

【별지 제4호서식】

동 일 형 식 일 랑 표

사업장명	KITO CORP.	개정일자 및 번호	2013.10.12	인증번호			
형식 및 모델번호		동일형식 항목 및 내역					
형식번호	모델번호	동일형식 항목1	동일형식 항목2	동일형식 항목3	동일형식 항목4		
KML- SHER2-010	KITO-SHER2SP010L	Lift max 30m	권상모타 0.9kW	횡행모타 없음	Trolley 있음		
	KITO-SHER2SP010IL		권상모타 0.9kW		Trolley + 수동체인		
	KITO-SHER2SG010L						
	KITO-SHER2SG010IL						
	KITO-SHER2M010L-S		Lift max 30m	권상모타 0.9kW	횡행모터 0.4kW .S : 24m/min .L: 12m/min .IS:24/4m/min .IL:12/2m/min	전기Trolley 결합 type	
	KITO-SHER2M010L-L						
	KITO-SHER2M010L-IS						
	KITO-SHER2M010L-IL						
	KITO-SHER2M010IL-S						
	KITO-SHER2M010IL-L						
	KITO-SHER2M010IL-IS						
	KITO-SHER2M010IL-IL						
	KITO-C-SHER2M010L-S			권상모타 0.9kW			전기Trolley 결합 Clean type
	KITO-C-SHER2M010L-L						
	KITO-C-SHER2M010L-IS						
	KITO-C-SHER2M010L-IL						
	KITO-C-SHER2M010IL-S						
	KITO-C-SHER2M010IL-L						
KITO-C-SHER2M010IL-IS	권상모타 0.9kW						
KITO-C-SHER2M010IL-IL							



제 2012-BJ-0009 호



안 전 인 증 서

정호엔지니어링

경기도 광명시 노온사동 440-5

위 사업장에서 제조하는 아래의 품목이 산업안전보건법 제34조 및 같은 법 시행규칙 제58조의4제4항에 따른 안전인증 심사 결과 안전·보건기준에 적합하므로 안전인증표시의 사용을 인증합니다.

품 목

양중기용 과부하방지장치

형식·모델/용량·등급/인증번호

형식·모델	용량·등급	인증번호
JDL-100	J-2	12-AV2BJ-0009

인 증 기 준

방호장치 의무안전인증 고시(고용노동부고시 제2010-36호)

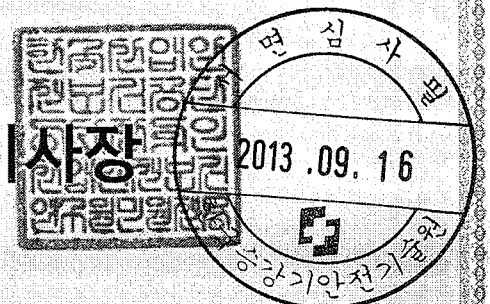
인 증 조 건

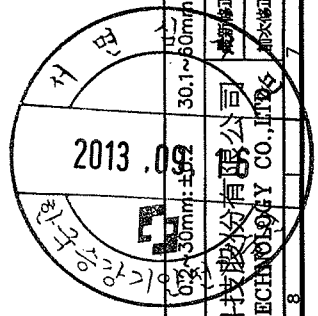
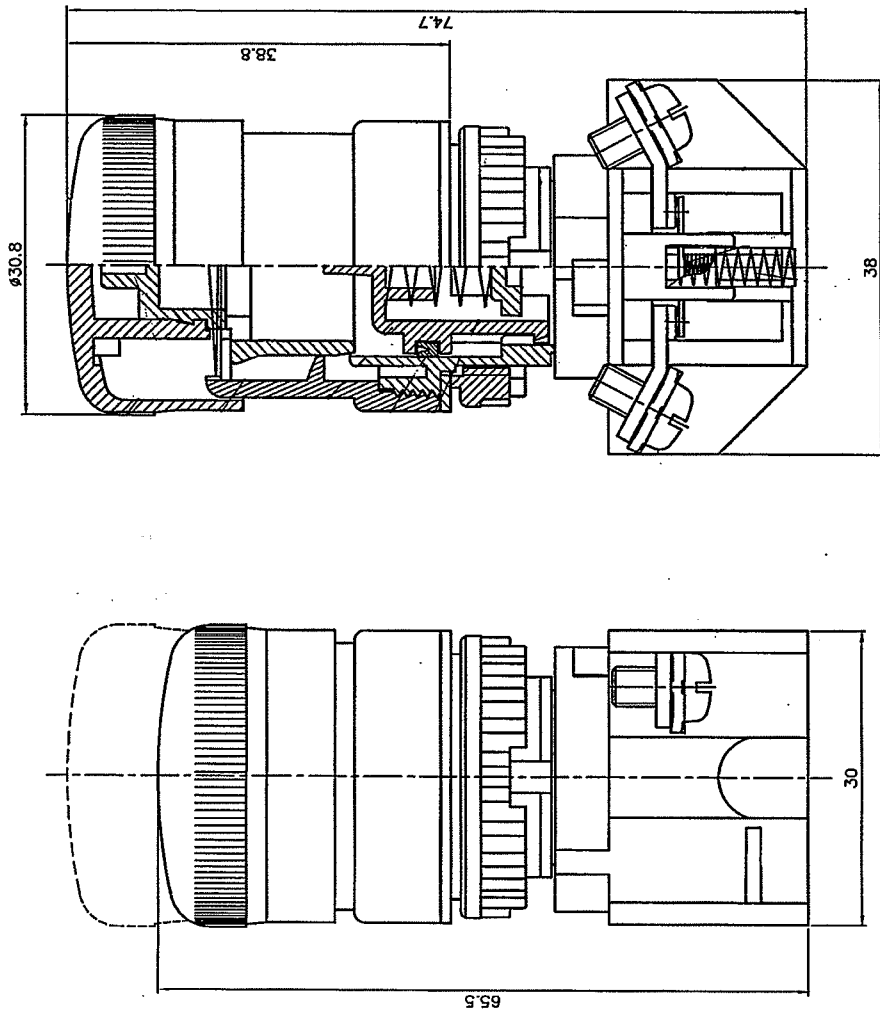
아래 주소에서 생산되는 제품에 한함.

정호엔지니어링, 경기도 광명시 노온사동 440-5

2012년 06월 11일

한국산업안전보건공단 이사장

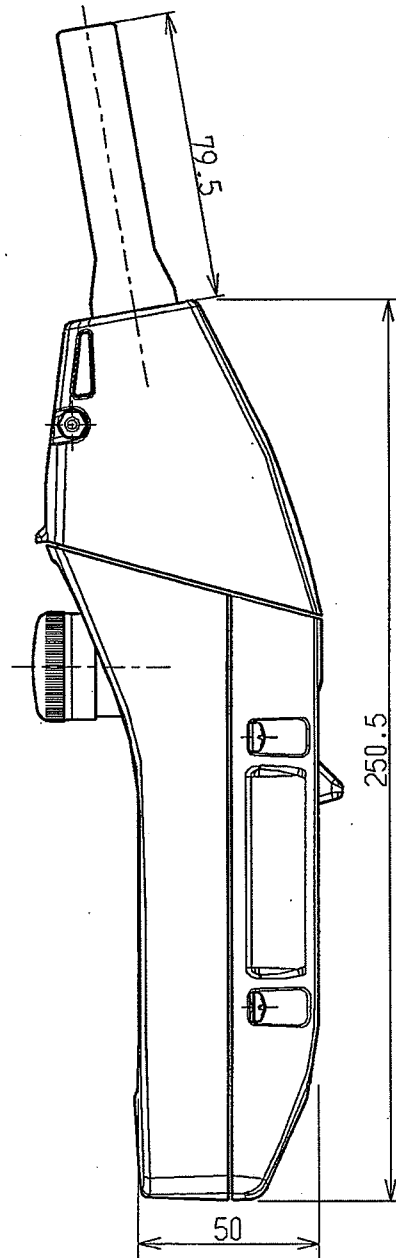
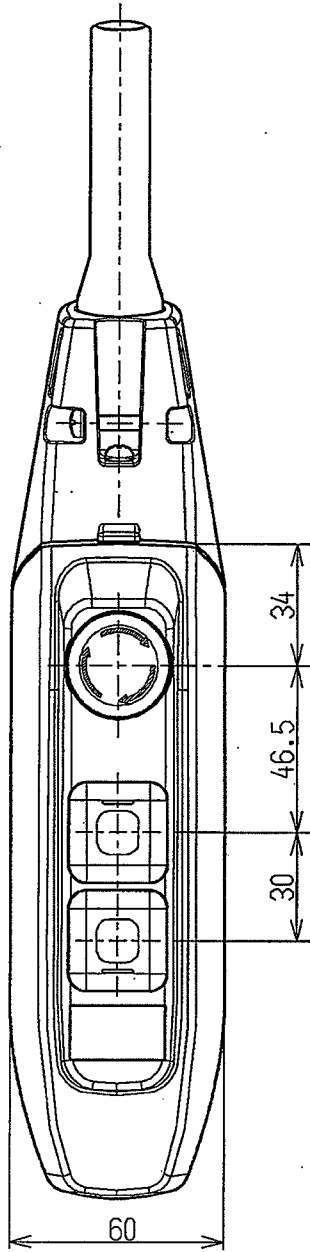




