

■ 산업안전보건법 시행규칙 [별지 제10호의 5서식]



심사결과 통지서

신청인	사업장명	(주)KITO	사업장관리번호	2010E110010
	사업자등록번호	010-E1-10010	대표자 성명	KITO YOSHIO
	소재지	2000, Tsujiarai, Showa-Cho, Nakakoma-Gun, Yamanashi, Japan		

안전인증대상기계·기구명	호이스트
형식(규격)	KML-ER2-018

용량(등급) 1.8 ton

「산업안전보건법」 제34조 및 같은 법 시행규칙 제58조의4제4항에 따라 실시한

[] 예비심사

[■] 서면심사

[] 기술능력 및 생산체계 심사 결과가 [■] 적합 함을 통지합니다.
[] 부적합

[] 개별 제품심사

[] 형식별 제품심사

2014년 03월 31일

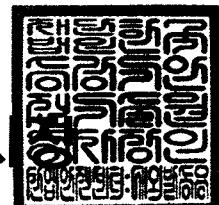
인증심사원

최 창 일

이정우 (인)

이정우

이정우 (인)



한국승강기안전기술원 이사장

【별지 제4호서식】

동일형식 일람표

사업장명	KITO CORP.	개정일자 및 번호	2014.03.21	인증번호	
형식 및 모델번호	동일형식 항목 및 내역				
형식번호	모델번호	동일형식 항목1	동일형식 항목2	동일형식 항목3	동일형식 항목4
KML-ER2-018	KITO-ER2-018L	Lift max 30m	권상모터 1.8kW	횡행모터 없음	Trolley 고정형
	KITO-ER2-018IL		권상모터 1.8kW		Trolley 있음
	KITO-ER2SP018L		권상모터 1.8kW		Trolley + 수동체인
	KITO-ER2SP018IL		권상모터 1.8kW		전기Trolley 결합 type
	KITO-ER2SG018L		권상모터 1.8kW		
	KITO-ER2SG018IL		권상모터 1.8kW		
	KITO-ER2M018L-S		권상모터 1.8kW		
	KITO-ER2M018L-L		권상모터 1.8kW		
	KITO-ER2M018L-IS		권상모터 1.8kW		
	KITO-ER2M018L-IL		권상모터 1.8kW		
	KITO-ER2M018IL-S		권상모터 1.8kW		전기Trolley 결합 Clean type
	KITO-ER2M018IL-L		권상모터 1.8kW		
	KITO-ER2M018IL-IS		권상모터 1.8kW		
	KITO-ER2M018IL-IL		권상모터 1.8kW		
	KITO-C-ER2M018L-S		권상모터 1.8kW		
	KITO-C-ER2M018L-L		권상모터 1.8kW		
	KITO-C-ER2M018L-IS		권상모터 1.8kW		
	KITO-C-ER2M018L-IL		권상모터 1.8kW		
	KITO-C-ER2M018IL-S		권상모터 1.8kW		
	KITO-C-ER2M018IL-L		권상모터 1.8kW		
	KITO-C-ER2M018IL-IS		권상모터 1.8kW		2014.03.21 전기Trolley 결합 Clean 바라 type
	KITO-C-ER2M018IL-IL		권상모터 1.8kW		
	KITO-CZ-ER2M018L-S		권상모터 1.8kW		
	KITO-CZ-ER2M018L-L		권상모터 1.8kW		
	KITO-CZ-ER2M018L-IS		권상모터 1.8kW		



안 전 인 증 서

정호엔지니어링

경기도 광명시 노온사동 440-5

위 사업장에서 제조하는 아래의 품목이 산업안전보건법 제34조 및 같은 법 시행규칙 제58조의4제4항에 따른 안전인증 심사 결과 안전·보건기준에 적합하므로 안전인증표시의 사용을 인증합니다.

품 목

양중기용 과부하방지장치

형식·모델/용량·등급/인증번호

형식·모델	용량·등급	인증번호
JDL-100	J-2	12-AV2BJ-0009

인 증 기 준

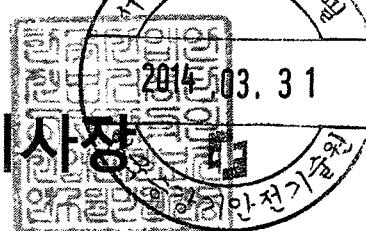
방호장치 의무안전인증 고시(고용노동부고시 제2010-36호)

인 증 조 건

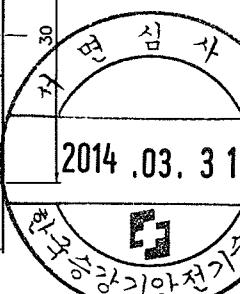
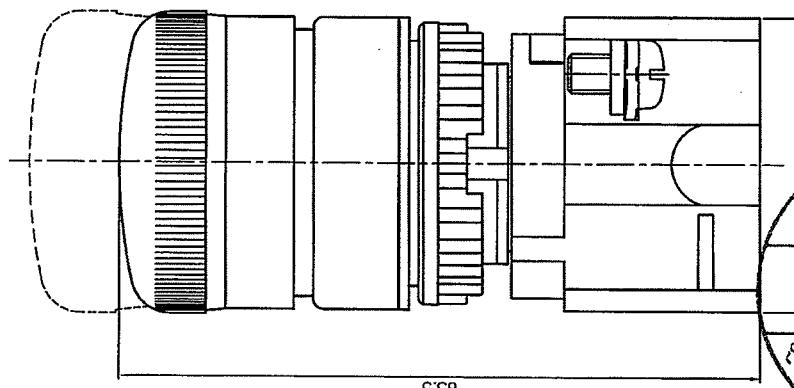
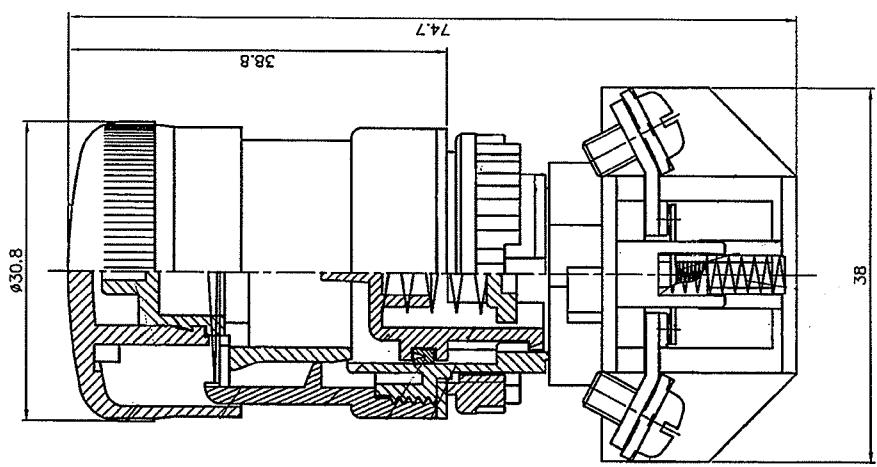
아래 주소에서 생산되는 제품에 한함.

정호엔지니어링, 경기도 광명시 노온사동 440-5

2012년 06월 11일



한국산업안전보건공단 이사장



一般公差		0.2~ ^{0.3} _{0.1}	30mm	± 0.2	30.1~360mm: ± 0.3	60.1~300mm: ± 0.5	视图标注	模具体积	模具体积	单位	mm	材质	图號	T2-BKH
天得科科技股份有限公司		最新修正			品保	保制	研磨机 95.05.24	研磨机 95.05.24	研磨机 95.05.24	比例	2:1	表面處理		
TEND TECHNOLOGY CO.,LTD.		前次修正					研磨机 95.05.24	研磨机 95.05.24	研磨机 95.05.24	投影法	◎	顏色	品名	T2-BKH 連鎖開關
14/2014.03.31											3		2	
14/2014.03.31											4			

面示:A

A

B

C

D

E

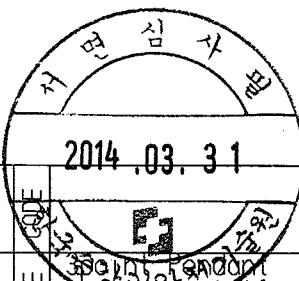
F

G

H

Revision	Incidence	Description		Date	Charge	Approved

⑥	⑤	④	③	②	①	NOTE	
APPROVED							
ISHIKAWA							
08.02.08							
CHECKED							
FURIYA							
08.02.08							
DESIGNED							
KOBAYASHI							
08.02.08							
DRAWN							
KOBAYASHI							
08.02.08							
SCALE							
-							
DWG. NO., MATERIAL							
NAME							
CODE							
2014.03.31							
SPECIAL STANDARD control station MX subassembly							
SWD2X00AA1							
Date issued							



Revision	Incidence	Description	Date	Charge	Approved
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A
B
C
D
E
F
G
H

The lifting and lowering push buttons are marked with for single speed or for dual speed.
The traveling push buttons are marked with E W or N S depending on the installed direction.

⑥	NOTE				
⑤					
④					
③					
②					
①					
APPROVED		H.FURIYA	CHECKED	T.HATANO	DESIGNED
		09.04.21	09.04.21	09.04.21	KOBAYASHI
Date issued		DRAIN		KOBAYASHI	SCALE
		09.04.21		09.04.21	-
Dwg. No.		S W D 2 X X 0 A A 1			
MATERIAL		NAME: 100E 50mtr Pendant Control Station MXX subassembly			
DATE		2014.03.31			

2

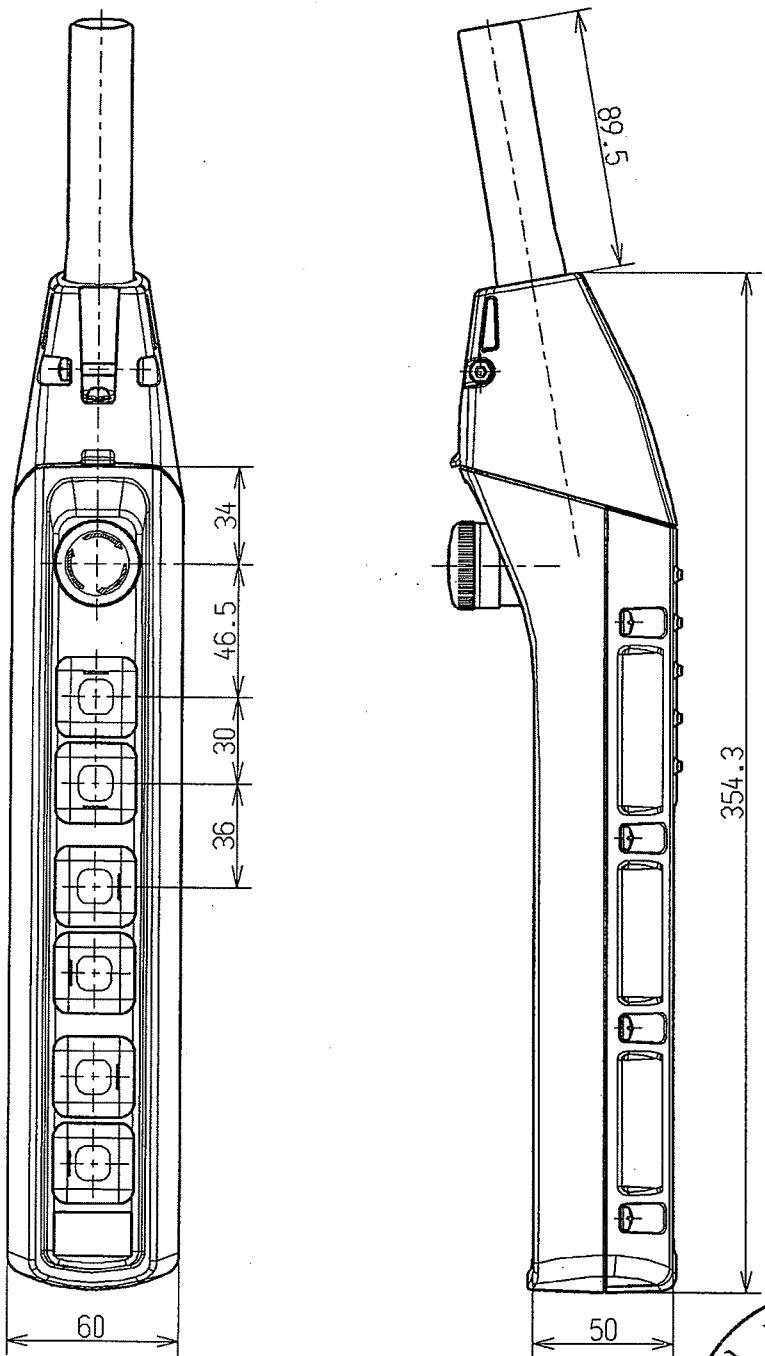
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4

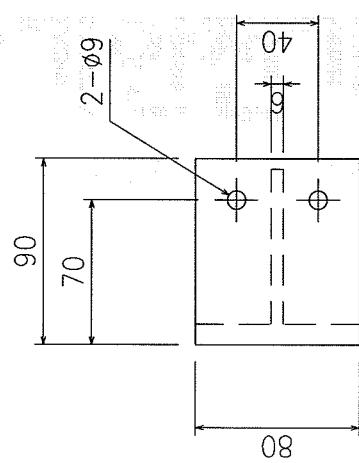
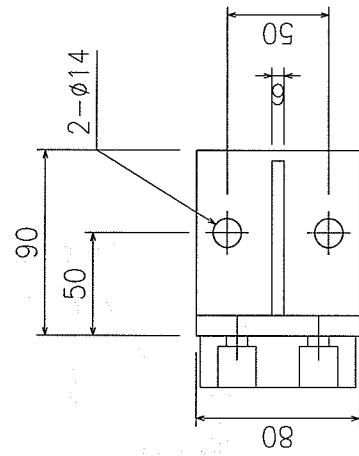
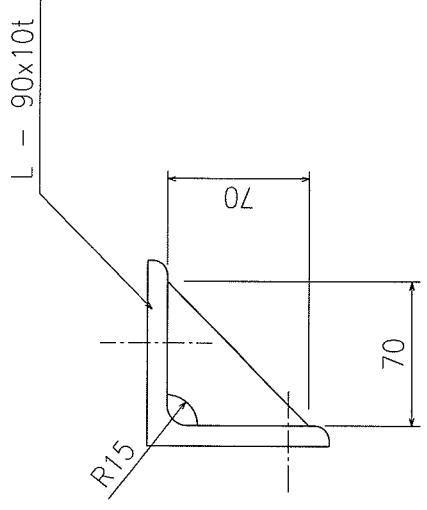
5

6

Revision	Incidence	Description	Date	Charge	Approved	
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⑥	
⑤	
④	
③	
②	
①	
Date issued _____	



No.	Part Name	Description	Mat'l	Unit	Qty	Weight (kg)	Remark
TITLE				SS000	4	DRAWN	
	J . S . CHO	W.H.EUN					STOPPER – traversing
							Part No.
							Dwg No.
							SCALE
							REV.

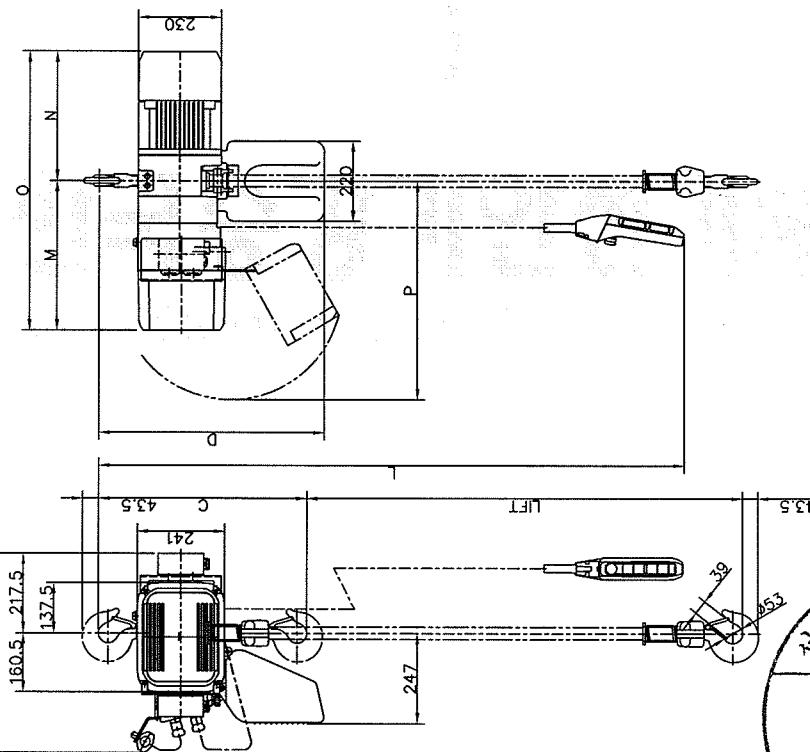
KTO (주) KTO KOREA



9
8
7
6
5
4
3
2
1

형식번호 : KML-ER2-018			
Particulars			
Model number.	Size	本体尺寸	ER2-E
KITO-ER2-018L	Nominal Capacity	重量	1.8t
KITO-ER2-018L	Lift	提升高度	3m(max 30m)

Dimensions			
M	N	O	P
1속 저속	308	646	488
2속 저속	372	710	561



B チェーンリフタ
Chain Lifter

C チェーンリフタ
Chain Lifter

D チェーンリフタ
Chain Lifter

E チェーンリフタ
Chain Lifter

F チェーンリフタ
Chain Lifter

G チェーンリフタ
Chain Lifter

H チェーンリフタ
Chain Lifter

I チェーンリフタ
Chain Lifter

J チェーンリフタ
Chain Lifter

K チェーンリフタ
Chain Lifter

L チェーンリフタ
Chain Lifter

M チェーンリフタ
Chain Lifter

N チェーンリフタ
Chain Lifter

O チェーンリフタ
Chain Lifter

P チェーンリフタ
Chain Lifter

Q チェーンリフタ
Chain Lifter

R チェーンリフタ
Chain Lifter

S チェーンリフタ
Chain Lifter

T チェーンリフタ
Chain Lifter

U チェーンリフタ
Chain Lifter

V チェーンリフタ
Chain Lifter

W チェーンリフタ
Chain Lifter

X チェーンリフタ
Chain Lifter

Y チェーンリフタ
Chain Lifter

Z チェーンリフタ
Chain Lifter

A チェーンリフタ
Chain Lifter

B チェーンリフタ
Chain Lifter

C チェーンリフタ
Chain Lifter

D チェーンリフタ
Chain Lifter

E チェーンリフタ
Chain Lifter

F チェーンリフタ
Chain Lifter

G チェーンリフタ
Chain Lifter

H チェーンリフタ
Chain Lifter

I チェーンリフタ
Chain Lifter

J チェーンリフタ
Chain Lifter

K チェーンリフタ
Chain Lifter

L チェーンリフタ
Chain Lifter

M チェーンリフタ
Chain Lifter

N チェーンリフタ
Chain Lifter

O チェーンリフタ
Chain Lifter

P チェーンリフタ
Chain Lifter

Q チェーンリフタ
Chain Lifter

R チェーンリフタ
Chain Lifter

S チェーンリフタ
Chain Lifter

T チェーンリフタ
Chain Lifter

U チェーンリフタ
Chain Lifter

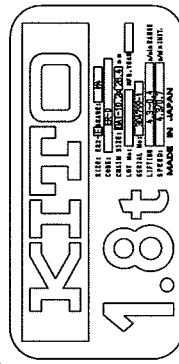
V チェーンリフタ
Chain Lifter

W チェーンリフタ
Chain Lifter

X チェーンリフタ
Chain Lifter

Y チェーンリフタ
Chain Lifter

Z チェーンリフタ
Chain Lifter

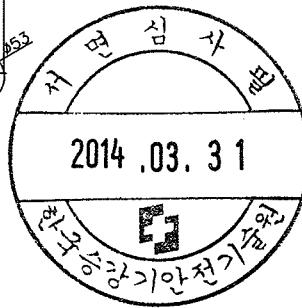


Name Plate For Chain Hoist

The trolley be installed on either tapered or flat flange.
A figure in () is available for only flat flange.

형식번호 : KML-ER2-018
*자세히는 음선 시기입니다.

NAME	ER2-S	SCALE	NOT
Code No.	KML-ER2-018-001	Rev. 0	
Date	2014.03.31	Drawn	Approved



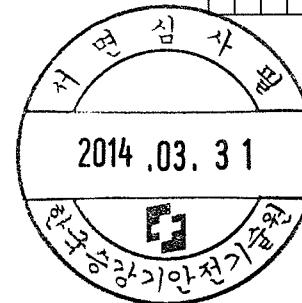
1 2 3 4 5 6 7 8 9

형식번호 : KML-ER2-018				Particulars	
Model number.				基本本体	
KITO-ER2SP018L KITO-ER2SP018LL				Size	ER2-E
				Nominal Capacity	1.8t
				Lift	3m(max 30m)
				Chain size	Φ10x2.1
				Type	手動ブリーフ
				Material	レバーブリッジより ブリックまで的小面積
				Min. Headroom	C
				Phase	電圧
				Voltage	3φ 220/208V 60Hz
				Power	380.440V 60Hz
				Current	1.8kW x 4P
				Dimensions	M N O P
1속 속도	308	338	646	488	
2속 속도	372	710	561		

A	Model number. KITO-ER2SP018L KITO-ER2SP018LL	Size	ER2-E
		Nominal Capacity	1.8t
		Lift	3m(max 30m)
		Chain size	Φ10x2.1
B	Dimensions M N O P	レバーブリッジより ブリックまで的小面積	手動ブリーフ
	1속 속도 308	646	488
	2속 속도 372	710	561
C	Phase • Voltage	3φ 220/208V 60Hz	380.440V 60Hz
	モーター出力 • 反復定格 等級	卷上モータ for Lifting	1.8kW x 4P
	Duty Output Classification	L	1.8kW x 4P
	卷上速度 Lifting Speed	2 속도(엔진 속도) Push Button Cord	IL 4.3/0.7 m/min
		Push Button Cord	2.7 m(max 29.7m)
D	Length of Power Supply Cable ケーブル長さ	(2.5)m	(2.5)m
E	レバーブリッジよりチエーン バットまでの寸法 Chain Container Distance from Bottom of Beam	D	690mm(max 1000)
	適用 レバーブリッジ Fence Width	B	82~153mm
	トロリ 大巾 Max. Dimension of Trolley Width	G	630mm
	Mass Pointing Color	H	91 kg マス Munsell 7.5R/14

The trolley be installed on either tapered
or flat flange.
A figure in () is available for only flat flange.

Name Plate
Name Plate For Chain Hoist



형식번호 : KML-ER2-018
*자비라는 옵션 사용됨
Title 1.8t ER2 Series Electric Chain Hoist
WITH PLAN TROLLEY

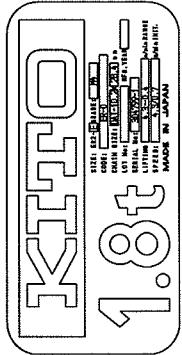
改訂 REV.	数 QTY	CONTENTS	年.月.일 DATE	担当 DRAWN APPROVED	承認 DRAWN APPROVED	検査 CHECKED	設計 DESIGNED	製造 MANUFACTURED	尺規 SCALE	NOT REV.
				H.SAITO	NAITO	SHIMURA	SHIMURA	10.07.13	10.07.13	0

二角法 厚さ : mm

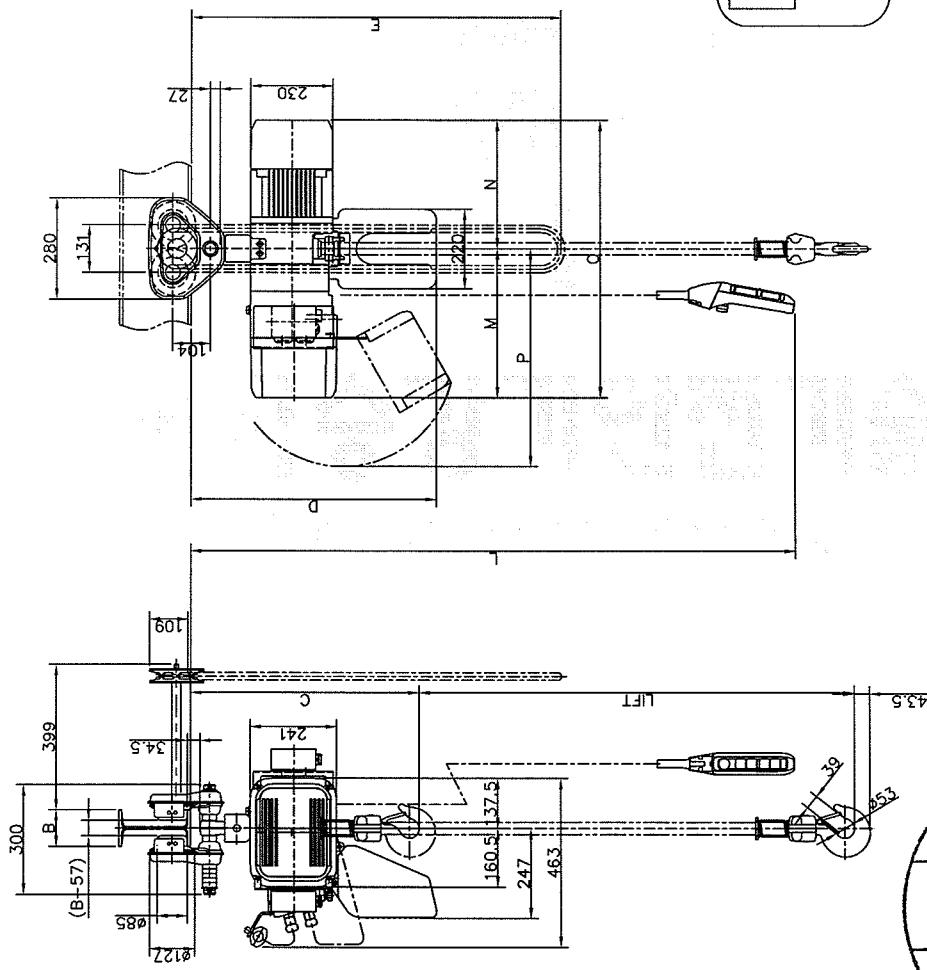
00000 - 10

諸 條		Particulars		ER2-E	
基 本 本 体 Size					
定 品 名	電 磁 集 磁 量	1.8t			
定 品 名	電 磁 集 磁 量	3m(max 30m)			
電 磁 集 磁 量	電 磁 集 磁 量	ø10.2x11			
チ ョ ー ナ イ ズ Chain Size					
レ リ ル 下 面 より フ ッ ク マ ザ の 小 さ き 距 間 Min. Headroom	相 電 壓	C	635mm		
Phase • Voltage	相 電 壓	3φ 220/380V 60Hz	380,440V 60Hz		
モーダル出力 • 反復定格 Motor Output Duty Rating Classification	卷上モータ For Lifting	L	1.8kW x 4P		
		L	1.8kW x 4P		
巻上速度 Lifting Speed	2.98(巻上げ速度) 1秒毎 360度	L	4.3/0.7m/min 4.4 m/min		
		L	4.3/0.7m/min 4.4 m/min		
オジボンコード長さ : L ケーブル長さ Length of Power Supply Cable	2.7m(max 29.7m)				
キヤブリヤギ電 ケーブル長さ Length of Power Supply Cable	(2.5)m				
ケーブル下端よりチーン ハケットまでの距離 Chain Container Distance from Bottom of Beam	D	690mm(max 1000)			
肩用 レフテッド : B		82~153mm			
トロリ 大 尺寸 : G New Dimension of Trolley Width	G	630mm			
質量 Mass	kg	91 kg			
漆装色 Painting Color		マット 7.5%R/14 Mattel 7.5%R/14 Munsell 7.5%R/14			

The trolley be installed on either tapered or flat flange.



