma 1 = \sigma v1 + \sigma v3 = 0.455$$

0.455	<	1.279	ton/cm ²	O.K
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$$\Sigma \sigma 2 = \sigma v2 + \sigma v4 = 0.555$$

0.555	<	1.6	ton/cm ²	O.K
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용접효율 : 80% 응력 1391x80% x 115% = 1.279 ton/cm²

켄틀레버는 용접부 없음 1391*작업시1.15(풍하중포함) = 1.600 ton/cm²

10. 휴지시 풍하중에 의한 BENDING MOMENT

휴지시 V=45m/s, q = 158.0 kg/m² h(최고양정) = 30 m

휴지시 q = M x ⁴√h = 67.5 x ⁴√30 = 158.0 kg/m²

M = V² / 30 = 67.5

*휴지시 풍하중에 의한 수평하중

PITCH내 풍하중 = F x L x H x q x 1.6 = 1.11x8x0.25x158x1.6 = 561 kg

켄틀레버에 대한 풍하중 = F x L₁ x H x q x 1.3 = 1.11x3x0.25x158x1.3 = 171 kg

$$MM_{G1} = \frac{0.561 \times 800}{8} + \frac{0.171 \times 300}{2} = 30.45 \text{ ton.cm}$$

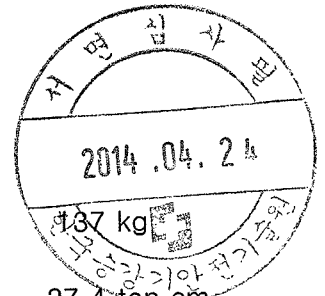
$$MM_1 = \frac{0.171 \times 300}{2} = 25.65 \text{ ton.cm}$$

HOIST에 대한 풍하중 = F x H_B x H_H x q x 1.2 = 1.11x0.65x1x158x1.2 =

*PITCH내 MM_{H0} = 137 x 800 / 4 = 109600 KG.CM = 27.4 ton.cm

*켄틀레버 MM_{H1} = 137 x 300 = 41100 KG.CM = 41.1 ton.cm

* COMBINED MOMENT



$$MM_2 = MM_{G1} + MM_{H0} = 30.45 + 27.4 = 57.85 \text{ ton.cm}$$

$$MM_4 = MM_1 + MM_{H1} = 25.65 + 41.1 = 66.75 \text{ ton.cm}$$

* BENDING STRESS

$$\Sigma\sigma_3 = MM_2 / Z_y = 57.85 / 67.7 = 0.855 \text{ ton/cm}^2 < 1.447 \text{ ton/cm}^2 \text{---O.K}$$

$$\Sigma\sigma_4 = MM_4 / Z_y = 66.75 / 67.7 = 0.986 \text{ ton/cm}^2 < 1.808 \text{ ton/cm}^2 \text{---O.K}$$

PITCH내 휴지시 응력 1391 x 80% x 130% = 1447 ton/cm²
 켄틀레버는 용접부 없음 휴지시 응력 1391 x 130% = 1808 ton/cm²

11. DEFLECTION OF GIRDER

* PITCH 내

1) DUE TO RATED & TROLLEY LOAD

$$.D1 = \frac{(Q+Q2) \times (L)^3}{48 \times E \times I_x} = 0.8172 \text{ cm}$$

2) DUE TO DEAD LOAD

$$.D2 = \frac{5 \times W_g \times L^3}{384 \times E \times I_x} = 0.1127 \text{ cm}$$

3) TOTAL DEFLECTION

$$.s1 = D1 + D2 = 0.930 \text{ cm}$$

$$\text{RATIO : } D3/L = 1/860 < 800 \text{ --- O.K}$$

* 켄틀레버

1) DUE TO RATED & TROLLEY LOAD

$$.D1 = \frac{(Q+Q2) \times (L1-L2)^3}{3 \times E \times I_x} = 0.5027 \text{ cm}$$

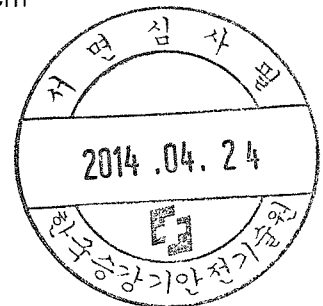
2) DUE TO DEAD LOAD

$$.D2 = \frac{W_{g1} \times L1^3}{8 \times E \times I_x} = 0.0214 \text{ cm}$$

3) TOTAL DEFLECTION

$$.s2 = D1 + D2 = 0.5241 \text{ cm}$$

$$\text{RATIO : } D3/L = 1/572 < 500 \text{ --- O.K}$$

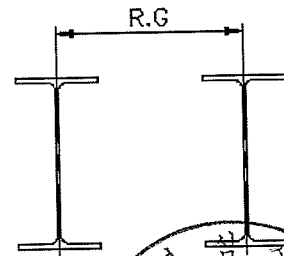
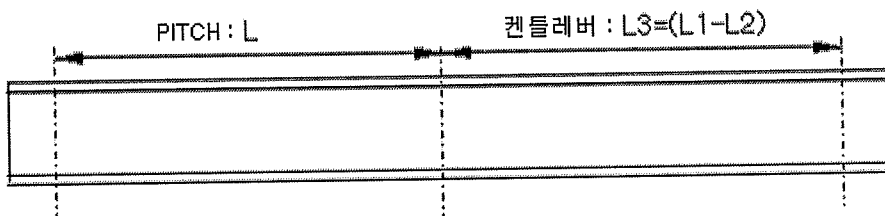


7. H-BEAM 최대허용 가능 SPAN 계산

1. SPECIFICATION

.정격하중	-----	Q =	1	ton
.SPAN (PITCH)	-----	L =	1000	cm
.켄틸리버	-----	L1 =	400	cm
.TROLLEY WHEEL BASE	-----	B =	13.2	cm
.WEIGHT OF HOIST	-----	Wh =	0.16	ton
.PITCH내의 GIRDER 무게	-----	Wg =	0.494	ton
.켄틸리버의 GIRDER 무게	-----	Wg1 =	0.198	ton
.H-BEAM SIZE	-----		350x175x7/11t	
		Ix =	13500	cm ⁴
		Iy =	984	cm ⁴
		Zx =	771	cm ³
		Zy =	112	cm ³
		A =	62.91	cm ²
		Wb =	49.4	kg/m
.HOOK APPROACH	-----	L2 =	30	cm
탄성계수	-----	E =	2100000	kg/cm ²
동하중 계수(작업계수)	-----	M(Φ) =	1.11	
정하중 계수(충격계수)	-----	F(Ψ) =	1.10	