一般公差	0.12~30mm: ±0.3	60.1~300mm: ±0.5	模具材質	模具處理	模具孔數	單位	mm	材質	圖號	T2-BKH
	30.1~50mm: ±0.3		核准	校對	繪圖	比例	2:1	表面處理	品名	T2 BKH 連續開關
			品保部 95.05.24 林建宏	研發部 95.05.24 周啟祥	設計課 95.05.24 吳宗達	投影法	第一角	顏色		

附:A

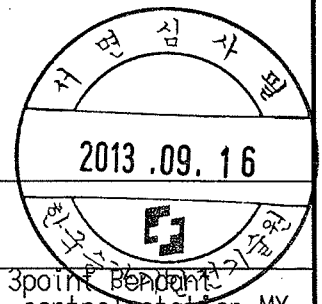
Revision	Incidence	Description	Date	Change	Approved



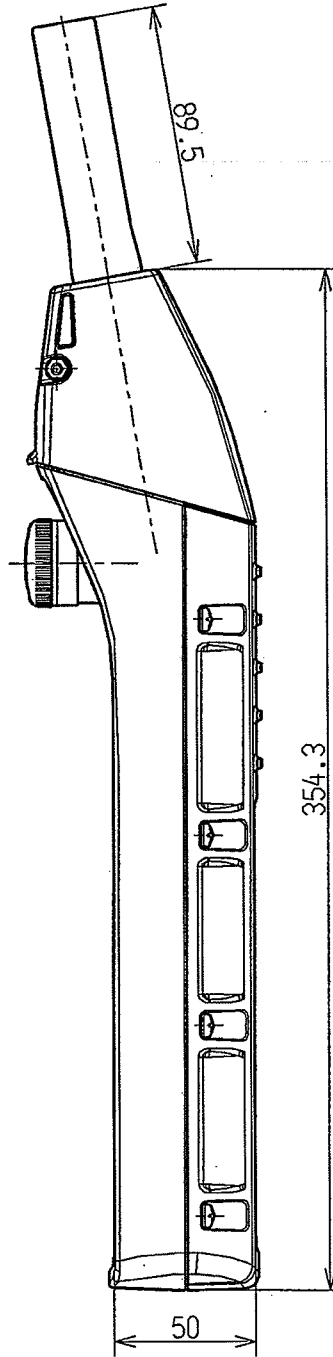
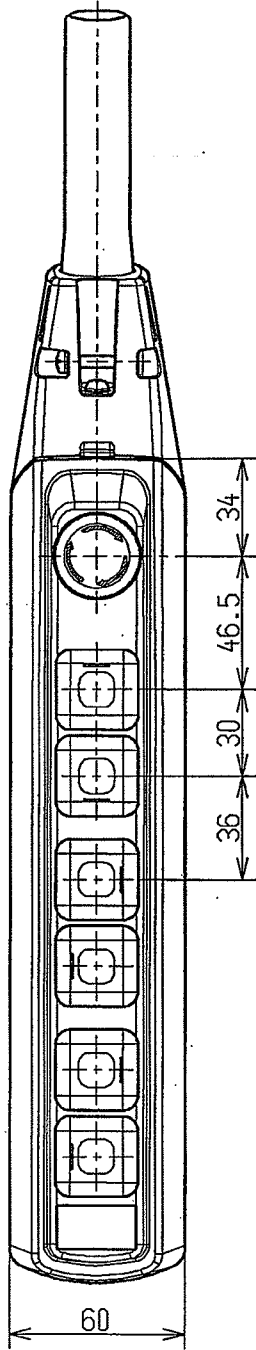
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NOTE

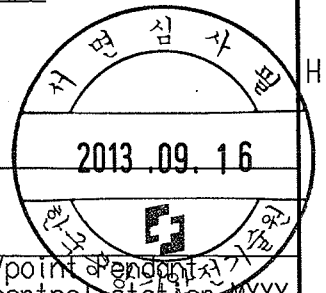
APPROVED	ISHIKAWA	CHECKED	FURIYA	DESIGNED	KOBAYASHI	DRAWN	KOBAYASHI	SCALE	-	DWG. NO.	SWD2X00AA1	NAME CODE	3point Pendant control station MX subassembly
Date issued	08.02.08		08.02.08		08.02.08		08.02.08						



Revision	Incidence	Description	Date	Charge	Approved



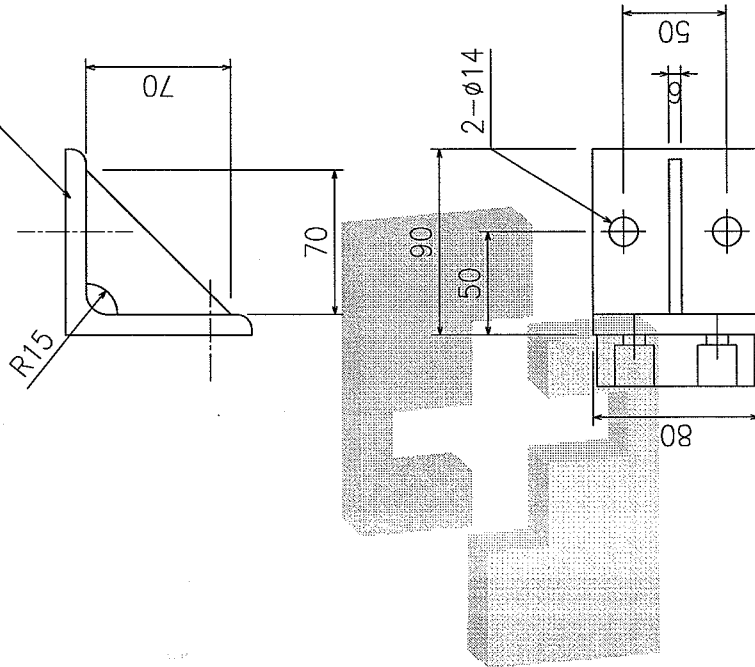
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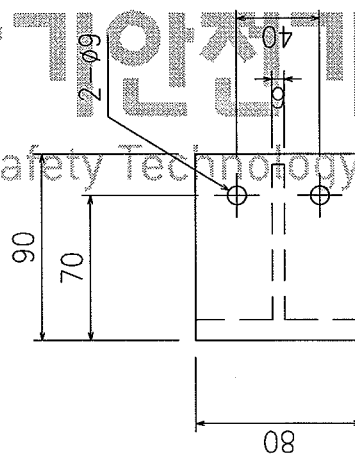
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DATE ISSUED						APPROVED		CHECKED		DESIGNED		DRAWN		SCALE		DWG. NO.		MATERIAL		NAME		CODE	
08.02.08						ISHIKAWA		FURIYA		KOBAYASHI		KOBAYASHI		-		SWD2XXXXAA1		-		7point of control station MXXX subassembly		-	
Date issued						08.02.08		08.02.08		08.02.08		08.02.08		-		SWD2XXXXAA1		-		7point of control station MXXX subassembly		-	

L - 90x10t



한국승강기안전기술원
Korea Elevator Safety Technology Institute



No	Part Name	Description	Mat'l	Unit	Q'ty	Weight (kg)	Remark
	TITLE	STOPPER - traversing	SS400		4		
	Part No.	STOPPER					SCALE
	DWG No.						REV.