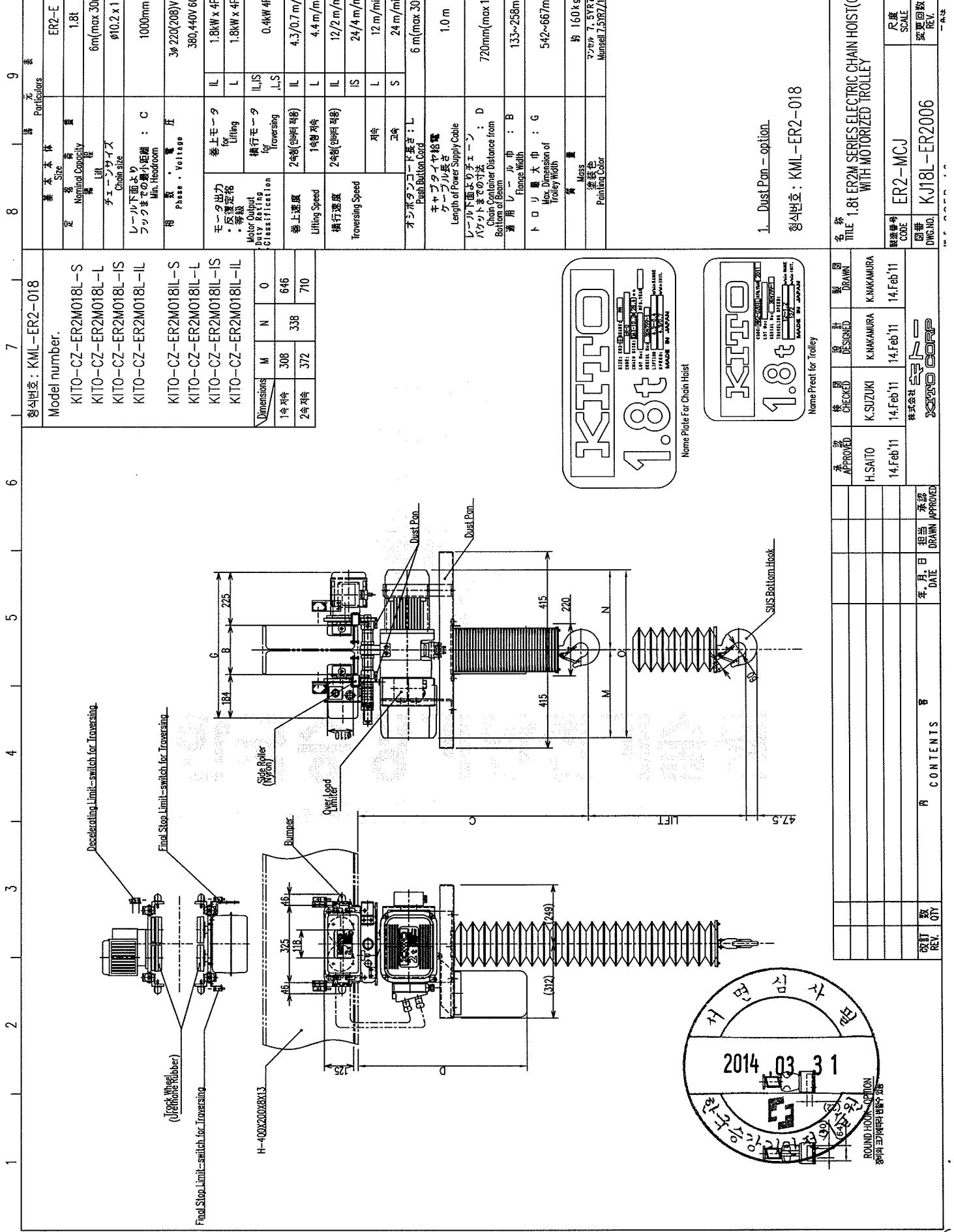
Name Plate For Chain Hoist

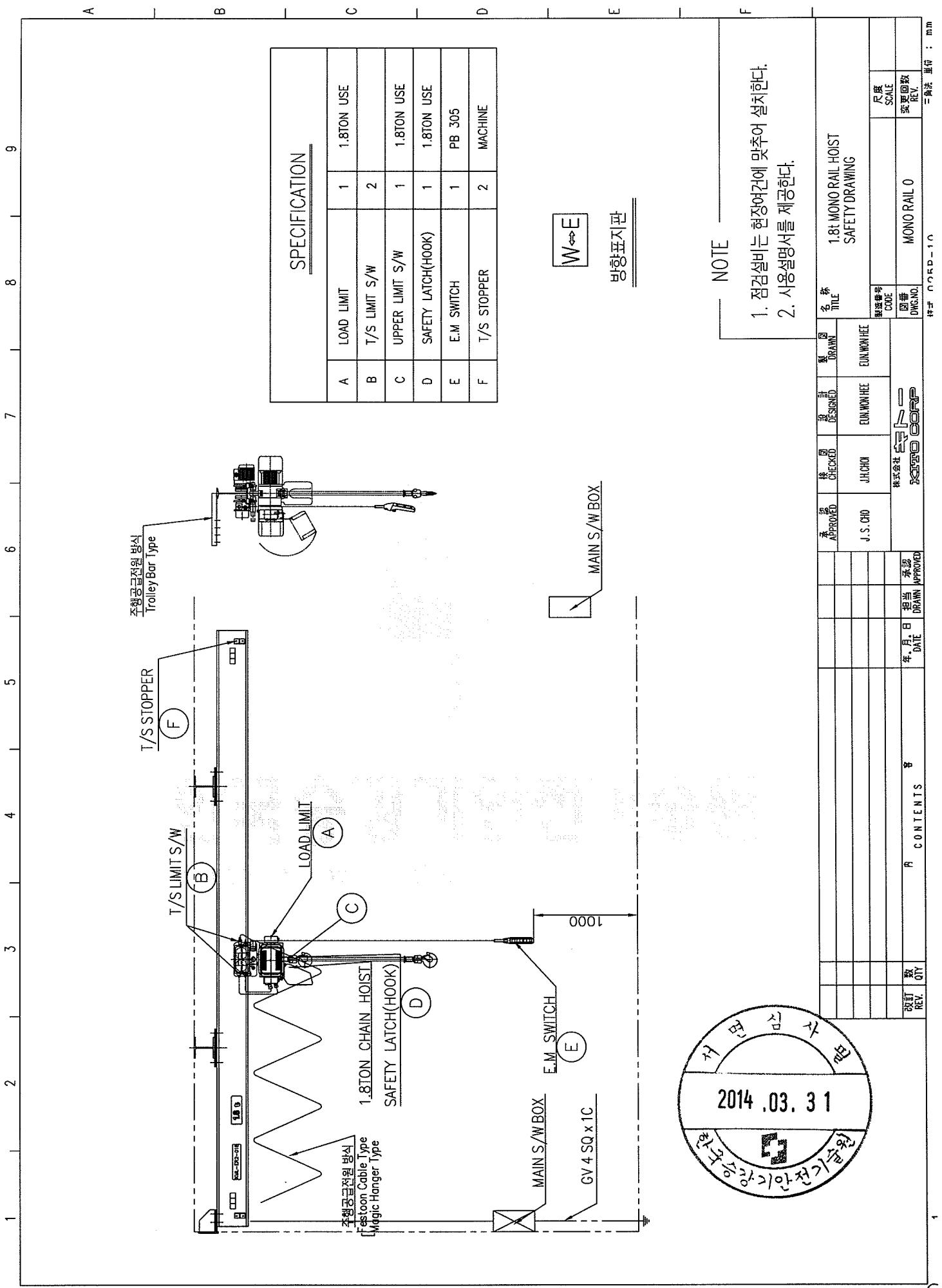


2014 .03. 31

Technical Data									
	Dimensions			Performance			Electrical		
Model number : KML-ER2-018									
Dimensions			Performance			Electrical			
1. 起重机尺寸			2. 起重机性能			3. 电源			
1.1. 高度			1.2. 起升速度			1.3. 电源电压			
1.2. 宽度			1.4. 行走速度			1.5. 电源频率			
1.3. 深度			1.6. 起升高度			1.7. 电源容量			
1.4. 总重量			1.8. 起升力矩			1.9. 电源线长度			
1.5. 总功率			1.10. 起升时间			1.11. 电源线直径			
1.6. 起升时间			1.12. 行走时间			1.13. 电源线重量			
1.7. 行走时间			1.14. 行走距离			1.15. 电源线重量			
1.8. 行走距离			1.16. 行走速度			1.17. 电源线重量			
1.9. 行走速度			1.18. 行走时间			1.19. 电源线重量			
1.10. 行走时间			1.20. 行走距离			1.21. 电源线重量			
1.11. 行走距离			1.22. 行走速度			1.23. 电源线重量			
1.12. 行走速度			1.24. 行走时间			1.25. 电源线重量			
1.13. 行走时间			1.26. 行走距离			1.27. 电源线重量			
1.14. 行走距离			1.28. 行走速度			1.29. 电源线重量			
1.15. 行走速度			1.30. 行走时间			1.31. 电源线重量			
1.16. 行走时间			1.32. 行走距离			1.33. 电源线重量			
1.17. 行走距离			1.34. 行走速度			1.35. 电源线重量			
1.18. 行走速度			1.36. 行走时间			1.37. 电源线重量			
1.19. 行走时间			1.38. 行走距离			1.39. 电源线重量			
1.20. 行走距离			1.40. 行走速度			1.41. 电源线重量			
1.21. 行走速度			1.42. 行走时间			1.43. 电源线重量			
1.22. 行走时间			1.44. 行走距离			1.45. 电源线重量			
1.23. 行走距离			1.46. 行走速度			1.47. 电源线重量			
1.24. 行走速度			1.48. 行走时间			1.49. 电源线重量			
1.25. 行走时间			1.50. 行走距离			1.51. 电源线重量			
1.26. 行走距离			1.52. 行走速度			1.53. 电源线重量			
1.27. 行走速度			1.54. 行走时间			1.55. 电源线重量			
1.28. 行走时间			1.56. 行走距离			1.57. 电源线重量			
1.29. 行走距离			1.58. 行走速度			1.59. 电源线重量			
1.30. 行走速度			1.60. 行走时间			1.61. 电源线重量			
1.31. 行走时间			1.62. 行走距离			1.63. 电源线重量			
1.32. 行走距离			1.64. 行走速度			1.65. 电源线重量			
1.33. 行走速度			1.66. 行走时间			1.67. 电源线重量			
1.34. 行走时间			1.68. 行走距离			1.69. 电源线重量			
1.35. 行走距离			1.70. 行走速度			1.71. 电源线重量			
1.36. 行走速度			1.72. 行走时间			1.73. 电源线重量			
1.37. 行走时间			1.74. 行走距离			1.75. 电源线重量			
1.38. 行走距离			1.76. 行走速度			1.77. 电源线重量			
1.39. 行走速度			1.78. 行走时间			1.79. 电源线重量			
1.40. 行走时间			1.80. 行走距离			1.81. 电源线重量			
1.41. 行走距离			1.82. 行走速度			1.83. 电源线重量			
1.42. 行走速度			1.84. 行走时间			1.85. 电源线重量			
1.43. 行走时间			1.86. 行走距离			1.87. 电源线重量			
1.44. 行走距离			1.88. 行走速度			1.89. 电源线重量			
1.45. 行走速度			1.90. 行走时间			1.91. 电源线重量			
1.46. 行走时间			1.92. 行走距离			1.93. 电源线重量			
1.47. 行走距离			1.94. 行走速度			1.95. 电源线重量			
1.48. 行走速度			1.96. 行走时间			1.97. 电源线重量			
1.49. 行走时间			1.98. 行走距离			1.99. 电源线重量			
1.50. 行走距离			1.100. 行走速度			1.101. 电源线重量			
1.51. 行走速度			1.102. 行走时间			1.103. 电源线重量			
1.52. 行走时间			1.104. 行走距离			1.105. 电源线重量			
1.53. 行走距离			1.106. 行走速度			1.107. 电源线重量			
1.54. 行走速度			1.108. 行走时间			1.109. 电源线重量			
1.55. 行走时间			1.110. 行走距离			1.111. 电源线重量			
1.56. 行走距离			1.112. 行走速度			1.113. 电源线重量			
1.57. 行走速度			1.114. 行走时间			1.115. 电源线重量			
1.58. 行走时间			1.116. 行走距离			1.117. 电源线重量			
1.59. 行走距离			1.118. 行走速度			1.119. 电源线重量			
1.60. 行走速度			1.120. 行走时间			1.121. 电源线重量			
1.61. 行走时间			1.122. 行走距离			1.123. 电源线重量			
1.62. 行走距离			1.124. 行走速度			1.125. 电源线重量			
1.63. 行走速度			1.126. 行走时间			1.127. 电源线重量			
1.64. 行走时间			1.128. 行走距离			1.129. 电源线重量			
1.65. 行走距离			1.130. 行走速度			1.131. 电源线重量			
1.66. 行走速度			1.132. 行走时间			1.133. 电源线重量			
1.67. 行走时间			1.134. 行走距离			1.135. 电源线重量			
1.68. 行走距离			1.136. 行走速度			1.137. 电源线重量			
1.69. 行走速度			1.138. 行走时间			1.139. 电源线重量			
1.70. 行走时间			1.140. 行走距离			1.141. 电源线重量			
1.71. 行走距离			1.142. 行走速度			1.143. 电源线重量			
1.72. 行走速度			1.144. 行走时间			1.145. 电源线重量			
1.73. 行走时间			1.146. 行走距离			1.147. 电源线重量			
1.74. 行走距离			1.148. 行走速度			1.149. 电源线重量			
1.75. 行走速度			1.150. 行走时间			1.151. 电源线重量			
1.76. 行走时间			1.152. 行走距离			1.153. 电源线重量			
1.77. 行走距离			1.154. 行走速度			1.155. 电源线重量			
1.78. 行走速度			1.156. 行走时间			1.157. 电源线重量			
1.79. 行走时间			1.158. 行走距离			1.159. 电源线重量			
1.80. 行走距离			1.160. 行走速度			1.161. 电源线重量			
1.81. 行走速度			1.162. 行走时间						

25/201





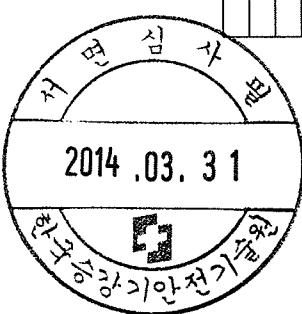
mm : 三角法

17-005-RD-10

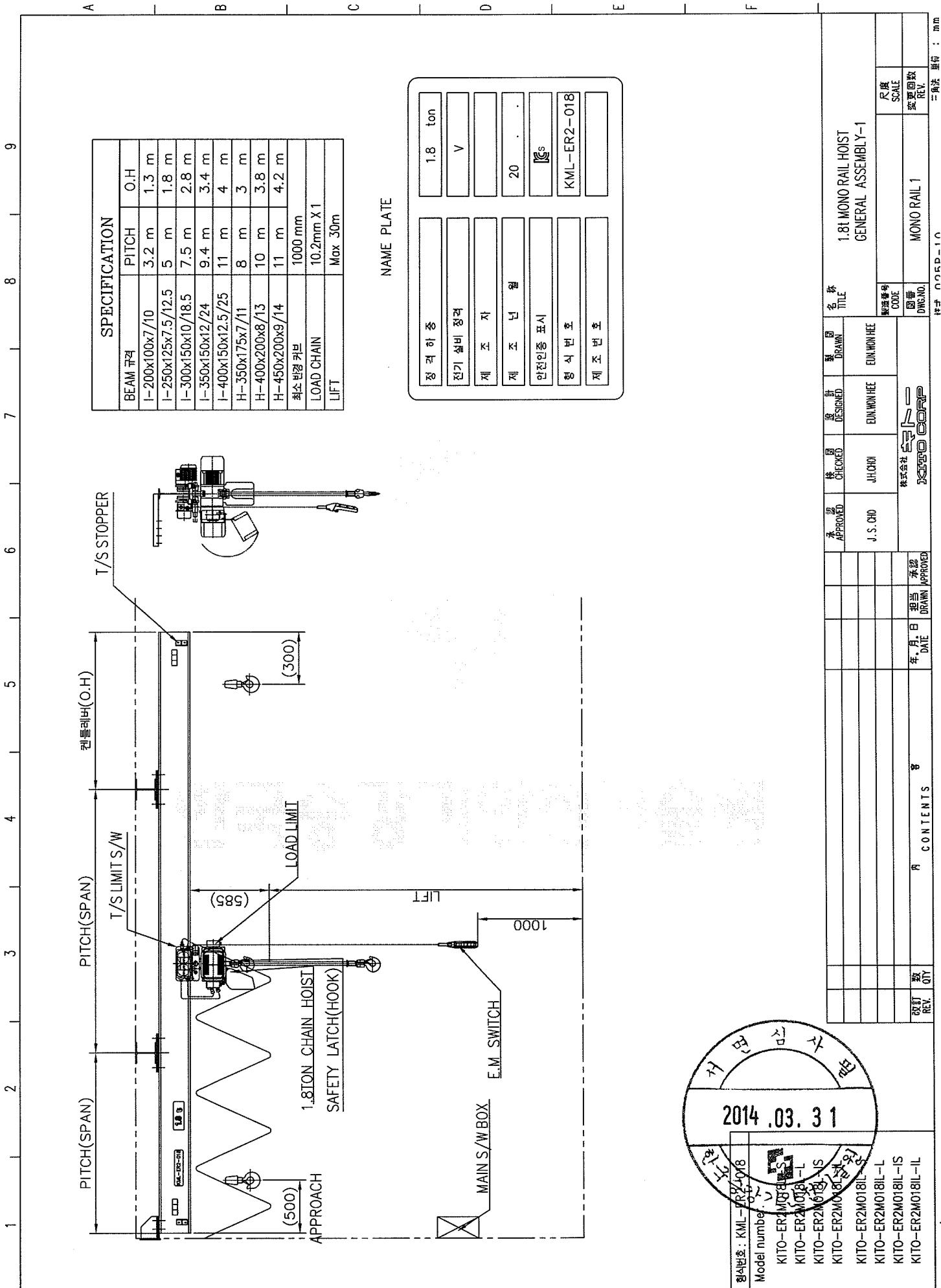
REV. 01

1

1

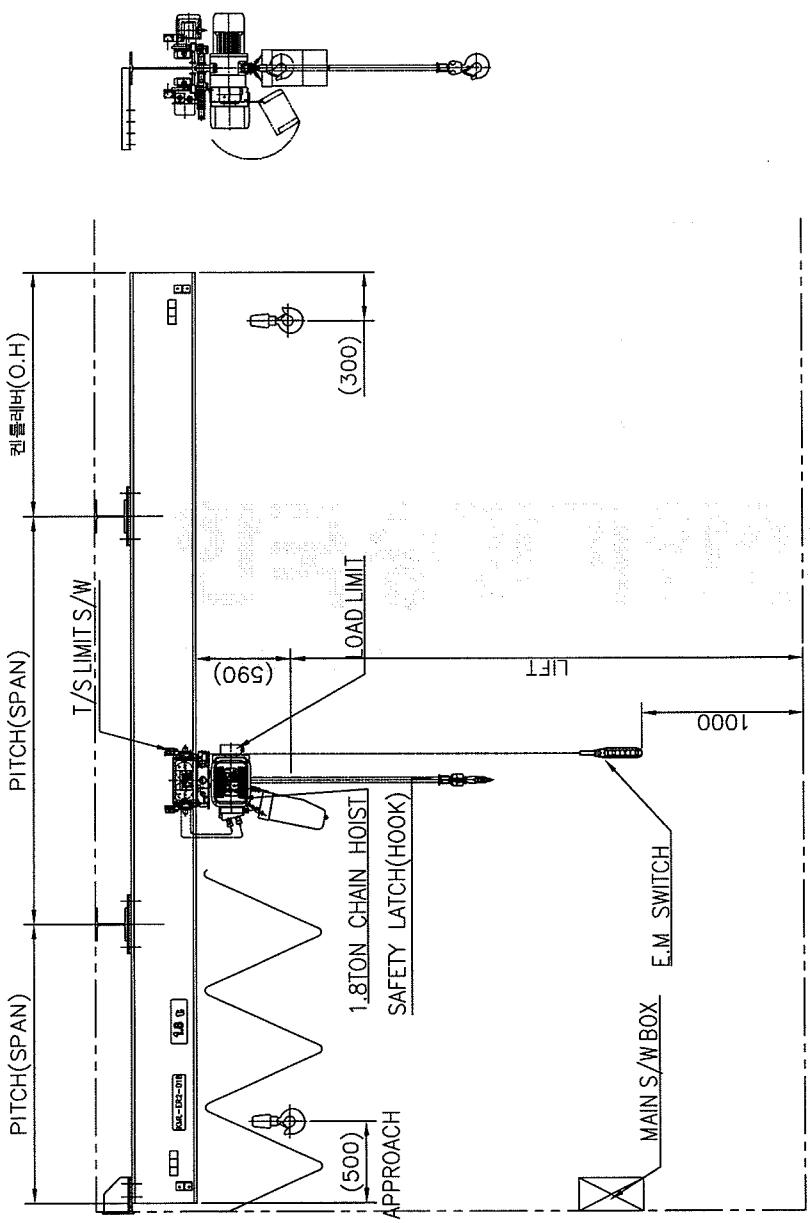


35/201



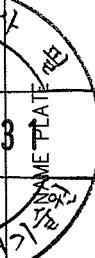
36/201

SPECIFICATION				
BEAM ट्रैक	PITCH	O.H		
H-300x150x6.5/9	6 m	2 m		
H-350x175x7/11	8 m	3 m		
H-400x200x8/13	10 m	3.8 m		
H-450x200x9/14	11 m	4.2 m		
LOAD CHAIN	10.2mm X 1			
LIFT	Max 30m			

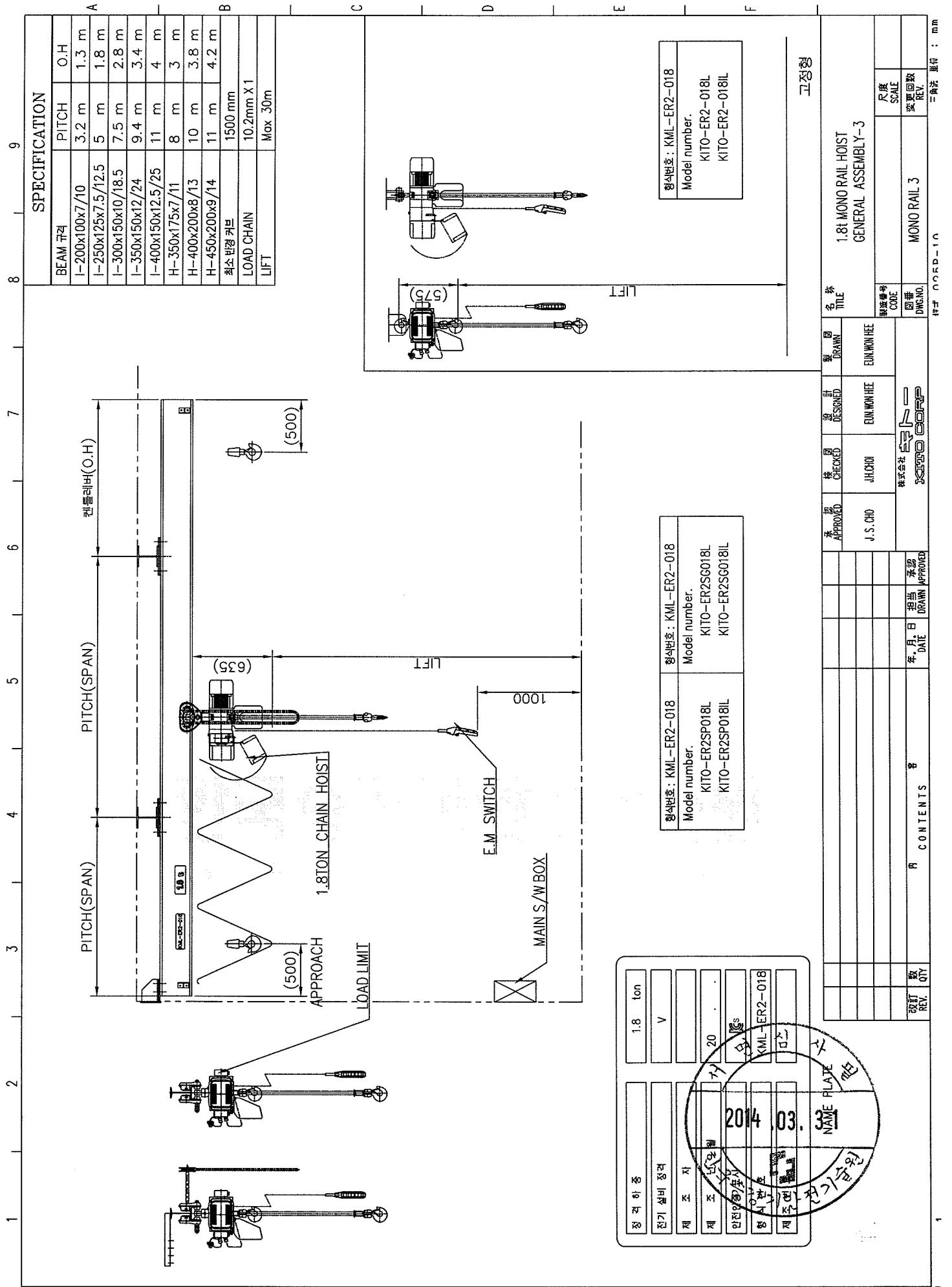


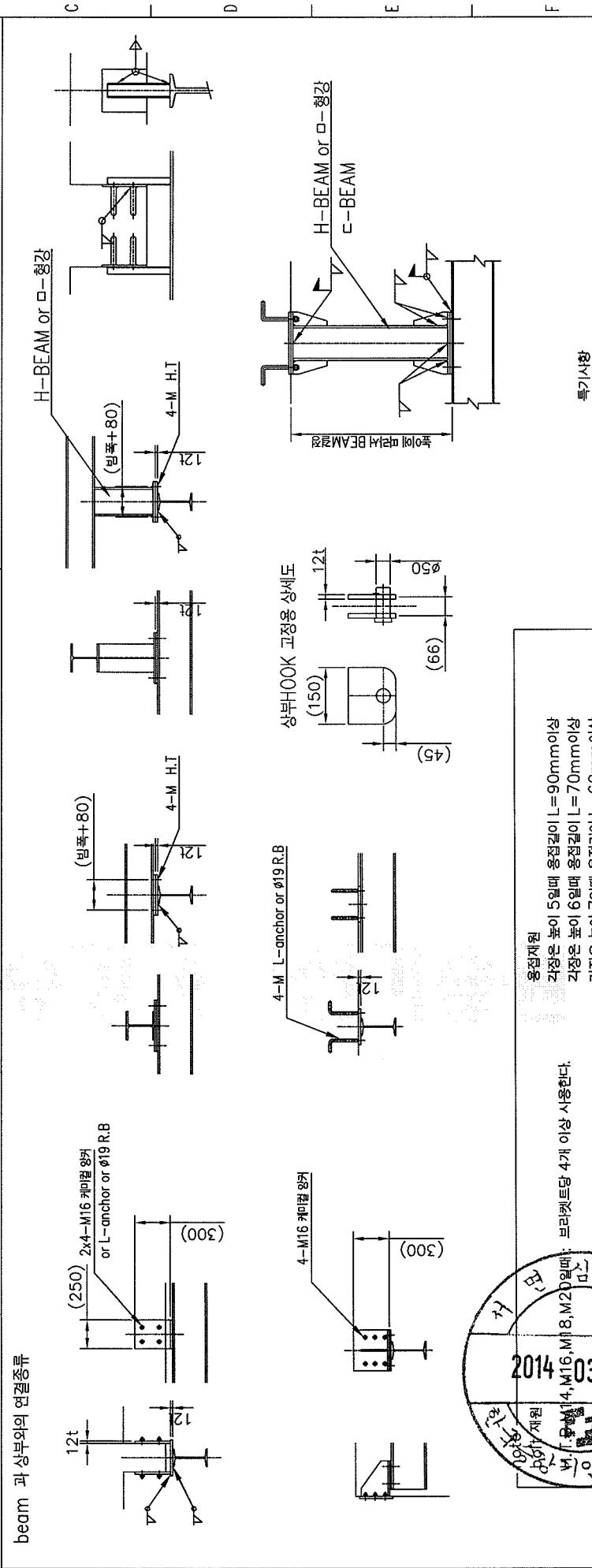
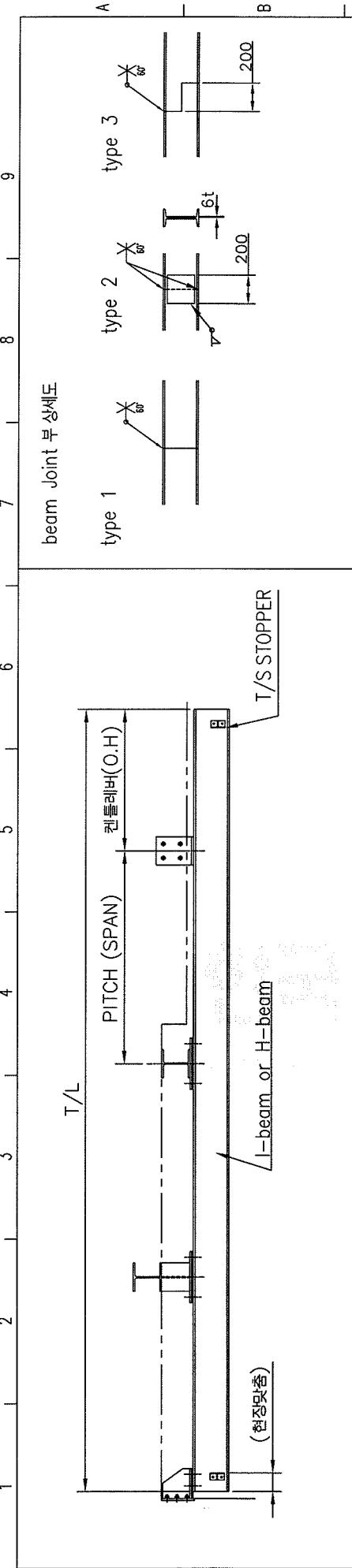
형식번호 : KML-ER2-018	설명번호 : KML-ER2-018
Model number.	Model number.
K10-C-ER2M018L-S	K10-CZ-ER2M018L-S
K10-C-ER2M018L-L	K10-CZ-ER2M018L-L
K10-C-ER2M018L-IS	K10-CZ-ER2M018L-IS
K10-C-ER2M018L-IL	K10-CZ-ER2M018L-IL
K10-C-ER2M018L-S	K10-CZ-ER2M018L-S
K10-C-ER2M018L-L	K10-CZ-ER2M018L-L
K10-C-ER2M018L-IS	K10-CZ-ER2M018L-IS
K10-C-ER2M018L-IL	K10-CZ-ER2M018L-IL
F	
1	名 称 TITLE
	1.8t MONO RAIL HOIST GENERAL ASSEMBLY-Y-2
2	製造番号 CODE
3	規格番号 DIN NO.
4	尺度 SCALE
5	変更回数 REV.
三脚式	
mm	

장 력 하 종	1.8 ton
전기 설비 설치	V
제 조 지	
제조 년 월	20 . .
판정인 표시	KMS
판정일	2014.03.
판정인	KML-ER2-018



37/201





39/201

4. 전기도면

- 1) ELECTRICAL SPECIFICATION
- 2) SYMBOL LIST
- 3) 배선배관도 & 접지계통도
- 4) 전기회로도
- 5) PANEL 관련도



LOAD SUMMARY 1 - (ER2-018IL-IS/IL TYPE)

*POWER SOURCE : AC 3Φ 220(208)V

OBJECT	HOISTING	TRAVERSING	CONTROL CIRCUIT
MOTOR OUTPUT	1.8KW x 4P	0.4KW x 4P	
FULL LOAD CURRENT	11.2 (A)	3 (A)	0.5 (A)

*크레인 하중상태를 HOIST의 정격 LOAD의 100(%)를 사용했을때를 기준으로 작성하였음.

*** NOMAL 전류값 ***

권상과 횡행시 : HOISTING + TRAVERSING + CONTROL CIRCUIT =

14.7 A

*** PEAK 전류값 ***

K= NAMAL 전류치가 50A미만일때 1.25, 50A이상일때 1.1적용

NOMAL 전류값 * K = 14.7 * 1.25 = 18.3 A

*POWER SOURCE : AC 3Φ 380(440)V

OBJECT	HOISTING	TRAVERSING	CONTROL CIRCUIT
MOTOR OUTPUT	1.8KW x 4P	0.4KW x 4P	
FULL LOAD CURRENT	5.1 (A)	2.5 (A)	0.5 (A)

*크레인 하중상태를 HOIST의 정격 LOAD의 100(%)를 사용했을때를 기준으로 작성하였음.

*** NOMAL 전류값 ***

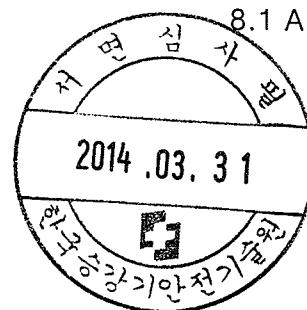
권상과 횡행시 : HOISTING + TRAVERSING + CONTROL CIRCUIT =

8.1 A

*** PEAK 전류값 ***

K= NAMAL 전류치가 50A미만일때 1.25, 50A이상일때 1.1적용

NOMAL 전류값 * K = 8.1 * 1.25 = 10.125 A



LOAD SUMMARY 2 – (ER2-018IL TYPE)

*POWER SOURCE : AC 3Φ 220(208)V

OBJECT	HOISTING	TRAVERSING	CONTROL CIRCUIT
MOTOR OUTPUT	1.8KW x 4P	-	
FULL LOAD CURRENT	11.2 (A)	0 (A)	0.5 (A)

*크레인 하중상태를 HOIST의 정격 LOAD의 100(%)를 사용했을때를 기준으로 작성하였음.

*** NOMAL 전류값 ***

$$\text{권상시} : \text{HOISTING + CONTROL CIRCUIT} = 11.7 \text{ A}$$

*** PEAK 전류값 ***

K= NAMAL 전류치가 50A미만일때 1.25, 50A이상일때 1.1적용

$$\text{NOMAL 전류값} * K = 11.7 * 1.25 = 14.6 \text{ A}$$

*POWER SOURCE : AC 3Φ 380(440)V

OBJECT	HOISTING	TRAVERSING	CONTROL CIRCUIT
MOTOR OUTPUT	1.8KW x 4P	-	
FULL LOAD CURRENT	5.1 (A)	0 (A)	0.5 (A)

*크레인 하중상태를 HOIST의 정격 LOAD의 100(%)를 사용했을때를 기준으로 작성하였음.

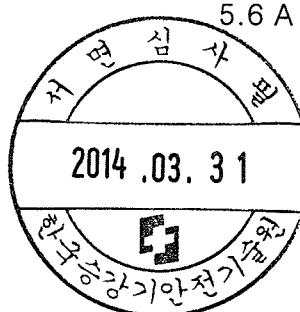
*** NOMAL 전류값 ***

$$\text{권상시} : \text{HOISTING + CONTROL CIRCUIT} = 5.6 \text{ A}$$

*** PEAK 전류값 ***

K= NAMAL 전류치가 50A미만일때 1.25, 50A이상일때 1.1적용

$$\text{NOMAL 전류값} * K = 5.6 * 1.25 = 7 \text{ A}$$



LOAD SUMMARY 3 – (ER2-018IL-S/L TYPE)

*POWER SOURCE : AC 3Φ 220(208)V

OBJECT	HOISTING	TRAVERSING	CONTROL CIRCUIT
MOTOR OUTPUT	1.8KW x 4P	0.4KW x 4P	
FULL LOAD CURRENT	11.2 (A)	3 (A)	0.5 (A)

*크레인 하중상태를 HOIST의 정격 LOAD의 100(%)를 사용했을때를 기준으로 작성하였음.

*** NOMAL 전류값 ***

$$\text{권상시} : \text{HOISTING + CONTROL CIRCUIT} = 14.7 \text{ A}$$

*** PEAK 전류값 ***

K= NAMAL 전류치가 50A미만일때 1.25, 50A이상일때 1.1적용

$$\text{NOMAL 전류값} * K = 14.7 * 1.25 = 18.3 \text{ A}$$

*POWER SOURCE : AC 3Φ 380(440)V

OBJECT	HOISTING	TRAVERSING	CONTROL CIRCUIT
MOTOR OUTPUT	1.8KW x 4P	0.4KW x 4P	
FULL LOAD CURRENT	5.1 (A)	2.2 (A)	0.5 (A)

*크레인 하중상태를 HOIST의 정격 LOAD의 100(%)를 사용했을때를 기준으로 작성하였음.

*** NOMAL 전류값 ***

$$\text{권상시} : \text{HOISTING + CONTROL CIRCUIT} = 7.8 \text{ A}$$

*** PEAK 전류값 ***

K= NAMAL 전류치가 50A미만일때 1.25, 50A이상일때 1.1적용

$$\text{NOMAL 전류값} * K = 7.8 * 1.25 = 9.7 \text{ A}$$



LOAD SUMMARY 4 - (ER2-018L-L/S TYPE)

*POWER SOURCE : AC 3Φ 220(208)V

OBJECT	HOISTING	TRaversing	CONTROL CIRCUIT
MOTOR OUTPUT	1.8KW x 4P	0.4KW x 4P	
FULL LOAD CURRENT	8.4 (A)	3 (A)	0.5 (A)

*크레인 하중상태를 HOIST의 정격 LOAD의 100(%)를 사용했을때를 기준으로 작성하였음.

*** NOMAL 전류값 ***

권상과 횡행시 : HOISTING + TRAVERSING + CONTROL CIRCUIT = 11.9 A

*** PEAK 전류값 ***

K= NAMAL 전류치가 50A미만일때 1.25, 50A이상일때 1.1적용

NOMAL 전류값 * K = 11.9 * 1.25 = 14.8 A

*POWER SOURCE : AC 3Φ 380(440)V

OBJECT	HOISTING	TRAVERSING	CONTROL CIRCUIT
MOTOR OUTPUT	1.8KW x 4P	0.4KW x 4P	
FULL LOAD CURRENT	4.6 (A)	2.2 (A)	0.5 (A)

*크레인 하중상태를 HOIST의 정격 LOAD의 100(%)를 사용했을때를 기준으로 작성하였음.