1. DESIGN



2. ONE SIDE H-BEAM에 작용하는 하중

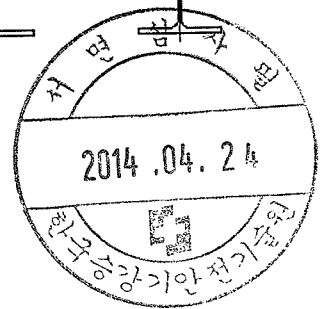
$$P = (Q + Wh)/2 = (1 + 0.16)/2 = 0.58 \text{ ton}$$

3. 수직하중에 의한 BENDING MOMENT

1) PITCH 지점내 BENDING MOMENT

$$M_{h1} = F \times M \times P \times (L-B/2)^2 / (4 \times L)$$

$$= 1.11 \times 1.1 \times 0.58 \times (1000 - 13.2/2)^2 / (4 \times 1000)$$



9022

$$= 174.7 \quad \text{ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$Mh2 = F \times M \times P \times L3$$

$$= 1.11 \times 1.1 \times 0.58 \times (400-30) = 262.03 \quad \text{ton.cm}$$

4. 자중에 의한 BENDING MOMENT(I-BEAM)

1) PITCH 지점내 BENDING MOMENT

$$Mt1 = F \times Wg \times L / 8 = 1.11 \times 0.494 \times 1000 / 8 = 68.543 \quad \text{ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$Mt2 = F \times Wg1 \times L1 / 2 = 1.11 \times 0.198 \times 400 / 2 = 43.96 \quad \text{ton.cm}$$

5. 수직하중에 의한 최대 BENDING MOMENT

$$Mmax1 = Mh1 + Mt1 = 243.3 \quad \text{ton.cm}$$

$$Mmax2 = Mh2 + Mt2 = 306 \quad \text{ton.cm}$$

6. 풍하중에 의한 수평하중의 BENDING MOMENT

(1) GIRDER에 의한 수직 풍하중

$$\text{작업시 } V=16\text{m/s}, q = 19.9 \text{ kg/m}^2 \quad h(\text{최고양정}) = 30 \text{ m}$$

$$\text{작업시 } q = 8.5 \times \sqrt{h} = 8.5 \times \sqrt{42} = 19.9 \text{ kg/m}^2$$

$$H(\text{GIRDER높이}) = 0.25 \text{ m}$$

$$\text{PITCH내 풍하중} = F \times L \times H \times q \times 1.7 = 1.11 \times 10 \times 0.25 \times 19.9 \times 1.7 = 94 \text{ kg}$$

$$\text{캔틀레버 풍하중} = F \times L1 \times H \times q \times 1.4 = 1.11 \times 4 \times 0.25 \times 19.9 \times 1.4 = 31 \text{ kg}$$

$$M_{FGG} = \frac{0.094 \times 1000}{8} - \frac{0.031 \times 400}{2} = 5.55 \quad \text{ton.cm}$$

$$M_{FG1} = \frac{0.031 \times 400}{2} = 6.2 \quad \text{ton.cm}$$

(2) HOIST에 의한 풍하중

* 작업시

$$HH = 1.0 \text{ m} \quad HB = 0.65 \text{ m}$$

$$\text{풍하중} = F \times HB \times HH \times q \times 1.2 = 1.11 \times 0.65 \times 1 \times 19.9 \times 1.2 =$$

*PITCH내 풍하중

$$M_{FHG} = 17 \times 1000 / 4 = 4250 \text{ kg.cm} =$$

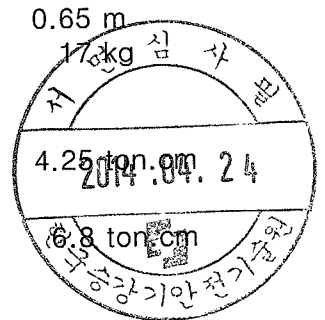
*캔틀레버 풍하중

$$M_{FH1} = 17 \times 400 = 6800 \text{ kg.cm} =$$

7. COMBINED MOMENT

*PITCH내

$$M_{HCG} = M_{FGG} + M_{FHG} = 5.55 + 4.25 = 9.800 \text{ ton.cm}$$



*컨트레버

$$M_{HC1} = M_{FG1} + M_{FH1} = 6.2 + 6.8 = 13.000 \text{ ton.cm}$$

8. BENDING STRESS

A. VERTICAL BENDING STRESS

1. PITCH 내

$$\sigma v1 = M_{max1} / Z_x = 243.3 / 771 = 0.316 \text{ ton/cm}^2$$

2. 컨트레버

$$\sigma v2 = M_{max2} / Z_x = 306 / 771 = 0.397 \text{ ton/cm}^2$$

B. 수평 최대 응력

1. PITCH 내

$$\sigma v3 = M_{HCG} / Z_y = 9.8 / 112 = 0.088 \text{ ton/cm}^2$$

2. 컨트레버

$$\sigma v4 = M_{HC1} / Z_y = 13 / 112 = 0.116 \text{ ton/cm}^2$$

9. 합성 응력

$$\Sigma \sigma 1 = \sigma v1 + \sigma v3 = 0.403$$

0.403	<	1.279	ton/cm ²	☞	O.K
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$$\Sigma \sigma 2 = \sigma v2 + \sigma v4 = 0.513$$

0.513	<	1.6	ton/cm ²	☞	O.K
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용접효율 : 80% 응력 1391x80% x 115% = 1.279 ton/cm²

컨트레버는 용접부 없음 1391*작업시1.15(풍하중포함) = 1.600 ton/cm²

10. 휴지시 풍하중에 의한 BENDING MOMENT

휴지시 V=45m/s , q = 158.0 kg/m ²	h(최고양정) = 30 m
휴지시 q = M x ⁴ √h = 67.5 x ⁴ √30 = 158.0 kg/m ²	
M = V ² / 30 = 67.5	

*휴지시 풍하중에 의한 수평하중

PITCH내 풍하중 = F x L x H x q x 1.7 = 1.11x10x0.25x158x1.7 = 745 kg

컨트레버에 대한 풍하중 = FxL1xHxqx1.4 = 1.11x4x0.25x158x1.4 = 246 kg

$$MM_{G1} = \frac{0.745 \times 1000}{8} + \frac{0.246 \times 400}{2} = 43.925 \text{ ton.cm}$$

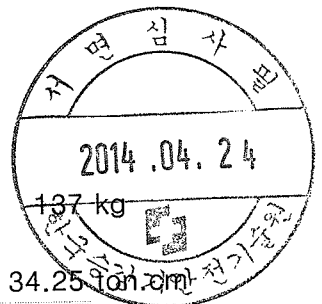
$$MM_1 = \frac{0.246 \times 400}{2} = 49.2 \text{ ton.cm}$$