APPROVED	CHECKED	DESIGNED	DRAWN
J. S. CHO	J. S. CHO	W.H.LEUN	W.H.LEUN

DATE	DRAWN	APPROVED

REV.	CONTENTS

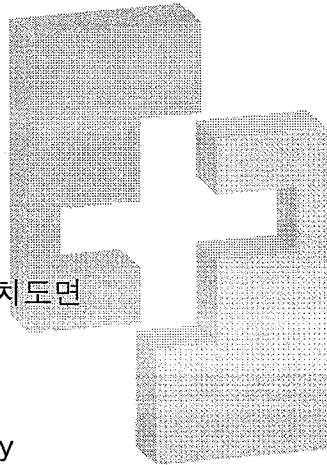
3) HOIST 기계도면

3-1. 조립도

- 1) 전체 조립도

3-2. 상세도

- 1) ER2 Friction Clutch
- 2) ER2 Motor Brake Structure & Brake Gap Limit
- 3) Traversing wheel
- 4) Bottom Hook
- 5) Bottom yoke

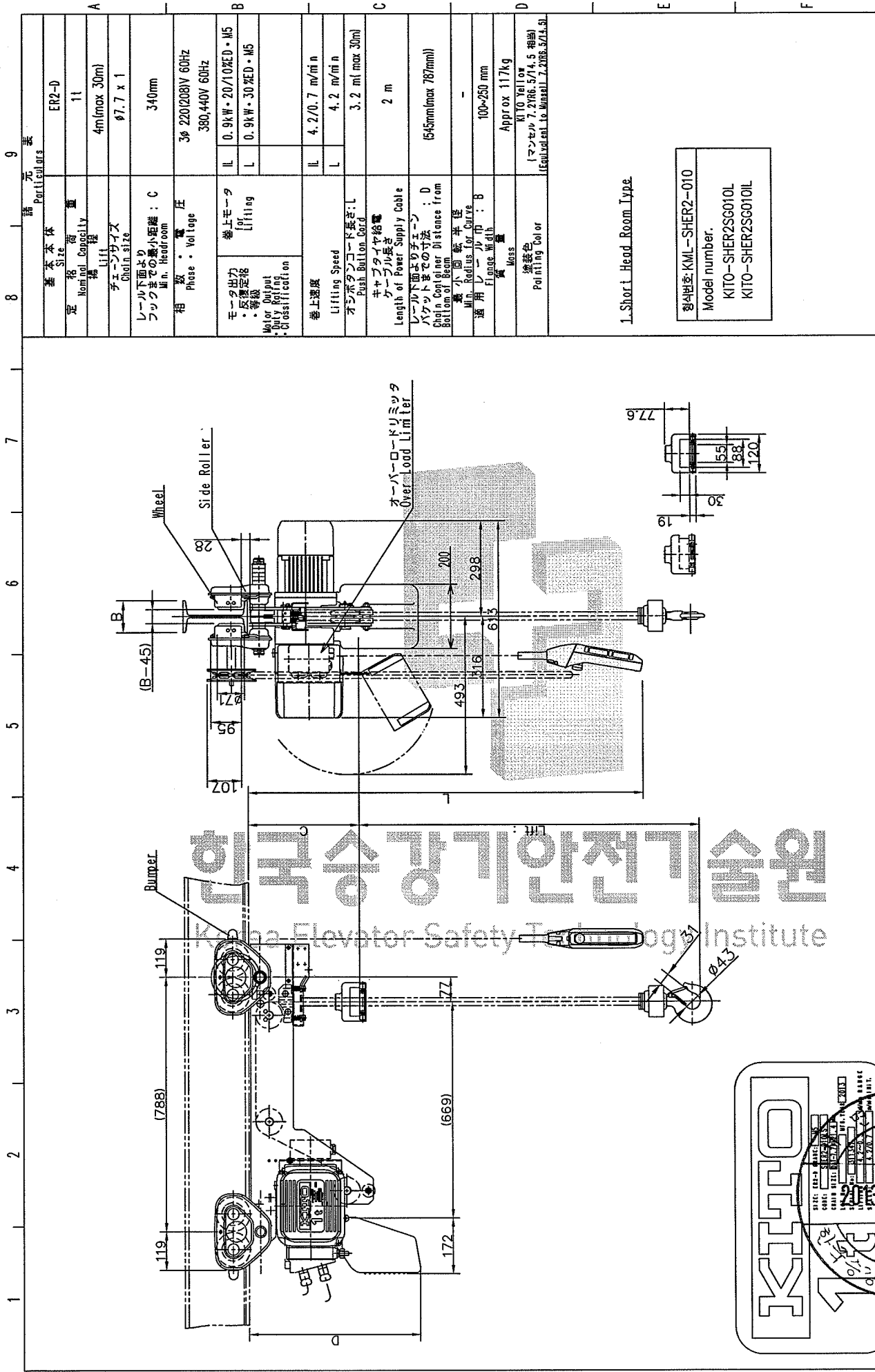


3-3. MONO RAIL 설치도면

- 1) Safety Dwg.
- 2) General assembly
- 3) Girder detail

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Korea Elevator Safety Technology Institute





基本仕様		Particulars	
基本仕様	ER2-D	基本仕様	ER2-D
定格容量	11	定格容量	11
リフト高	4m(max 30m)	リフト高	4m(max 30m)
チェーンサイズ	φ7.7 x 1	チェーンサイズ	φ7.7 x 1
レール下面よりフックまでの最小距離	340mm	レール下面よりフックまでの最小距離	340mm
相電圧	3φ 220/208V 60Hz	相電圧	3φ 220/208V 60Hz
	380/440V 60Hz		380/440V 60Hz
モーター出力	IL 0.9kW・20/10%ED・M5	モーター出力	IL 0.9kW・20/10%ED・M5
モーター定格電圧	L 0.9kW・30%ED・M5	モーター定格電圧	L 0.9kW・30%ED・M5
モーター出力	IL 4.2/0.7 mv/min	モーター出力	IL 4.2/0.7 mv/min
モーター出力	L 4.2 mv/min	モーター出力	L 4.2 mv/min
モーター出力	3.2 ml max 30ml	モーター出力	3.2 ml max 30ml
モーター出力	2 m	モーター出力	2 m
モーター出力	(55mm(max 78mm))	モーター出力	(55mm(max 78mm))
モーター出力	100~250 mm	モーター出力	100~250 mm
モーター出力	Approx 117kg	モーター出力	Approx 117kg
モーター出力	KITO Yellow	モーター出力	KITO Yellow
モーター出力	(同モデル 7.2TR6.5/74.5 相当)	モーター出力	(同モデル 7.2TR6.5/74.5 相当)
モーター出力	(Equivalent to Model 7.2TR6.5/74.5)	モーター出力	(Equivalent to Model 7.2TR6.5/74.5)

1.Short Head Room Type

型番: KML-SHER2-010
 Model number:
 KITO-SHER2SG010L
 KITO-SHER2SG010L

名称	1t ER2M SERIES ELECTRIC CHAIN HOIST WITH GEAR TROLLEY		
製造番号	SHER2-SG	尺度	NOT
図番	KML-SHER2-010-002	変更回数	0
図番	KML-SHER2-010-002	変更回数	0

承認	設計	製図	製図
A. Saito	K. Suzuki	K. Horiuchi	—
13, 8, 23	13, 8, 23	13, 8, 23	13, 8, 23

承認	承認	承認	承認
K. Saito	K. Saito	K. Saito	K. Saito
13, 8, 26	13, 8, 26	13, 8, 26	13, 8, 26