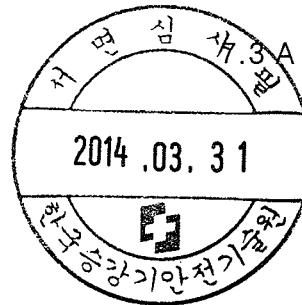
*** NOMAL 전류값 ***

권상과 횡행시 : HOISTING + TRAVERSING + CONTROL CIRCUIT =

*** PEAK 전류값 ***

K= NAMAL 전류치가 50A미만일때 1.25, 50A이상일때 1.1적용

NOMAL 전류값 * K = 7.3 * 1.25 = 9.125 A



LOAD SUMMARY 5 – (ER2-018L TYPE)

*POWER SOURCE : AC 3Φ 220(208)V

OBJECT	HOISTING	TRAVERSING	CONTROL CIRCUIT
MOTOR OUTPUT	1.8KW x 4P	-	
FULL LOAD CURRENT	8.4 (A)	0 (A)	0.5 (A)

*크레인 하중상태를 HOIST의 정격 LOAD의 100(%)를 사용했을때를 기준으로 작성하였음.

*** NOMAL 전류값 ***

$$\text{권상시} : \text{HOISTING} + \text{CONTROL CIRCUIT} = 8.9 \text{ A}$$

*** PEAK 전류값 ***

K= NAMAL 전류치가 50A미만일때 1.25, 50A이상일때 1.1적용

$$\text{NOMAL 전류값} * K = 8.9 * 1.25 = 11.1 \text{ A}$$

*POWER SOURCE : AC 3Φ 380(440)V

OBJECT	HOISTING	TRAVERSING	CONTROL CIRCUIT
MOTOR OUTPUT	1.8KW x 4P	-	
FULL LOAD CURRENT	4.6 (A)	0 (A)	0.5 (A)

*크레인 하중상태를 HOIST의 정격 LOAD의 100(%)를 사용했을때를 기준으로 작성하였음.

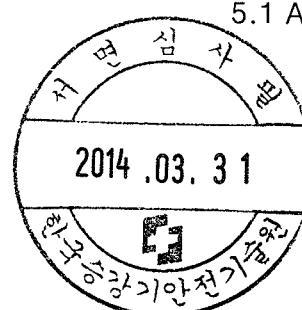
*** NOMAL 전류값 ***

$$\text{권상시} : \text{HOISTING} + \text{CONTROL CIRCUIT} = 5.1 \text{ A}$$

*** PEAK 전류값 ***

K= NAMAL 전류치가 50A미만일때 1.25, 50A이상일때 1.1적용

$$\text{NOMAL 전류값} * K = 5.1 * 1.25 = 6.3 \text{ A}$$



LOAD SUMMARY 6 – (ER2-018L-IL/IS TYPE)

*POWER SOURCE : AC 3Φ 220(208)V

OBJECT	HOISTING	TRAVERSING	CONTROL CIRCUIT
MOTOR OUTPUT	1.8KW x 4P	0.4KW x 4P	
FULL LOAD CURRENT	8.4 (A)	3 (A)	0.5 (A)

*크레인 하중상태를 HOIST의 정격 LOAD의 100(%)를 사용했을때를 기준으로 작성하였음.

*** NOMAL 전류값 ***

$$\text{권상시} : \text{HOISTING + CONTROL CIRCUIT} = 11.9 \text{ A}$$

*** PEAK 전류값 ***

K= NAMAL 전류치가 50A미만일때 1.25, 50A이상일때 1.1적용

$$\text{NOMAL 전류값} * K = 11.9 * 1.25 = 14.8 \text{ A}$$

*POWER SOURCE : AC 3Φ 380(440)V

OBJECT	HOISTING	TRAVERSING	CONTROL CIRCUIT
MOTOR OUTPUT	1.8KW x 4P	0.4KW x 4P	
FULL LOAD CURRENT	4.6 (A)	2.5 (A)	0.5 (A)

*크레인 하중상태를 HOIST의 정격 LOAD의 100(%)를 사용했을때를 기준으로 작성하였음.

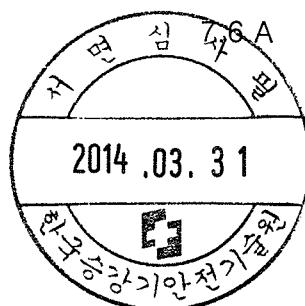
*** NOMAL 전류값 ***

$$\text{권상시} : \text{HOISTING + CONTROL CIRCUIT} =$$

*** PEAK 전류값 ***

K= NAMAL 전류치가 50A미만일때 1.25, 50A이상일때 1.1적용

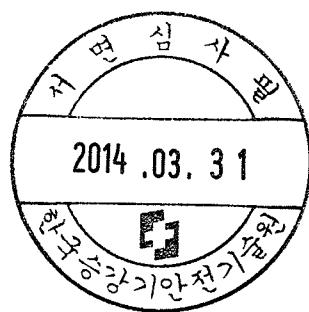
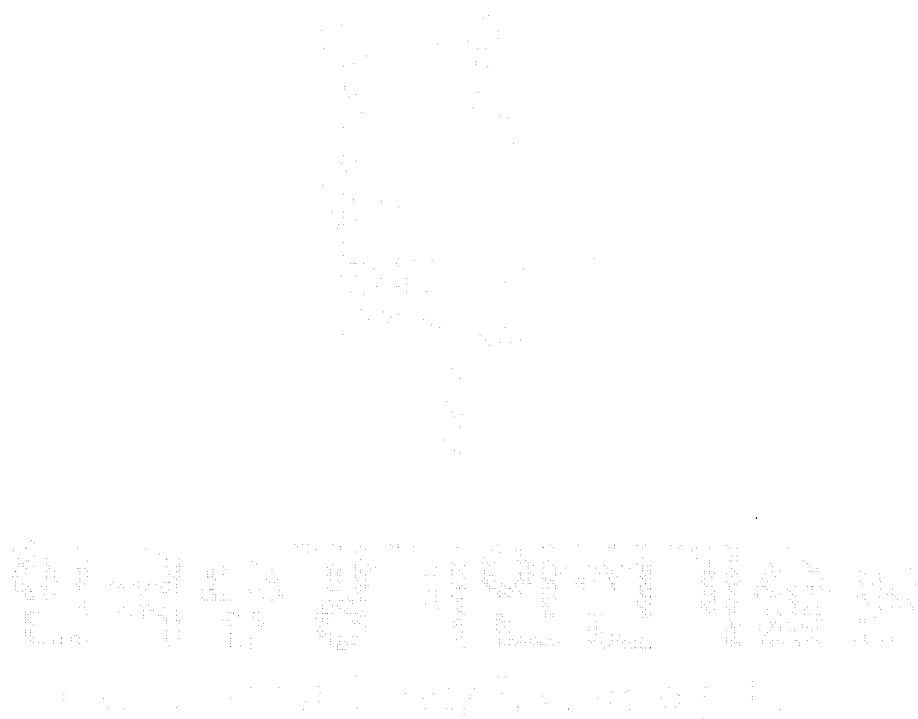
$$\text{NOMAL 전류값} * K = 7.6 * 1.25 = 9.5 \text{ A}$$

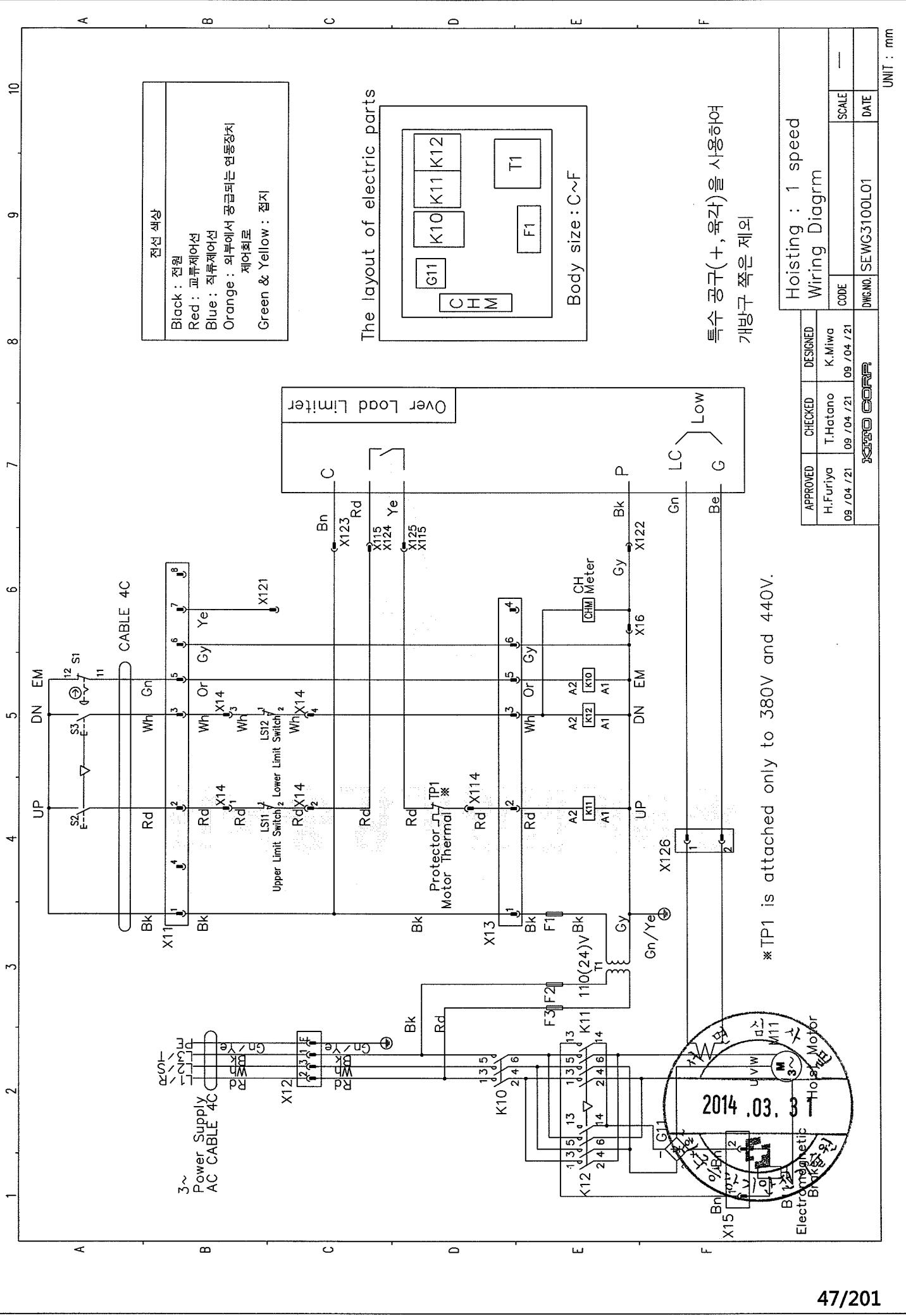


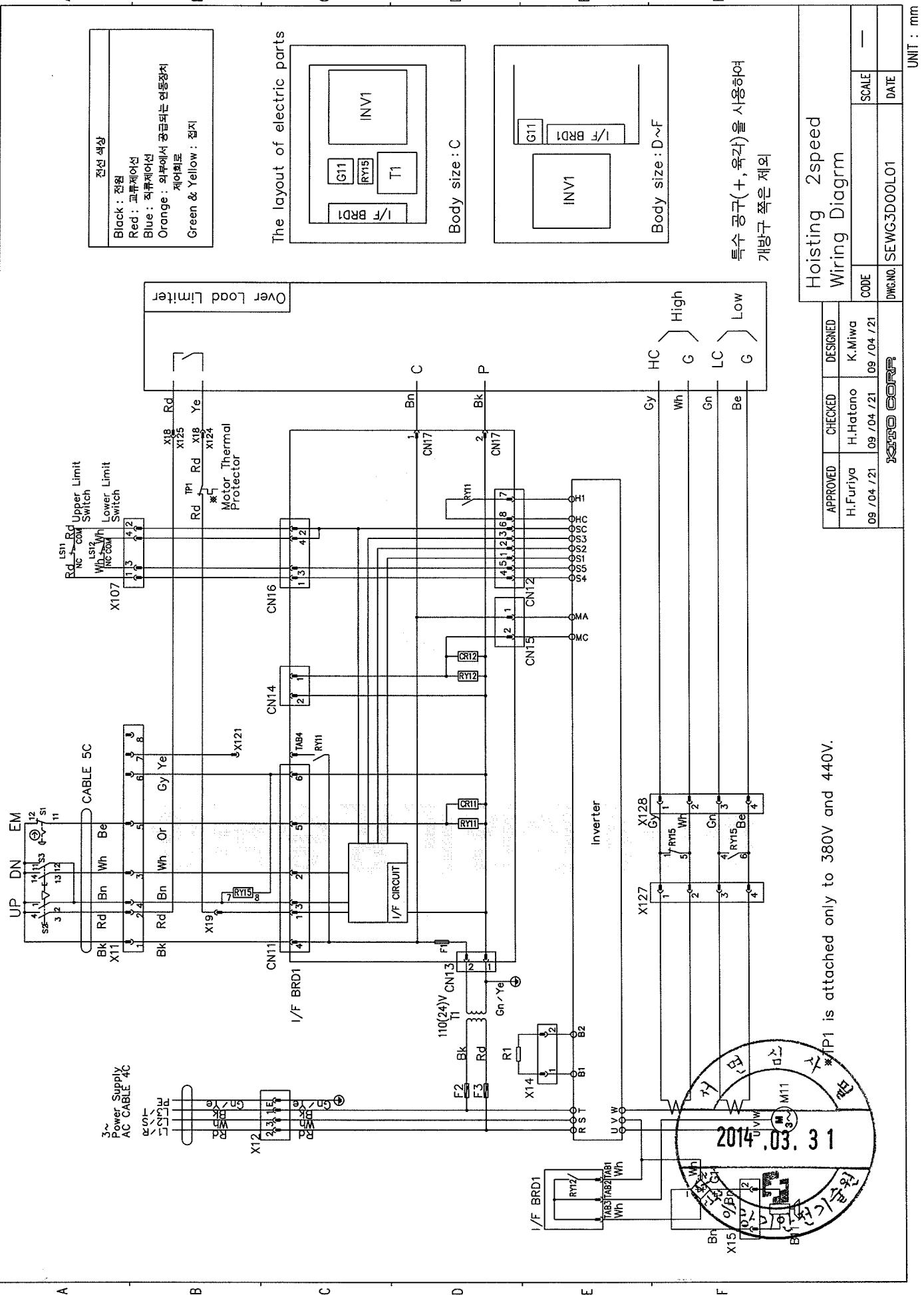


(1) 2점식 전기도면

- . 1속형 hoisting
- . 2속형(INVERTER) hoisting

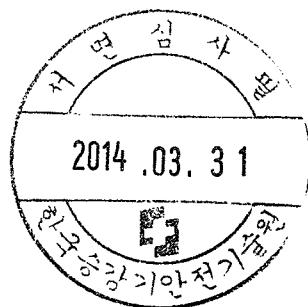
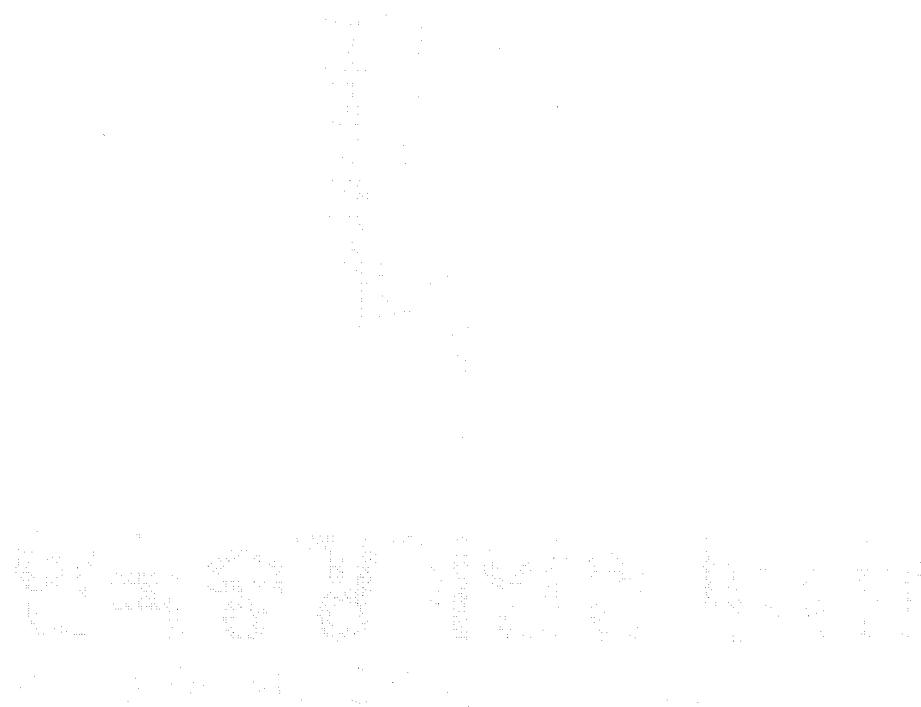






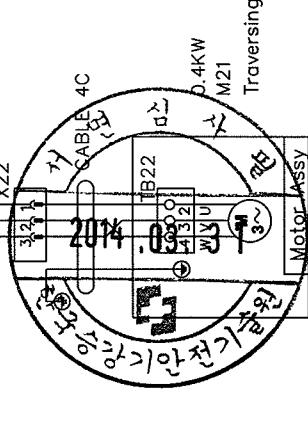
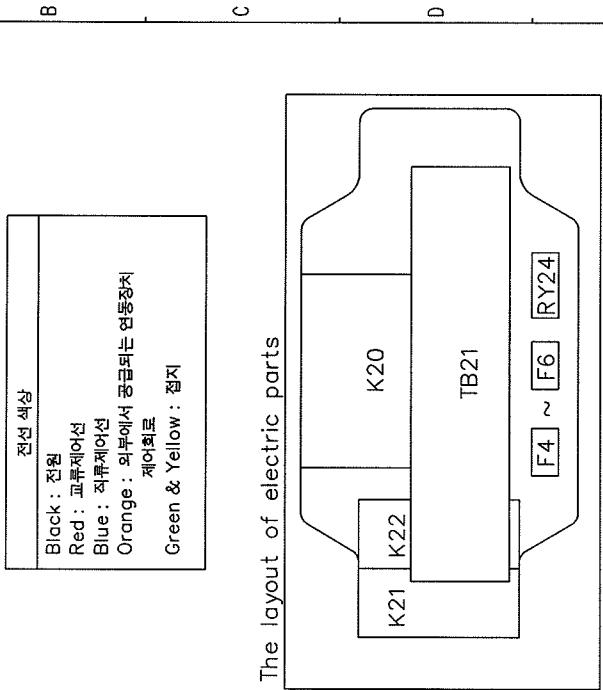
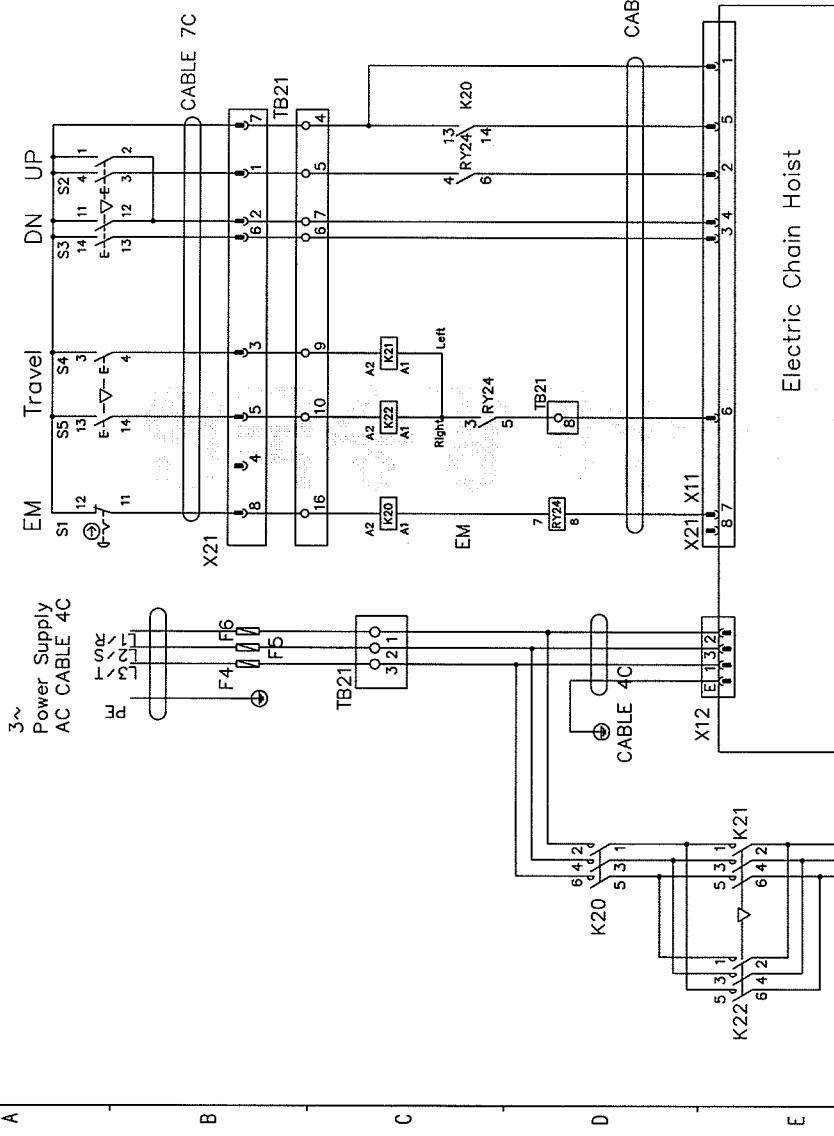
(2) 4점식이상 전기도면

- . 1속형 traversing
- . 2속형(INVERTER) traversing
- . 1속형 hoisting
- . 2속형(INVERTER) hoisting



. 1속형 hoisting/. 1속형 traversing

1 2 3 4 5 6 7 8 9 10

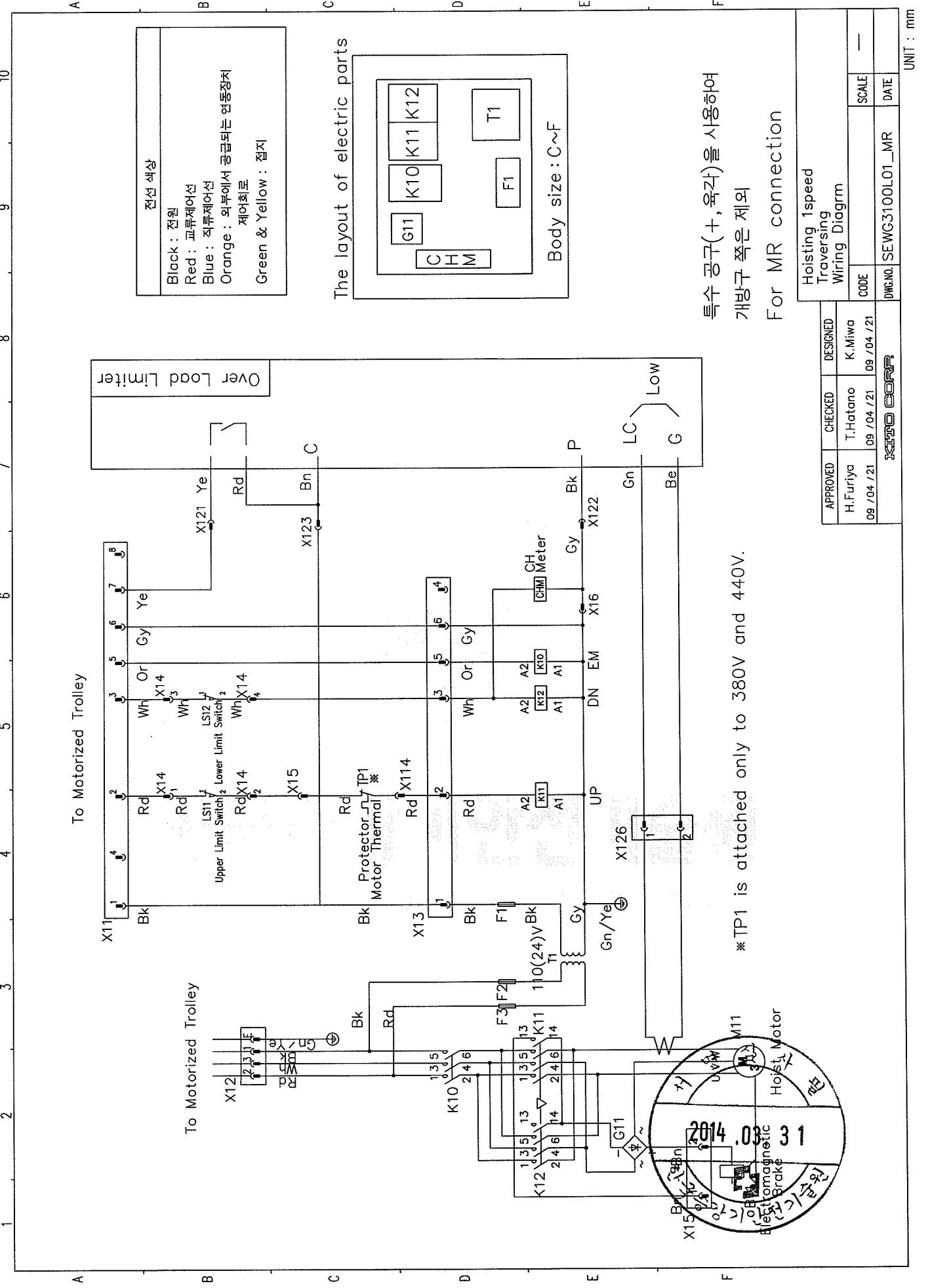


특수 공구(+, 끌자)을 사용하여
개방구 쪽은 제외

Traversing 1 speed

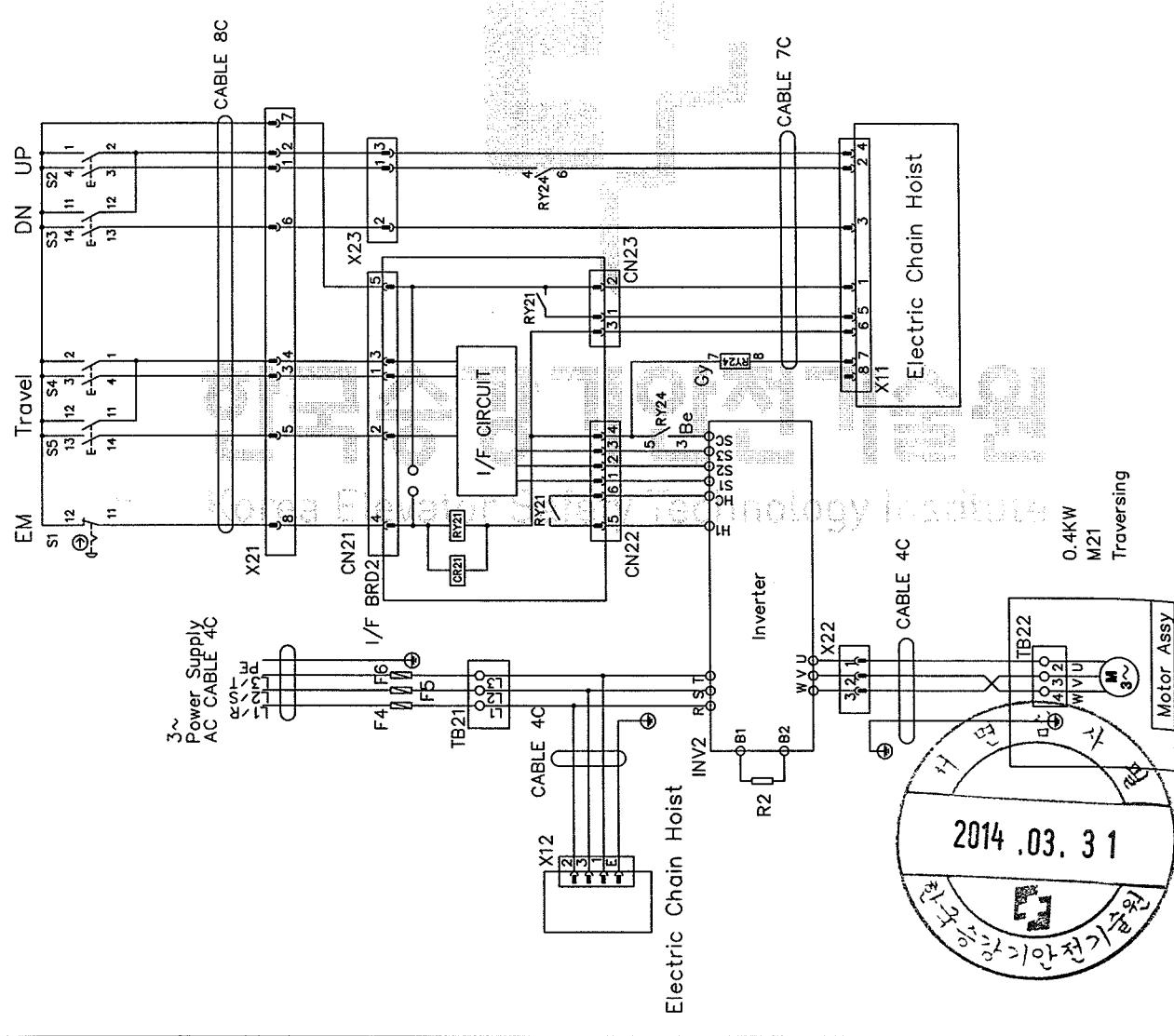
APPROVED	CHECKED	DESIGNED	Wiring Diagram	SCALE	DATE
H.Furiya 09/04/21 SEWTO CORP	H.Hatano 09/04/21	K.Miwa 09/04/21	Dwg. No. SEWC3DD0L01	—	—

UNIT : mm



52/201

. 1속형 hoisting/. 2속형 traversing



특수 공구(+, 육각)을 사용하여
개방구 쪽은 제외

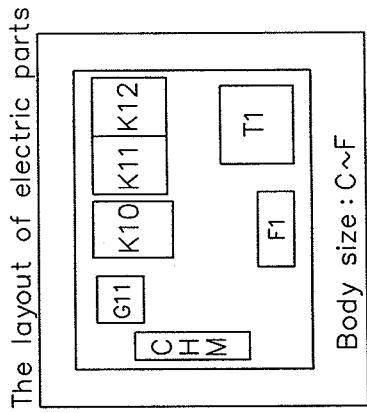
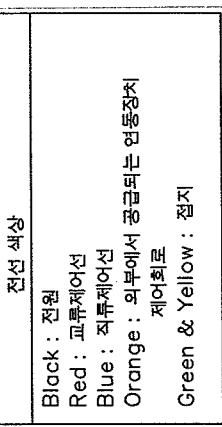
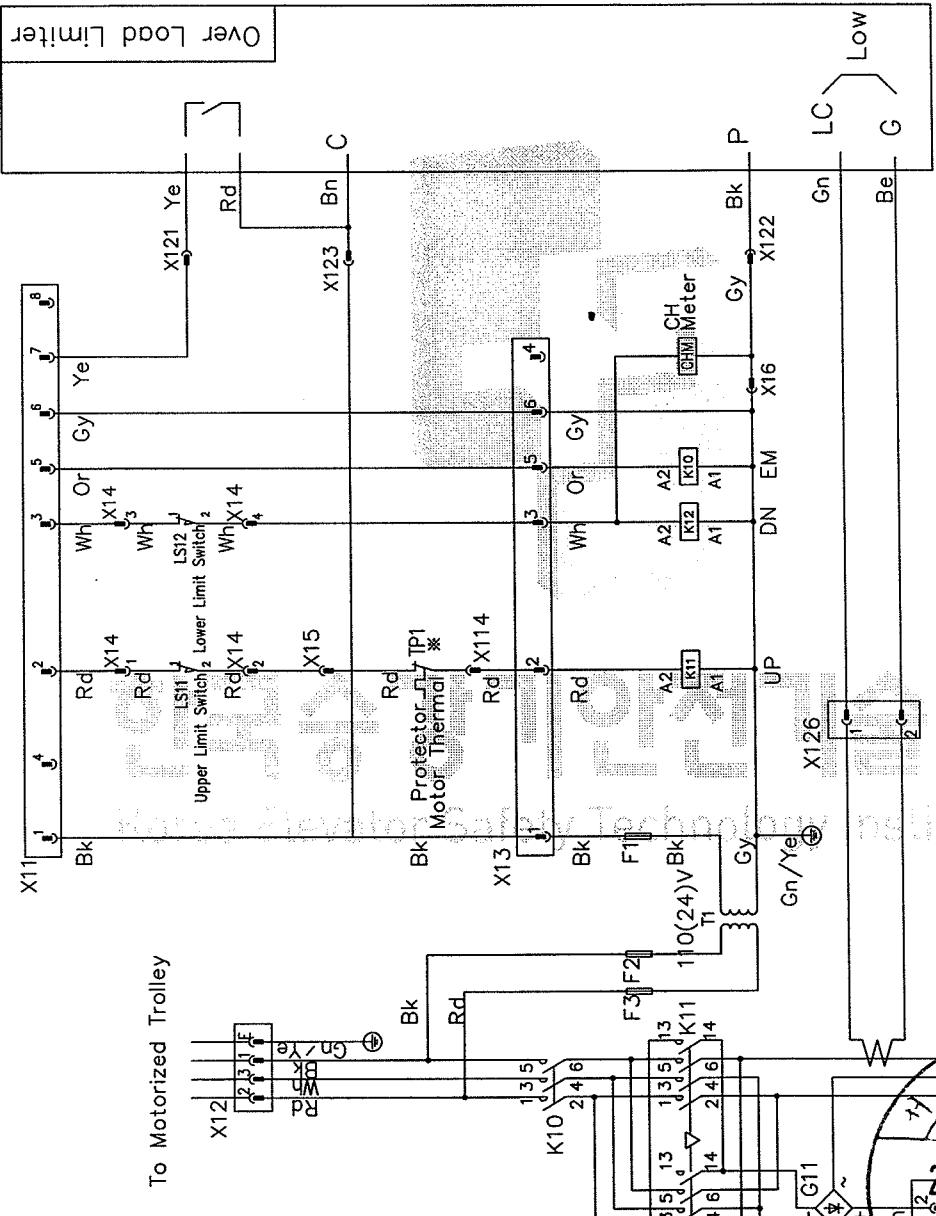
Traversing 2speed
Wiring Diagram

APPROVED CHECKED DE
 H.Furiya H.Hatano K
 09/04/21 09/04/21 09/04/21
KITTO CORP.