HOIST에 대한 풍하중 = FxHBxHHxqx1.2 = 1.11x0.65x1x158x1.2 = 137 kg

*PITCH내 MM_{H0} = 137 x 1000 / 4 = 34250 KG.CM = 34.25 ton.cm

*컨트레버 MM_{H1} = 137 x 400 = 54800 KG.CM = 54.8 ton.cm

* COMBINED MOMENT



2014

$$MM_2 = MM_{G1} + MM_{H0} = 43.925 + 34.25 = 78.175 \text{ ton.cm}$$

$$MM_4 = MM_1 + MM_{H1} = 49.2 + 54.8 = 104.00 \text{ ton.cm}$$

* BENDING STRESS

$$\Sigma\sigma_3 = MM_2 / Z_y = 78.175 / 112 = 0.698 \text{ ton/cm}^2 < 1.447 \text{ ton/cm}^2 \text{---O.K}$$

$$\Sigma\sigma_4 = MM_4 / Z_y = 104 / 112 = 0.929 \text{ ton/cm}^2 < 1.808 \text{ ton/cm}^2 \text{---O.K}$$

PITCH내 휴지시 응력 1391 x 80% x 130% = 1447 ton/cm²

컨트레버는 용접부 없음 휴지시 응력 1391 x 130% = 1808 ton/cm²

11. DEFLECTION OF GIRDER

* PITCH 내

1) DUE TO RATED & TROLLEY LOAD

$$.D1 = \frac{(Q+Q2)x(L)^3}{48xEx Ix} = 0.8524 \text{ cm}$$

2) DUE TO DEAD LOAD

$$.D2 = \frac{5 x Wgx L^3}{384 x E x Ix} = 0.2269 \text{ cm}$$

3) TOTAL DEFLECTION

$$.d1 = D1 + D2 = 1.079 \text{ cm}$$

$$\text{RATIO : } D3/L = 1 / 927 < 800 \text{ --- O.K}$$

* 컨트레버

1) DUE TO RATED & TROLLEY LOAD

$$.D1 = \frac{(Q+Q2)x(L1-L2)^3}{3xEx Ix} = 0.6909 \text{ cm}$$

2) DUE TO DEAD LOAD

$$.D2 = \frac{Wg1x L1^3}{8 x E x Ix} = 0.0559 \text{ cm}$$

3) TOTAL DEFLECTION

$$.d2 = D1 + D2 = 0.7468 \text{ cm}$$

$$\text{RATIO : } D3/L = 1 / 536 < 500 \text{ --- O.K}$$

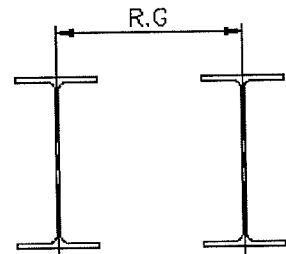
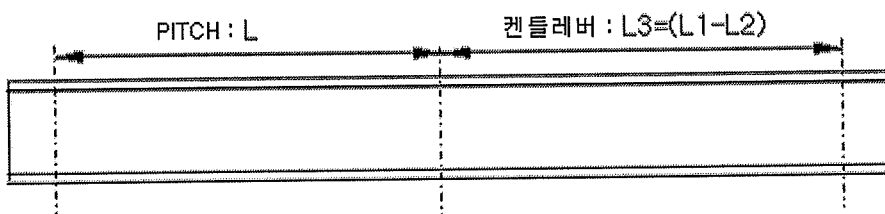


8. H-BEAM 최대허용 가능 SPAN 계산

1. SPECIFICATION

.정격하중	-----	Q =	1	ton
.SPAN (PITCH)	-----	L=	1100	cm
.켄틸리버	-----	L1=	450	cm
.TROLLEY WHEEL BASE	-----	B=	13.2	cm
.WEIGHT OF HOIST	-----	Wh=	0.16	ton
.PITCH내의 GIRDER 무게	-----	Wg=	0.719	ton
.켄틸리버의 GIRDER 무게	-----	Wg1=	0.294	ton
.H-BEAM SIZE	-----		400x200x8/13t	
		Ix =	23500	cm ⁴
		Iy =	1740	cm ⁴
		Zx =	1170	cm ³
		Zy =	174	cm ³
		A =	83.37	cm ²
		Wb =	65.4	kg/m
.HOOK APPROACH	-----	L2=	30	cm
탄성계수	-----	E=	2100000	kg/cmf
동하중 계수(작업계수)	-----	M(Φ)=	1.11	
정하중 계수(충격계수)	-----	F(Ψ)=	1.10	

1. DESIGN



2. ONE SIDE H-BEAM에 작용하는 하중

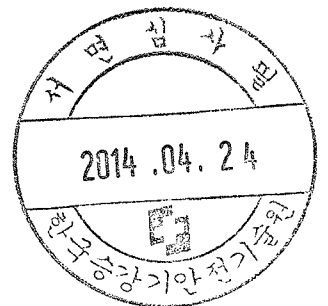
$$P = (Q + Wh)/2 = (1 + 0.16)/2 = 0.58 \text{ ton}$$

3. 수직하중에 의한 BENDING MOMENT

1) PITCH 지점내 BENDING MOMENT

$$Mh1 = F \times M \times P \times (L-B/2)^2 / (4 \times L)$$

$$= 1.11 \times 1.1 \times 0.58 \times (1100 - 13.2/2)^2 / (4 \times 1100)$$



$$= 192.4 \text{ ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$Mh2 = F \times M \times P \times L3$$

$$= 1.11 \times 1.1 \times 0.58 \times (450 - 30) = 297.44 \text{ ton.cm}$$

4. 자중에 의한 BENDING MOMENT(I-BEAM)

1) PITCH 지점내 BENDING MOMENT

$$Mt1 = F \times Wg \times L / 8 = 1.11 \times 0.719 \times 1100 / 8 = 109.737 \text{ ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$Mt2 = F \times Wg1 \times L1 / 2 = 1.11 \times 0.294 \times 450 / 2 = 73.43 \text{ ton.cm}$$