内容	258→208	数量	1
内容	24/4(mv/min)→12/2(mv/min)	数量	1

KITO

09.16

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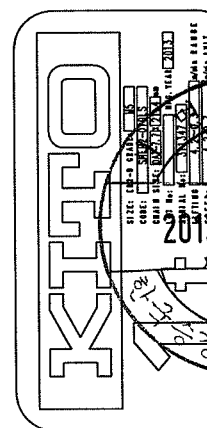
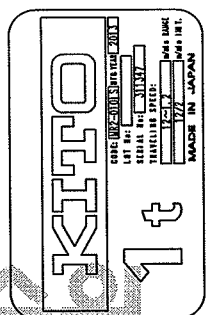
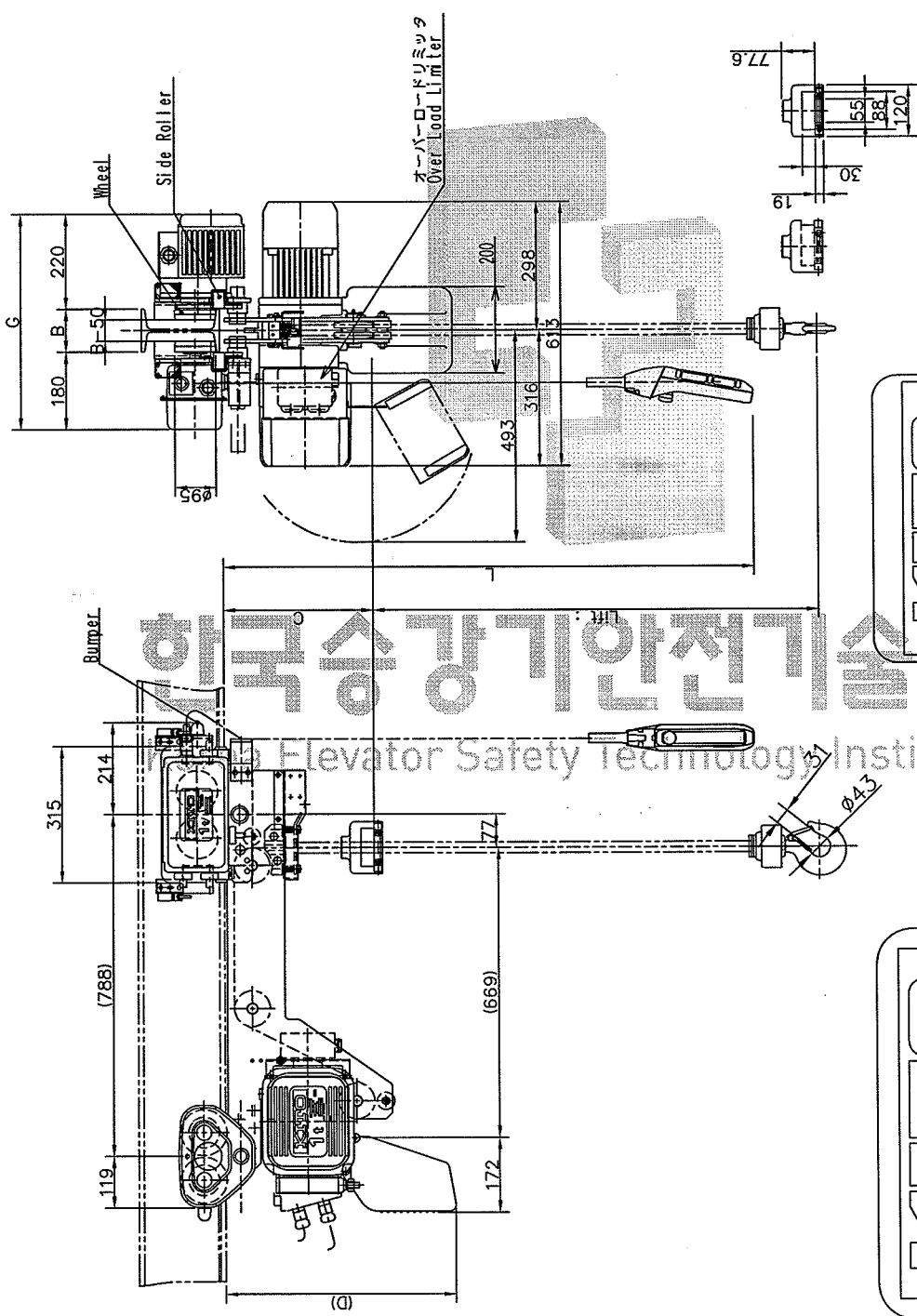
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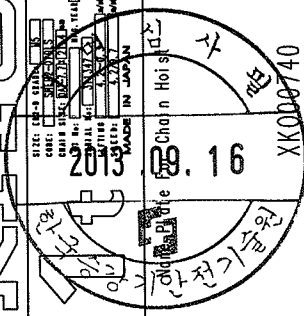
Particulars	
基本本体 Size	ER2-D
定荷 Nominal Capacity	1t
チェーンサイズ Chain size	4m(max 30m) φ7.7 x 1
チェーン下より フックまでの最小距離 : C Min. Headroom	340mm
相数・電圧 Phase・Voltage	3φ 220/208V 60Hz 380,440V 60Hz
巻上モータ [Lifting Motor Output ・反復定格 ・等級 Repeating Classification]	IL 0.9kW・20/10%ED・M5 L 0.9kW・30%ED・M5
横行モータ [Traversing Motor Output ・反復定格 ・等級 Repeating Classification]	IS,IL 0.4kW・27/13%ED SL 0.4kW・40%ED
巻上速度 Lifting Speed	IL 4.2/0.7 m/min L 4.2 m/min
横行速度 Traversing Speed	IS 24/4 m/min IL 12/2 m/min
横行速度 Traversing Speed	S 24 m/min L 12 m/min
押しボタン Push Button	3.2 m(max 30m)
ケーブル長さ Cable Length	2 m
チェーンコンテナ Chain Container	645mm(max 787mm)
最小回転半径 Min. Radius for Curve	100×250 mm
トロリ最大巾 Trolley Width	500×650 mm
質量 Mass	Approx 134kg
塗装色 Painting Color	KITO Yellow [マンセル 7.2R6.5/14.5 相当] [Equivalent to Munsell 7.2R6.5/14.5]
1. Short Head Room Type	型番: KML-SHER2-010 Model number: KITO-SHER2M010L-S KITO-SHER2M010L-L KITO-SHER2M010L-IS KITO-SHER2M010L-IL KITO-SHER2M010L-S KITO-SHER2M010L-L KITO-SHER2M010L-IS KITO-SHER2M010L-IL

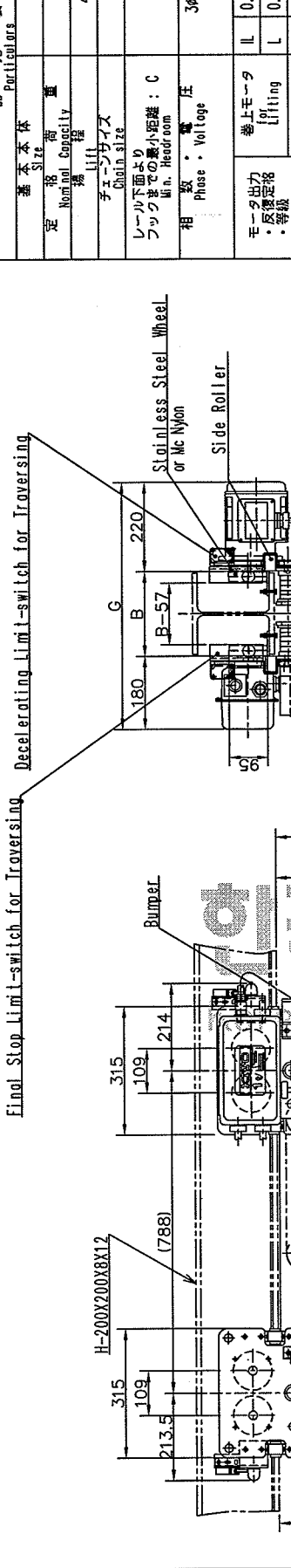
名称 TITLE	1t ER2M SERIES ELECTRIC CHAIN HOIST WITH MOTORIZED TROLLEY		
製造番号 CODE	SHER2-M	尺度 SCALE	NOT
図番 Dwg. No.	KML-SHER2-010-003	変更回数 REV.	0

形式: 075P-19 三角法単位: mm



承認 APPROVED	設計 DESIGNED	製図 DRAWN
A. Saito	K. Suzuki	—
13, 8, 23	13, 8, 23	13, 8, 23
13, 8, 26	13, 8, 26	13, 8, 26
年 月 日 DATE	担当 承認 DRAWN APPROVED	
258→208	24/4(m/min)→12/2(m/min)	
設計 数 REV.	内 容 CONTENTS	
1		





諸元表 Particulars	
基本本体 Nominal Capacity	ER2-D 1t
チェーンサイズ Chain size	4m(max 30m) 97.7 x 1
レール下面よりフックまでの最小距離: C Min. Headroom	340mm
相数・電圧 Phase・Voltage	3φ 220/208/1V 60Hz 360/440V 60Hz
モーター出力・反巻定格 Motor Output for Reversing	IL 0.9kW・20/10%ED・M5 L 0.9kW・30%ED・M5
モーター出力・反巻定格 Motor Output for Reversing	ISIL 0.4kW・27/13%ED SL 0.4kW・40%ED
巻上速度 Lifting Speed	IL 4.2/0.7 m/min L 4.2 m/min
横行速度 Standard speed	IS 24/4 m/min IL 12/2 m/min
横行速度 Traversing Speed	S 24 m/min L 12 m/min
オンボタコード長さ: L Push Button Cord	3.2 m(max 30m)
ケーブル給電 Length of Power Supply Cable	2 m
レール下面よりチェーンの寸法 Chain dimension from Bottom of Beam	1545mm(max 787mm)
最小回転半径 Min. Radius for Curve	158~258 mm
フロア幅 Floor Width	557~657 mm
質量 Mass	Approx 140kg
塗装色 Painting Color	KITO Yellow (マンセル: 7.2YR6.5/14.5 相当) (Equivalent to Munsell 7.2R6.5/14.5)
1. Short Head Room Type	型式記号: KML-SHER2-010 Model number: KITO-C-SHER2M010L-S KITO-C-SHER2M010L-L KITO-C-SHER2M010L-IS KITO-C-SHER2M010L-IL KITO-C-SHER2M010L-S KITO-C-SHER2M010L-L KITO-C-SHER2M010L-IS KITO-C-SHER2M010L-IL

承認 APPROVED		検閲 CHECKED		設計 DESIGNED		製図 DRAWN	
A. Saito		K. Suzuki		K. Horiiuchi		—	
13, 8, 23	13, 8, 23	13, 8, 23	13, 8, 23	13, 8, 23	13, 8, 23	13, 8, 23	13, 8, 23
年, 月, 日	年, 月, 日	年, 月, 日	年, 月, 日	年, 月, 日	年, 月, 日	年, 月, 日	年, 月, 日

承認 APPROVED		検閲 CHECKED		設計 DESIGNED		製図 DRAWN	
E. Horiechi		A. Saito		K. Horiiuchi		—	
13, 8, 26	13, 8, 26	13, 8, 26	13, 8, 26	13, 8, 26	13, 8, 26	13, 8, 26	13, 8, 26
年, 月, 日	年, 月, 日	年, 月, 日	年, 月, 日	年, 月, 日	年, 月, 日	年, 月, 日	年, 月, 日