51/201

A 10
9
8
7
6
5
4
3
2
1

To Motorized Trolley



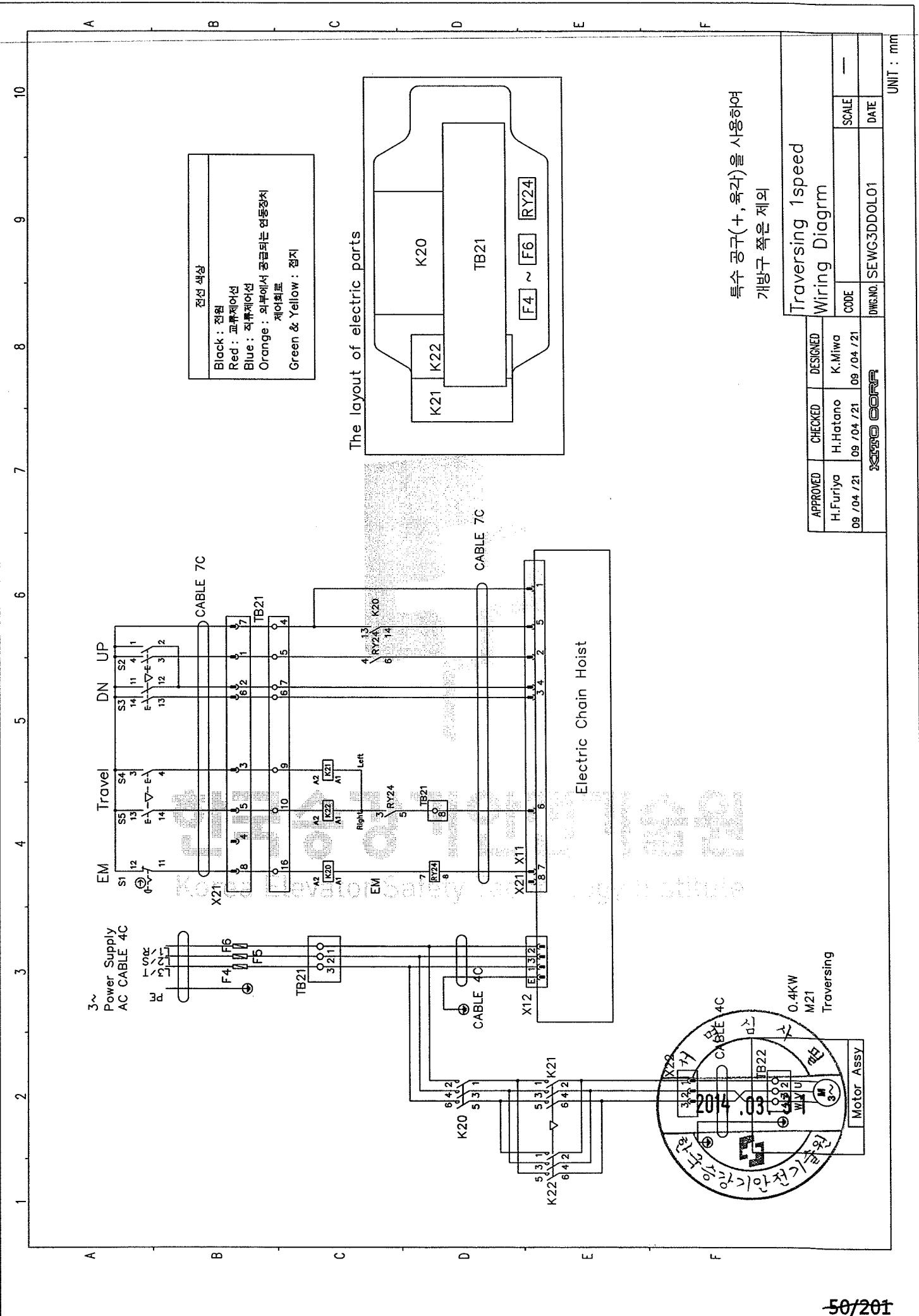
특수 공구(+, 육각)을 사용하여
개별구 쪽은 제외

For MR connection

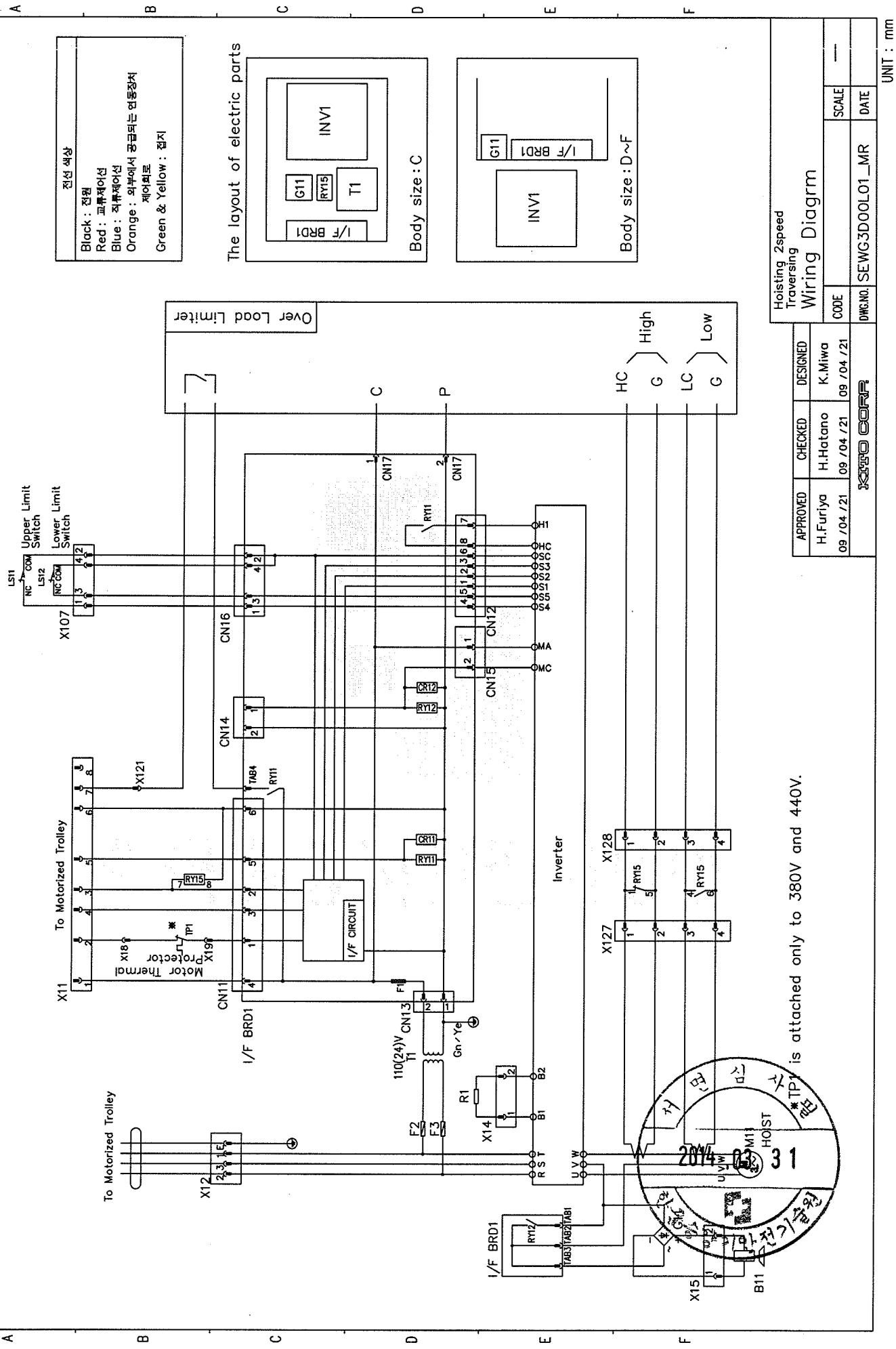
Hoisting 1 speed			
Traversing			
Wiring Diagram			
CODE		SCALE	—
DWG NO.	SEWNG3100L01_MR	DATE	

UNIT : mm

. 2속형 hoisting/. 1속형 traversing



1 2 3 4 5 6 7 8 9 10



. 2속형 hoisting/. 2속형 traversing

10
9
8
7
6
5
4
3
2
1

A

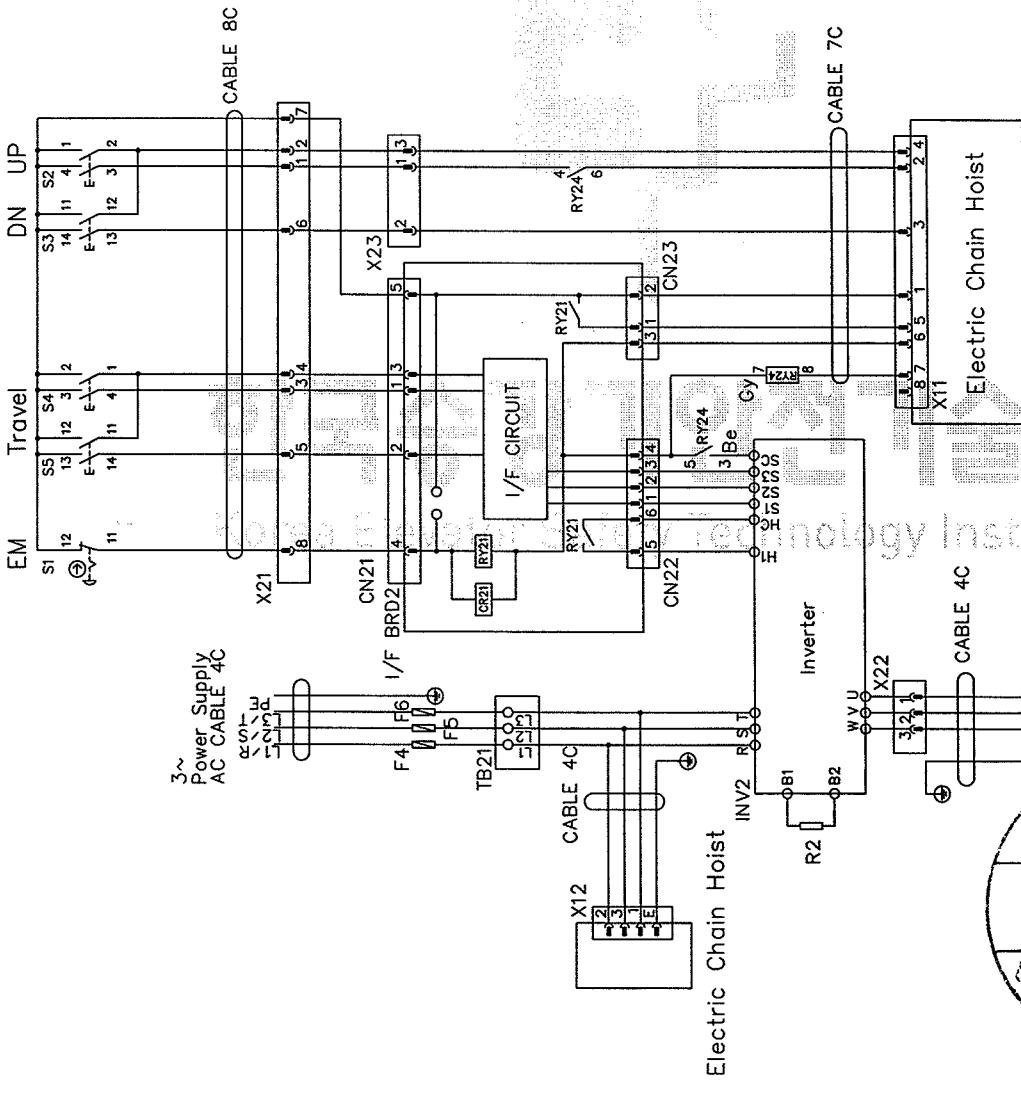
B

C

D

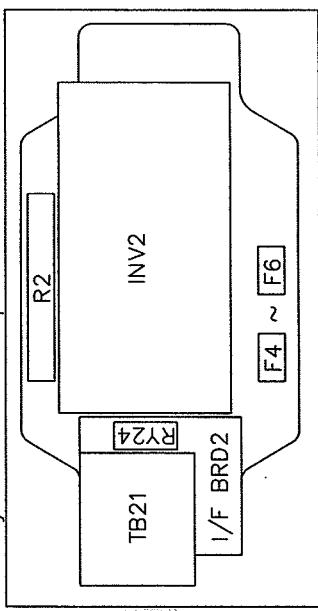
E

F



전선 색상	
Black	전원
Red	교류제어선
Blue	직류제어선
Orange	외부에서 공급되는 연동장치
Green & Yellow	제어회로
	접지

The layout of electric parts



특수 공구(+, 육각)을 사용하여
개방구 쪽은 제외

Traversing 2speed Wiring Diagram

APPROVED	CHECKED	DESIGNED	DATE
H.Furiya	H.Hatano	K.Miwa	—
09.04.21	09.04.21	09.04.21	
09.04.21	09.04.21	09.04.21	

DWG. NO. SEWC3DD01

UNIT : mm

10

9

8

7

6

5

4

3

2

1

A

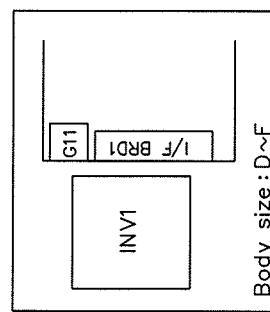
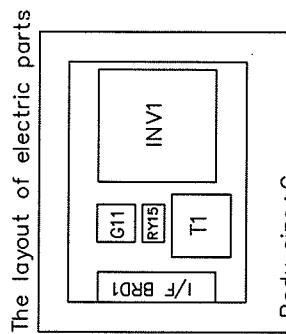
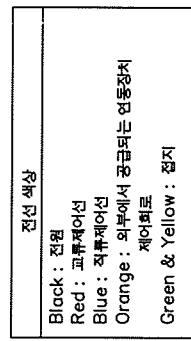
B

C

D

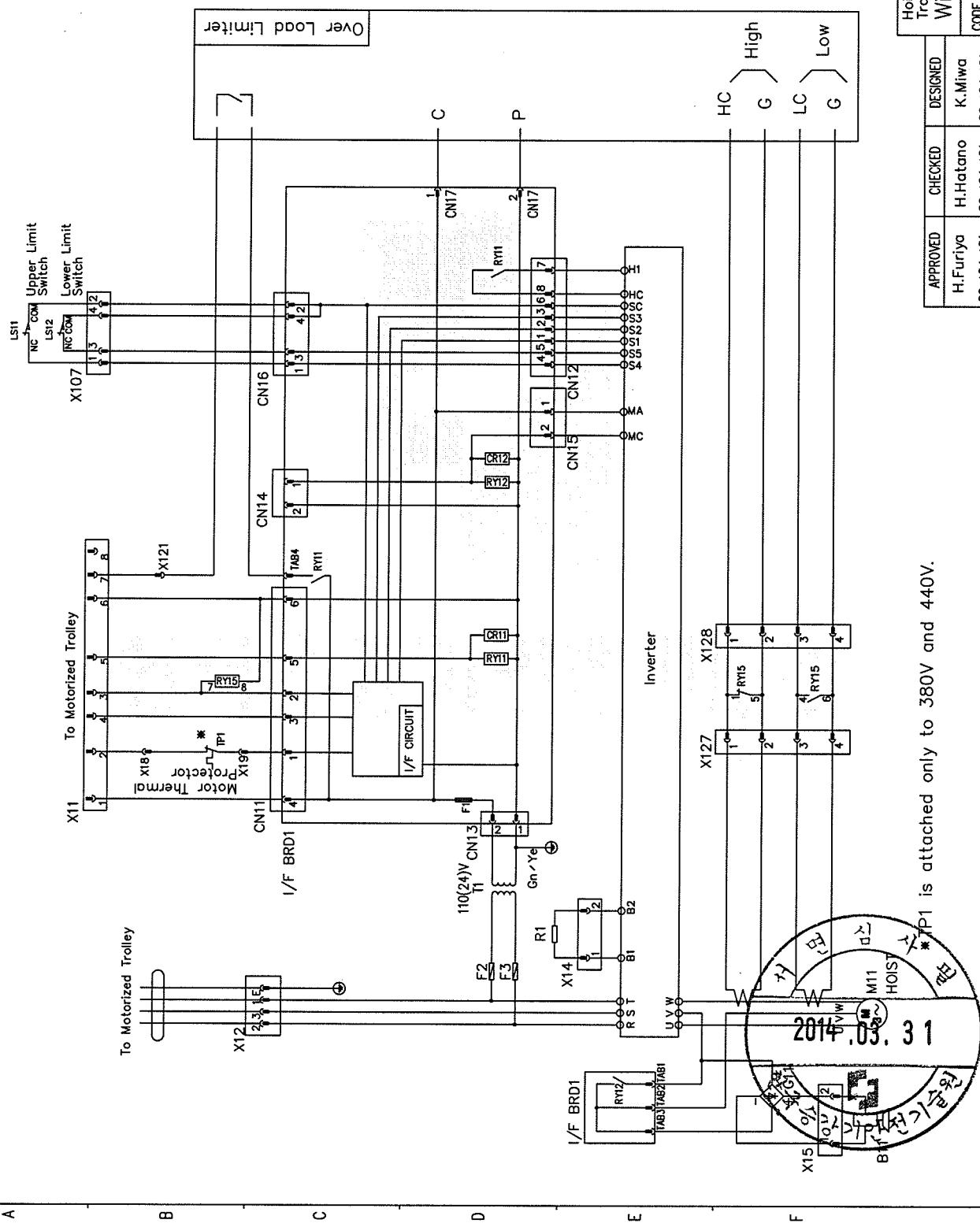
E

F



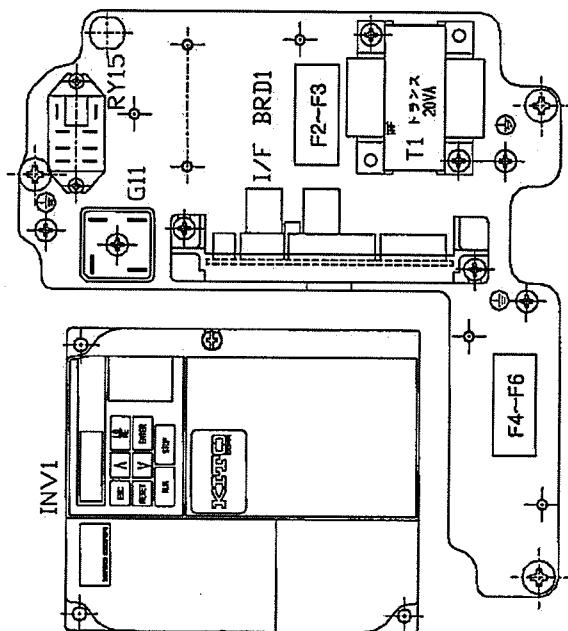
Body size : D~F

Hoisting 2speed Traversing Wiring Diagram		
APPROVED	CHECKED	DESIGNED
H.Furiya 09 /04 /21	H.Hatano 09 /04 /21	K.Miwa 09 /04 /21
DWGNO : SEWG3D00L01	CODE : —	SCALE : —
도면번호 : 0001	DATE : —	UNIT : mm



• 헉이스트 CONTROL BOX 배치도 (ER2-018IL)

HOISTING CONTROL BOX



ENCLOSURE : HOIST BODY – IP55
PUSH BUTTON – IP65

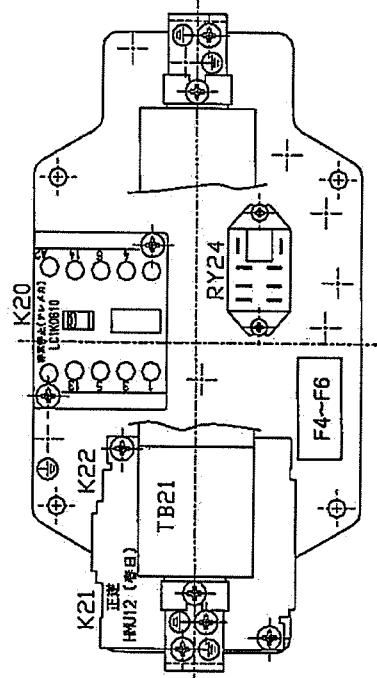
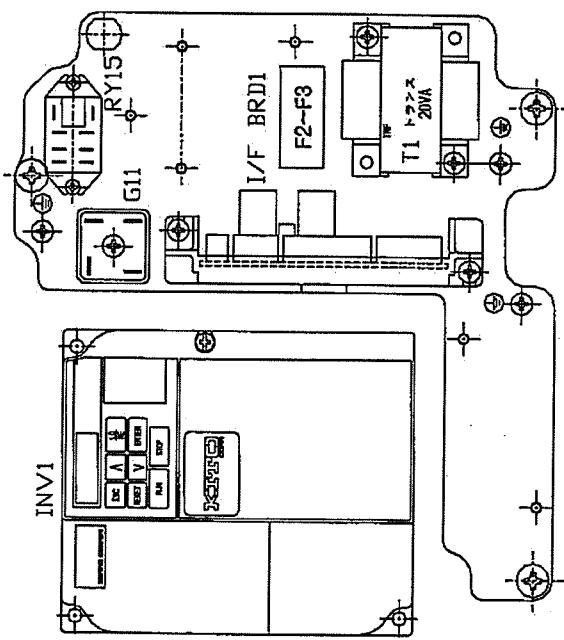
MARK	DESCRIPTION	TYPE OF MODEL	Q'TY	MAKER	REMARKS	
INV1	INVERTER	220V V1000	380V V1000	440V V1000	1 1	YASKAWA YASKAWA
I/F T1 2014	TRANSFORMER	220V/24V 20VA	380V/24V 20VA	440V/24V 20VA	1 1	KITO KITO
G11	BRIDGE DIODE	S15VB60	S15VB60	S15VB60	1 1	SHINDENGEN SHINDENGEN
I/F BRD1 F2~F3 F4~F6 RY15	INTERFACE BOARD	10~15A	10~15A	10~15A	1 1 1 1	KITO FUJI FUJI OMRON
	GLASS FUSE	10A	10A	10A	2 3 1	FUJI FUJI HIGH/LOW
	GLASS FUSE	20A	10A	10A	3	
	RELAY	24V	24V	24V	1	



호이스트 CONTROL BOX 배치도(ER2-018IL-S/L)

HOISTING CONTROL BOX

TRaversing Control Box

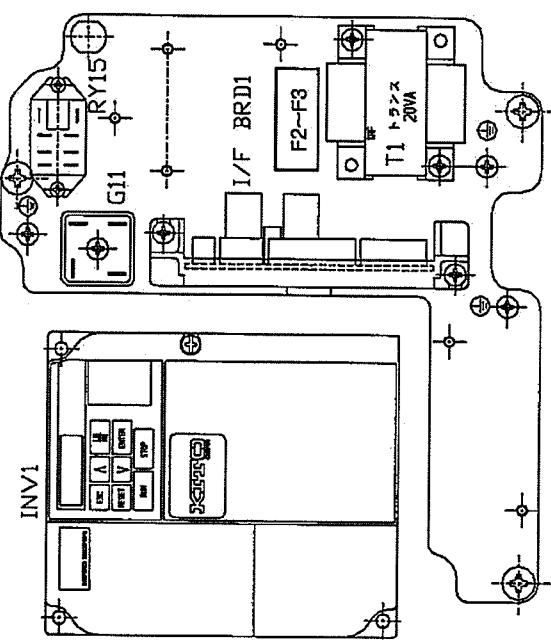


ENCLOSURE : HOIST BODY – IP55
PUSH BUTTON – IP65

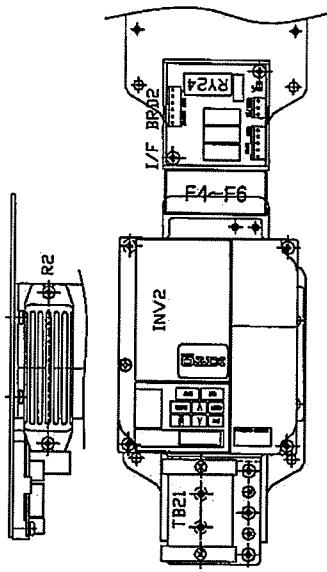
MARK	DESCRIPTION	TYPE OF MODEL	QTY	MAKER	REMARKS
INV1	INVERTER	220V V1000	380V V1000	440V V1000	YASKAWA
T1	TRANSFORMER	220V/24V 20VA	380V/24V 20VA	440V/24V 20VA	1 1 1
G11	BRIDGE DIODE	S15VB60	S15VB60	S15VB60	KITO SHINDENGEN
I/F BRD	INTERFACE BOARD	10~15A	10~15A	10~15A	KITO
F2~F3	Glass FUSE	10A	10A	10A	FUJI
F4~F6	Glass FUSE	20A	10A	10A	FUJI
RY15	RELAY	24V	24V	24V	OMRON
K20	MAGNET CONTACTOR	LCK0610B7	LCK0610B7	LCK0610B7	TELEMECANIQUE
K21	MAGNET CONTACTOR	HMU12	HMU12	HMU12	KASUGA
RY24	RELAY	24V	24V	24V	OMRON
TB21	TERMINAL BOARD 21	10~15A	10~15A	10~15A	KITO

• 헉이스 트리뷴 콘트롤 박스 배치도 (ER2-018IL-IS/IL)

HOISTING CONTROL BOX



TRAVERSING CONTROL BOX



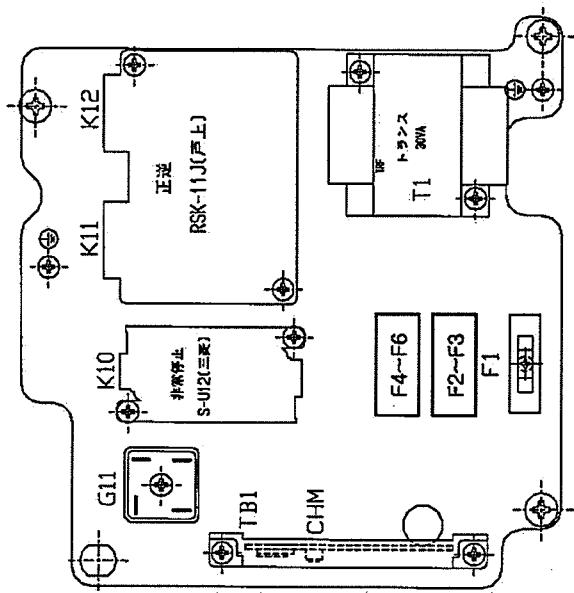
ENCLOSURE : HOIST BODY – IP55
PUSH BUTTON – IP65

MARK	DESCRIPTION	TYPE OF MODEL			Q'TY	MAKER	REMARKS
		220V	380V	440V			
INV1	INVERTER	V1000	V1000	V1000	1	YASKAWA	UP/DOWN
T1	TRANSFORMER	220V/24V 20VA	380V/24V 20VA	440V/24V 20VA	1	KITO	CONTROL CIRCUIT
G11	BRIDGE DIODE	S15VB60	S15VB60	S15VB60	1	SHINDENGEN	
I/F BRD1	INTERFACE BOARD	10~15A	10~15A	10~15A	1	KITO	
F3-F5	GLASS FUSE	10A	10A	10A	10A	FUJI	
F6	GLASS FUSE	20A	10A	10A	10A	FUJI	
RY15	RELAY	24V	24V	24V	1	OMRON	HIGH/LOW
-	INV2	V1000	V1000	V1000	1	YASKAWA	RIGHT/LEFT
J/F BRD2	INTERFACE BOARD	10~15A	10~15A	10~15A	1	KITO	
RY24	RELAY	24V	24V	24V	1	OMRON	EMERGENCY STOP
TB21	TERMINAL BOARD 21	10~15A	10~15A	10~15A	1	KITO	



호이스 콘트롤 박스 배치도 (ER2-018L)

HOISTING CONTROL BOX



ENCLOSURE : HOIST BODY – IP55
PUSH BUTTON – IP65

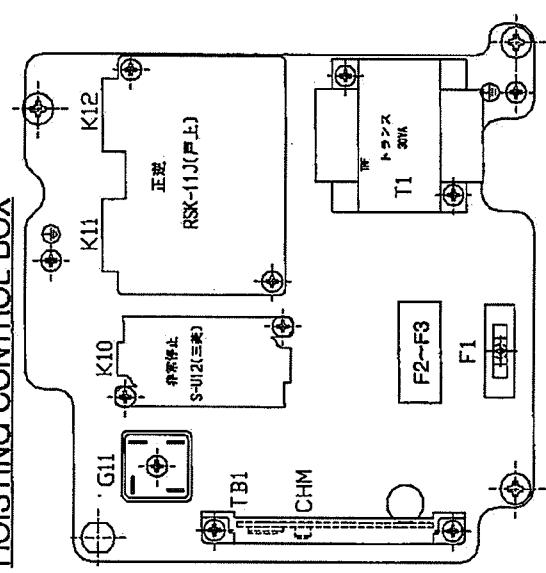
MARK	DESCRIPTION	TYPE OF MODEL			Q'TY	MAKER	REMARKS
		220V	380V	440V			
K10	MAGNET CONTACTOR	S-U12	S-U12	S-U12	1	MITSUBISHI	EMERGENCY STOP
K11,K12	MAGNET CONTACTOR	RSK-11J-S95	RSK-11J-S95	RSK-11J-S95	1	TOGAMI	UP/DOWN
T1	TRANSFORMER	220V/24V 30VA	380V/24V 30VA	440V/24V 30VA	1	KITO	CONTROL CIRCUIT
G11	GLASS FUSE	2A	2A	2A	1	FUJI	
F2-F3	GLASS FUSE	10A	10A	10A	2	FUJI	
F4-F6	GLASS FUSE	20A	10A	10A	3	FUJI	
G11	BRIDGE DIODE	S15VB60	S15VB60	S15VB60	1	SHINDENGEN	
TERMINAL BOARD		10~15A	10~15A	10~15A	1	KITO	
CHM	COUNTER HOUR METER	ECP91CHA1-3	ECP91CHA1-3	ECP91CHA1-3	1	OTEC	사용회수, 시간 기록

2014.03.31

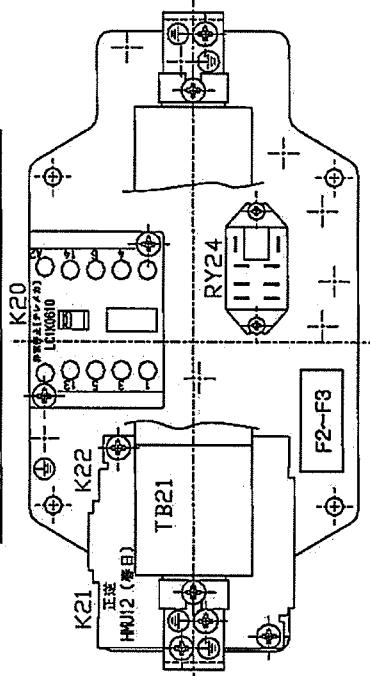


• 헉이스 트리뷴트 컨트롤 박스 배치도 (ER2-018L-L/S)