5. 수직하중에 의한 최대 BENDING MOMENT

$$Mmax1 = Mh1 + Mt1 = 302.2 \text{ ton.cm}$$

$$Mmax2 = Mh2 + Mt2 = 370.9 \text{ ton.cm}$$

6. 풍하중에 의한 수평하중의 BENDING MOMENT

(1) GIRDER에 의한 수직 풍하중

$$\text{작업시 } V=16\text{m/s}, q = 19.9 \text{ kg/m}^2 \quad h(\text{최고양정}) = 30 \text{ m}$$

$$\text{작업시 } q = 8.5 \times \sqrt{h} = 8.5 \times \sqrt{42} = 19.9 \text{ kg/m}^2$$

$$H(\text{GIRDER높이}) = 0.25 \text{ m}$$

$$\text{PITCH내 풍하중} = F \times L \times H \times q \times 1.6 = 1.11 \times 11 \times 0.25 \times 19.9 \times 1.6 = 97 \text{ kg}$$

$$\text{캔틀레버 풍하중} = F \times L1 \times H \times q \times 1.4 = 1.11 \times 4.5 \times 0.25 \times 19.9 \times 1.4 = 35 \text{ kg}$$

$$M_{FGG} = \frac{0.097 \times 1100}{8} - \frac{0.035 \times 450}{2} = 5.463 \text{ ton.cm}$$

$$M_{FG1} = \frac{0.035 \times 450}{2} = 7.875 \text{ ton.cm}$$

(2) HOIST에 의한 풍하중

* 작업시

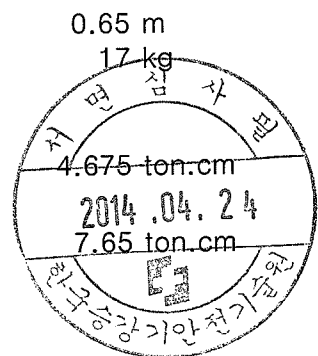
$$\text{풍하중} = F \times HB \times HH \times q \times 1.2 = 1.11 \times 0.65 \times 1 \times 19.9 \times 1.2 =$$

*PITCH내 풍하중

$$M_{FHG} = 17 \times 1100 / 4 = 4675 \text{ kg.cm} =$$

*캔틀레버 풍하중

$$M_{FH1} = 17 \times 450 = 7650 \text{ kg.cm} =$$



7. COMBINED MOMENT

*PITCH내

$$M_{HCG} = M_{FGG} + M_{FHG} = 5.463 + 4.675 = 10.138 \text{ ton.cm}$$

90-21

*컨트레버

$$M_{HC1} = M_{FG1} + M_{FH1} = 7.875 + 7.65 = 15.525 \text{ ton.cm}$$

8. BENDING STRESS

A. VERTICAL BENDING STRESS

1. PITCH 내

$$\sigma v1 = M_{max1} / Zx = 302.2 / 1170 = 0.258 \text{ ton/cm}^2$$

2. 컨트레버

$$\sigma v2 = M_{max2} / Zx = 370.9 / 1170 = 0.317 \text{ ton/cm}^2$$

B. 수평 최대 응력

1. PITCH 내

$$\sigma v3 = M_{HCG} / Zy = 10.138 / 174 = 0.058 \text{ ton/cm}^2$$

2. 컨트레버

$$\sigma v4 = M_{HC1} / Zy = 15.525 / 174 = 0.089 \text{ ton/cm}^2$$

9. 합성 응력

$$\Sigma \sigma 1 = \sigma v1 + \sigma v3 = 0.317$$

$$0.317 < 1.279 \text{ ton/cm}^2 \quad \text{O.K}$$

$$\Sigma \sigma 2 = \sigma v2 + \sigma v4 = 0.406$$

$$0.406 < 1.6 \text{ ton/cm}^2 \quad \text{O.K}$$

용접효율 : 80% 응력 $1391 \times 80\% \times 115\% = 1.279 \text{ ton/cm}^2$
 컨트레버는 용접부 없음 $1391 \times \text{작업시} 1.15 (\text{풍하중포함}) = 1.600 \text{ ton/cm}^2$

10. 휴지시 풍하중에 의한 BENDING MOMENT

휴지시 $V=45\text{m/s}$, $q = 158.0 \text{ kg/m}^2$ $h(\text{최고양정}) = 30 \text{ m}$
 휴지시 $q = M \times \sqrt[4]{h} = 67.5 \times \sqrt[4]{30} = 158.0 \text{ kg/m}^2$
 $M = V^2 / 30 = 67.5$

*휴지시 풍하중에 의한 수평하중

PITCH내 풍하중 = $F \times L \times H \times q \times 1.6 = 1.11 \times 11 \times 0.25 \times 158 \times 1.6 = 772 \text{ kg}$

컨트레버에 대한 풍하중 = $F \times L1 \times H \times q \times 1.4 = 1.11 \times 4.5 \times 0.25 \times 158 \times 1.4 = 276 \text{ kg}$

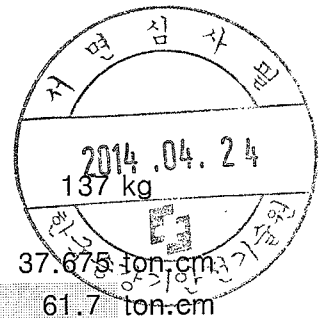
$$MM_{G1} = \frac{0.772 \times 1100}{8} - \frac{0.276 \times 450}{2} = 44.05 \text{ ton.cm}$$

$$MM_1 = \frac{0.276 \times 450}{2} = 62.1 \text{ ton.cm}$$

HOIST에 대한 풍하중 = $F \times HB \times HH \times q \times 1.2 = 1.11 \times 0.65 \times 1 \times 158 \times 1.2 =$

*PITCH내 $MM_{H0} = 137 \times 1100 / 4 = 150700 \text{ KG.CM} = 37.675 \text{ ton.cm}$

*컨트레버 $MM_{H1} = 137 \times 450 = 61650 \text{ KG.CM} = 61.7 \text{ ton.cm}$



* COMBINED MOMENT

$$\begin{aligned} MM_2 &= MM_{G1} + MM_{H0} = 44.05 + 37.675 = 81.725 \text{ ton.cm} \\ MM_4 &= MM_1 + MM_{H1} = 62.1 + 61.7 = 123.80 \text{ ton.cm} \end{aligned}$$

* BENDING STRESS

$$\begin{aligned} \Sigma\sigma_3 &= MM_2 / Zy = 81.725 / 174 = 0.470 \text{ ton/cm}^2 < 1.447 \text{ ton/cm}^2 \text{---O.K} \\ \Sigma\sigma_4 &= MM_4 / Zy = 123.8 / 174 = 0.711 \text{ ton/cm}^2 < 1.808 \text{ ton/cm}^2 \text{---O.K} \end{aligned}$$

PITCH내 휴지시 응력 1391 x 80% x 130% = 1447 ton/cm²
 캔틀레버는 용접부 없음 휴지시 응력 1391 x 130% = 1808 ton/cm²