承認 APPROVED		検閲 CHECKED		設計 DESIGNED		製図 DRAWN	
E. Horiechi		A. Saito		K. Horiiuchi		—	
13, 8, 26	13, 8, 26	13, 8, 26	13, 8, 26	13, 8, 26	13, 8, 26	13, 8, 26	13, 8, 26
年, 月, 日	年, 月, 日	年, 月, 日	年, 月, 日	年, 月, 日	年, 月, 日	年, 月, 日	年, 月, 日

承認 APPROVED		検閲 CHECKED		設計 DESIGNED		製図 DRAWN	
E. Horiechi		A. Saito		K. Horiiuchi		—	
13, 8, 26	13, 8, 26	13, 8, 26	13, 8, 26	13, 8, 26	13, 8, 26	13, 8, 26	13, 8, 26
年, 月, 日	年, 月, 日	年, 月, 日	年, 月, 日	年, 月, 日	年, 月, 日	年, 月, 日	年, 月, 日

承認 APPROVED		検閲 CHECKED		設計 DESIGNED		製図 DRAWN	
E. Horiechi		A. Saito		K. Horiiuchi		—	
13, 8, 26	13, 8, 26	13, 8, 26	13, 8, 26	13, 8, 26	13, 8, 26	13, 8, 26	13, 8, 26
年, 月, 日	年, 月, 日	年, 月, 日	年, 月, 日	年, 月, 日	年, 月, 日	年, 月, 日	年, 月, 日

承認 APPROVED		検閲 CHECKED		設計 DESIGNED		製図 DRAWN	
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13, 8, 26	13, 8, 26	13, 8, 26	13, 8, 26	13, 8, 26	13, 8, 26	13, 8, 26	13, 8, 26
年, 月, 日	年, 月, 日	年, 月, 日	年, 月, 日	年, 月, 日	年, 月, 日	年, 月, 日	年, 月, 日

承認 APPROVED		検閲 CHECKED		設計 DESIGNED		製図 DRAWN	
E. Horiechi		A. Saito		K. Horiiuchi		—	
13, 8, 26	13, 8, 26	13, 8, 26	13, 8, 26	13, 8, 26	13, 8, 26	13, 8, 26	13, 8, 26
年, 月, 日	年, 月, 日	年, 月, 日	年, 月, 日	年, 月, 日	年, 月, 日	年, 月, 日	年, 月, 日

承認 APPROVED		検閲 CHECKED		設計 DESIGNED		製図 DRAWN	
E. Horiechi		A. Saito		K. Horiiuchi		—	
13, 8, 26	13, 8, 26	13, 8, 26	13, 8, 26	13, 8, 26	13, 8, 26	13, 8, 26	13, 8, 26
年, 月, 日	年, 月, 日	年, 月, 日	年, 月, 日	年, 月, 日	年, 月, 日	年, 月, 日	年, 月, 日

承認 APPROVED		検閲 CHECKED		設計 DESIGNED		製図 DRAWN	
E. Horiechi		A. Saito		K. Horiiuchi		—	
13, 8, 26	13, 8, 26	13, 8, 26	13, 8, 26	13, 8, 26	13, 8, 26	13, 8, 26	13, 8, 26
年, 月, 日	年, 月, 日	年, 月, 日	年, 月, 日	年, 月, 日	年, 月, 日	年, 月, 日	年, 月, 日

承認 APPROVED		検閲 CHECKED		設計 DESIGNED		製図 DRAWN	
E. Horiechi		A. Saito		K. Horiiuchi		—	
13, 8, 26	13, 8, 26	13, 8, 26	13, 8, 26	13, 8, 26	13, 8, 26	13, 8, 26	13, 8, 26
年, 月, 日	年, 月, 日	年, 月, 日	年, 月, 日	年, 月, 日	年, 月, 日	年, 月, 日	年, 月, 日

承認 APPROVED		検閲 CHECKED		設計 DESIGNED		製図 DRAWN	
E. Horiechi		A. Saito		K. Horiiuchi		—	
13, 8, 26	13, 8, 26	13, 8, 26	13, 8, 26	13, 8, 26	13, 8, 26	13, 8, 26	13, 8, 26
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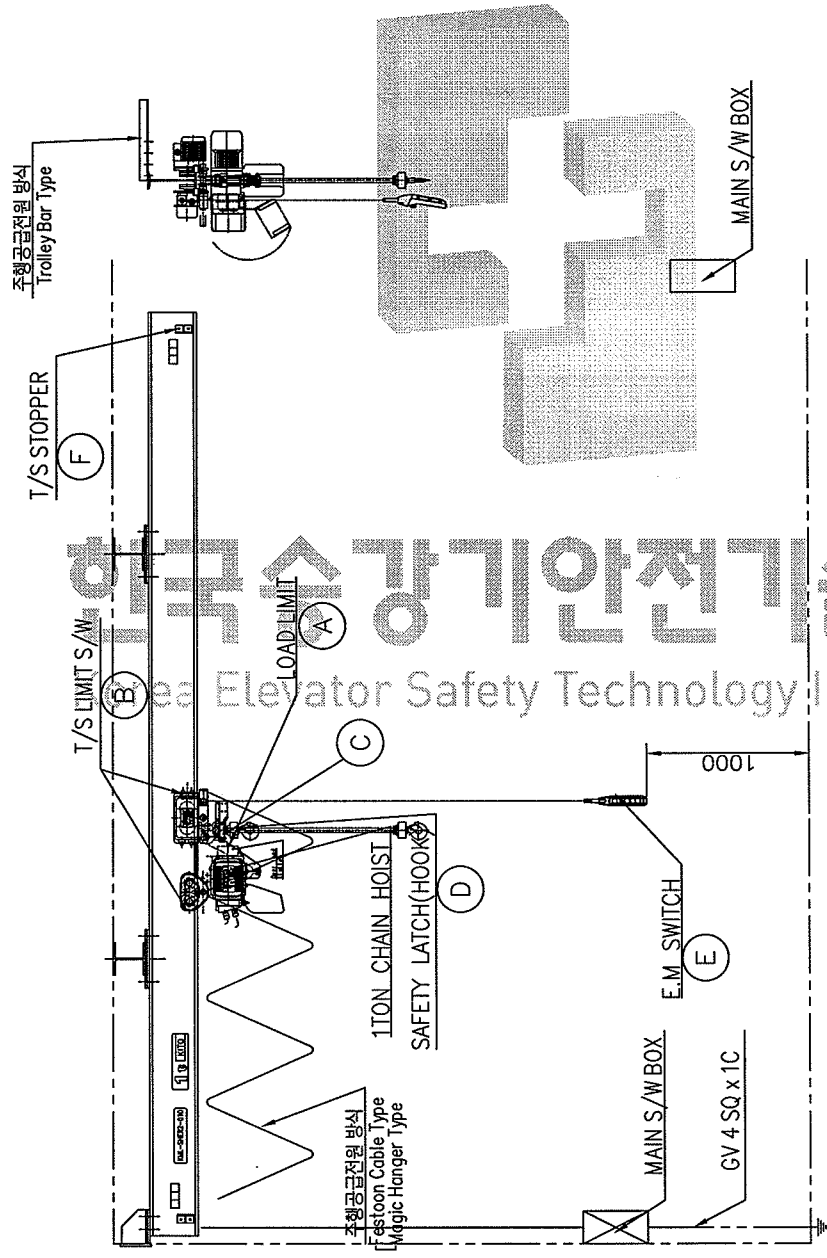
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E. Horiechi		A. Saito		K. Horiiuchi		—	
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A B C D E F



SPECIFICATION		
A	LOAD LIMIT	1 1TON USE
B	T/S LIMIT S/W	2
C	UPPER LIMIT S/W	1 1TON USE
D	SAFETY LATCH(HOOK)	1 1TON USE
E	E.M SWITCH	1 PB 305
F	T/S STOPPER	2 MACHINE



방향표지판

NOTE

1. 점검설비는 현장여건에 맞추어 설치한다.
2. 시용설명서를 제공한다.