HOISTING CONTROL BOX

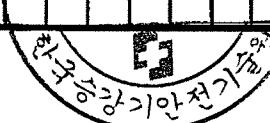


TRaversing CONTROL BOX



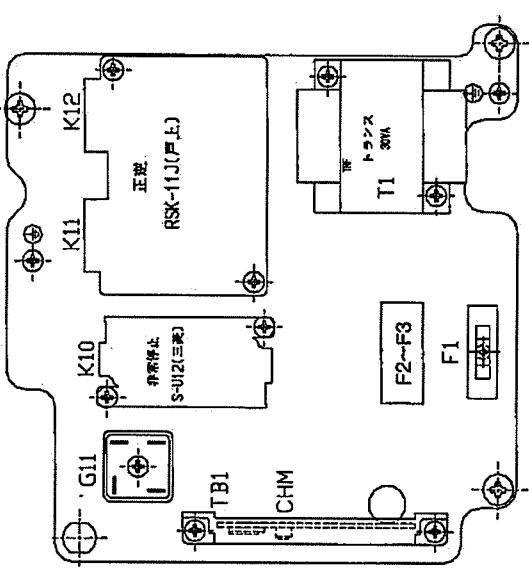
ENCLOSURE : HOIST BODY – IP55
PUSH BUTTON – IP65

MARK	DESCRIPTION	TYPE OF MODEL	Q'TY	MAKER	REMARKS	
K10	MAGNET CONTACTOR	S-U12	1	MITSUBISHI	EMERGENCY STOP	
K11,K12	MAGNET CONTACTOR	RSK-11J-S95	1	TOGAMI	UP/DOWN	
T1	TRANSFORMER	380V/24V 30VA	440V/24V 30VA	1	KITO	CONTROL CIRCUIT
F1	GLASS FUSE	2A	2A	1	FUJI	
F2~F3	GLASS FUSE	10A	10A	2	FUJI	
F4~F5	GLASS FUSE	20A	10A	10A	3	FUJI
G11	BRIDGE DIODE	\$15VB60	\$15VB60	1	SHINDENGEN	
TB1~TB8	TERMINAL BOARD	10~15A	10~15A	10~15A	1	KITO
C1~C3	COUNTER HOUR METER	ECP91CHA1-3	ECP91CHA1-3	1	OTEC	사용회수, 시간 기록
K20	MAGNET CONTACTOR	LC1K0610B7	LC1K0610B7	1	TELEMECANIQUE	EMERGENCY STOP
K24,K25	MAGNET CONTACTOR	HMU12	HMU12	1	KASIGA	RIGHT/LEFT
RY24	RELAY	24V	24V	1	OMRON	EMERGENCY STOP
TB21	TERMINAL BOARD 21	10~15A	10~15A	10~15A	1	KITO

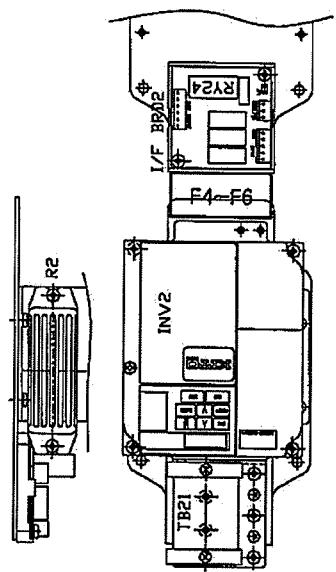


• 흐이스트 CONTROL BOX 배치도 (ER2-018L-IS/IL)

HOISTING CONTROL BOX



TRaversing CONTROL BOX

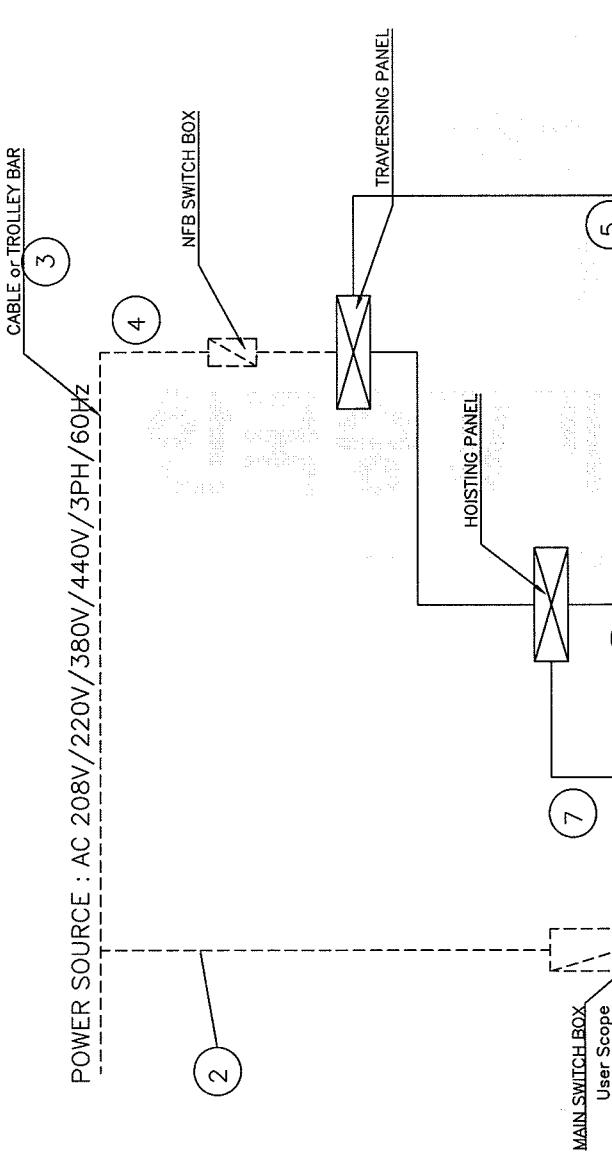


ENCLOSURE : HOIST BODY – IP55
PUSH BUTTON – IP65

MARK	DESCRIPTION	TYPE OF MODEL		Q'TY	MAKER	REMARKS	
		220V	380V				
K10	MAGNET CONTACTOR	S-U12	S-U12	1	MITSUBISHI	EMERGENCY STOP	
K11,K12	MAGNET CONTACTOR	RSK-11J-S95	RSK-11J-S95	1	TOGAMI	UP/DOWN	
T1	TRANSFORMER	220V/24V 30VA	380V/24V 30VA	1	KITO	CONTROL CIRCUIT	
F1	GLASS FUSE	2A	2A	1	FUJI		
F2-F3	GLASS FUSE	10A	10A	10A	2	FUJI	
F4-F6	GLASS FUSE	20A	10A	10A	3	FUJI	
G11	BRIDGE DIODE	S15VB60	S15VB60	1	SHINDENGEN		
TERMINAL BOARD	TERMINAL BOARD	10~15A	10~15A	10~15A	1	KITO	
CHM	COUNTER HOUR METER	ECP91CHA1-3	ECP91CHA1-3	1	OTEC	사용회수, 시간 기록	
INV2	INVERTER	V1000	V1000	V1000	1	YASKAWA	RIGHT/LEFT
I/F-BRD2	INTERFACE BOARD	10~15A	10~15A	10~15A	1	KITO	
RY24	RELAY	24V	24V	24V	1	OMRON	EMERGENCY STOP
TB2T	TERMINAL BOARD 21	10~15A	10~15A	10~15A	1	KITO	



1 2 3 4 5 6 7 8 9



첨지설비 시공방법

1. 전동기의 외형, 제어반 등을 접지를 해야 하며 그 접지상은 다른 의 규정을 따른다

접지공사	
3종 접지공사	400V 이하 100 Ω이하
특3종 접지공사	400V 이상 10 Ω이하

단 병목자역은 전압관계없이 10 Ω이하일것
2. 접지전용 Trolley Duct 및 전선은 당해 전기기기 기구에 대하여 충분한 용량과 전기적, 기계적인 강도를 가져야 함.

3. 접지선이 오상을 받을 우려가 있는 경우는 전선관, 합성수지관 등과 함께 사용한다

4. 접지공사는 지표면에서 최저 75cm이상의 깊이에 접지통을 박고 접지통에는 접지판을 연결한다.

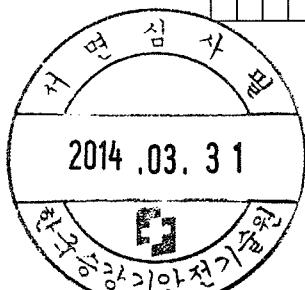
CABLE 종류 및 굽기

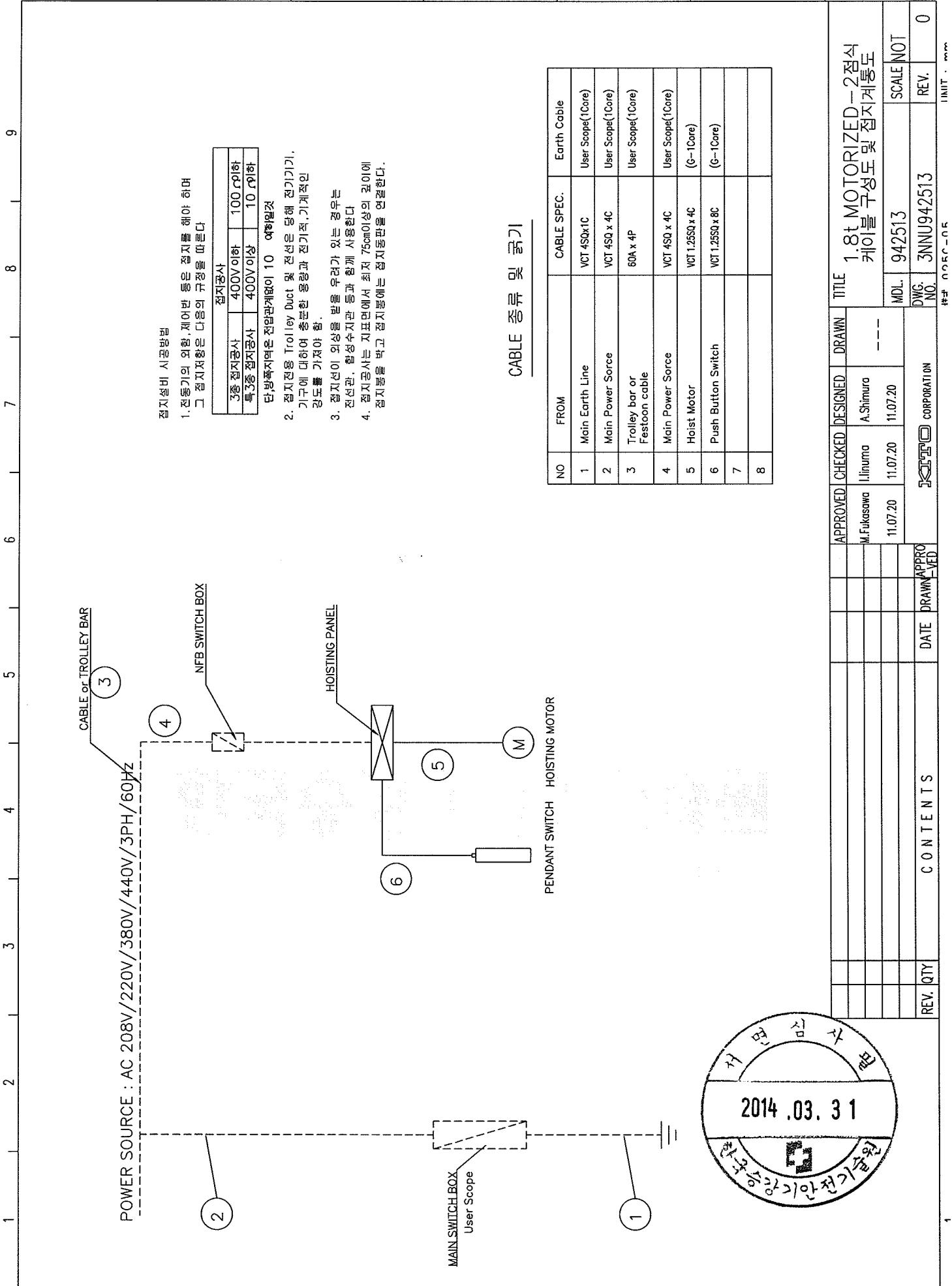
NO	FROM	CABLE SPEC.	Earth Cable
1	Main Earth Line	VCT 4SQx1G	User Scope(1Core)
2	Main Power Source	VCT 4SQ x 4C	User Scope(1Core)
3	Trolley bar or Festoon cable	60A x 4P	User Scope(1Core)
4	Main Power Source	VCT 4SQ x 4C	User Scope(1Core)
5	Traversing Motor	VCT 1.25SQ x 6C	(G-1Core)
6	Hoist Motor	VCT 1.25SQ x 4C	(G-1Core)
7	Push Button Switch	VCT 1.25SQ x 8C	(G-1Core)
8			

TITLE 1.8t MOTORIZED - 4점식 케이블 구성도 및 접지계통도			
MDL.	942513	SCALE	NOT
DWG.	3NNU942513	REV.	0
NO.	KITO CORPORATION		
DATE DRAWN APPROVED			
CONTENTS			
REV. QTY			

10050 - 05

1





CABLE 구성도 및 사용 - 권상 용량 1.8kW

CABLE SPECIFICATION FOR ER2M – 4점

NO	ITEM	TYPE	SIZE	ER2M018
①	Power Line	VCT	2sq x 4C	
②	Push Button Switch	VCT	1.25sq x 8C	
③	Load Limit	VCT	0.75sq x 8C	
④	Power Line for ER	VCT	1.25sq x 4C	
⑤	Control Line for ER	VCT	1.25sq x 6C	
⑥	Traversing Motor With Earth	VCT	1.25sq x 4C	

CABLE SPECIFICATION FOR ER2M – 2점

NO	ITEM	TYPE	SIZE	ER2M018
①	Power Line	VCT	2sq x 4C	
②	Push Button Switch	VCT	1.25sq x 8C	
③	Load Limit	VCT	0.75sq x 8C	
④	Power Line for ER	VCT	1.25sq x 4C	
⑤	Control Line for ER	VCT	1.25sq x 6C	

(3Φ 20(20)X / 380V / 440V 60Hz)

허용 최대 SPAN 적용표 (I-BEAM, H-BEAM)

PROJECT NAME : KML-ER2-018
 RATED LOAD : 1.8 ton
 DESCRIPTION : LIFT(max) 30 m

NO.	I-BEAM-SIZE (B*H*11*12)		Ix cm ⁴	Iy cm ⁴	Zx cm ³	Zy cm ³	A cm ²	Wb kg/m	Wh ton	Wg1 ton	L cm	L1 cm	L2 cm	b cm	E kg/cm ²	Φ °	Ψ °	$\Sigma\sigma_1$ TON/cm ²	$\Sigma\sigma_2$ TON/cm ²	$\Sigma\sigma_3$ TON/cm ²	$\Sigma\sigma_4$ TON/cm ²	61 < L / 800 -> O.K	62 < L / 500 -> O.K	61 < L / 800	62 < L / 500	결과
I - BEAM																										
1	200x100x7/10t	2170	138	217	27.7	33.06	26	0.16	0.083	0.034	320	130	30	13.2	2100000	1.11	1.1	0.927	1.215	0.505	0.816	0.301	0.400	0.1454	0.26 O.K	
2	250x125x7.5/12.5t	5180	337	414	53.9	48.79	38.3	0.16	0.192	0.069	500	180	30	13.2	2100000	1.11	1.1	0.804	0.962	0.553	0.630	0.498	0.625	0.2073	0.36 O.K	
3	300x150x10/18.5t	12700	886	849	118	83.47	65.5	0.16	0.491	0.183	750	280	30	13.2	2100000	1.11	1.1	0.638	0.802	0.472	0.515	0.747	0.938	0.4016	0.56 O.K	
4	350x150x12/24t	22400	1180	1280	158	111.1	87.2	0.16	0.82	0.296	940	340	30	13.2	2100000	1.11	1.1	0.582	0.686	0.516	0.504	0.910	1.175	0.4447	0.68 O.K	
5	400x150x12.5/25t	31700	1240	1580	165	122.1	95.8	0.16	1.054	0.383	1100	400	30	13.2	2100000	1.11	1.1	0.590	0.693	0.613	0.630	1.091	1.375	0.5431	0.80 O.K	
H - BEAM																										
6	300x150x6.5/9t	7210	508	481	67.7	40.8	32	0.16	0.192	0.064	600	200	30	13.2	2100000	1.11	1.1	0.839	0.931	0.602	0.573	0.602	0.750	0.2162	0.40 O.K	
7	350x175x7/11t	13500	984	771	112	62.91	49.4	0.16	0.395	0.148	800	300	30	13.2	2100000	1.11	1.1	0.732	0.945	0.517	0.596	0.830	1.000	0.4712	0.60 O.K	
8	400x200x8/13t	23500	1170	1170	174	83.37	65.4	0.16	0.654	0.249	1000	380	30	13.2	2100000	1.11	1.1	0.640	0.827	0.464	0.536	1.000	1.250	0.6022	0.76 O.K	
9	450x200x9/14t	32900	1460	1870	187	95.43	74.9	0.16	0.824	0.315	1100	420	30	13.2	2100000	1.11	1.1	0.594	0.761	0.501	0.576	0.993	1.375	0.6031	0.84 O.K	

- Ix X축의 단면 2차 모멘트
 Iy Y축의 단면 2차 모멘트
 Zx X축의 단면 계수
 Zy Y축의 단면 계수
 A BEAM의 단면 적
 Wb BEAM의 단위 중량
 Wh HOIST 자중
 Wg GIRDER 자중
 Lbbbbb 캔들레버 GIRDER 자중
 SPAN-PITCH LENGTH
 Lb LENGTH
 L1 캔들레버 LENGTH
 L2 HOOK APPROACH
 b WHEEL BASE OF HOIST
 E 중 탄성 계수
 Φ = M
 ψ = F
 정하중 계수
 중 탄성 계수
 정하중 계수
 (으)전기설비
 2014.03.31
1. PITCH내 계산응력 < 1.279 TON/CM² 이하일 경우 "O.K" (SS400 용접효율을 80% 적용, 풍하중 115% 적용)
 작업시 풍하중을 고려 허용응력에 115% 적용
 2. 작업시 풍하중을 고려 계산응력 < 1.600 TON/CM² 이하일 경우 "O.K" (SS400, 풍하중 115% 적용)
 (캔들레버 용접부위 없음)
 3. 휴지시 PITCH내 휴지시 풍하중을 고려 허용응력에 115% 적용
 PITCH내 휴지시 풍하중을 고려 계산응력 < 1.447 TON/CM² 이하일 경우 "O.K" (SS400, 풍하중 130% 적용)
 3. 휴지시 캔들레버 계산응력 < 1.808 TON/CM² 이하일 경우 "O.K" (SS400, 풍하중 130% 적용)
 (캔들레버 BEAM 연결 부위 없음)
 4. $\sigma_1 = (\text{PITCH내 계산응력}) \times 1.15$
 $\sigma_2 = (\text{캔들레버 계산응력}) \times 1.15$
 $\sigma_3 = (\text{PITCH내 계산응력}) \times 1.30$
 $\sigma_4 = (\text{캔들레버 계산응력}) \times 1.30$
 $\sigma_5 = (\text{캔들레버 계산응력}) \times 1.30$
 $\sigma_6 = (\text{캔들레버 계산응력}) \times 1.30$
 $\sigma_7 = (\text{캔들레버 계산응력}) \times 1.30$
 $\sigma_8 = (\text{캔들레버 계산응력}) \times 1.30$
 $\sigma_9 = (\text{캔들레버 계산응력}) \times 1.30$
 $\sigma_{10} = (\text{캔들레버 계산응력}) \times 1.30$
 $\sigma_{11} = (\text{캔들레버 계산응력}) \times 1.30$
 $\sigma_{12} = (\text{캔들레버 계산응력}) \times 1.30$
 IN DOOR or OUT DOOR
 AREA CLASSIFICATION :

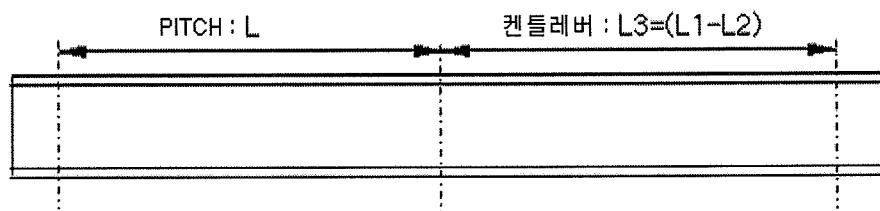
< L / 800 이하일 경우 "O.K"
 < L / 500 이하일 경우 "O.K"

1. I-BEAM 최대허용 가능 SPAN 계산

1. SPECIFICATION

.정격하중	-----	Q =	1.8	ton
.SPAN (PITCH)	-----	L=	320	cm
.켄틸리버	-----	L1=	130	cm
.TROLLEY WHEEL BASE	-----	B=	13.2	cm
.WEIGHT OF HOIST	-----	Wh=	0.16	ton
.PITCH내의 GIRDER 무게	-----	Wg=	0.083	ton
.켄틸리버의 GIRDER 무게	-----	Wg1=	0.034	ton
.I-BEAM SIZE	-----	200x100x7/10t		
		I _x =	2170	cm ⁴
		I _y =	138	cm ⁴
		Z _x =	217	cm ³
		Z _y =	27.7	cm ³
		A =	33.06	cm ²
		W _b =	26	kg/m
.HOOK APPROACH	-----	L ₂ =	30	cm
탄성계수	-----	E=	2100000	kg/cm ²
동하중 계수(작업계수)	-----	M(ϕ)=	1.11	
정하중 계수(총격계수)	-----	F(ψ)=	1.10	

1. DESIGN



2. I-BEAM에 작용하는 하중

$$P = Q + Wh = 1.8 + 0.16 = 1.96 \text{ ton}$$

3. 수직하중에 의한 BENDING MOMENT

1) PITCH 지점내 BENDING MOMENT

$$\begin{aligned} M_{h1} &= F \times M \times P \times (L - B/2)^2 / (4 \times L) \\ &= 1.11 \times 1.1 \times 1.96 \times (320 - 13.2/2)^2 / (4 \times 320) \end{aligned}$$



$$= 183.6 \text{ ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$M_{h2} = F \times M \times P \times L_3$$

$$= 1.11 \times 1.1 \times 1.96 \times (130 - 30) = 239.32 \text{ ton.cm}$$

4. 자중에 의한 BENDING MOMENT(I-BEAM)

1) PITCH 지점내 BENDING MOMENT

$$M_{t1} = F \times W_g \times L / 8 = 1.11 \times 0.083 \times 320 / 8 = 3.685 \text{ ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$M_{t2} = F \times W_g \times L_1 / 2 = 1.11 \times 0.034 \times 130 / 2 = 2.45 \text{ ton.cm}$$

5. 수직하중에 의한 최대 BENDING MOMENT

$$M_{max1} = M_{h1} + M_{t1} = 187.3 \text{ ton.cm}$$

$$M_{max2} = M_{h2} + M_{t2} = 241.8 \text{ ton.cm}$$

6. 풍하중에 의한 수평하중의 BENDING MOMENT

(1) GIRDER에 의한 수직 풍하중

$$\text{작업시 } V=16\text{m/s}, q = 19.9 \text{ kg/m}^2 \quad h(\text{최고양정}) = 30 \text{ M}$$

$$\text{작업시 } q = 8.5 \times 4 \sqrt{h} = 8.5 \times 4 \sqrt{42} = 19.9 \text{ kg/m}^2$$

$$H=(\text{GIRDER 높이}) \quad 0.25 \text{ m}$$

$$\text{PITCH내 풍하중} = F \times L \times H \times q \times 1.4 = 1.11 \times 3.2 \times 0.25 \times 19.9 \times 1.4 = 25 \text{ kg}$$

$$\text{캔들레버 풍하중} = F \times L_1 \times H \times q \times 1.3 = 1.11 \times 1.3 \times 0.25 \times 19.9 \times 1.3 = 9 \text{ kg}$$

$$M_{FGG} = \frac{0.025 \times 320}{8} = \frac{0.009 \times 130}{2} = 0.415 \text{ ton.cm}$$

$$M_{FG1} = \frac{0.009 \times 130}{2} = 0.585 \text{ ton.cm}$$

(2) HOIST에 의한 풍하중

* 작업시

$$HH= 1.0 \text{ m} \quad HB= 0.65 \text{ m} \quad \text{풍하중} = F \times HB \times HH \times q \times 1.2 = 1.11 \times 0.65 \times 1 \times 19.9 \times 1.2 =$$

*PITCH내 풍하중

$$M_{FHG} = 17 \times 320 / 4 = 1360 \text{ kg.cm} =$$

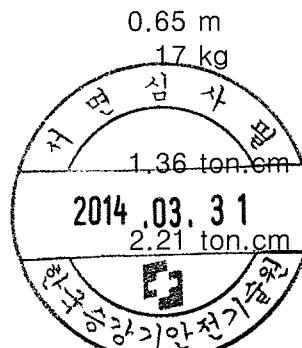
*캔들레버 풍하중

$$M_{FH1} = 17 \times 130 = 2210 \text{ kg.cm} =$$

7. COMBINED MOMENT

*PITCH내

$$M_{HCG} = M_{FGG} + M_{FHG} = 0.415 + 1.36 = 1.775 \text{ ton.cm}$$



*肯들레버

$$M_{HC1} = M_{FG1} + M_{FH1} = 0.585 + 2.21 = 2.795 \text{ ton.cm}$$

8. BENDING STRESS

A. VERTICAL BENDING STRESS

1. PITCH 내

$$\sigma v1 = M_{max1}/Zx = 187.3 / 217 = 0.863 \text{ ton/cm}^2$$

2. 켄들레버

$$\sigma v2 = M_{max2}/Zx = 241.8 / 217 = 1.114 \text{ ton/cm}^2$$

B. 수평 최대 응력

1. PITCH 내

$$\sigma v3 = M_{HCG}/Zy = 1.775 / 27.7 = 0.064 \text{ ton/cm}^2$$

2. 켄들레버

$$\sigma v4 = M_{HCl}/Zy = 2.795 / 27.7 = 0.101 \text{ ton/cm}^2$$

9. 합성 응력

$$\Sigma \sigma 1 = \sigma v1 + \sigma v3 = 0.927 \\ 0.927 < 1.279 \text{ ton/cm}^2 \quad \text{OK}$$

$$\Sigma \sigma 2 = \sigma v2 + \sigma v4 = 1.215 \\ 1.215 < 1.6 \text{ ton/cm}^2 \quad \text{OK}$$

용접효율 : 80% 응력 $1391 \times 80\% \times 115\% = 1.279 \text{ ton/cm}^2$
켄들레버는 용접부 없음 $1391 \times 1.15(\text{풍하중포함}) = 1.600 \text{ ton/cm}^2$

10. 휴지시 풍하중에 의한 BENDING MOMENT

휴지시 $V=45 \text{ m/s}$, $q = 158.0 \text{ kg/m}^2$, $h(\text{최고양정}) = 30 \text{ m}$
 $q = M \times 4 \sqrt{h} = 67.5 \times 4 \sqrt{30} = 158.0 \text{ kg/m}^2$
 $M = V^2 / 30 = 67.5$

* 휴지시 풍하중에 의한 수평하중

$$\text{PITCH내 풍하중} = F \times L \times H \times q \times 1.4 = 1.11 \times 3.2 \times 0.25 \times 158 \times 1.4 = 196 \text{ kg}$$

$$\text{켄들레버에 대한 풍하중} = F \times L \times H \times q \times 1.3 = 1.11 \times 1.3 \times 0.25 \times 158 \times 1.3 = 74 \text{ kg}$$

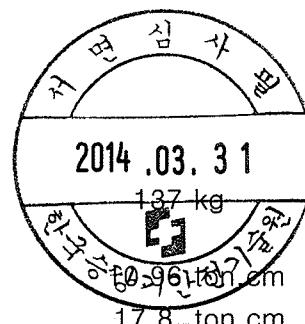
$$MM_{G1} = \frac{0.196 \times 320}{8} - \frac{0.074 \times 130}{2} = 3.03 \text{ ton.cm}$$

$$MM_1 = \frac{0.074 \times 130}{2} = 4.81 \text{ ton.cm}$$

$$\text{HOIST에 대한 풍하중} = F \times H \times B \times H \times q \times 1.2 = 1.11 \times 0.65 \times 1 \times 158 \times 1.2 =$$

$$* \text{PITCH내 } MM_{H0} = 137 \times 320 / 4 = 43840 \text{ KG.CM} =$$

$$* \text{켄들레버 } MM_{H1} = 137 \times 130 = 17810 \text{ KG.CM} =$$



* COMBINED MOMENT

$$\begin{aligned} MM_2 &= MM_{G1} + MM_{H0} = & 3.03 + 10.96 &= 13.99 \text{ ton.cm} \\ MM_4 &= MM_1 + MM_{H1} = & 4.81 + 17.8 &= 22.61 \text{ ton.cm} \end{aligned}$$

* BENDING STRESS

$$\begin{aligned} \Sigma\sigma_3 = MM_2 / Zy &= 13.99 / 27.7 = 0.505 \text{ ton/cm}^2 &< 1.447 \text{ ton/cm}^2 &\text{--O.K} \\ \Sigma\sigma_4 = MM_4 / Zy &= 22.61 / 27.7 = 0.816 \text{ ton/cm}^2 &< 1.808 \text{ ton/cm}^2 &\text{--O.K} \end{aligned}$$

PITCH내 휴지시 응력 1391 x 80% x130% = 1447 ton/cm²
 켄틀레버는 용접부 없음 휴지시 응력 1391 x 130% = 1808 ton/cm²

11. DEFLECTION OF GIRDER

* PITCH 내

1) DUE TO RATED & TROLLEY LOAD

$$D1 = \frac{(Q+Q2)x(L)^3}{48xEIx} = 0.2936 \text{ cm}$$

2) DUE TO DEAD LOAD

$$D2 = \frac{5 \times Wgx L^3}{384 \times E \times Ix} = 0.0078 \text{ cm}$$

3) TOTAL DEFLECTION

$$.81 = D1 + D2 = 0.301 \text{ cm}$$

$$\text{RATIO : } D3/L = 1/1062 < 800 \text{ ---- O.K}$$

* 켄틀레버

1) DUE TO RATED & TROLLEY LOAD

$$D1 = \frac{(Q+Q2)x(L1-L2)^3}{3xEIx} = 0.1434 \text{ cm}$$

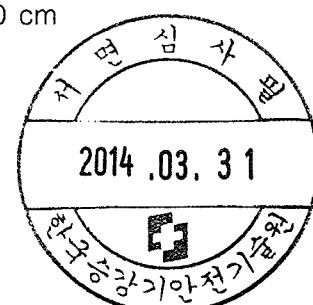
2) DUE TO DEAD LOAD

$$D2 = \frac{Wg1x L1^3}{8 \times E \times Ix} = 0.0020 \text{ cm}$$

3) TOTAL DEFLECTION

$$.82 = D1 + D2 = 0.1454 \text{ cm}$$

$$\text{RATIO : } D3/L = 1/894 < 500 \text{ ---- O.K}$$

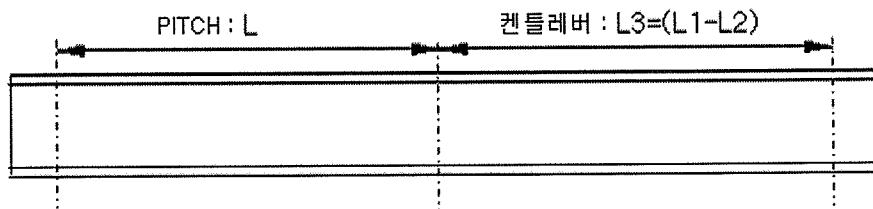


2. I-BEAM 최대허용 가능 SPAN 계산

1. SPECIFICATION

.정격하중	-----	Q =	1.8	ton
.SPAN (PITCH)	-----	L=	500	cm
.켄틸리버	-----	L1=	180	cm
.TROLLEY WHEEL BASE	-----	B=	13.2	cm
.WEIGHT OF HOIST	-----	Wh=	0.16	ton
.PITCH내의 GIRDER 무게	-----	Wg=	0.192	ton
.켄틸리버의 GIRDER 무게	-----	Wg1=	0.069	ton
.I-BEAM SIZE	-----	250x125x7.5/12.5t		
		I _x =	5180	cm ⁴
		I _y =	337	cm ⁴
		Z _x =	414	cm ³
		Z _y =	53.9	cm ³
		A =	48.79	cm ²
		W _b =	38.3	kg/m
.HOOK APPROACH	-----	L ₂ =	30	cm
탄성계수	-----	E=	2100000	kg/cm ²
동하중 계수(작업계수)	-----	M(ϕ)=	1.11	
정하중 계수(총격계수)	-----	F(ψ)=	1.10	

1. DESIGN



2. I-BEAM에 작용하는 하중

$$P = Q + Wh = 1.8 + 0.16 = 1.96 \text{ ton}$$

3. 수직하중에 의한 BENDING MOMENT

1) PITCH 지점내 BENDING MOMENT

$$\begin{aligned} M_{h1} &= F \times M \times P \times (L - B/2)^2 / (4 \times L) \\ &= 1.11 \times 1.1 \times 1.96 \times (500 - 13.2/2)^2 / (4 \times 500) \end{aligned}$$



$$= 291.3 \text{ ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$M_{h2} = F \times M \times P \times L_3$$

$$= 1.11 \times 1.1 \times 1.96 \times (180 - 30) = 358.97 \text{ ton.cm}$$

4. 자중에 의한 BENDING MOMENT(I-BEAM)