11. DEFLECTION OF GIRDER

* PITCH 내

1) DUE TO RATED & TROLLEY LOAD

$$.D1 = \frac{(Q+Q2)x(L)^3}{48xEx Ix} = 0.6518 \text{ cm}$$

2) DUE TO DEAD LOAD

$$.D2 = \frac{5 \times Wgx L^3}{384 \times E \times Ix} = 0.2525 \text{ cm}$$

3) TOTAL DEFLECTION

$$.d1 = D1 + D2 = 0.904 \text{ cm}$$

$$\text{RATIO : } D3/L = 1/1216 < 800 \text{ ---O.K}$$

* 캔틀레버

1) DUE TO RATED & TROLLEY LOAD

$$.D1 = \frac{(Q+Q2)x(L1-L2)^3}{3xEx Ix} = 0.5805 \text{ cm}$$

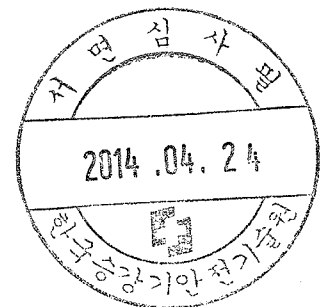
2) DUE TO DEAD LOAD

$$.D2 = \frac{Wg1x L1^3}{8 \times E \times Ix} = 0.0679 \text{ cm}$$

3) TOTAL DEFLECTION

$$.d2 = D1 + D2 = 0.6484 \text{ cm}$$

$$\text{RATIO : } D3/L = 1/694 < 500 \text{ ---O.K}$$

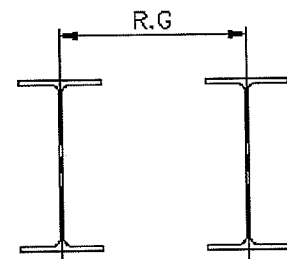
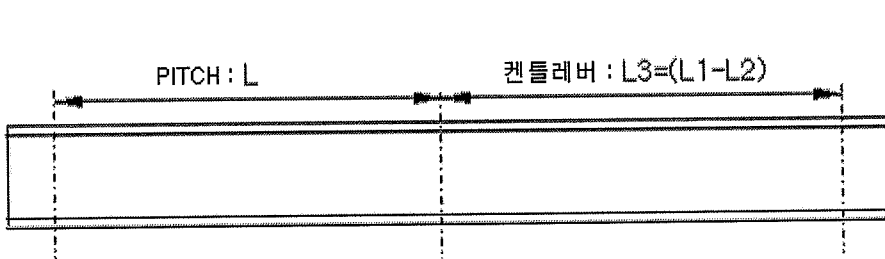


9. H-BEAM 최대허용 가능 SPAN 계산

1. SPECIFICATION

.정격하중	-----	Q =	1	ton
.SPAN (PITCH)	-----	L=	1200	cm
.켄틸리버	-----	L1=	500	cm
.TROLLEY WHEEL BASE	-----	B=	13.2	cm
.WEIGHT OF HOIST	-----	Wh=	0.16	ton
.PITCH내의 GIRDER 무게	-----	Wg=	0.899	ton
.켄틸리버의 GIRDER 무게	-----	Wg1=	0.375	ton
.H-BEAM SIZE	-----		450x200x9/14t	
		Ix =	32900	cm ⁴
		Iy =	1870	cm ⁴
		Zx =	1460	cm ³
		Zy =	187	cm ³
		A =	95.43	cm ²
		Wb =	74.9	kg/m
.HOOK APPROACH	-----	L2=	30	cm
탄성계수	-----	E=	2100000	kg/cm ²
동하중 계수(작업계수)	-----	M(Φ) =	1.11	
정하중 계수(충격계수)	-----	F(Ψ) =	1.10	

1. DESIGN



2. ONE SIDE H-BEAM에 작용하는 하중

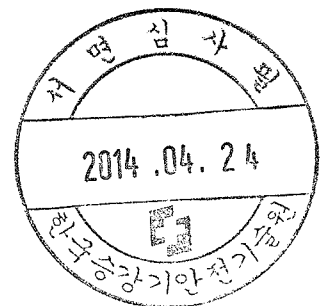
$$P = (Q + Wh)/2 = (1 + 0.16)/2 = 0.58 \text{ ton}$$

3. 수직하중에 의한 BENDING MOMENT

1) PITCH 지점내 BENDING MOMENT

$$Mh1 = F \times M \times P \times (L-B/2)^2 / (4 \times L)$$

$$= 1.11 \times 1.1 \times 0.58 \times (1200 - 13.2/2)^2 / (4 \times 1200)$$



$$= 210.1 \quad \text{ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$M_{h2} = F \times M \times P \times L3$$

$$= 1.11 \times 1.1 \times 0.58 \times (500-30) = 332.84 \quad \text{ton.cm}$$

4. 자중에 의한 BENDING MOMENT(I-BEAM)

1) PITCH 지점내 BENDING MOMENT

$$M_{t1} = F \times Wg \times L / 8 = 1.11 \times 0.899 \times 1200 / 8 = 149.684 \quad \text{ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$M_{t2} = F \times Wg1 \times L1 / 2 = 1.11 \times 0.375 \times 500 / 2 = 104.06 \quad \text{ton.cm}$$

5. 수직하중에 의한 최대 BENDING MOMENT

$$M_{max1} = M_{h1} + M_{t1} = 359.8 \quad \text{ton.cm}$$

$$M_{max2} = M_{h2} + M_{t2} = 436.9 \quad \text{ton.cm}$$

6. 풍하중에 의한 수평하중의 BENDING MOMENT

(1) GIRDER에 의한 수직 풍하중

$$\text{작업시 } V=16\text{m/s}, q = 19.9 \text{ kg/m}^2 \quad h(\text{최고양정}) = 30 \text{ m}$$

$$\text{작업시 } q = 8.5 \times \sqrt{h} = 8.5 \times \sqrt{42} = 19.9 \text{ kg/m}^2$$

$$H(\text{GIRDER높이}) = 0.25 \text{ m}$$

$$\text{PITCH내 풍하중} = F \times L \times H \times q \times 1.6 = 1.11 \times 12 \times 0.25 \times 19.9 \times 1.6 = 106 \text{ kg}$$

$$\text{캔틀레버 풍하중} = F \times L1 \times H \times q \times 1.4 = 1.11 \times 5 \times 0.25 \times 19.9 \times 1.4 = 39 \text{ kg}$$

$$M_{FGG} = \frac{0.106 \times 1200}{8} + \frac{0.039 \times 500}{2} = 6.15 \quad \text{ton.cm}$$

$$M_{FG1} = \frac{0.039 \times 500}{2} = 9.75 \quad \text{ton.cm}$$

(2) HOIST에 의한 풍하중

* 작업시

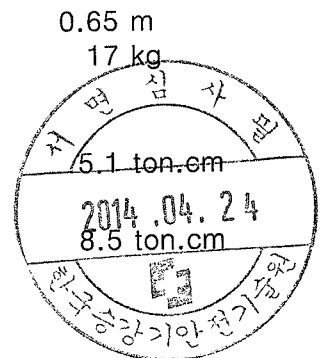
$$\text{풍하중} = F \times HB \times HH \times q \times 1.2 = 1.11 \times 0.65 \times 1 \times 19.9 \times 1.2 =$$