한국승강기안전기술원
Korea Elevator Safety Technology Institute



형식번호: KMS-SHER2-010

承認 APPROVED	檢査 CHECKED	設計 DESIGNED	製圖 DRAWN	名稱 TITLE
J. S. CHO	J.H.CHOI	EDMON HEE	EDMON HEE	1T MONO RAIL HOIST SAFETY DRAWING
承認 APPROVED	檢査 CHECKED	設計 DESIGNED	製圖 DRAWN	尺碼 SCALE
				縮小圖紙 REV.
年 月 日 DATE	圖號 Dwg. NO.	MONO RAIL 0		

CONTENTS

REV. QTY

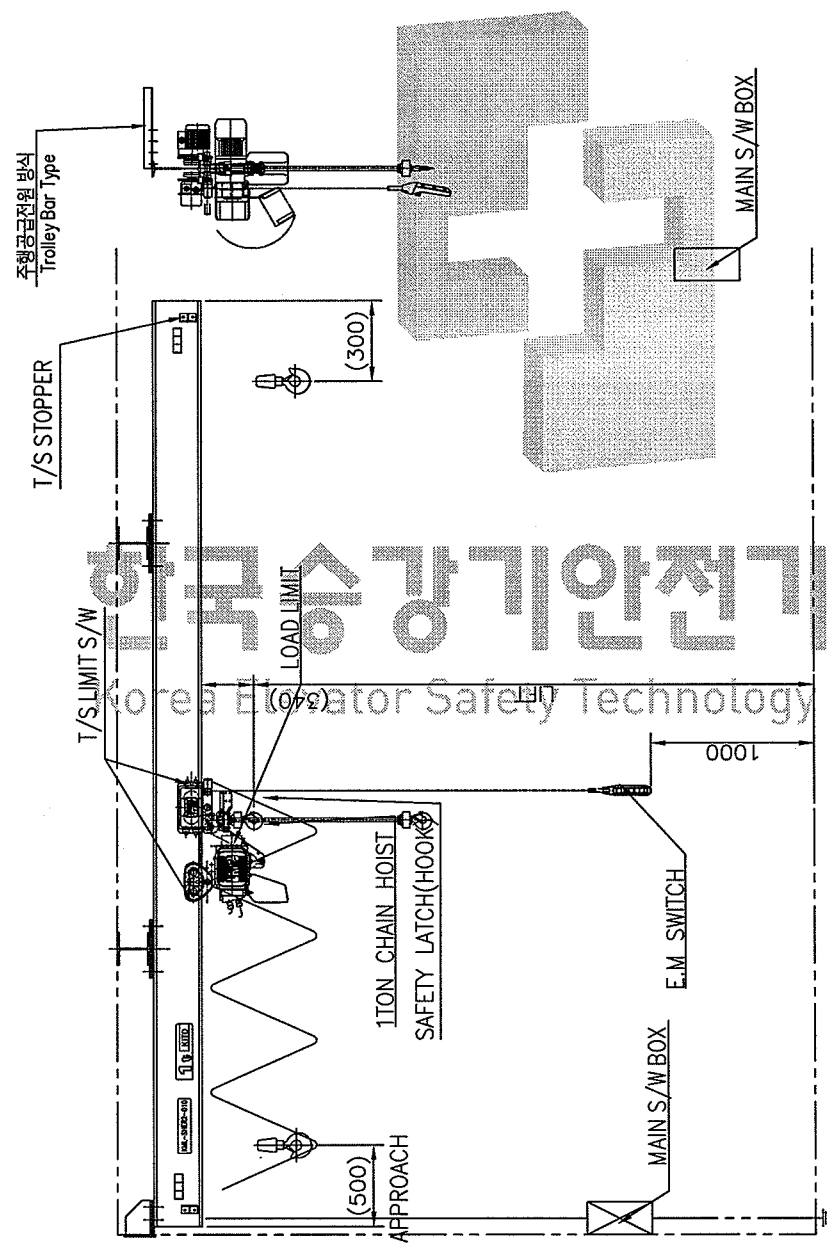
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SPECIFICATION

BEAM 규격	PITCH	O.H
I-200x100x7/10	4.7 m	1.9 m
I-250x125x7.5/12.5	7 m	2.7 m
LOAD CHAIN	7.7mm X 1	
LIFT	Max 30m	

NAME PLATE

정격 하중	1.0 ton
전기 설비 정격	V
제조 자	
제조 변월	20
안전인증 표시	KS
형식 번호	KML-SHER2-010
제조 번호	



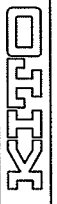
한국승강기안전기술원
 Korea Elevator Safety Technology Institute

2013 .09. 16

형식번호: KML-SHER2-010	Model number.
KITO-SHER2M010L-S	
KITO-SHER2M010L	
KITO-SHER2M010L	
KITO-SHER2M010L	
KITO-SHER2M010L-S	
KITO-SHER2M010L-IL	

1t MONO RAIL HOIST
 GENERAL ASSEMBLY-1

核准 APPROVED	設計 DESIGNED	製圖 DRAWN	名稱 TITLE
J. S. CHO	EDMUNION HEE	EDMUNION HEE	1t MONO RAIL HOIST GENERAL ASSEMBLY-1
承認 DRAWN APPROVED	年月日 DATE	圖號 DWG. NO.	尺碼 SCALE
		MONO RAIL 1	變更次數 REV.

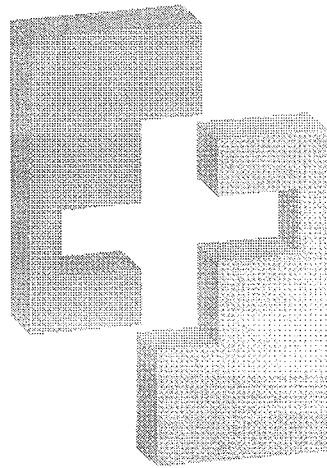


CONTENTS

REV. QTY

4. 전 기 도 면

- 1) ELECTRICAL SPECIFICATION
- 2) SYMBOL LIST
- 3) 배선배관도 & 접지계통도
- 4) 전기회로도
- 5) PANEL 관련도



한국승강기안전기술원

Korea Elevator Safety Technology Institute



LOAD SUMMARY 1 – INVERTER사양(IL)

*POWER SOURCE : AC 3Φ 220(208)V

OBJECT	HOISTING	TRAVERSING	CONTROL CIRCUIT
MOTOR OUTPUT	0.9KW x 4P	0.4KW x 4P	
FULL LOAD CURRENT	5.7 (A)	3 (A)	0.5 (A)

*크레인 하중상태를 HOIST의 정격 LOAD의 100(%)를 사용했을때를 기준으로 작성하였음.

*** NOMAL 전류값 ***

권상과 횡행시 : HOISTING + TRAVERSING + CONTROL CIRCUIT = 9.2 A

*** PEAK 전류값 ***

K= NOMAL 전류치가 50A미만일때 1.25, 50A이상일때 1.1적용

NOMAL 전류값 * K = 9.2 * 1.25 = 11.5 A

*POWER SOURCE : AC 3Φ 380(440)V

OBJECT	HOISTING	TRAVERSING	CONTROL CIRCUIT
MOTOR OUTPUT	0.9KW x 4P	0.4KW x 4P	
FULL LOAD CURRENT	3.6 (A)	2.5 (A)	0.5 (A)

Korea Elevator Safety Technology Institute

*크레인 하중상태를 HOIST의 정격 LOAD의 100(%)를 사용했을때를 기준으로 작성하였음.

*** NOMAL 전류값 ***

권상과 횡행시 : HOISTING + TRAVERSING + CONTROL CIRCUIT = 6.6 A

*** PEAK 전류값 ***

K= NOMAL 전류치가 50A미만일때 1.25, 50A이상일때 1.1적용

NOMAL 전류값 * K = 6.6 * 1.25 = 8.25 A



LOAD SUMMARY 2 – INVERTER사양(IL)

*POWER SOURCE : AC 3Φ 220(208)V

OBJECT	HOISTING	TRAVERSING	CONTROL CIRCUIT
MOTOR OUTPUT	0.9KW x 4P	-	
FULL LOAD CURRENT	5.7 (A)	0 (A)	0.5 (A)

*크레인 하중상태를 HOIST의 정격 LOAD의 100(%)를 사용했을때를 기준으로 작성하였음.

*** NOMAL 전류값 ***

권상시 : HOISTING + CONTROL CIRCUIT = 6.2 A

*** PEAK 전류값 ***

K= NOMAL 전류치가 50A미만일때 1.25, 50A이상일때 1.1적용

NOMAL 전류값 * K = 6.2 * 1.25 = 7.75 A

*POWER SOURCE : AC 3Φ 380(440)V

OBJECT	HOISTING	TRAVERSING	CONTROL CIRCUIT
MOTOR OUTPUT	0.9KW x 4P	-	
FULL LOAD CURRENT	3.6 (A)	0 (A)	0.5 (A)

*크레인 하중상태를 HOIST의 정격 LOAD의 100(%)를 사용했을때를 기준으로 작성하였음.

*** NOMAL 전류값 ***

권상시 : HOISTING + CONTROL CIRCUIT = 4.1 A

*** PEAK 전류값 ***

K= NOMAL 전류치가 50A미만일때 1.25, 50A이상일때 1.1적용

NOMAL 전류값 * K = 4.1 * 1.25 = 5.125 A



LOAD SUMMARY 3 – 1속저속형사양(L)

*POWER SOURCE : AC 3Φ 220(208)V

OBJECT	HOISTING	TRAVERSING	CONTROL CIRCUIT
MOTOR OUTPUT	0.9KW x 4P	0.4KW x 4P	
FULL LOAD CURRENT	4.7 (A)	3 (A)	0.5 (A)

*크레인 하중상태를 HOIST의 정격 LOAD의 100(%)를 사용했을때를 기준으로 작성하였음.