1) PITCH 지점내 BENDING MOMENT

$$M_{t1} = F \times W_g \times L / 8 = 1.11 \times 0.192 \times 500 / 8 = 13.32 \text{ ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$M_{t2} = F \times W_g \times L_1 / 2 = 1.11 \times 0.069 \times 180 / 2 = 6.89 \text{ ton.cm}$$

5. 수직하중에 의한 최대 BENDING MOMENT

$$M_{max1} = M_{h1} + M_{t1} = 304.6 \text{ ton.cm}$$

$$M_{max2} = M_{h2} + M_{t2} = 365.9 \text{ ton.cm}$$

6. 풍하중에 의한 수평하중의 BENDING MOMENT

(1) GIRDER에 의한 수직 풍하중

$$\text{작업시 } V=16\text{m/s}, q = 19.9 \text{ kg/m}^2 \quad h(\text{최고양정}) = 30 \text{ M}$$

$$\text{작업시 } q = 8.5 \times 4 \sqrt{h} = 8.5 \times 4 \sqrt{42} = 19.9 \text{ kg/m}^2$$

$$H(\text{GIRDER 높이}) = 0.25 \text{ m}$$

$$\text{PITCH내 풍하중} = F \times L \times H \times q \times 1.6 = 1.11 \times 5 \times 0.25 \times 19.9 \times 1.6 = 44 \text{ kg}$$

$$\text{켄틀레버 풍하중} = F \times L_1 \times H \times q \times 1.3 = 1.11 \times 1.8 \times 0.25 \times 19.9 \times 1.3 = 13 \text{ kg}$$

$$M_{FGG} = \frac{0.044 \times 500}{8} = \frac{0.013 \times 180}{2} = 1.58 \text{ ton.cm}$$

$$M_{FG1} = \frac{0.013 \times 180}{2} = 1.17 \text{ ton.cm}$$

(2) HOIST에 의한 풍하중

* 작업시

$$HH = 1.0 \text{ m} \quad HB = 0.65 \text{ m} \quad 풍하중 = F \times HB \times HH \times q \times 1.2 = 1.11 \times 0.65 \times 1 \times 19.9 \times 1.2 =$$

*PITCH내 풍하중

$$M_{FHG} = 17 \times 500 / 4 = 2125 \text{ kg.cm} =$$

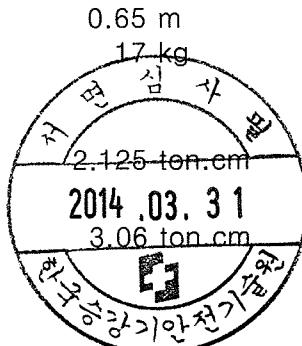
*肯틀레버 풍하중

$$M_{FH1} = 17 \times 180 = 3060 \text{ kg.cm} =$$

7. COMBINED MOMENT

*PITCH내

$$M_{HCG} = M_{FGG} + M_{FHG} = 1.58 + 2.125 = 3.705 \text{ ton.cm}$$



*肯들레버

$$M_{HC1} = M_{FG1} + M_{FH1} = 1.17 + 3.06 = 4.230 \text{ ton.cm}$$

8. BENDING STRESS

A. VERTICAL BENDING STRESS

1. PITCH 내

$$\sigma v1 = M_{max1}/Zx = 304.6 / 414 = 0.736 \text{ ton/cm}^2$$

2. 켄들레버

$$\sigma v2 = M_{max2}/Zx = 365.9 / 414 = 0.884 \text{ ton/cm}^2$$

B. 수평 최대 응력

1. PITCH 내

$$\sigma v3 = M_{HCG}/Zy = 3.705 / 53.9 = 0.069 \text{ ton/cm}^2$$

2. 켄들레버

$$\sigma v4 = M_{HC1}/Zy = 4.23 / 53.9 = 0.078 \text{ ton/cm}^2$$

9. 합성 응력

$$\Sigma \sigma 1 = \sigma v1 + \sigma v3 = 0.804 \\ 0.804 < 1.279 \text{ ton/cm}^2 \quad \text{OK}$$

$$\Sigma \sigma 2 = \sigma v2 + \sigma v4 = 0.962 \\ 0.962 < 1.6 \text{ ton/cm}^2 \quad \text{OK}$$

용접효율 : 80% 응력 $1391 \times 80\% \times 115\% = 1.279 \text{ ton/cm}^2$
 켄들레버는 용접부 없음 $1391 \times 1.15(\text{풍하중포함}) = 1.600 \text{ ton/cm}^2$

10. 휴지시 풍하중에 의한 BENDING MOMENT

휴지시 $V=45 \text{ m/s}$, $q = 158.0 \text{ kg/m}^2$, $h(\text{최고양정}) = 30 \text{ m}$
 $q = M \times 4 \sqrt{h} = 67.5 \times 4 \sqrt{30} = 158.0 \text{ kg/m}^2$
 $M = V^2 / 30 = 67.5$

*휴지시 풍하중에 의한 수평하중

$$\text{PITCH내 풍하중} = F \times L \times H \times q \times 1.6 = 1.11 \times 5 \times 0.25 \times 158 \times 1.6 = 351 \text{ kg}$$

$$\text{켄들레버에 대한 풍하중} = F \times L \times H \times q \times 1.3 = 1.11 \times 1.8 \times 0.25 \times 158 \times 1.3 = 103 \text{ kg}$$

$$MM_{G1} = \frac{0.351 \times 500}{8} - \frac{0.103 \times 180}{2} = 12.6675 \text{ ton.cm}$$

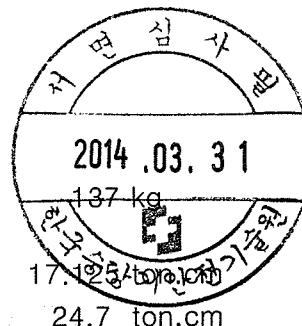
$$MM_1 = \frac{0.103 \times 180}{2} = 9.27 \text{ ton.cm}$$

$$\text{HOIST에 대한 풍하중} = F \times H \times B \times H \times q \times 1.2 = 1.11 \times 0.65 \times 1 \times 158 \times 1.2 =$$

$$* \text{PITCH내 } MM_{H0} = 137 \times 500 / 4 = 68500 \text{ KG.CM} =$$

$$* \text{켄들레버 } MM_{H1} = 137 \times 180 = 24660 \text{ KG.CM} =$$

* COMBINED MOMENT



$$\begin{aligned} MM_2 &= MM_{G1} + MM_{H0} = & 12.6675 + 17.125 &= 29.7925 \text{ ton.cm} \\ MM_4 &= MM_1 + MM_{H1} = & 9.27 + 24.7 &= 33.97 \text{ ton.cm} \end{aligned}$$

* BENDING STRESS

$$\begin{aligned} \Sigma \sigma_3 &= MM_2 / Zy = 29.7925 / 53.9 = 0.553 \text{ ton/cm}^2 &< 1.447 \text{ ton/cm}^2 \text{---O.K} \\ \Sigma \sigma_4 &= MM_4 / Zy = 33.97 / 53.9 = 0.630 \text{ ton/cm}^2 &< 1.808 \text{ ton/cm}^2 \text{---O.K} \end{aligned}$$

PITCH내 휴지시 응력 1391 x 80% x130% = 1447 ton/cm²
 켄틀레버는 용접부 없음 휴지시 응력 1391 x 130% = 1808 ton/cm²

11. DEFLECTION OF GIRDERS

* PITCH 내

1) DUE TO RATED & TROLLEY LOAD

$$D1 = \frac{(Q+Q2)x(L)^3}{48 \times E \times I_x} = 0.4692 \text{ cm}$$

2) DUE TO DEAD LOAD

$$D2 = \frac{5 \times Wgx L^3}{384 \times E \times I_x} = 0.0287 \text{ cm}$$

3) TOTAL DEFLECTION

$$.\delta_1 = D1 + D2 = 0.498 \text{ cm}$$

$$\text{RATIO : } D3/L = 1/1004 < 800 \text{ ----- O.K}$$

* 켄틀레버

1) DUE TO RATED & TROLLEY LOAD

$$D1 = \frac{(Q+Q2)x(L1-L2)^3}{3 \times E \times I_x} = 0.2027 \text{ cm}$$

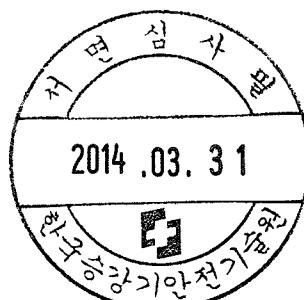
2) DUE TO DEAD LOAD

$$D2 = \frac{Wg1x L1^3}{8 \times E \times I_x} = 0.0046 \text{ cm}$$

3) TOTAL DEFLECTION

$$.\delta_2 = D1 + D2 = 0.2073 \text{ cm}$$

$$\text{RATIO : } D3/L = 1/868 < 500 \text{ ----- O.K}$$



3. I-BEAM 최대허용 가능 SPAN 계산

1. SPECIFICATION

.정격하중	-----	Q =	1.8	ton
.SPAN (PITCH)	-----	L=	750	cm
.肯탈리버	-----	L1=	280	cm
.TROLLEY WHEEL BASE	-----	B=	13.2	cm
.WEIGHT OF HOIST	-----	Wh=	0.16	ton
.PITCH내의 GIRDER 무게	-----	Wg=	0.491	ton
.肯탈리버의 GIRDER 무게	-----	Wg1=	0.183	ton
.I-BEAM SIZE	-----	300x150x10/18.5t		
		I _x =	12700	cm ⁴
		I _y =	886	cm ⁴
		Z _x =	849	cm ³
		Z _y =	118	cm ³
		A =	83.47	cm ²
		W _b =	65.5	kg/m
.HOOK APPROACH	-----	L ₂ =	30	cm
탄성계수	-----	E=	2100000	kg/cm ²
동하중 계수(작업계수)	-----	M(ϕ)=	1.11	
정하중 계수(충격계수)	-----	F(ψ)=	1.10	

1. DESIGN



2. I-BEAM에 작용하는 하중

$$P = Q + Wh = 1.8 + 0.16 = 1.96 \text{ ton}$$

3. 수직하중에 의한 BENDING MOMENT

1) PITCH 지점내 BENDING MOMENT

$$M_{h1} = F \times M \times P \times (L - B/2)^2 / (4 \times L)$$

$$= 1.11 \times 1.1 \times 1.96 \times (750 - 13.2/2)^2 / (4 \times 750)$$



$$= 440.9 \text{ ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$M_{h2} = F \times M \times P \times L_3$$

$$= 1.11 \times 1.1 \times 1.96 \times (280 - 30) = 598.29 \text{ ton.cm}$$

4. 자중에 의한 BENDING MOMENT(I-BEAM)

1) PITCH 지점내 BENDING MOMENT

$$M_{t1} = F \times W_g \times L / 8 = 1.11 \times 0.491 \times 750 / 8 = 51.095 \text{ ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$M_{t2} = F \times W_g \times L_1 / 2 = 1.11 \times 0.183 \times 280 / 2 = 28.44 \text{ ton.cm}$$

5. 수직하중에 의한 최대 BENDING MOMENT

$$M_{max1} = M_{h1} + M_{t1} = 491.9 \text{ ton.cm}$$

$$M_{max2} = M_{h2} + M_{t2} = 626.7 \text{ ton.cm}$$

6. 풍하중에 의한 수평하중의 BENDING MOMENT

(1) GIRDER에 의한 수직 풍하중

$$\text{작업시 } V=16\text{m/s}, q = 19.9 \text{ kg/m}^2 \quad h(\text{최고양정}) = 30 \text{ M}$$

$$\text{작업시 } q = 8.5 \times 4 \sqrt{h} = 8.5 \times 4 \sqrt{42} = 19.9 \text{ kg/m}^2$$

$$H = (\text{GIRDER 높이}) = 0.25 \text{ m}$$

$$\text{PITCH내 풍하중} = F \times L \times H \times q \times 1.7 = 1.11 \times 7.5 \times 0.25 \times 19.9 \times 1.7 = 70 \text{ kg}$$

$$\text{켄틀레버 풍하중} = F \times L_1 \times H \times q \times 1.3 = 1.11 \times 2.8 \times 0.25 \times 19.9 \times 1.3 = 20 \text{ kg}$$

$$M_{FGG} = \frac{0.07 \times 750}{8} - \frac{0.02 \times 280}{2} = 3.763 \text{ ton.cm}$$

$$M_{FG1} = \frac{0.02 \times 280}{2} = 2.8 \text{ ton.cm}$$

(2) HOIST에 의한 풍하중

* 작업시

$$HH = 1.0 \text{ m} \quad HB = 0.65 \text{ m} \quad \text{풍하중} = F \times HB \times HH \times q \times 1.2 = 1.11 \times 0.65 \times 1 \times 19.9 \times 1.2 =$$

* PITCH내 풍하중

$$M_{FHG} = 17 \times 750 / 4 = 3187.5 \text{ kg.cm} =$$

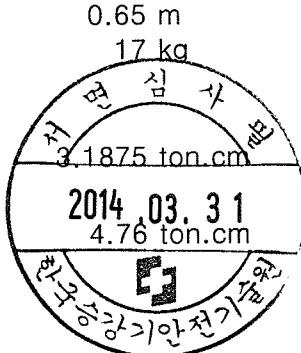
* 켄틀레버 풍하중

$$M_{FH1} = 17 \times 280 = 4760 \text{ kg.cm} =$$

7. COMBINED MOMENT

* PITCH내

$$M_{HCG} = M_{FGG} + M_{FHG} = 3.763 + 3.1875 = 6.951 \text{ ton.cm}$$



*肯틀레버

$$M_{HC1} = M_{FG1} + M_{FH1} = 2.8 + 4.76 = 7.560 \text{ ton.cm}$$

8. BENDING STRESS

A. VERTICAL BENDING STRESS

1. PITCH 내

$$\sigma v1 = M_{max1}/Zx = 491.9 / 849 = 0.579 \text{ ton/cm}^2$$

2. 켄틀레버

$$\sigma v2 = M_{max2}/Zx = 626.7 / 849 = 0.738 \text{ ton/cm}^2$$

B. 수평 최대 응력

1. PITCH 내

$$\sigma v3 = M_{HCG}/Zy = 6.9505 / 118 = 0.059 \text{ ton/cm}^2$$

2. 켄틀레버

$$\sigma v4 = M_{HC1}/Zy = 7.56 / 118 = 0.064 \text{ ton/cm}^2$$

9. 합성 응력

$$\Sigma \sigma 1 = \sigma v1 + \sigma v3 = 0.638 \\ 0.638 < 1.279 \text{ ton/cm}^2 \quad \text{OK}$$

$$\Sigma \sigma 2 = \sigma v2 + \sigma v4 = 0.802 \\ 0.802 < 1.6 \text{ ton/cm}^2 \quad \text{OK}$$

용접효율 : 80% 응력 $1391 \times 80\% \times 115\% = 1.279 \text{ ton/cm}^2$
 켄틀레버는 용접부 없음 $1391 \times 1.15(\text{풍하중포함}) = 1.600 \text{ ton/cm}^2$

10. 휴지시 풍하중에 의한 BENDING MOMENT

휴지시 $V=45 \text{ m/s}$, $q = 158.0 \text{ kg/m}^2$, $h(\text{최고양정}) = 30 \text{ m}$
 $q = M \times 4 \sqrt{h} = 67.5 \times 4 \sqrt{30} = 158.0 \text{ kg/m}^2$
 $M = V^2 / 30 = 67.5$

* 휴지시 풍하중에 의한 수평하중

$$\text{PITCH내 풍하중} = F \times L \times H \times q \times 1.7 = 1.11 \times 7.5 \times 0.25 \times 158 \times 1.7 = 559 \text{ kg}$$

$$\text{켄틀레버에 대한 풍하중} = F \times L \times H \times q \times 1.3 = 1.11 \times 2.8 \times 0.25 \times 158 \times 1.3 = 160 \text{ kg}$$

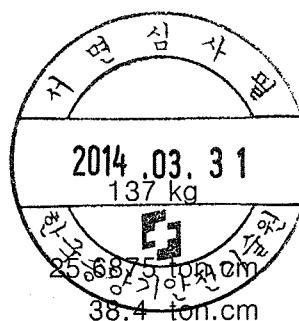
$$MM_{G1} = \frac{0.559 \times 750}{8} - \frac{0.16 \times 280}{2} = 30.00625 \text{ ton.cm}$$

$$MM_1 = \frac{0.16 \times 280}{2} = 22.4 \text{ ton.cm}$$

$$\text{HOIST에 대한 풍하중} = F \times H \times B \times H \times q \times 1.2 = 1.11 \times 0.65 \times 1 \times 158 \times 1.2 =$$

$$* \text{PITCH내 } MM_{H0} = 137 \times 750 / 4 = 102750 \text{ KG.CM} =$$

$$* \text{켄틀레버 } MM_{H1} = 137 \times 280 = 38360 \text{ KG.CM} =$$



* COMBINED MOMENT

$$MM_2 = MM_{G1} + MM_{H0} = 30.00625 + 25.6875 = 55.69375 \text{ ton.cm}$$

$$MM_4 = MM_1 + MM_{H1} = 22.4 + 38.4 = 60.80 \text{ ton.cm}$$

* BENDING STRESS

$$\Sigma\sigma_3 = MM_2 / Zy = 55.69375 / 118 = 0.472 \text{ ton/cm}^2 < 1.447 \text{ ton/cm}^2 \text{--O.K}$$

$$\Sigma\sigma_4 = MM_4 / Zy = 60.8 / 118 = 0.515 \text{ ton/cm}^2 < 1.808 \text{ ton/cm}^2 \text{--O.K}$$

PITCH내 휴지시 응력 1391 x 80% x130% = 1447 ton/cm²
 켄들레버는 용접부 없음 휴지시 응력 1391 x 130% = 1808 ton/cm²

11. DEFLECTION OF GIRDER

* PITCH 내

1) DUE TO RATED & TROLLEY LOAD

$$D1 = \frac{(Q+Q2)x(L)^3}{48xExIx} = 0.6459 \text{ cm}$$

2) DUE TO DEAD LOAD

$$D2 = \frac{5 \times Wgx L^3}{384 \times E \times Ix} = 0.1011 \text{ cm}$$

3) TOTAL DEFLECTION

$$\delta_1 = D1 + D2 = 0.747 \text{ cm}$$

$$\text{RATIO : } D3/L = 1/1004 < 800 \text{ ---- O.K}$$

* 켄들레버

1) DUE TO RATED & TROLLEY LOAD

$$D1 = \frac{(Q+Q2)x(L1-L2)^3}{3xExIx} = 0.3828 \text{ cm}$$

2) DUE TO DEAD LOAD

$$D2 = \frac{Wg1x L1^3}{8 \times E \times Ix} = 0.0188 \text{ cm}$$

3) TOTAL DEFLECTION

$$\delta_2 = D1 + D2 = 0.4016 \text{ cm}$$

$$\text{RATIO : } D3/L = 1/697 < 500 \text{ ---- O.K}$$

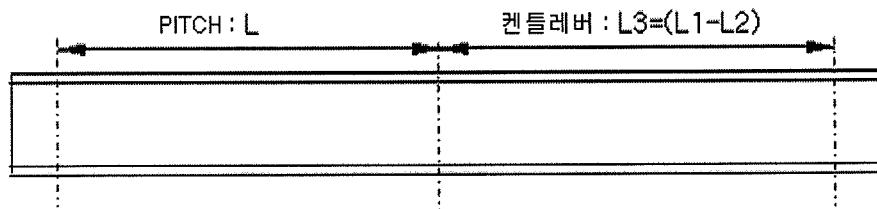


4. I-BEAM 최대허용 가능 SPAN 계산

1. SPECIFICATION

.정격하중	-----	Q =	1.8	ton
.SPAN (PITCH)	-----	L=	940	cm
.켄틸리버	-----	L1=	340	cm
.TROLLEY WHEEL BASE	-----	B=	13.2	cm
.WEIGHT OF HOIST	-----	Wh=	0.16	ton
.PITCH내의 GIRDER 무게	-----	Wg=	0.82	ton
.켄틸리버의 GIRDER 무게	-----	Wg1=	0.296	ton
.I-BEAM SIZE	-----	I _x =	22400	cm ⁴
		I _y =	1180	cm ⁴
		Z _x =	1280	cm ³
		Z _y =	158	cm ³
		A =	111.1	cm ²
		W _b =	87.2	kg/m
.HOOK APPROACH	-----	L ₂ =	30	cm
탄성계수	-----	E=	2100000	kg/cm ²
동하중 계수(작업계수)	-----	M(ϕ)=	1.11	
정하중 계수(충격계수)	-----	F(ψ)=	1.10	

1. DESIGN



2. I-BEAM에 작용하는 하중

$$P = Q + Wh = 1.8 + 0.16 = 1.96 \text{ ton}$$

3. 수직하중에 의한 BENDING MOMENT

1) PITCH 지점내 BENDING MOMENT

$$M_{h1} = F \times M \times P \times (L - B/2)^2 / (4 \times L)$$

$$= 1.11 \times 1.1 \times 1.96 \times (940 - 13.2/2)^2 / (4 \times 940)$$



$$= 554.5 \text{ ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$M_{h2} = F \times M \times P \times L_3$$

$$= 1.11 \times 1.1 \times 1.96 \times (340 - 30) = 741.88 \text{ ton.cm}$$

4. 자중에 의한 BENDING MOMENT(I-BEAM)

1) PITCH 지점내 BENDING MOMENT

$$M_{t1} = F \times W_g \times L / 8 = 1.11 \times 0.82 \times 940 / 8 = 106.949 \text{ ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$M_{t2} = F \times W_g \times L_1 / 2 = 1.11 \times 0.296 \times 340 / 2 = 55.86 \text{ ton.cm}$$

5. 수직하중에 의한 최대 BENDING MOMENT

$$M_{max1} = M_{h1} + M_{t1} = 661.5 \text{ ton.cm}$$

$$M_{max2} = M_{h2} + M_{t2} = 797.7 \text{ ton.cm}$$

6. 풍하중에 의한 수평하중의 BENDING MOMENT

(1) GIRDER에 의한 수직 풍하중

$$\text{작업시 } V=16\text{m/s}, q = 19.9 \text{ kg/m}^2 \quad h(\text{최고양정}) = 30 \text{ M}$$

$$\text{작업시 } q = 8.5 \times 4 \sqrt{h} = 8.5 \times 4 \sqrt{42} = 19.9 \text{ kg/m}^2$$

$$H=(\text{GIRDER높이}) \quad 0.25 \text{ m}$$

$$\text{PITCH내 풍하중} = F \times L \times H \times q \times 1.7 = 1.11 \times 9.4 \times 0.25 \times 19.9 \times 1.7 = 88 \text{ kg}$$

$$\text{캔틸레버 풍하중} = F \times L_1 \times H \times q \times 1.3 = 1.11 \times 3.4 \times 0.25 \times 19.9 \times 1.3 = 24 \text{ kg}$$

$$M_{FGG} = \frac{0.088 \times 940}{8} - \frac{0.024 \times 340}{2} = 6.26 \text{ ton.cm}$$

$$M_{FG1} = \frac{0.024 \times 340}{2} = 4.08 \text{ ton.cm}$$

(2) HOIST에 의한 풍하중

* 작업시

$$HH=1.0 \text{ m} \quad HB=0.65 \text{ m} \quad 17 \text{ kg} \\ \text{풍하중} = F \times HB \times HH \times q \times 1.2 = 1.11 \times 0.65 \times 1 \times 19.9 \times 1.2 = 3.995 \text{ kg.cm} = 3.995 \text{ ton.cm}$$

*PITCH내 풍하중

$$M_{FHG} = 17 \times 940 / 4 = 3995 \text{ kg.cm} = 3.995 \text{ ton.cm}$$

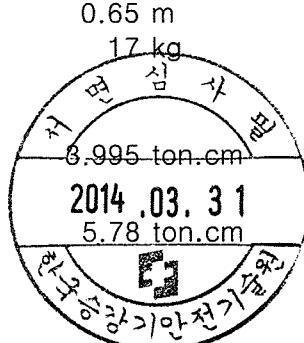
*캔틸레버 풍하중

$$M_{FH1} = 17 \times 340 = 5780 \text{ kg.cm} = 5.78 \text{ ton.cm}$$

7. COMBINED MOMENT

*PITCH내

$$M_{HCG} = M_{FGG} + M_{FHG} = 6.26 + 3.995 = 10.255 \text{ ton.cm}$$



*肯들레버

$$M_{HC1} = M_{FG1} + M_{FH1} = 4.08 + 5.78 = 9.860 \text{ ton.cm}$$

8. BENDING STRESS

A. VERTICAL BENDING STRESS

1. PITCH 내

$$\sigma v1 = M_{max1}/Zx = 661.5 / 1280 = 0.517 \text{ ton/cm}^2$$

2. 켄들레버

$$\sigma v2 = M_{max2}/Zx = 797.7 / 1280 = 0.623 \text{ ton/cm}^2$$

B. 수평 최대 응력

1. PITCH 내

$$\sigma v3 = M_{HCG}/Zy = 10.255 / 158 = 0.065 \text{ ton/cm}^2$$

2. 켄들레버

$$\sigma v4 = M_{HC1}/Zy = 9.86 / 158 = 0.062 \text{ ton/cm}^2$$

9. 합성 응력

$$\Sigma \sigma 1 = \sigma v1 + \sigma v3 = 0.582 \\ 0.582 < 1.279 \text{ ton/cm}^2 \quad \text{OK}$$

$$\Sigma \sigma 2 = \sigma v2 + \sigma v4 = 0.686 \\ 0.686 < 1.6 \text{ ton/cm}^2 \quad \text{OK}$$

용접효율 : 80% 응력 $1391 \times 80\% \times 115\% = 1.279 \text{ ton/cm}^2$
 켄들레버는 용접부 없음 $1391 \times 1.15(\text{풍하중포함}) = 1.600 \text{ ton/cm}^2$

10. 휴지시 풍하중에 의한 BENDING MOMENT

휴지시 $V=45\text{m/s}$, $q = 158.0 \text{ kg/m}^2$, $h(\text{최고양정}) = 30 \text{ m}$
 $q = M \times 4 \sqrt{h} = 67.5 \times 4 \sqrt{30} = 158.0 \text{ kg/m}^2$
 $M = V^2 / 30 = 67.5$

*휴지시 풍하중에 의한 수평하중

$$\text{PITCH내 풍하중} = F \times L \times H \times q \times 1.7 = 1.11 \times 9.4 \times 0.25 \times 158 \times 1.7 = 701 \text{ kg}$$

$$\text{켄들레버에 대한 풍하중} = F \times L \times H \times q \times 1.3 = 1.11 \times 3.4 \times 0.25 \times 158 \times 1.3 = 194 \text{ kg}$$

$$MM_{G1} = \frac{0.701 \times 940}{8} - \frac{0.194 \times 340}{2} = 49.3875 \text{ ton.cm}$$

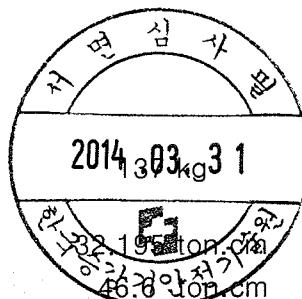
$$MM_1 = \frac{0.194 \times 340}{2} = 32.98 \text{ ton.cm}$$

$$\text{HOIST에 대한 풍하중} = F \times H \times B \times H \times q \times 1.2 = 1.11 \times 0.65 \times 1 \times 158 \times 1.2 =$$

$$* \text{PITCH내 } MM_{H0} = 137 \times 940 / 4 = 128780 \text{ KG.CM} =$$

$$* \text{켄들레버 } MM_{H1} = 137 \times 340 = 46580 \text{ KG.CM} =$$

* COMBINED MOMENT



$$MM_2 = MM_{G1} + MM_{H0} = 49.3875 + 32.195 = 81.5825 \text{ ton.cm}$$

$$MM_4 = MM_1 + MM_{H1} = 32.98 + 46.6 = 79.58 \text{ ton.cm}$$

* BENDING STRESS

$$\Sigma \sigma 3 = MM_2 / Zy = 81.5825 / 158 = 0.516 \text{ ton/cm}^2 < 1.447 \text{ ton/cm}^2 \text{--O.K}$$

$$\Sigma \sigma 4 = MM_4 / Zy = 79.58 / 158 = 0.504 \text{ ton/cm}^2 < 1.808 \text{ ton/cm}^2 \text{--O.K}$$

PITCH내 휴지시 응력 1391 x 80% x 130% = 1447 ton/cm²
 켄틀레버는 용접부 없음 휴지시 응력 1391 x 130% = 1808 ton/cm²

11. DEFLECTION OF GIRDER

* PITCH 내

1) DUE TO RATED & TROLLEY LOAD

$$D1 = \frac{(Q+Q2)x(L)^3}{48xEIx} = 0.7210 \text{ cm}$$

2) DUE TO DEAD LOAD

$$D2 = \frac{5 \times Wgx L^3}{384 \times E \times Ix} = 0.1885 \text{ cm}$$

3) TOTAL DEFLECTION

$$\delta 1 = D1 + D2 = 0.910 \text{ cm}$$

$$\text{RATIO : } D3/L = 1/1034 < 800 \text{ ---- O.K}$$

* 켄틀레버

1) DUE TO RATED & TROLLEY LOAD

$$D1 = \frac{(Q+Q2)x(L1-L2)^3}{3xEIx} = 0.4138 \text{ cm}$$

2) DUE TO DEAD LOAD

$$D2 = \frac{Wg1x L1^3}{8 \times E \times Ix} = 0.0309 \text{ cm}$$

3) TOTAL DEFLECTION

$$\delta 2 = D1 + D2 = 0.4447 \text{ cm}$$

$$\text{RATIO : } D3/L = 1/765 < 500 \text{ ---- O.K}$$

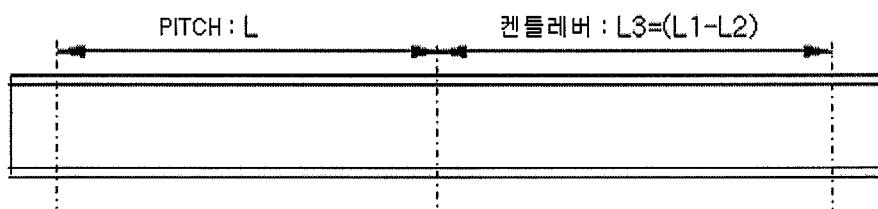


5. I-BEAM 최대허용 가능 SPAN 계산

1. SPECIFICATION

.정격하중	-----	Q =	1.8	ton
.SPAN (PITCH)	-----	L=	1100	cm
.켄틸리버	-----	L1=	400	cm
.TROLLEY WHEEL BASE	-----	B=	13.2	cm
.WEIGHT OF HOIST	-----	Wh=	0.16	ton
.PITCH내의 GIRDER 무게	-----	Wg=	1.054	ton
.켄틸리버의 GIRDER 무게	-----	Wg1=	0.383	ton
.I-BEAM SIZE	-----	400x150x12.5/25t		
		I _x =	31700	cm ⁴
		I _y =	1240	cm ⁴
		Z _x =	1580	cm ³
		Z _y =	165	cm ³
		A =	122.1	cm ²
		W _b =	95.8	kg/m
.HOOK APPROACH	-----	L ₂ =	30	cm
탄성계수	-----	E=	2100000	kg/cm ²
동하중 계수(작업계수)	-----	M(Φ)=	1.11	
정하중 계수(충격계수)	-----	F(Ψ)=	1.10	

1. DESIGN



2. |-BEAM에 작용하는 하중

$$P = Q + Wh = 1.8 + 0.16 = 1.96 \text{ ton}$$

3. 수직하중에 의한 BENDING MOMENT

1) PITCH 지점내 BENDING MOMENT

$$Mh1 = F \times M \times P \times (L-B/2)^2 / (4 \times L)$$

$$= 1.11 * 1.1 * 1.96 * (1100 - 13.2/2)^2 / (4 * 1100)$$



$$= 650.3 \text{ ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$M_{h2} = F \times M \times P \times L_3$$

$$= 1.11 \times 1.1 \times 1.96 \times (400 - 30) = 885.47 \text{ ton.cm}$$

4. 자중에 의한 BENDING MOMENT(I-BEAM)

1) PITCH 지점내 BENDING MOMENT

$$M_{t1} = F \times W_g \times L / 8 = 1.11 \times 1.054 \times 1100 / 8 = 160.867 \text{ ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$M_{t2} = F \times W_g \times L_1 / 2 = 1.11 \times 0.383 \times 400 / 2 = 85.03 \text{ ton.cm}$$