*PITCH내 풍하중

$$M_{FHG} = 17 \times 1200 / 4 = 5100 \text{ kg.cm} =$$

*캔틀레버 풍하중

$$M_{FH1} = 17 \times 500 = 8500 \text{ kg.cm} =$$



7. COMBINED MOMENT

*PITCH내

$$M_{HCG} = M_{FGG} + M_{FHG} = 6.15 + 5.1 = 11.250 \text{ ton.cm}$$

*켄틀레버

$$M_{HC1} = M_{FG1} + M_{FH1} = 9.75 + 8.5 = 18.250 \text{ ton.cm}$$

8. BENDING STRESS

A. VERTICAL BENDING STRESS

1. PITCH 내

$$\sigma_1 = M_{max1} / Z_x = 359.8 / 1460 = 0.246 \text{ ton/cm}^2$$

2. 켄틀레버

$$\sigma_2 = M_{max2} / Z_x = 436.9 / 1460 = 0.299 \text{ ton/cm}^2$$

B. 수평 최대 응력

1. PITCH 내

$$\sigma_3 = M_{HCG} / Z_y = 11.25 / 187 = 0.060 \text{ ton/cm}^2$$

2. 켄틀레버

$$\sigma_4 = M_{HC1} / Z_y = 18.25 / 187 = 0.098 \text{ ton/cm}^2$$

9. 합성 응력

$$\Sigma\sigma_1 = \sigma_1 + \sigma_3 = 0.307$$

0.307	<	1.279	ton/cm ²	O.K
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$$\Sigma\sigma_2 = \sigma_2 + \sigma_4 = 0.397$$

0.397	<	1.6	ton/cm ²	O.K
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용접효율 : 80% 응력 1391x80% x 115% = 1.279 ton/cm²
 켄틀레버는 용접부 없음 1391*작업시1.15(풍하중포함) = 1.600 ton/cm²

10. 휴지시 풍하중에 의한 BENDING MOMENT

휴지시 V=45m/s, q = 158.0 kg/m² h(최고양정) = 30 m
 휴지시 q = M x ⁴√h = 67.5 x ⁴√30 = 158.0 kg/m²
 M = V² / 30 = 67.5

*휴지시 풍하중에 의한 수평하중

PITCH내 풍하중 = F x L x H x q x 1.6 = 1.11x12x0.25x158x1.6 = 842 kg

켄틀레버에 대한 풍하중 = F x L₁ x H x q x 1.4 = 1.11x5x0.25x158x1.4 = 307 kg

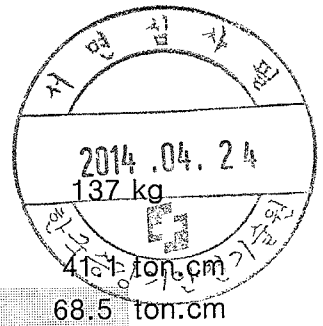
$$MM_{G1} = \frac{0.842 \times 1200}{8} - \frac{0.307 \times 500}{2} = 49.55 \text{ ton.cm}$$

$$MM_1 = \frac{0.307 \times 500}{2} = 76.75 \text{ ton.cm}$$

HOIST에 대한 풍하중 = F x H_B x H_H x q x 1.2 = 1.11x0.65x1x158x1.2 =

*PITCH내 MM_{H0} = 137 x 1200 / 4 = 164400 KG.CM =

*켄틀레버 MM_{H1} = 137 x 500 = 68500 KG.CM =



* COMBINED MOMENT

$$\begin{aligned} MM_2 &= MM_{G1} + MM_{H0} = 49.55 + 41.1 = 90.65 \text{ ton.cm} \\ MM_4 &= MM_1 + MM_{H1} = 76.75 + 68.5 = 145.25 \text{ ton.cm} \end{aligned}$$

* BENDING STRESS

$$\begin{aligned} \Sigma\sigma_3 &= MM_2 / Z_y = 90.65 / 187 = 0.485 \text{ ton/cm}^2 < 1.447 \text{ ton/cm}^2 \text{---O.K} \\ \Sigma\sigma_4 &= MM_4 / Z_y = 145.25 / 187 = 0.777 \text{ ton/cm}^2 < 1.808 \text{ ton/cm}^2 \text{---O.K} \end{aligned}$$

PITCH내 휴지시 응력 1391 x 80% x 130% = 1447 ton/cm²
 캔틀레버는 용접부 없음 휴지시 응력 1391 x 130% = 1808 ton/cm²

11. DEFLECTION OF GIRDER

* PITCH 내

1) DUE TO RATED & TROLLEY LOAD

$$.D1 = \frac{(Q+Q2)x(L)^3}{48xEx Ix} = 0.6044 \text{ cm}$$

2) DUE TO DEAD LOAD

$$.D2 = \frac{5 \times Wgx L^3}{384 \times E \times Ix} = 0.2928 \text{ cm}$$

3) TOTAL DEFLECTION

$$.d1 = D1 + D2 = 0.897 \text{ cm}$$

$$\text{RATIO : } D3/L = 1/1337 < 800 \text{ ---O.K}$$

* 캔틀레버

1) DUE TO RATED & TROLLEY LOAD

$$.D1 = \frac{(Q+Q2)x(L1-L2)^3}{3xEx Ix} = 0.5811 \text{ cm}$$

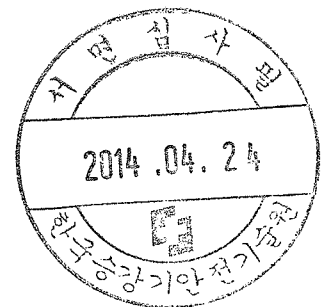
2) DUE TO DEAD LOAD

$$.D2 = \frac{Wg1x L1^3}{8 \times E \times Ix} = 0.0848 \text{ cm}$$

3) TOTAL DEFLECTION

$$.d2 = D1 + D2 = 0.6659 \text{ cm}$$

$$\text{RATIO : } D3/L = 1/751 < 500 \text{ ---O.K}$$



SUSPENSION MONORAIL 볼트 및 용접강도계산

1. 볼트로 고정시의 강도계산

* 허용 최대 SPAN 및 하중은 최악의 조건으로 계산한다.
(허용 최대 스팬 12 M, 400x150x12.5/25t, HOIST자중 : 160 KG)