*** NOMAL 전류값 ***

권상과 횡행시 : HOISTING + TRAVERSING + CONTROL CIRCUIT = 8.2 A

*** PEAK 전류값 ***

K= NAMAL 전류치가 50A미만일때 1.25, 50A이상일때 1.1적용

NOMAL 전류값 * K = 8.2 * 1.25 = 10.25 A

*POWER SOURCE : AC 3Φ 380(440)V

OBJECT	HOISTING	TRAVERSING	CONTROL CIRCUIT
MOTOR OUTPUT	0.9KW x 4P	0.4KW x 4P	
FULL LOAD CURRENT	2.6 (A)	2.2 (A)	0.5 (A)

*크레인 하중상태를 HOIST의 정격 LOAD의 100(%)를 사용했을때를 기준으로 작성하였음.

*** NOMAL 전류값 ***

권상과 횡행시 : HOISTING + TRAVERSING + CONTROL CIRCUIT = 5.3 A

*** PEAK 전류값 ***

K= NAMAL 전류치가 50A미만일때 1.25, 50A이상일때 1.1적용

NOMAL 전류값 * K = 5.3 * 1.25 = 6.625 A



LOAD SUMMARY 4 - 1속 저속형사양(L)

*POWER SOURCE : AC 3Φ 220(208)V

OBJECT	HOISTING	TRAVERSING	CONTROL CIRCUIT
MOTOR OUTPUT	0.9KW x 4P	-	
FULL LOAD CURRENT	4.7 (A)	0 (A)	0.5 (A)

*크레인 하중상태를 HOIST의 정격 LOAD의 100(%)를 사용했을때를 기준으로 작성하였음.

*** NOMAL 전류값 ***

권상시 : HOISTING + CONTROL CIRCUIT = 5.2 A

*** PEAK 전류값 ***

K= NAMAL 전류치가 50A미만일때 1.25, 50A이상일때 1.1적용

NOMAL 전류값 * K = 5.2 * 1.25 = 6.5 A

*POWER SOURCE : AC 3Φ 380(440)V

OBJECT	HOISTING	TRAVERSING	CONTROL CIRCUIT
MOTOR OUTPUT	0.9KW x 4P	-	
FULL LOAD CURRENT	2.6 (A)	0 (A)	0.5 (A)

*크레인 하중상태를 HOIST의 정격 LOAD의 100(%)를 사용했을때를 기준으로 작성하였음.

*** NOMAL 전류값 ***

권상시 : HOISTING + CONTROL CIRCUIT = 3.1 A

*** PEAK 전류값 ***

K= NAMAL 전류치가 50A미만일때 1.25, 50A이상일때 1.1적용

NOMAL 전류값 * K = 3.1 * 1.25 = 3.875 A



ROTATING MACHINE

- SYNCHRONOUS GENERATOR, 3-PHASE
- AC INDUCTION MOTOR, 3-PHASE
 - * N : NORMAL DUTY
 - S : STAND-BY
- DC MOTOR

LIGHTNING ARRESTERS

- LA : LIGHTNING ARRESTER
- SA : SURGE ARRESTER
- SS : SURGE SUPPRESSOR
- DISCHARGE COUNTER

INSTRUMENT TRANSFORMERS

- CURRENT TRANSFORMER
- ZERO PHASE CURRENT TRANSFORMER
- POTENTIAL TRANSFORMER

CIRCUIT BREAKERS

- POWER CIRCUIT BREAKER, FIXED TYPE
 - GB : SF6 GAS CIRCUIT BREAKER
 - VCB : VACUUM CIRCUIT BREAKER
 - ACB : AIR CIRCUIT BREAKER
- POWER CIRCUIT BREAKER, DRAWOUT TYPE

SWITCHES

- DISCONNECTOR SWITCH, SINGLE THROW MANUALLY OPERATED
- LOAD BREAK SWITCH, SINGLE THROW MANUALLY OPERATED
- EARTHING SWITCH, SINGLE THROW MANUALLY OPERATED
- DISCONNECTOR SWITCH, SINGLE THROW MOTOR OPERATED
- EARTHING SWITCH, SINGLE THROW MOTOR OPERATED
- VACUUM CIRCUIT SWITCH
- FUSED DISCONNECTOR SWITCH
- FUSE-SWITCH
- LIMIT SWITCH (MAKE CONTACT)
- LIMIT SWITCH (BREAK CONTACT)
- PUSH BUTTON, NORMALLY OPEN MOMENTARY CONTACT
- PUSH BUTTON, NORMALLY CLOSED MOMENTARY CONTACT
- PUSH BUTTON, NORMALLY OPEN PUSH TO LOCK, RELEASED BY KEY
- MANUAL SELECTOR SWITCH (LOCKED)

CONTACTORS AND STARTERS

- AUX. CONTACT, NORMALLY OPEN WHEN MAIN SWITCHING DEVICE IS DE-ENERGIZED
- AUX. CONTACT, NORMALLY CLOSED WHEN MAIN SWITCHING DEVICE IS DE-ENERGIZED
- MAGNETIC CONTACTOR, ELECTRICALLY OPERATED
- COMBINATION STARTER, FULL VOLTAGE, NON-REVERSING, DRAWOUT TYPE, WITH ELECTRICALLY OPERATED CONTACTORS, WITH MAGNETIC MOTOR CIRCUIT BREAKER, BUILT IN ELECTRONIC OVER-CURRENT RELAY WITH ADJUSTABLE TRIP RATING
- COMBINATION STARTER, FULL VOLTAGE, NON-REVERSING, FIXED TYPE, WITH ELECTRICALLY OPERATED CONTACTORS, WITH MAGNETIC MOTOR CIRCUIT BREAKER, BUILT IN THERMAL OVER-CURRENT RELAY WITH ADJUSTABLE TRIP RATING

GRAPHIC SYMBOLS

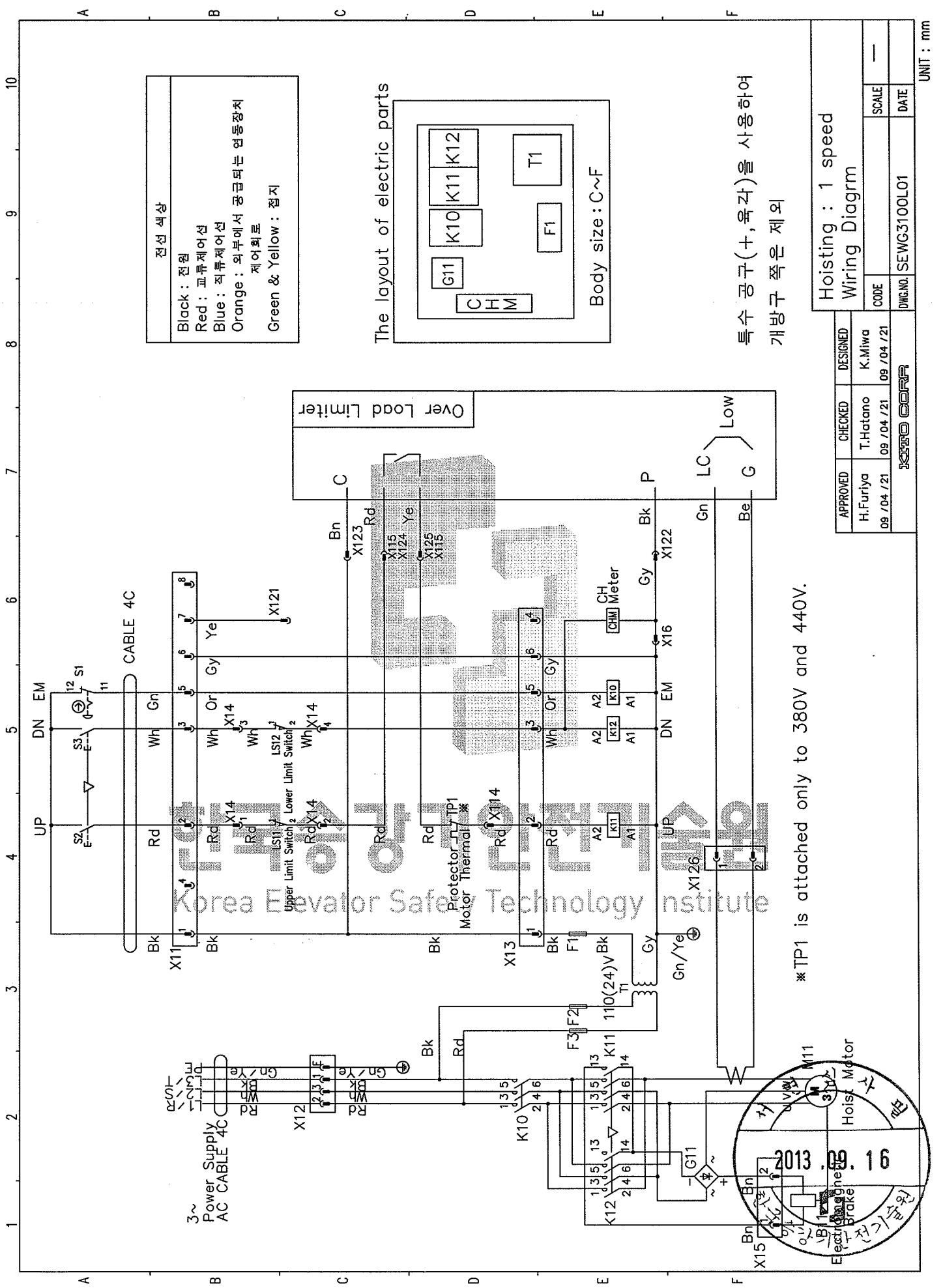
- GENERAL OPERATING COIL
- CAPACITOR
- CAPACITOR VOLTAGE TRANSFORMER(CVT)
- RESISTOR
- DIODE