5. 수직하중에 의한 최대 BENDING MOMENT

$$M_{max1} = M_{h1} + M_{t1} = 811.1 \text{ ton.cm}$$

$$M_{max2} = M_{h2} + M_{t2} = 970.5 \text{ ton.cm}$$

6. 풍하중에 의한 수평하중의 BENDING MOMENT

(1) GIRDER에 의한 수직 풍하중

$$\text{작업시 } V=16\text{m/s}, q = 19.9 \text{ kg/m}^2 \quad h(\text{최고양정}) = 30 \text{ m}$$

$$\text{작업시 } q = 8.5 \times 4 \sqrt{h} = 8.5 \times 4 \sqrt{42} = 19.9 \text{ kg/m}^2$$

$$H=(\text{GIRDER 높이}) \quad 0.25 \text{ m}$$

$$\text{PITCH내 풍하중} = F \times L \times H \times q \times 1.7 = 1.11 \times 11 \times 0.25 \times 19.9 \times 1.7 = 103 \text{ kg}$$

$$\text{캔틸레버 풍하중} = F \times L_1 \times H \times q \times 1.4 = 1.11 \times 4 \times 0.25 \times 19.9 \times 1.4 = 31 \text{ kg}$$

$$M_{FGG} = \frac{0.103 \times 1100}{8} \times \frac{0.031 \times 400}{2} = 7.963 \text{ ton.cm}$$

$$M_{FG1} = \frac{0.031 \times 400}{2} = 6.2 \text{ ton.cm}$$

(2) HOIST에 의한 풍하중

* 작업시

$$HH= 1.0 \text{ m} \quad HB= 0.65 \text{ m}$$

$$\text{풍하중} = F \times HB \times HH \times q \times 1.2 = 1.11 \times 0.65 \times 1 \times 19.9 \times 1.2 = 4.675 \text{ ton.cm}$$

*PITCH내 풍하중

$$M_{FHG} = 17 \times 1100 / 4 = 4675 \text{ kg.cm} = 4.675 \text{ ton.cm}$$

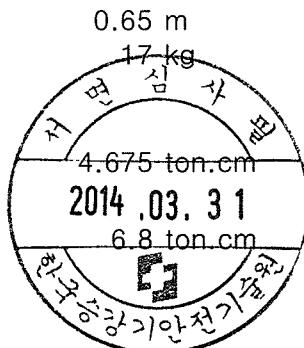
*캔틸레버 풍하중

$$M_{FH1} = 17 \times 400 = 6800 \text{ kg.cm} = 6.8 \text{ ton.cm}$$

7. COMBINED MOMENT

*PITCH내

$$M_{HCG} = M_{FGG} + M_{FHG} = 7.963 + 4.675 = 12.638 \text{ ton.cm}$$



*肯틀레버

$$M_{HC1} = M_{FG1} + M_{FH1} = 6.2 + 6.8 = 13.000 \text{ ton.cm}$$

8. BENDING STRESS

A. VERTICAL BENDING STRESS

1. PITCH 내

$$\sigma v1 = M_{max1}/Zx = 811.1 / 1580 = 0.513 \text{ ton/cm}^2$$

2. 켄틀레버

$$\sigma v2 = M_{max2}/Zx = 970.5 / 1580 = 0.614 \text{ ton/cm}^2$$

B. 수평 최대 응력

1. PITCH 내

$$\sigma v3 = M_{HCG}/Zy = 12.638 / 165 = 0.077 \text{ ton/cm}^2$$

2. 켄틀레버

$$\sigma v4 = M_{HC1}/Zy = 13 / 165 = 0.079 \text{ ton/cm}^2$$

9. 합성 응력

$$\Sigma \sigma 1 = \sigma v1 + \sigma v3 = 0.590 \quad 0.590 < 1.279 \text{ ton/cm}^2 \quad \text{OK}$$

$$\Sigma \sigma 2 = \sigma v2 + \sigma v4 = 0.693 \quad 0.693 < 1.6 \text{ ton/cm}^2 \quad \text{OK}$$

용접효율 : 80% 응력 $1391 \times 80\% \times 115\% = 1.279 \text{ ton/cm}^2$

肯틀레버는 용접부 없음 1391×1.15 (풍하중포함) = 1.600 ton/cm^2

10. 휴지시 풍하중에 의한 BENDING MOMENT

휴지시 $V=45\text{m/s}$, $q = 158.0 \text{ kg/m}^2$, $h(\text{최고양정}) = 30 \text{ m}$
 $q = M \times 4 \sqrt{h} = 67.5 \times 4 \sqrt{30} = 158.0 \text{ kg/m}^2$
 $M = V^2 / 30 = 67.5$

* 휴지시 풍하중에 의한 수평하중

$$\text{PITCH내 풍하중} = F \times L \times H \times q \times 1.7 = 1.11 \times 11 \times 0.25 \times 158 \times 1.7 = 820 \text{ kg}$$

$$\text{켄틀레버에 대한 풍하중} = F \times L \times H \times q \times 1.4 = 1.11 \times 4 \times 0.25 \times 158 \times 1.4 = 246 \text{ kg}$$

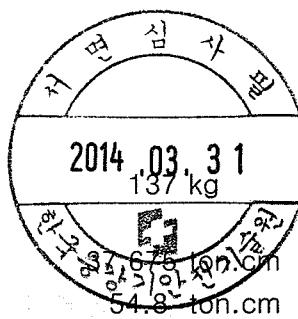
$$MM_{G1} = \frac{0.82 \times 1100}{8} - \frac{0.246 \times 400}{2} = 63.55 \text{ ton.cm}$$

$$MM_1 = \frac{0.246 \times 400}{2} = 49.2 \text{ ton.cm}$$

$$\text{HOIST에 대한 풍하중} = F \times H \times B \times H \times q \times 1.2 = 1.11 \times 0.65 \times 1 \times 158 \times 1.2 =$$

$$* \text{PITCH내 } MM_{H0} = 137 \times 1100 / 4 = 150700 \text{ KG.CM} =$$

$$* \text{켄틀레버 } MM_{H1} = 137 \times 400 = 54800 \text{ KG.CM} =$$



* COMBINED MOMENT

$$\begin{aligned} MM_2 &= MM_{G1} + MM_{H0} = & 63.55 + 37.675 &= 101.225 \text{ ton.cm} \\ MM_4 &= MM_1 + MM_{H1} = & 49.2 + 54.8 &= 104.00 \text{ ton.cm} \end{aligned}$$

* BENDING STRESS

$$\begin{aligned} \Sigma \sigma 3 &= MM_2 / Zy = 101.225 / 165 = 0.613 \text{ ton/cm}^2 &< 1.447 \text{ ton/cm}^2 \text{---O.K} \\ \Sigma \sigma 4 &= MM_4 / Zy = 104 / 165 = 0.630 \text{ ton/cm}^2 &< 1.808 \text{ ton/cm}^2 \text{---O.K} \end{aligned}$$

PITCH내 휴지시 응력 1391 x 80% x130% = 1447 ton/cm²
 켄틀레버는 용접부 없음 휴지시 응력 1391 x 130% = 1808 ton/cm²

11. DEFLECTION OF GIRDER

* PITCH 내

1) DUE TO RATED & TROLLEY LOAD

$$D1 = \frac{(Q+Q2)x(L)^3}{48 \times E \times I_x} = 0.8164 \text{ cm}$$

2) DUE TO DEAD LOAD

$$D2 = \frac{5 \times Wgx L^3}{384 \times E \times I_x} = 0.2744 \text{ cm}$$

3) TOTAL DEFLECTION

$$.81 = D1 + D2 = 1.091 \text{ cm}$$

$$\text{RATIO : } D3/L = 1/1008 < 800 \text{ ----- O.K}$$

* 켄틀레버

1) DUE TO RATED & TROLLEY LOAD

$$D1 = \frac{(Q+Q2)x(L1-L2)^3}{3 \times E \times I_x} = 0.4971 \text{ cm}$$

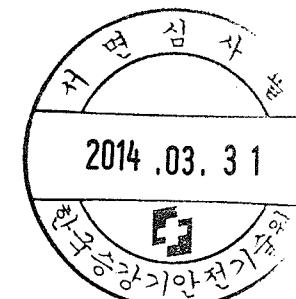
2) DUE TO DEAD LOAD

$$D2 = \frac{Wg1x L1^3}{8 \times E \times I_x} = 0.0460 \text{ cm}$$

3) TOTAL DEFLECTION

$$.82 = D1 + D2 = 0.5431 \text{ cm}$$

$$\text{RATIO : } D3/L = 1/737 < 500 \text{ ----- O.K}$$



6. H-BEAM 최대허용 가능 SPAN 계산

1. SPECIFICATION

.정격하중	-----	Q =	1.8	ton
.SPAN (PITCH)	-----	L=	600	cm
.켄틸리버	-----	L1=	200	cm
.TROLLEY WHEEL BASE	-----	B=	13.2	cm
.WEIGHT OF HOIST	-----	Wh=	0.16	ton
.PITCH내의 GIRDER 무게	-----	Wg=	0.192	ton
.켄틸리버의 GIRDER 무게	-----	Wg1=	0.064	ton
.H-BEAM SIZE	-----	300x150x6.5/9t		
		I _x =	7210	cm ⁴
		I _y =	508	cm ⁴
		Z _x =	481	cm ³
		Z _y =	67.7	cm ³
		A =	40.8	cm ²
		W _b =	32	kg/m
.HOOK APPROACH	-----	L ₂ =	30	cm
탄성계수	-----	E=	2100000	kg/cm ²
동하중 계수(작업계수)	-----	M(Φ)=	1.11	
정하중 계수(총격계수)	-----	F(Ψ)=	1.10	

1. DESIGN



2. H-BEAM에 작용하는 하중

$$P = Q + Wh = 1.8 + 0.16 = 1.96 \text{ ton}$$

3. 수직하중에 의한 BENDING MOMENT

1) PITCH 지점내 BENDING MOMENT

$$M_{h1} = F \times M \times P \times (L - B/2)^2 / (4 \times L)$$

$$= 1.11 \times 1.1 \times 1.96 \times (600 - 13.2/2)^2 / (4 \times 600)$$



$$= 351.1 \text{ ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$M_{h2} = F \times M \times P \times L_3$$

$$= 1.11 \times 1.1 \times 1.96 \times (200 - 30) = 406.84 \text{ ton.cm}$$

4. 자중에 의한 BENDING MOMENT(I-BEAM)

1) PITCH 지점내 BENDING MOMENT

$$M_{t1} = F \times W_g \times L / 8 = 1.11 \times 0.192 \times 600 / 8 = 15.984 \text{ ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$M_{t2} = F \times W_g \times L_1 / 2 = 1.11 \times 0.064 \times 200 / 2 = 7.1 \text{ ton.cm}$$

5. 수직하중에 의한 최대 BENDING MOMENT

$$M_{max1} = M_{h1} + M_{t1} = 367.1 \text{ ton.cm}$$

$$M_{max2} = M_{h2} + M_{t2} = 413.9 \text{ ton.cm}$$

6. 풍하중에 의한 수평하중의 BENDING MOMENT

(1) GIRDER에 의한 수직 풍하중

$$\text{작업시 } V=16\text{m/s}, q = 19.9 \text{ kg/m}^2 \quad h(\text{최고양정}) = 30 \text{ m}$$

$$\text{작업시 } q = 8.5 \times 4 \sqrt{h} = 8.5 \times 4 \sqrt{42} = 19.9 \text{ kg/m}^2$$

$$H=(\text{GIRDER 높이}) \quad 0.25 \text{ m}$$

$$\text{PITCH내 풍하중} = F \times L \times H \times q \times 1.6 = 1.11 \times 6 \times 0.25 \times 19.9 \times 1.6 = 53 \text{ kg}$$

$$\text{캔틸레버 풍하중} = F \times L_1 \times H \times q \times 1.3 = 1.11 \times 2 \times 0.25 \times 19.9 \times 1.3 = 14 \text{ kg}$$

$$M_{FGG} = \frac{0.053 \times 600}{8} = \frac{0.014 \times 200}{2} = 2.575 \text{ ton.cm}$$

$$M_{FG1} = \frac{0.014 \times 200}{2} = 1.4 \text{ ton.cm}$$

(2) HOIST에 의한 풍하중

* 작업시

$$HH=1.0 \text{ m} \quad HB=0.65 \text{ m} \quad \text{풍하중} = F \times HB \times HH \times q \times 1.2 = 1.11 \times 0.65 \times 1 \times 19.9 \times 1.2 =$$

*PITCH내 풍하중

$$M_{FHG} = 17 \times 600 / 4 = 2550 \text{ kg.cm} =$$

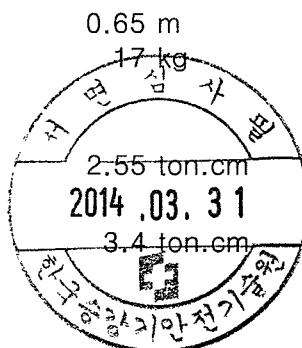
*캔틸레버 풍하중

$$M_{FH1} = 17 \times 200 = 3400 \text{ kg.cm} =$$

7. COMBINED MOMENT

*PITCH내

$$M_{HCG} = M_{FGG} + M_{FHG} = 2.575 + 2.55 = 5.125 \text{ ton.cm}$$



*肯틀레버

$$M_{HC1} = M_{FG1} + M_{FH1} = 1.4 + 3.4 = 4.800 \text{ ton.cm}$$

8. BENDING STRESS

A. VERTICAL BENDING STRESS

1. PITCH 내

$$\sigma v1 = M_{max1}/Zx = 367.1 / 481 = 0.763 \text{ ton/cm}^2$$

2. 켄틀레버

$$\sigma v2 = M_{max2}/Zx = 413.9 / 481 = 0.860 \text{ ton/cm}^2$$

B. 수평 최대 응력

1. PITCH 내

$$\sigma v3 = M_{HCG}/Zy = 5.125 / 67.7 = 0.076 \text{ ton/cm}^2$$

2. 켄틀레버

$$\sigma v4 = M_{HC1}/Zy = 4.8 / 67.7 = 0.071 \text{ ton/cm}^2$$

9. 합성 응력

$$\Sigma \sigma 1 = \sigma v1 + \sigma v3 = 0.839 \\ 0.839 < 1.279 \text{ ton/cm}^2 \quad \text{OK}$$

$$\Sigma \sigma 2 = \sigma v2 + \sigma v4 = 0.931 \\ 0.931 < 1.6 \text{ ton/cm}^2 \quad \text{OK}$$

용접효율 : 80% 응력 $1391 \times 80\% \times 115\% = 1.279 \text{ ton/cm}^2$

肯틀레버는 용접부 없음 $1391 \times 1.15(\text{풍하중포함}) = 1.600 \text{ ton/cm}^2$

10. 휴지시 풍하중에 의한 BENDING MOMENT

$$\text{휴지시 } V=45 \text{ m/s}, q = 158.0 \text{ kg/m}^2 \quad h(\text{최고양정}) = 30 \text{ m} \\ q = M \times \frac{4}{\sqrt{h}} = 67.5 \times \frac{4}{\sqrt{30}} = 158.0 \text{ kg/m}^2 \\ M = V^2 / 30 = 67.5$$

* 휴지시 풍하중에 의한 수평하중

$$\text{PITCH내 풍하중} = F \times L \times H \times q \times 1.6 = 1.11 \times 6 \times 0.25 \times 158 \times 1.6 = 421 \text{ kg}$$

$$\text{켄틀레버에 대한 풍하중} = F \times L_1 \times H \times q \times 1.3 = 1.11 \times 2 \times 0.25 \times 158 \times 1.3 = 114 \text{ kg}$$

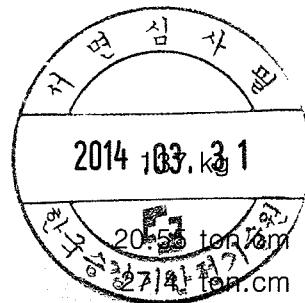
$$MM_{G1} = \frac{0.421 \times 600}{8} - \frac{0.114 \times 200}{2} = 20.175 \text{ ton.cm}$$

$$MM_1 = \frac{0.114 \times 200}{2} = 11.4 \text{ ton.cm}$$

$$\text{HOIST에 대한 풍하중} = F \times H \times B \times H \times q \times 1.2 = 1.11 \times 0.65 \times 1 \times 158 \times 1.2 =$$

$$* \text{PITCH내 } MM_{H0} = 137 \times 600 / 4 = 82200 \text{ KG.CM} =$$

$$* \text{켄틀레버 } MM_{H1} = 137 \times 200 = 27400 \text{ KG.CM} =$$



* COMBINED MOMENT

$$\begin{aligned} MM_2 &= MM_{G1} + MM_{H0} = & 20.175 + 20.55 &= 40.725 \text{ ton.cm} \\ MM_4 &= MM_1 + MM_{H1} = & 11.4 + 27.4 &= 38.80 \text{ ton.cm} \end{aligned}$$

* BENDING STRESS

$$\begin{aligned} \Sigma\sigma_3 = MM_2 / Zy &= 40.725 / 67.7 = 0.602 \text{ ton/cm}^2 &< 1.447 \text{ ton/cm}^2 \text{---O.K} \\ \Sigma\sigma_4 = MM_4 / Zy &= 38.8 / 67.7 = 0.573 \text{ ton/cm}^2 &< 1.808 \text{ ton/cm}^2 \text{---O.K} \end{aligned}$$

PITCH내 휴지시 응력 1391 x 80% x130% = 1447 ton/cm²
 켄틀레버는 용접부 없음 휴지시 응력 1391 x 130% = 1808 ton/cm²

11. DEFLECTION OF GIRDER

* PITCH 내

1) DUE TO RATED & TROLLEY LOAD

$$D1 = \frac{(Q+Q2)x(L)^3}{48xEIx} = 0.5825 \text{ cm}$$

2) DUE TO DEAD LOAD

$$D2 = \frac{5 \times Wgx L^3}{384 \times E \times Ix} = 0.0357 \text{ cm}$$

3) TOTAL DEFLECTION

$$\delta_1 = D1 + D2 = 0.618 \text{ cm}$$

$$\text{RATIO : } D3/L = 1/971 < 800 \text{ ----- O.K}$$

* 켄틀레버

1) DUE TO RATED & TROLLEY LOAD

$$D1 = \frac{(Q+Q2)x(L1-L2)^3}{3xEIx} = 0.2120 \text{ cm}$$

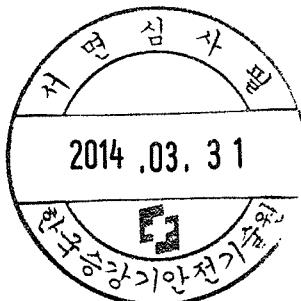
2) DUE TO DEAD LOAD

$$D2 = \frac{Wg1x L1^3}{8 \times E \times Ix} = 0.0042 \text{ cm}$$

3) TOTAL DEFLECTION

$$\delta_2 = D1 + D2 = 0.2162 \text{ cm}$$

$$\text{RATIO : } D3/L = 1/925 < 500 \text{ ----- O.K}$$

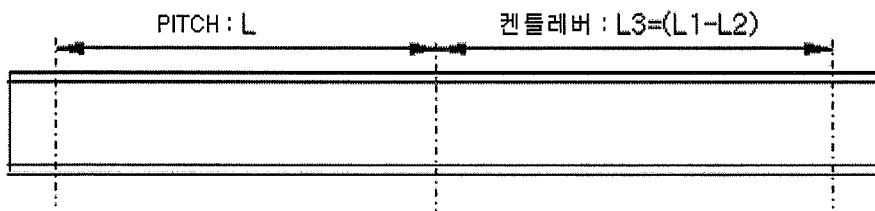


7. H-BEAM 최대허용 가능 SPAN 계산

1. SPECIFICATION

.정격하중	-----	Q =	1.8	ton
.SPAN (PITCH)	-----	L=	800	cm
.켄틸리버	-----	L1=	300	cm
.TROLLEY WHEEL BASE	-----	B=	13.2	cm
.WEIGHT OF HOIST	-----	Wh=	0.16	ton
.PITCH내의 GIRDER 무게	-----	Wg=	0.395	ton
.켄틸리버의 GIRDER 무게	-----	Wg1=	0.148	ton
.H-BEAM SIZE	-----	350x175x7/11t		
		I _x =	13500	cm ⁴
		I _y =	984	cm ⁴
		Z _x =	771	cm ³
		Z _y =	112	cm ³
		A =	62.91	cm ²
		W _b =	49.4	kg/m
.HOOK APPROACH	-----	L ₂ =	30	cm
탄성계수	-----	E=	2100000	kg/cm ²
동하중 계수(작업계수)	-----	M(Φ)=	1.11	
정하중 계수(충격계수)	-----	F(Ψ)=	1.10	

1. DESIGN



2. H-BEAM에 작용하는 하중

$$P = Q + Wh = 1.8 + 0.16 = 1.96 \text{ ton}$$

3. 수직하중에 의한 BENDING MOMENT

1) PITCH 지점내 BENDING MOMENT

$$M_{h1} = F \times M \times P \times (L - B/2)^2 / (4 \times L)$$

$$= 1.11 \times 1.1 \times 1.96 \times (800 - 13.2/2)^2 / (4 \times 800)$$



$$= 470.8 \text{ ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$M_{h2} = F \times M \times P \times L_3$$

$$= 1.11 \times 1.1 \times 1.96 \times (300 - 30) = 646.15 \text{ ton.cm}$$

4. 자중에 의한 BENDING MOMENT(I-BEAM)

1) PITCH 지점내 BENDING MOMENT

$$M_{t1} = F \times W_g \times L / 8 = 1.11 \times 0.395 \times 800 / 8 = 43.845 \text{ ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$M_{t2} = F \times W_g \times L_1 / 2 = 1.11 \times 0.148 \times 300 / 2 = 24.64 \text{ ton.cm}$$

5. 수직하중에 의한 최대 BENDING MOMENT

$$M_{max1} = M_{h1} + M_{t1} = 514.6 \text{ ton.cm}$$

$$M_{max2} = M_{h2} + M_{t2} = 670.8 \text{ ton.cm}$$

6. 풍하중에 의한 수평하중의 BENDING MOMENT

(1) GIRDER에 의한 수직 풍하중

$$\text{작업시 } V=16\text{m/s}, q = 19.9 \text{ kg/m}^2, h(\text{최고양정}) = 30 \text{ m}$$

$$\text{작업시 } q = 8.5 \times 4 \sqrt{h} = 8.5 \times 4 \sqrt{42} = 19.9 \text{ kg/m}^2$$

$$H = (\text{GIRDER 높이}) = 0.25 \text{ m}$$

$$\text{PITCH내 풍하중} = F \times L \times H \times q \times 1.6 = 1.11 \times 8 \times 0.25 \times 19.9 \times 1.6 = 71 \text{ kg}$$

$$\text{켄틀레버 풍하중} = F \times L_1 \times H \times q \times 1.3 = 1.11 \times 3 \times 0.25 \times 19.9 \times 1.3 = 22 \text{ kg}$$

$$M_{FGG} = \frac{0.071 \times 800}{8} = \frac{0.022 \times 300}{2} = 3.8 \text{ ton.cm}$$

$$M_{FG1} = \frac{0.022 \times 300}{2} = 3.3 \text{ ton.cm}$$

(2) HOIST에 의한 풍하중

* 작업시

$$HH = 1.0 \text{ m}, HB = 0.65 \text{ m}, \text{풍하중} = F \times HB \times HH \times q \times 1.2 = 1.11 \times 0.65 \times 1 \times 19.9 \times 1.2 =$$

*PITCH내 풍하중

$$M_{FHG} = 17 \times 800 / 4 = 3400 \text{ kg.cm} =$$

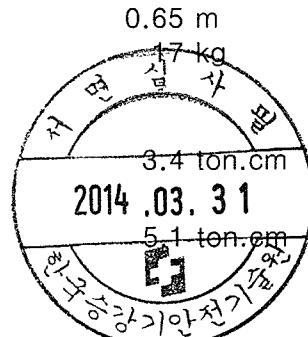
*肯틀레버 풍하중

$$M_{FH1} = 17 \times 300 = 5100 \text{ kg.cm} =$$

7. COMBINED MOMENT

*PITCH내

$$M_{HCG} = M_{FGG} + M_{FHG} = 3.8 + 3.4 = 7.200 \text{ ton.cm}$$



*肯들레버

$$M_{HC1} = M_{FG1} + M_{FH1} = 3.3 + 5.1 = 8.400 \text{ ton.cm}$$

8. BENDING STRESS

A. VERTICAL BENDING STRESS

1. PITCH 내

$$\sigma v1 = M_{max1}/Zx = 514.6 / 771 = 0.667 \text{ ton/cm}^2$$

2. 켄들레버

$$\sigma v2 = M_{max2}/Zx = 670.8 / 771 = 0.870 \text{ ton/cm}^2$$

B. 수평 최대 응력

1. PITCH 내

$$\sigma v3 = M_{HCG}/Zy = 7.2 / 112 = 0.064 \text{ ton/cm}^2$$

2. 켄들레버

$$\sigma v4 = M_{HC1}/Zy = 8.4 / 112 = 0.075 \text{ ton/cm}^2$$

9. 합성 응력

$$\Sigma \sigma 1 = \sigma v1 + \sigma v3 = 0.732 \\ 0.732 < 1.279 \text{ ton/cm}^2 \quad \text{OK}$$

$$\Sigma \sigma 2 = \sigma v2 + \sigma v4 = 0.945 \\ 0.945 < 1.6 \text{ ton/cm}^2 \quad \text{OK}$$

용접효율 : 80% 응력 $1391 \times 80\% \times 115\% = 1.279 \text{ ton/cm}^2$

肯들레버는 용접부 없음 $1391 \times 1.15(\text{풍하중포함}) = 1.600 \text{ ton/cm}^2$

10. 휴지시 풍하중에 의한 BENDING MOMENT

$$\text{휴지시 } V=45 \text{ m/s}, q = 158.0 \text{ kg/m}^2 \quad h(\text{최고양정}) = 30 \text{ m} \\ q = M \times 4 \sqrt{h} = 67.5 \times 4 \sqrt{30} = 158.0 \text{ kg/m}^2 \\ M = V^2 / 30 = 67.5$$

* 휴지시 풍하중에 의한 수평하중

$$\text{PITCH내 풍하중} = F \times L \times H \times q \times 1.6 = 1.11 \times 8 \times 0.25 \times 158 \times 1.6 = 561 \text{ kg}$$

$$\text{켄들레버에 대한 풍하중} = F \times L \times H \times q \times 1.3 = 1.11 \times 3 \times 0.25 \times 158 \times 1.3 = 171 \text{ kg}$$

$$MM_{G1} = \frac{0.561 \times 800}{8} - \frac{0.171 \times 300}{2} = 30.45 \text{ ton.cm}$$

$$MM_1 = \frac{0.171 \times 300}{2} = 25.65 \text{ ton.cm}$$

$$\text{HOIST에 대한 풍하중} = F \times H \times B \times H \times q \times 1.2 = 1.11 \times 0.65 \times 1 \times 158 \times 1.2 =$$

$$* \text{PITCH내 } MM_{H0} = 137 \times 800 / 4 = 109600 \text{ KG.CM} =$$

$$* \text{켄들레버 } MM_{H1} = 137 \times 300 = 41100 \text{ KG.CM} =$$

* COMBINED MOMENT



$$MM_2 = MM_{G1} + MM_{H0} = 30.45 + 27.4 = 57.85 \text{ ton.cm}$$

$$MM_4 = MM_1 + MM_{H1} = 25.65 + 41.1 = 66.75 \text{ ton.cm}$$

* BENDING STRESS

$$\Sigma\sigma_3 = MM_2 / Zy = 57.85 / 112 = 0.517 \text{ ton/cm}^2 < 1.447 \text{ ton/cm}^2 \text{---O.K}$$

$$\Sigma\sigma_4 = MM_4 / Zy = 66.75 / 112 = 0.596 \text{ ton/cm}^2 < 1.808 \text{ ton/cm}^2 \text{---O.K}$$

PITCH내 휴지시 응력 1391 x 80% x130% = 1447 ton/cm²
 켄틀레버는 용접부 없음 휴지시 응력 1391 x 130% = 1808 ton/cm²

11. DEFLECTION OF GIRDER

* PITCH 내

1) DUE TO RATED & TROLLEY LOAD

$$D1 = \frac{(Q+Q2)x(L)^3}{48xExIx} = 0.7374 \text{ cm}$$

2) DUE TO DEAD LOAD

$$D2 = \frac{5 \times Wgx L^3}{384 \times E \times Ix} = 0.0929 \text{ cm}$$

3) TOTAL DEFLECTION

$$.81 = D1 + D2 = 0.830 \text{ cm}$$

$$\text{RATIO : } D3/L = 1/964 < 800 \text{ ---- O.K}$$

* 켄틀레버

1) DUE TO RATED & TROLLEY LOAD

$$D1 = \frac{(Q+Q2)x(L1-L2)^3}{3xEIx} = 0.4536 \text{ cm}$$

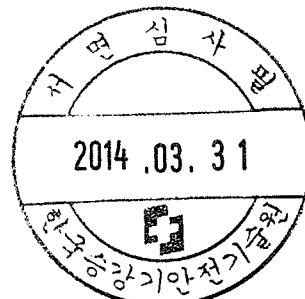
2) DUE TO DEAD LOAD

$$D2 = \frac{Wg1x L1^3}{8 \times E \times Ix} = 0.0176 \text{ cm}$$

3) TOTAL DEFLECTION

$$.82 = D1 + D2 = 0.4712 \text{ cm}$$

$$\text{RATIO : } D3/L = 1/637 < 500 \text{ ---- O.K}$$

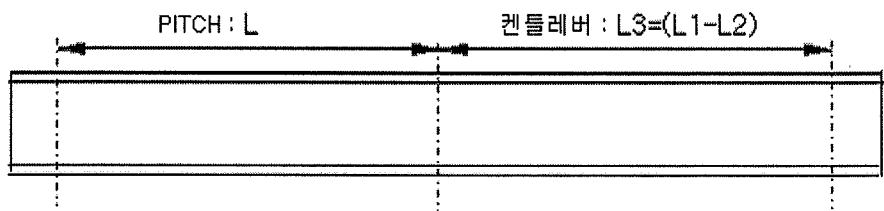


8. H-BEAM 최대허용 가능 SPAN 계산

1. SPECIFICATION

.정격하중	-----	Q =	1.8	ton
.SPAN (PITCH)	-----	L=	1000	cm
.肯틸리버	-----	L1=	380	cm
.TROLLEY WHEEL BASE	-----	B=	13.2	cm
.WEIGHT OF HOIST	-----	Wh=	0.16	ton
.PITCH내의 GIRDER 무게	-----	Wg=	0.654	ton
.肯틸리버의 GIRDER 무게	-----	Wg1=	0.249	ton
.H-BEAM SIZE	-----	400x200x8/13t		
		I _x =	23500	cm ⁴
		I _y =	1740	cm ⁴
		Z _x =	1170	cm ³
		Z _y =	174	cm ³
		A =	83.37	cm ²
		W _b =	65.4	kg/m
.HOOK APPROACH	-----	L ₂ =	30	cm
탄성계수	-----	E=	2100000	kg/cm ²
동하중 계수(작업계수)	-----	M(Φ)=	1.11	
정하중 계수(충격계수)	-----	F(Ψ)=	1.10	

1. DESIGN



2. H-BEAM에 작용하는 하중

$$P = Q + Wh = 1.8 + 0.16 = 1.96 \text{ ton}$$

3. 수직하중에 의한 BENDING MOMENT

1) PITCH 지점내 BENDING MOMENT

$$\begin{aligned} M_{h1} &= F \times M \times P \times (L - B/2)^2 / (4 \times L) \\ &= 1.11 \times 1.1 \times 1.96 \times (1000 - 13.2/2)^2 / (4 \times 1000) \end{aligned}$$



$$= 590.4 \text{ ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$M_{h2} = F \times M \times P \times L_3$$

$$= 1.11 \times 1.1 \times 1.96 \times (380 - 30) = 837.61 \text{ ton.cm}$$

4. 자중에 의한 BENDING MOMENT(I-BEAM)

1) PITCH 지점내 BENDING MOMENT

$$M_{t1} = F \times W_g \times L / 8 = 1.11 \times 0.654 \times 1000 / 8 = 90.743 \text{ ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$M_{t2} = F \times W_g \times L_1 / 2 = 1.11 \times 0.249 \times 380 / 2 = 52.51 \text{ ton.cm}$$

5. 수직하중에 의한 최대 BENDING MOMENT

$$M_{max1} = M_{h1} + M_{t1} = 681.2 \text{ ton.cm}$$

$$M_{max2} = M_{h2} + M_{t2} = 890.1 \text{ ton.cm}$$

6. 풍하중에 의한 수평하중의 BENDING MOMENT

(1) GIRDER에 의한 수직 풍하중

$$\text{작업시 } V=16\text{m/s}, q = 19.9 \text{ kg/m}^2 \quad h(\text{최고양정}) = 30 \text{ m}$$

$$\text{작업시 } q = 8.5 \times 4 \sqrt{h} = 8.5 \times 4 \sqrt{42} = 19.9 \text{ kg/m}^2$$

$$H(\text{GIRDER높이}) = 0.25 \text{ m}$$

$$\text{PITCH내 풍하중} = F \times L \times H \times q \times 1.6 = 1.11 \times 10 \times 0.25 \times 19.9 \times 1.6 = 88 \text{ kg}$$

$$\text{켄틀레버 풍하중} = F \times L_1 \times H \times q \times 1.3 = 1.11 \times 3.8 \times 0.25 \times 19.9 \times 1.3 = 27 \text{ kg}$$

$$M_{FGG} = \frac{0.088 \times 1000}{8} - \frac{0.027 \times 380}{2} = 5.87 \text{ ton.cm}$$

$$M_{FG1} = \frac{0.027 \times 380}{2} = 5.13 \text{ ton.cm}$$

(2) HOIST에 의한 풍하중

* 작업시

$$HH = 1.0 \text{ m} \quad HB = 0.65 \text{ m} \quad 17 \text{ kg}$$

$$\text{풍하중} = F \times HB \times HH \times q \times 1.2 = 1.11 \times 0.65 \times 1 \times 19.9 \times 1.2 =$$