.Q = RATED LOAD= 1000 KG
.Q1=HOIST SELF WEIGHT= 160 KG
.Q2=I-BEAM WEIGHT = 1150 KG

BOLT 재질 :	H.T.B 9.8 이상 사용할것
BOLT 허용전단응력 :	2100 KG/CM ²

M14 일때 do= 1.1835 CM n= 4.4 . n = 유효산수(산수*80% 너트1종너트사용)
M16 일때 do= 1.3835 CM n= 5.2
M18 일때 do= 1.5294 CM n= 4.8
M20 일때 do= 1.7294 CM n= 5.1

$$P = 1.14*(Q+Q1)/2 + 1.1* Q2/2$$

$$= 1.14*(1000+160)/2 + 1.1* 1150/2$$

$$= 1294 \text{ KG}$$

$$(\sigma = 2100 \text{ KG/CM}^2)$$

$$4 \times P$$

1)인장(전단)강도 : $\sigma = P/A ; \sigma = \frac{4 \times P}{\pi \times do^2 \times Z}$ * $Z = \frac{4 \times P}{\pi \times do^2 \times \sigma}$

M16 일때 $Z = \frac{4 \times P}{\pi \times do^2 \times \sigma} = \frac{4 \times 1294}{835^2 \times 2100} = 0.41 \text{ 개}$

M18 일때 $Z = \frac{4 \times P}{\pi \times do^2 \times \sigma} = \frac{4 \times 1294}{294^2 \times 2100} = 0.34 \text{ 개}$

M20 일때 $Z = \frac{4 \times P}{\pi \times do^2 \times \sigma} = \frac{4 \times 1294}{294^2 \times 2100} = 0.26 \text{ 개}$

2)접촉 면압 강도 ($\sigma_a = 400 \text{ KG/CM}^2$)

$$\sigma = \frac{4 \times P}{\pi \times (d^2 - do^2) \times Z \times n} ; \quad Z = \frac{4 \times P}{\pi \times (d^2 - do^2) \times \sigma \times n}$$

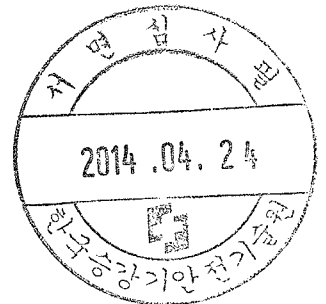
M14 일때 $Z = \frac{4 \times 1294}{\pi \times (1.4^2 - 1.1835^2) \times 400 \times 4.4} = 1.67 \text{ 개}$

M16 일때 $Z = \frac{4 \times 1294}{\pi \times (1.6^2 - 1.3835^2) \times 400 \times 5.2} = 1.23 \text{ 개}$

M18 일때 $Z = \frac{4 \times 1294}{\pi \times (1.8^2 - 1.5294^2) \times 400 \times 4.8} = 0.95 \text{ 개}$

M20 일때 $Z = \frac{4 \times 1294}{\pi \times (2^2 - 1.7294^2) \times 400 \times 5.1} = 0.8 \text{ 개}$

. do = 골경, P=브라켓 한 개에 작용하는 하중, Z = 볼트수
. n = 유효산수(산수*80%), 너트1종너트사용



90-34

3) 따라서 다음과 같이 적용한다

H.T.B M14, M16, M18, M20일때 : 브라켓트당 4개 이상 사용한다.

2. I-빔을 용접시공으로 고정시의 강도계산

* 허용 최대 SPAN 및 하중은 최악의 조건으로 계산한다.

(허용 최대 스팬 12 M, 400x150x12.5/25t, HOIST자중 : 160 KG)

.Q = RATED LOAD=	1000 KG	(h : 용접두께
.Q1=HOIST SELF WEIGHT=	160 KG	L : 용접길이)
.Q2=I-BEAM WEIGHT =	1150 KG	

1) 용접이음부 인장강도 계산 (용접두께 : 45° 용접부위)

$$\sigma = \frac{1.414 \times P}{h \times L} ; (\sigma_a = 1200 \text{ KG/CM}^2) \quad L = \frac{1.414 \times P}{h \times \sigma}$$

(1) h 가 5일때

$$L = \frac{1.414 \times 1294}{0.5 \times 1200} = 3.05 \text{ CM} = 30.5 \text{ mm}$$

(2) h 가 6일때

$$L = \frac{1.414 \times 1294}{0.6 \times 1200} = 2.54 \text{ CM} = 25.4 \text{ mm}$$

(3) h 가 7일때

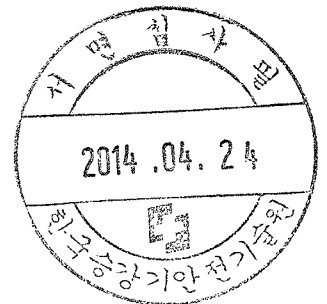
$$L = \frac{1.414 \times 1294}{0.7 \times 1200} = 2.18 \text{ CM} = 21.8 \text{ mm}$$

2) 적용

h = 5일때, 한 브라켓트당 용접길이 L = 50mm 이상 용접한다.

h = 6일때, 한 브라켓트당 용접길이 L = 40mm 이상 용접한다.

h = 7일때, 한 브라켓트당 용접길이 L = 30mm 이상 용접한다.



Date: 2009/04/14

Certificate of Compliance

We certify that the ER2 protection degrees conform to the IP rating as follows:

Hoist body - IP55 based on JIS C 4034-5, "Rotating electrical machines – Part5: Classification of degrees of protection provided by enclosures of rotating electrical machines (IP code)".

Push button - IP65 based on JIS C 0920, "Tests to prove protection against ingress of water and degrees of protection against ingress of solid objects for electrical equipment".

Technical Control Group

Test Certificate

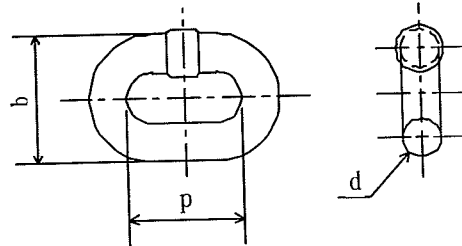
Messrs.