*PITCH내 풍하중

$$M_{FHG} = 17 \times 1000 / 4 = 4250 \text{ kg.cm} = 4.25 \text{ ton.cm}$$

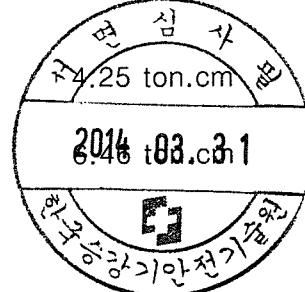
*肯틀레버 풍하중

$$M_{FH1} = 17 \times 380 = 6460 \text{ kg.cm} =$$

7. COMBINED MOMENT

*PITCH내

$$M_{HCG} = M_{FGG} + M_{FHG} = 5.87 + 4.25 = 10.120 \text{ ton.cm}$$



*肯들레버

$$M_{HC1} = M_{FG1} + M_{FH1} = 5.13 + 6.46 = 11.590 \text{ ton.cm}$$

8. BENDING STRESS

A. VERTICAL BENDING STRESS

1. PITCH 내

$$\sigma v1 = M_{max1}/Zx = 681.2 / 1170 = 0.582 \text{ ton/cm}^2$$

2. 켄들레버

$$\sigma v2 = M_{max2}/Zx = 890.1 / 1170 = 0.761 \text{ ton/cm}^2$$

B. 수평 최대 응력

1. PITCH 내

$$\sigma v3 = M_{HCG}/Zy = 10.12 / 174 = 0.058 \text{ ton/cm}^2$$

2. 켄들레버

$$\sigma v4 = M_{HC1}/Zy = 11.59 / 174 = 0.067 \text{ ton/cm}^2$$

9. 합성 응력

$$\Sigma \sigma 1 = \sigma v1 + \sigma v3 = 0.640 \\ 0.640 < 1.279 \text{ ton/cm}^2 \quad \text{OK}$$

$$\Sigma \sigma 2 = \sigma v2 + \sigma v4 = 0.827 \\ 0.827 < 1.6 \text{ ton/cm}^2 \quad \text{OK}$$

용접효율 : 80% 응력 $1391 \times 80\% \times 115\% = 1.279 \text{ ton/cm}^2$

肯들레버는 용접부 없음 $1391 \times 1.15(\text{풍하중포함}) = 1.600 \text{ ton/cm}^2$

10. 휴지시 풍하중에 의한 BENDING MOMENT

$$\text{휴지시 } V=45 \text{ m/s}, q = 158.0 \text{ kg/m}^2 \quad h(\text{최고양정}) = 30 \text{ m} \\ \text{휴지시 } q = M \times \frac{4}{\sqrt{h}} = 67.5 \times \frac{4}{\sqrt{30}} = 158.0 \text{ kg/m}^2 \\ M = V^2 \times \frac{1}{30} = 67.5$$

* 휴지시 풍하중에 의한 수평하중

$$\text{PITCH내 풍하중} = F \times L \times H \times q \times 1.6 = 1.11 \times 10 \times 0.25 \times 158 \times 1.6 = 702 \text{ kg}$$

$$\text{肯들레버에 대한 풍하중} = F \times L_1 \times H \times q \times 1.3 = 1.11 \times 3.8 \times 0.25 \times 158 \times 1.3 = 217 \text{ kg}$$

$$MM_{G1} = \frac{0.702 \times 1000}{8} - \frac{0.217 \times 380}{2} = 46.52 \text{ ton.cm}$$

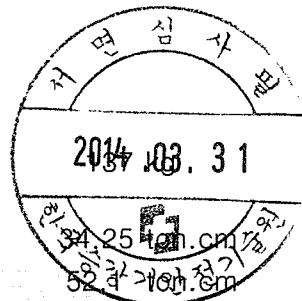
$$MM_1 = \frac{0.217 \times 380}{2} = 41.23 \text{ ton.cm}$$

$$\text{HOIST에 대한 풍하중} = F \times H_B \times H_H \times q \times 1.2 = 1.11 \times 0.65 \times 1 \times 158 \times 1.2 =$$

$$* \text{PITCH내 } MM_{H0} = 137 \times 1000 / 4 = 137000 \text{ KG.CM} =$$

$$* \text{肯들레버 } MM_{H1} = 137 \times 380 = 52060 \text{ KG.CM} =$$

* COMBINED MOMENT



$$MM_2 = MM_{G1} + MM_{H0} = 46.52 + 34.25 = 80.77 \text{ ton.cm}$$

$$MM_4 = MM_1 + MM_{H1} = 41.23 + 52.1 = 93.33 \text{ ton.cm}$$

* BENDING STRESS

$$\Sigma\sigma_3 = MM_2 / Zy = 80.77 / 174 = 0.464 \text{ ton/cm}^2 < 1.447 \text{ ton/cm}^2 \text{--O.K}$$

$$\Sigma\sigma_4 = MM_4 / Zy = 93.33 / 174 = 0.536 \text{ ton/cm}^2 < 1.808 \text{ ton/cm}^2 \text{--O.K}$$

PITCH내
肯틀레버는 용접부 없음

휴지시 응력 1391 x 80% x130% = 1447 ton/cm²
휴지시 응력 1391 x 130% = 1808 ton/cm²

11. DEFLECTION OF GIRDER

* PITCH 내

1) DUE TO RATED & TROLLEY LOAD

$$D1 = \frac{(Q+Q2)x(L)^3}{48xEIx} = 0.8274 \text{ cm}$$

2) DUE TO DEAD LOAD

$$D2 = \frac{5 \times Wgx L^3}{384 \times E \times Ix} = 0.1726 \text{ cm}$$

3) TOTAL DEFLECTION

$$.81 = D1 + D2 = 1.000 \text{ cm}$$

$$\text{RATIO : } D3/L = 1/1000 < 800 \text{ ---- O.K}$$

* 캔틀레버

1) DUE TO RATED & TROLLEY LOAD

$$D1 = \frac{(Q+Q2)x(L1-L2)^3}{3xEIx} = 0.5676 \text{ cm}$$

2) DUE TO DEAD LOAD

$$D2 = \frac{Wg1x L1^3}{8 \times E \times Ix} = 0.0346 \text{ cm}$$

3) TOTAL DEFLECTION

$$.82 = D1 + D2 = 0.6022 \text{ cm}$$

$$\text{RATIO : } D3/L = 1/631 < 500 \text{ ---- O.K}$$

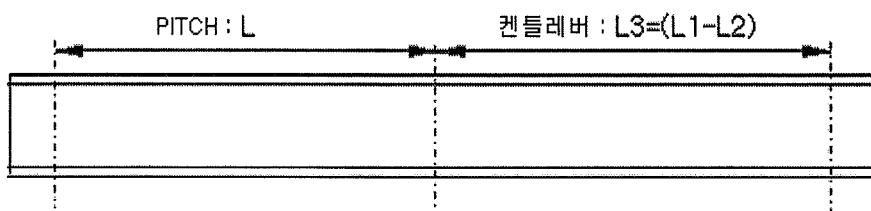


9. H-BEAM 최대허용 가능 SPAN 계산

1. SPECIFICATION

.정격하중	-----	Q =	1.8	ton
.SPAN (PITCH)	-----	L=	1100	cm
.켄틸리버	-----	L1=	420	cm
.TROLLEY WHEEL BASE	-----	B=	13.2	cm
.WEIGHT OF HOIST	-----	Wh=	0.16	ton
.PITCH내의 GIRDER 무게	-----	Wg=	0.824	ton
.켄틸리버의 GIRDER 무게	-----	Wg1=	0.315	ton
.H-BEAM SIZE	-----	450x200x9/14t		
		I _x =	32900	cm ⁴
		I _y =	1870	cm ⁴
		Z _x =	1460	cm ³
		Z _y =	187	cm ³
		A =	95.43	cm ²
		W _b =	74.9	kg/m
.HOOK APPROACH	-----	L ₂ =	30	cm
탄성계수	-----	E=	2100000	kg/cm ²
동하중 계수(작업계수)	-----	M(ϕ)=	1.11	
정하중 계수(총격계수)	-----	F(ψ)=	1.10	

1. DESIGN



2. H-BEAM에 작용하는 하중

$$P = Q + Wh = 1.8 + 0.16 = 1.96 \text{ ton}$$

3. 수직하중에 의한 BENDING MOMENT

1) PITCH 지점내 BENDING MOMENT

$$M_{h1} = F \times M \times P \times (L - B/2)^2 / (4 \times L)$$

$$= 1.11 \times 1.1 \times 1.96 \times (1100 - 13.2/2)^2 / (4 \times 1100)$$



$$= 650.3 \text{ ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$M_{h2} = F \times M \times P \times L_3$$

$$= 1.11 \times 1.1 \times 1.96 \times (420 - 30) = 933.33 \text{ ton.cm}$$

4. 자중에 의한 BENDING MOMENT(I-BEAM)

1) PITCH 지점내 BENDING MOMENT

$$M_{t1} = F \times W_g \times L / 8 = 1.11 \times 0.824 \times 1100 / 8 = 125.763 \text{ ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$M_{t2} = F \times W_g \times L_1 / 2 = 1.11 \times 0.315 \times 420 / 2 = 73.43 \text{ ton.cm}$$

5. 수직하중에 의한 최대 BENDING MOMENT

$$M_{max1} = M_{h1} + M_{t1} = 776 \text{ ton.cm}$$

$$M_{max2} = M_{h2} + M_{t2} = 1006.8 \text{ ton.cm}$$

6. 풍하중에 의한 수평하중의 BENDING MOMENT

(1) GIRDER에 의한 수직 풍하중

$$\text{작업시 } V=16\text{m/s}, q = 19.9 \text{ kg/m}^2 \quad h(\text{최고양정}) = 30 \text{ m}$$

$$\text{작업시 } q = 8.5 \times \sqrt{h} = 8.5 \times \sqrt{42} = 19.9 \text{ kg/m}^2$$

$$H=(\text{GIRDER 높이}) \quad 0.25 \text{ m}$$

$$\text{PITCH내 풍하중} = F \times L \times H \times q \times 1.6 = 1.11 \times 11 \times 0.25 \times 19.9 \times 1.6 = 97 \text{ kg}$$

$$\text{켄틀레버 풍하중} = F \times L_1 \times H \times q \times 1.3 = 1.11 \times 4.2 \times 0.25 \times 19.9 \times 1.3 = 30 \text{ kg}$$

$$M_{FGG} = \frac{0.097 \times 1100}{8} - \frac{0.03 \times 420}{2} = 7.038 \text{ ton.cm}$$

$$M_{FG1} = \frac{0.03 \times 420}{2} = 6.3 \text{ ton.cm}$$

(2) HOIST에 의한 풍하중

* 작업시

$$\text{HH} = 1.0 \text{ m} \quad \text{HB} = 0.65 \text{ m} \quad 17 \text{ kg}$$

$$\text{풍하중} = F \times HB \times HH \times q \times 1.2 = 1.11 \times 0.65 \times 1 \times 19.9 \times 1.2 =$$

*PITCH내 풍하중

$$M_{FHG} = 17 \times 1100 / 4 = 4675 \text{ kg.cm} =$$

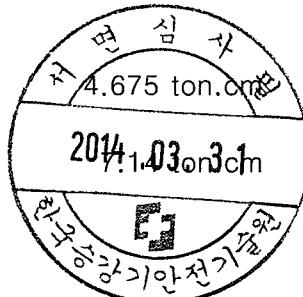
*켄틀레버 풍하중

$$M_{FH1} = 17 \times 420 = 7140 \text{ kg.cm} =$$

7. COMBINED MOMENT

*PITCH내

$$M_{HCG} = M_{FGG} + M_{FHG} = 7.038 + 4.675 = 11.713 \text{ ton.cm}$$



*肯들레버

$$M_{HC1} = M_{FG1} + M_{FH1} = 6.3 + 7.14 = 13.440 \text{ ton.cm}$$

8. BENDING STRESS

A. VERTICAL BENDING STRESS

1. PITCH 내

$$\sigma v1 = M_{max1}/Zx = 776 / 1460 = 0.532 \text{ ton/cm}^2$$

2. 켄들레버

$$\sigma v2 = M_{max2}/Zx = 1006.8 / 1460 = 0.690 \text{ ton/cm}^2$$

B. 수평 최대 응력

1. PITCH 내

$$\sigma v3 = M_{HCG}/Zy = 11.713 / 187 = 0.063 \text{ ton/cm}^2$$

2. 켄들레버

$$\sigma v4 = M_{HC1}/Zy = 13.44 / 187 = 0.072 \text{ ton/cm}^2$$

9. 합성 응력

$$\Sigma \sigma 1 = \sigma v1 + \sigma v3 = 0.594 \quad 0.594 < 1.279 \text{ ton/cm}^2 \quad \text{OK}$$

$$\Sigma \sigma 2 = \sigma v2 + \sigma v4 = 0.761 \quad 0.761 < 1.6 \text{ ton/cm}^2 \quad \text{OK}$$

용접 효율 : 80% 응력 $1391 \times 80\% \times 115\% = 1.279 \text{ ton/cm}^2$

肯들레버는 용접부 없음 $1391 \times 1.15(\text{풍하중포함}) = 1.600 \text{ ton/cm}^2$

10. 휴지시 풍하중에 의한 BENDING MOMENT

$$\begin{aligned} \text{휴지시 } V &= 45 \text{ m/s}, q = 158.0 \text{ kg/m}^2 & h(\text{최고양정}) &= 30 \text{ m} \\ q &= M \times 4 \sqrt{h} = 67.5 \times 4 \sqrt{30} = 158.0 \text{ kg/m}^2 \\ M &= V^2 / 30 = 67.5 \end{aligned}$$

* 휴지시 풍하중에 의한 수평하중

$$\text{PITCH내 풍하중} = F \times L \times H \times q \times 1.6 = 1.11 \times 11 \times 0.25 \times 158 \times 1.6 = 772 \text{ kg}$$

$$\text{肯들레버에 대한 풍하중} = F \times L \times H \times q \times 1.3 = 1.11 \times 4.2 \times 0.25 \times 158 \times 1.3 = 239 \text{ kg}$$

$$MM_{G1} = \frac{0.772 \times 1100}{8} - \frac{0.239 \times 420}{2} = 55.96 \text{ ton.cm}$$

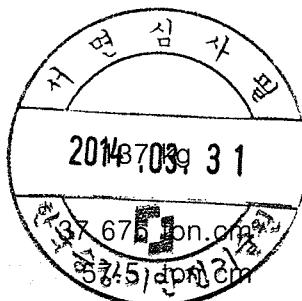
$$MM_1 = \frac{0.239 \times 420}{2} = 50.19 \text{ ton.cm}$$

$$\text{HOIST에 대한 풍하중} = F \times H \times B \times H \times q \times 1.2 = 1.11 \times 0.65 \times 1 \times 158 \times 1.2 =$$

$$* \text{PITCH내 } MM_{H0} = 137 \times 1100 / 4 = 150700 \text{ KG.CM} =$$

$$* \text{肯들레버 } MM_{H1} = 137 \times 420 = 57540 \text{ KG.CM} =$$

* COMBINED MOMENT



$$MM_2 = MM_{G1} + MM_{H0} = 55.96 + 37.675 = 93.635 \text{ ton.cm}$$

$$MM_4 = MM_1 + MM_{H1} = 50.19 + 57.5 = 107.69 \text{ ton.cm}$$

* BENDING STRESS

$$\Sigma\sigma_3 = MM_2 / Zy = 93.635 / 187 = 0.501 \text{ ton/cm}^2 < 1.447 \text{ ton/cm}^2 \text{--O.K}$$

$$\Sigma\sigma_4 = MM_4 / Zy = 107.69 / 187 = 0.576 \text{ ton/cm}^2 < 1.808 \text{ ton/cm}^2 \text{--O.K}$$

PITCH내 휴지시 응력 1391 x 80% x 130% = 1447 ton/cm²
 켄들레버는 용접부 없음 휴지시 응력 1391 x 130% = 1808 ton/cm²

11. DEFLECTION OF GIRDER

* PITCH 내

1) DUE TO RATED & TROLLEY LOAD

$$D1 = \frac{(Q+Q2)x(L)^3}{48xExIx} = 0.7866 \text{ cm}$$

2) DUE TO DEAD LOAD

$$D2 = \frac{5 \times Wgx L^3}{384 \times E \times Ix} = 0.2067 \text{ cm}$$

3) TOTAL DEFLECTION

$$.81 = D1 + D2 = 0.993 \text{ cm}$$

$$\text{RATIO : } D3/L = 1/1107 < 800 \text{ ---- O.K}$$

* 켄들레버

1) DUE TO RATED & TROLLEY LOAD

$$D1 = \frac{(Q+Q2)x(L1-L2)^3}{3xEIx} = 0.5609 \text{ cm}$$

2) DUE TO DEAD LOAD

$$D2 = \frac{Wg1x L1^3}{8 \times E \times Ix} = 0.0422 \text{ cm}$$

3) TOTAL DEFLECTION

$$.82 = D1 + D2 = 0.6031 \text{ cm}$$

$$\text{RATIO : } D3/L = 1/696 < 500 \text{ ---- O.K}$$



SUSPENSION MONORAIL 볼트 및 용접강도계산

1. 볼트로 고정시의 강도계산

* 허용 최대 SPAN 및 하중은 최악의 조건으로 계산한다.
(허용 최대 스판 11 M, 400x150x12.5/25t, HOIST자중 : 160 KG)

.Q = RATED LOAD= 1800 KG
.Q1=HOIST SELF WEIGHT= 160 KG
.Q2=I-BEAM WEIGHT = 1054 KG

BOLT 재질 : H.T.B	9.8 이상 사용할것
BOLT 허용전단응력 :	2100 KG/CM ²

M14 일때 do= 1.1835 CM n= 4.4 . n = 유효산수(산수*80%너트1종너트사용)
M16 일때 do= 1.3835 CM n= 5.2
M18 일때 do= 1.5294 CM n= 4.8
M20 일때 do= 1.7294 CM n= 5.1

$$P = 1.14*(Q+Q1) + 1.1* Q2/2
= 1.14*(1800+160) + 1.1* 1054/2
= 2814 \text{ KG} \quad (\sigma_a = 2100 \text{ KG/CM}^2)$$

$$1) \text{인장(전단)강도} : \sigma = P/A ; \sigma = \frac{4 \times P}{\pi \times do^2 \times Z} \quad * Z = \frac{\pi \times do^2 \times \sigma}{4 \times P}$$

$$\text{M16 일때} \quad Z = \frac{4 \times P}{\pi \times do^2 \times \sigma} = \frac{4 \times 2814}{835^2 \times 2100} = 0.89 \text{ 개}$$

$$\text{M18 일때} \quad Z = \frac{4 \times P}{\pi \times do^2 \times \sigma} = \frac{4 \times 2814}{294^2 \times 2100} = 0.73 \text{ 개}$$

$$\text{M20 일때} \quad Z = \frac{4 \times P}{\pi \times do^2 \times \sigma} = \frac{4 \times 2814}{294^2 \times 2100} = 0.57 \text{ 개}$$

2) 접촉 면압 강도 ($\sigma_a = 400 \text{ KG/CM}^2$)

$$\sigma = \frac{4 \times P}{\pi \times (d^2 - do^2) \times Z \times n} ; \quad Z = \frac{4 \times P}{\pi \times (d^2 - do^2) \times \sigma \times n}$$

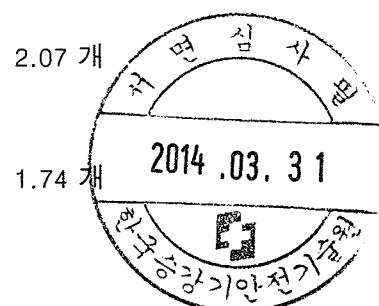
$$\text{M14 일때} \quad Z = \frac{4 \times 2814}{\pi \times (1.4^2 - 1.1835^2) \times 400 \times 4.4} = 3.64 \text{ 개}$$

$$\text{M16 일때} \quad Z = \frac{4 \times 2814}{\pi \times (1.6^2 - 1.3835^2) \times 400 \times 5.2} = 2.67 \text{ 개}$$

$$\text{M18 일때} \quad Z = \frac{4 \times 2814}{\pi \times (1.8^2 - 1.5294^2) \times 400 \times 4.8} = 2.07 \text{ 개}$$

$$\text{M20 일때} \quad Z = \frac{4 \times 2814}{\pi \times (2^2 - 1.7294^2) \times 400 \times 5.1} = 1.74 \text{ 개}$$

. do = 골경, P=브라켓 한 개에 작용하는 하중, Z = 볼트수
. n = 유효산수(산수*80%), 너트1종너트사용



3) 따라서 다음과 같이 적용한다

H.T.B M14,M16,M18, M20일때 : 브라켓트당 4개 이상 사용한다.

2. I-빔을 용접시공으로 고정시의 강도계산

* 허용 최대 SPAN 및 하중은 최악의 조건으로 계산한다.

(허용 최대 스판 11 M, 400x150x12.5/25t, HOIST자중 : 160 KG)

.Q = RATED LOAD= 1800 KG (h : 용접두께
.Q1=HOIST SELF WEIGHT= 160 KG L : 용접길이)
.Q2=I-BEAM WEIGHT = 1054 KG

1) 용접이음부 인장강도 계산 (용접두께 : 45° 용접부위)

$$\sigma = \frac{1.414 \times P}{h * L} ; (\sigma_a = 1200 \text{ KG/CM}^2) \quad L = \frac{1.414 \times P}{h * \sigma}$$

(1) h 가 5일때

$$L = \frac{1.414 \times 2814}{0.5 \times 1200} = 6.63 \text{ CM} = 66.3 \text{ mm}$$

(2) h 가 6일때

$$L = \frac{1.414 \times 2814}{0.6 \times 1200} = 5.53 \text{ CM} = 55.3 \text{ mm}$$

(3) h 가 7일때

$$L = \frac{1.414 \times 2814}{0.7 \times 1200} = 4.74 \text{ CM} = 47.4 \text{ mm}$$

2) 적용

h = 5일때, 한 브라켓트당 용접길이 L = 80mm 이상 용접한다.

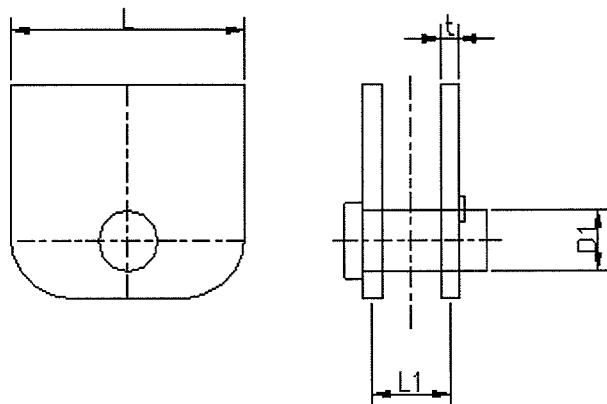
h = 6일때, 한 브라켓트당 용접길이 L = 70mm 이상 용접한다.

h = 7일때, 한 브라켓트당 용접길이 L = 50mm 이상 용접한다.



* 상후크 지지대 SHAFT & BRACKET

1. DESIGN DATA



1) DIMENSION

PIN직경	D1	4.0	판 두께	t2	1.2	
중심거리	L1	6.6	핀수량		1	EA
	L	15.0				

2) 허용 응력

SS41-PLATE S45C-SHAFT	허용압축응력	$\sigma_p =$	1400	KG/CM ²
	허용전단응력	$\sigma_t =$	800	KG/CM ²
	허용굽힘응력	$\sigma_b =$	1400	KG/CM ²

3) RATED LOAD

$$Q = 1800 \text{ KG}$$

4) CHAIN HOIST WEIGHT

$$G1 = 160 \text{ KG}$$

2. PIN의 전단응력, σ_t

$$\sigma_t = P / A,$$

$$P = (Q+G1)/2 = 980 \text{ KG}$$

$$\sigma_t = \frac{P}{\pi D^2/4} = \frac{P * 4}{\pi D^2}$$

$$\sigma_t = \frac{4 \times 980}{\pi \times 4^2} = 78 \text{ KG/CM}^2 < 800 \text{ KG/CM}^2 \text{ ---- O.K}$$

전단면 2곳

3. PIN의 굽힘응력, σ_b

$$P1 = Q+G1 = 1960 \text{ KG}$$

$$\sigma_b = M / Z = \frac{P \times L1 / 4}{\pi D^3 / 32} = \frac{32 \times P \times L1}{4 \times \pi \times D^3}$$

$$\sigma_b = \frac{32 \times 1960 \times 6.6}{4 \times \pi \times 4^3} = 515 \text{ KG/CM}^2 < 1400 \text{ KG/CM}^2 \text{ ---- O.K}$$

4. BRACKET의 지지압, σ_p

$$\sigma_p = P1 / A = \frac{P1}{(L-D) \times t \times E} \times f$$

$$\sigma_{p1} = \frac{P \times f}{(L3-D1) \times t1 \times 2} = \frac{1960 \times 2.5}{(15-4) \times 1.2 \times 2} = 186 \text{ KG/CM}^2 < 1400 \text{ KG/CM}^2 \text{ ---- O.K}$$

*E(PLATE 수령) : EA
*f(형상계수) : 2.5



6. FOR REFERENCE

- 1) LOAD CHAIN 시험성적서
- 2) MOTOR DATA SHEET
- 3) HOIST 사용설명서(operation manual)





Date: 2009/04/14

Certificate of Compliance

We certify that the ER2 protection degrees conform to the IP rating as follows:

Hoist body - IP55 based on JIS C 4034-5, "Rotating electrical machines – Part5: Classification of degrees of protection provided by enclosures of rotating electrical machines (IP code)".

Push button - IP65 based on JIS C 0920, "Tests to prove protection against ingress of water and degrees of protection against ingress of solid objects for electrical equipment".

Technical Control Group

Page 1 of 1

Test Certificate

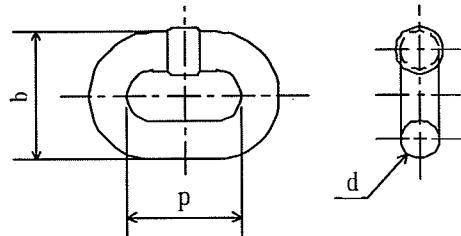
Messrs.

Commodity: NC Load Chain

Code : KER102

Lot No. : —

Quantity: — line(s)



1. Material: Manganese Alloy Steel

2. Dimensions

	d	p	b
Specified	10.2mm ± 0.4	28.4mm $^{+0.56}_0$	Max. 35.7mm
Result	Good	Good	Good

3. Breaking test

	Breaking load	Total ultimate elongation
Specified	Min. 131 (kN)	Min. 10 (%)
Result	Good	Good

4. Manufacturing Proof force test (Test load: 81.7 kN)

	Permanent elongation
Specified	0.25 (%)
Result	Good

General judgment: Satisfactory

KITO KITO CORP.

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Quality Assurance Group
Quality Assurance Department
Development & Technology Division

K. Kishimoto (Manager)

Certificate No.: MM080011e

Date of Issue: 2009/3/4

Messrs.

Motor Test Report for Electric Chain Hoist

Motor type : Three phase squirrel cage type induction motor.

Manufacturer : Yasukawa Electric Mfg. Co.

Production No. : -

Rating

Model	Output	Pole	Intermittent Rating	Voltage	Frequency
IBQ	1.8kW	4P	60%ED	220V	60Hz

Full load characteristics

Voltage	Frequency	220V 60Hz
Load	%	100
Current	A	8.4
Speed	rpm	1620

Insulation class E

The above characteristics are obtained from calculation where the motor is assembled with an electric chain hoist and the hoist is subjected to full load



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A handwritten signature in black ink that reads "M. Ogihara".

(Manager)

Certificate No.: MM080011f

Date of Issue: 2009/3/4

Messrs. _____

Motor Test Report for Electric Chain Hoist

Motor type : Three phase squirrel cage type induction motor.

Manufacturer : Yasukawa Electric Mfg. Co.

Production No. : -

Rating

Model	Output	Pole	Intermittent Rating	Voltage	Frequency
IBQ	1.8kW	4P	40/20%ED	220V	Speed Control by Inverter

Full load characteristics

Voltage	Frequency	220V Speed Control by Inverter
Load	%	100
Current	A	11.2
Speed	rpm	~

Insulation class E

The above characteristics are obtained from calculation where the motor is assembled with an electric chain hoist and the hoist is subjected to full load



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(Manager)

Certificate No.: MM070011e

Date of Issue: 2008/03/21

Messrs. _____

Motor Test Report for Electric Chain Hoist

Motor type : Three phase squirrel cage type induction motor.

Manufacturer : Yasukawa Electric Mfg. Co.

Production No. :

Rating

Model	Output	Pole	Intermittent Rating	Voltage	Frequency
IBQ	1.8kW	4P	60%ED	380 - 440V	60Hz

Full load characteristics

Voltage Frequency	380 - 440V 60Hz
Load %	100
Current A	4.6
Speed rpm	1610

Insulation class B

The above characteristics are obtained from calculation where the motor is assembled with an electric chain hoist and the hoist is subjected to full load



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(Manager)

K. Kishimoto

Certificate No.: MM070011f

Date of Issue: 2008/03/21

Messrs. _____

Motor Test Report for Electric Chain Hoist

Motor type : Three phase squirrel cage type induction motor.

Manufacturer : Yasukawa Electric Mfg. Co.

Production No. :

Rating

Model	Output	Pole	Intermittent Rating	Voltage	Frequency
IBQ	1.8kW	4P	60%ED	380 - 440V	Speed Control by Inverter

Full load characteristics

Voltage Frequency	380 - 440V Speed Control by Inverter
Load %	100
Current A	5.1
Speed rpm	~

Insulation class B

The above characteristics are obtained from calculation where the motor is assembled with an electric chain hoist and the hoist is subjected to full load



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Certificate No.: MM080012a

Date of Issue: 2009/3/4

Messrs.

Motor Test Report for Electric Trolley

Motor type : Three phase squirrel cage type induction motor.

Manufacturer : Yasukawa Electric Mfg. Co.

Production No. : -

Rating

Model	Output	Pole	Intermittent Rating	Voltage	Frequency
IBQ-T	0.4kW	4P	40%ED	220V	60Hz

Full load characteristics

Voltage	Frequency	220V 60Hz
Load	%	100
Current	A	3.0
Speed	rpm	1685

Insulation class E

The above characteristics are obtained from calculation where the motor is assembled with an electric trolley and the trolley is subjected to full load



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(Manager)

Certificate No.: MM080012b

Date of Issue: 2009/3/4

Messrs. _____

Motor Test Report for Electric Trolley

Motor type : Three phase squirrel cage type induction motor.

Manufacturer : Yasukawa Electric Mfg. Co.

Production No. : -

Rating

Model	Output	Pole	Intermittent Rating	Voltage	Frequency
IBQ-T	0.4kW	4P	27/13%ED	220V	Speed Control by Inverter

Full load characteristics

Voltage	Frequency	220V Speed Control by Inverter
Load	%	100
Current	A	3.0
Speed	rpm	~

Insulation class E

The above characteristics are obtained from calculation where the motor is assembled with an electric trolley and the trolley is subjected to full load



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A handwritten signature in black ink that reads "M. Ogihara".

(Manager)

Certificate No.: MM070013a

Date of Issue: 2008/03/21

Messrs. _____

Motor Test Report for End Carriage

Motor type : Three phase squirrel cage type induction motor.

Manufacturer : Yasukawa Electric Mfg. Co.

Production No. :

Rating

Model	Output	Pole	Intermittent Rating	Voltage	Frequency
IBQ-T	0.4kW	4P	40%ED	380 - 440V	60Hz

Full load characteristics

Voltage Frequency	380 - 440V 60Hz
Load %	100
Current A	2.2
Speed rpm	1670

Insulation class B

The above characteristics are obtained from calculation where the motor is assembled with an electric chain hoist and the hoist is subjected to full load



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(Manager)

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Certificate No.: MM070013b

Date of Issue: 2008/03/21

Messrs. _____

Motor Test Report for End Carriage

Motor type : Three phase squirrel cage type induction motor.

Manufacturer : Yasukawa Electric Mfg. Co.

Production No. :

Rating

Model	Output	Pole	Intermittent Rating	Voltage	Frequency
IBQ-T	0.4kW	4P	40%ED	380 - 440V	Speed Control by Inverter

Full load characteristics

Voltage	Frequency	220 – 230V	Speed Control by Inverter
Load	%	100	
Current	A	2.5	
Speed	rpm	~	

Insulation class B

The above characteristics are obtained from calculation where the motor is assembled with an electric chain hoist and the hoist is subjected to full load



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