Commodity: NC Load Chain

Code: KER077

Lot No.: —

Quantity: — line(s)



1. Material: Manganese Alloy Steel

2. Dimensions

	d	p	b
Specified	7.7mm ± 0.3	21.4mm $\begin{matrix} +0.46 \\ 0 \end{matrix}$	Max. 27.0mm
Result	Good	Good	Good

3. Breaking test

	Breaking load	Total ultimate elongation
Specified	Min. 75 (kN)	Min. 10 (%)
Result	Good	Good

4. Manufacturing Proof force test (Test load: 46.6 kN)

	Permanent elongation
Specified	0.25 (%)
Result	Good

General judgment: Satisfactory



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Quality Assurance Group
Quality Assurance Department
Development & Technology Division

K. Kishimoto (Manager)

Certificate No.: MM080011c

Date of Issue: 2009/3/4

Messrs. _____

Motor Test Report for Electric Chain Hoist

Motor type : Three phase squirrel cage type induction motor.

Manufacturer : Yasukawa Electric Mfg. Co.

Production No. : -

Rating

Model	Output	Pole	Intermittent Rating	Voltage	Frequency
IBQ	0.9kW	4P	60%ED	220V	60Hz

Full load characteristics

Voltage	Frequency	220V	60Hz
Load	%	100	
Current	A	4.7	
Speed	rpm	1660	

Insulation class E

The above characteristics are obtained from calculation where the motor is assembled with an electric chain hoist and the hoist is subjected to full load



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M. Ogihara (Manager)

Certificate No.: MM080011d

Date of Issue: 2009/3/4

Messrs. _____

Motor Test Report for Electric Chain Hoist

Motor type : Three phase squirrel cage type induction motor.

Manufacturer : Yasukawa Electric Mfg. Co.

Production No. : -

Rating

Model	Output	Pole	Intermittent Rating	Voltage	Frequency
IBQ	0.9kW	4P	40/20%ED	220V	Speed Control by Inverter

Full load characteristics

Voltage	Frequency	220V	Speed Control by Inverter
Load	%		100
Current	A		5.7
Speed	rpm		~

Insulation class E

The above characteristics are obtained from calculation where the motor is assembled with an electric chain hoist and the hoist is subjected to full load



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Quality Assurance Group
Quality Assurance Department
Development & Technology Division

M. Ogihara (Manager)

Certificate No.: MM070011c

Date of Issue: 2008/03/21

Messrs. _____

Motor Test Report for Electric Chain Hoist

Motor type : Three phase squirrel cage type induction motor.

Manufacturer : Yasukawa Electric Mfg. Co.

Production No. :

Rating

Model	Output	Pole	Intermittent Rating	Voltage	Frequency
IBQ	0.9kW	4P	60%ED	380 - 440V	60Hz

Full load characteristics

Voltage Frequency	380 - 440V 60Hz	
Load	%	100
Current	A	2.6
Speed	rpm	1640

Insulation class B

The above characteristics are obtained from calculation where the motor is assembled with an electric chain hoist and the hoist is subjected to full load



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(Manager)

K. Kishimoto

Certificate No.: MM070011d

Date of Issue: 2008/03/21

Messrs. _____

Motor Test Report for Electric Chain Hoist

Motor type : Three phase squirrel cage type induction motor.

Manufacturer : Yasukawa Electric Mfg. Co.

Production No. :

Rating

Model	Output	Pole	Intermittent Rating	Voltage	Frequency
IBQ	0.9kW	4P	60%ED	380 - 440V	Speed Control by Inverter

Full load characteristics

Voltage	Frequency	380 - 440V	Speed Control by Inverter
Load	%		100
Current	A		3.6
Speed	rpm		~

Insulation class B

The above characteristics are obtained from calculation where the motor is assembled with an electric chain hoist and the hoist is subjected to full load



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Date of Issue: 2009/3/4

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Motor Test Report for Electric Trolley

Motor type : Three phase squirrel cage type induction motor.

Manufacturer : Yasukawa Electric Mfg. Co.

Production No. : -

Rating

Model	Output	Pole	Intermittent Rating	Voltage	Frequency
IBQ-T	0.4kW	4P	40%ED	220V	60Hz

Full load characteristics

Voltage	Frequency	220V 60Hz
Load	%	100
Current	A	3.0
Speed	rpm	1685

Insulation class E

The above characteristics are obtained from calculation where the motor is assembled with an electric trolley and the trolley is subjected to full load



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Date of Issue: 2009/3/4

Messrs. _____

Motor Test Report for Electric Trolley

Motor type : Three phase squirrel cage type induction motor.

Manufacturer : Yasukawa Electric Mfg. Co.

Production No. : -

Rating

Model	Output	Pole	Intermittent Rating	Voltage	Frequency
IBQ-T	0.4kW	4P	27/13%ED	220V	Speed Control by Inverter

Full load characteristics

Voltage	Frequency	220V	Speed Control by Inverter
Load	%		100
Current	A		3.0
Speed	rpm		~

Insulation class E

The above characteristics are obtained from calculation where the motor is assembled with an electric trolley and the trolley is subjected to full load



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(Manager)

Certificate No.: MM070013a

Date of Issue: 2008/03/21

Messrs. _____

Motor Test Report for End Carriage

Motor type : Three phase squirrel cage type induction motor.

Manufacturer : Yasukawa Electric Mfg. Co.

Production No. :

Rating

Model	Output	Pole	Intermittent Rating	Voltage	Frequency
IBQ-T	0.4kW	4P	40%ED	380 - 440V	60Hz

Full load characteristics

Voltage	Frequency	380 - 440V 60Hz
Load	%	100
Current	A	2.2
Speed	rpm	1670

Insulation class B

The above characteristics are obtained from calculation where the motor is assembled with an electric chain hoist and the hoist is subjected to full load



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(Manager)

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Date of Issue: 2008/03/21

Messrs. _____

Motor Test Report for End Carriage

Motor type : Three phase squirrel cage type induction motor.

Manufacturer : Yasukawa Electric Mfg. Co.

Production No. :

Rating

Model	Output	Pole	Intermittent Rating	Voltage	Frequency
IBQ-T	0.4kW	4P	40%ED	380 - 440V	Speed Control by Inverter

Full load characteristics

Voltage	Frequency	220 - 230V	Speed Control by Inverter
Load	%		100
Current	A		2.5
Speed	rpm		~

Insulation class B

The above characteristics are obtained from calculation where the motor is assembled with an electric chain hoist and the hoist is subjected to full load



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