- CONTROLLED RECTIFIER
- DC-DC CONVERTER
- RECTIFIER, BATTERY CHARGER
- DC-AC INVERTER
- BATTERY BANK
- ELECTRIC HEATER, INDICATE 1st OR 3rd AND kW RATING, UNLESS OTHERWISE SPECIFIED, TO BE REGARDED AS 1st.
- EARTHING CONNECTION
- DISCONNECTION LINK
- CROSSING OF CONDUCTORS NOT CONNECTED
- JUNCTION OF CONDUCTORS OR WIRES
- BUS DUCT
 - SPB : SEGREGATED PHASE BUS DUCT
 - IPB : ISOLATED PHASE BUS DUCT
- CABLE HEAD AND CABLE CONNECTION
- AMMETER SWITCH
- VOLTMETER SWITCH
- SIGNAL LAMP
 - * R = RED
 - G = GREEN
 - W = WHITE
 - A = AMBER
 - C = CYAN

SYMBOL LIST

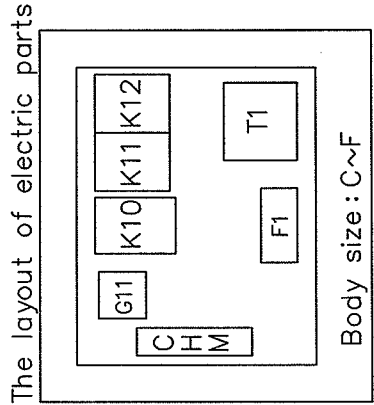
APPROVED	CHECKED	DESIGNED
KOTO CORP		
CODE	SYMBOL LIST	SCALE
DATE		





전선 색상

Black : 전원
 Red : 교류케이블선
 Blue : 직류케이블선
 Orange : 외부에서 공급되는 연동장치 제어회로
 Green & Yellow : 접지

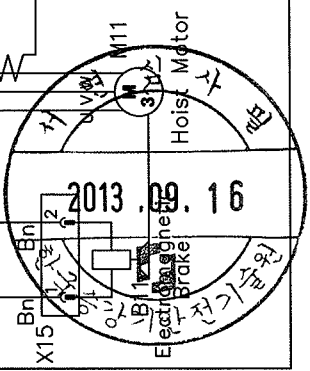


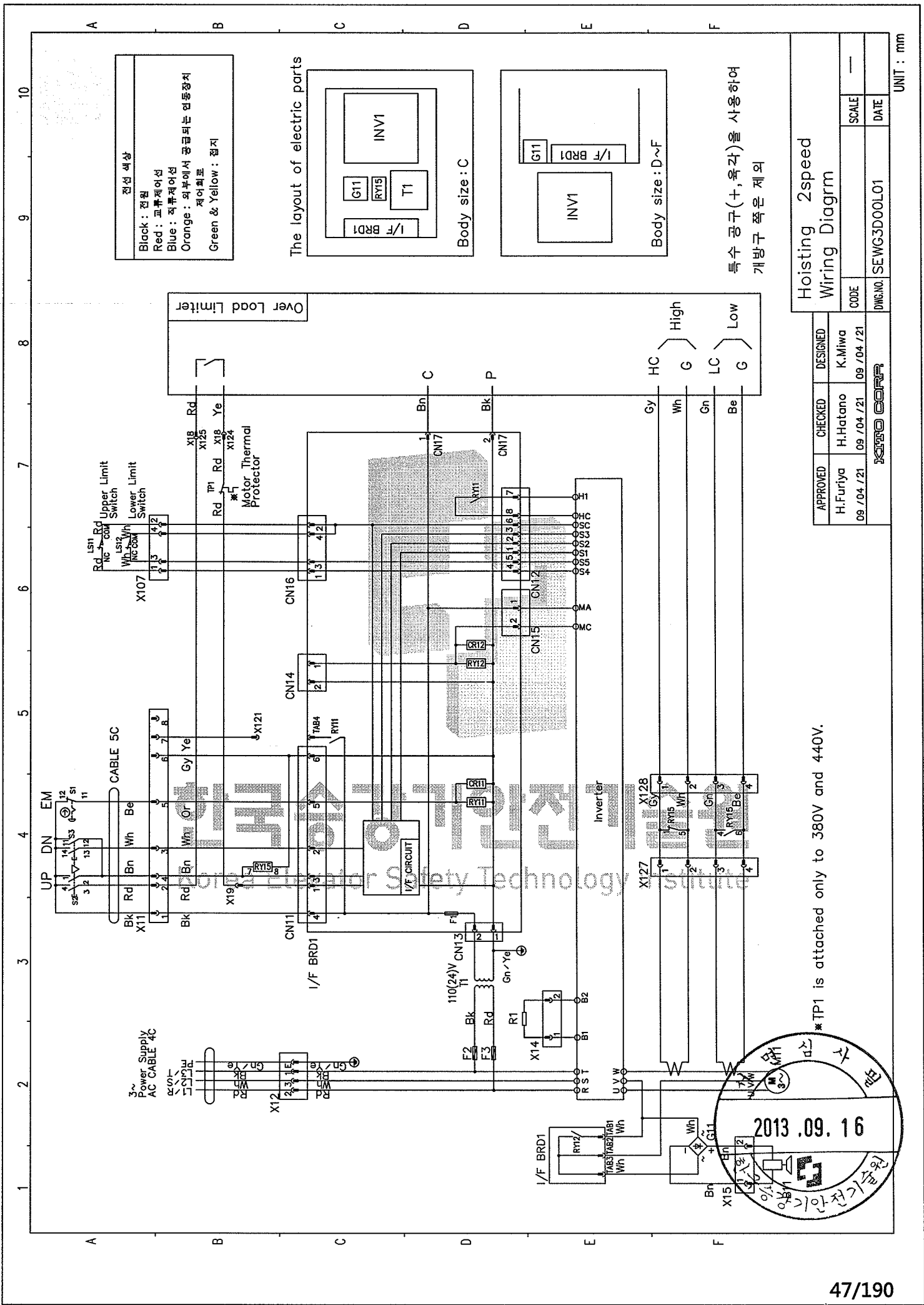
특수 공구(+, 육각)을 사용하여
 개방구 쪽은 제외

※ TP1 is attached only to 380V and 440V.

APPROVED	CHECKED	DESIGNED
H.Furiya	T.Hatano	K.Miwa
09./04./21	09./04./21	09./04./21
KOTO CORP		
Hoisting : 1 speed		
Wiring Diagram		
CODE	SCALE	DATE
DWG.NO	SEWG3100LO1	---

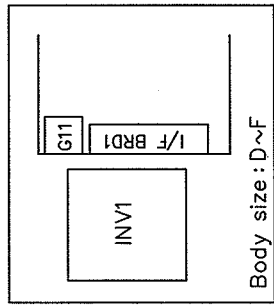
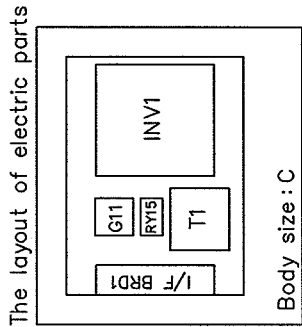
UNIT : mm





전선 색상

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 Blue : 직류제어선
 Orange : 외부에서 공급되는 연동장치 제어회로
 Green & Yellow : 접지



특수 광구(+,육각)을 사용하여
 개방구 쪽은 제외

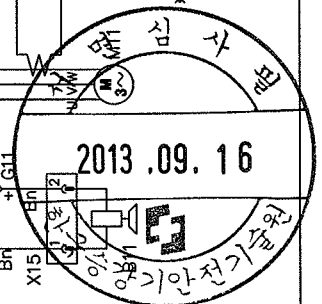
Hoisting 2speed
 Wiring Diagram

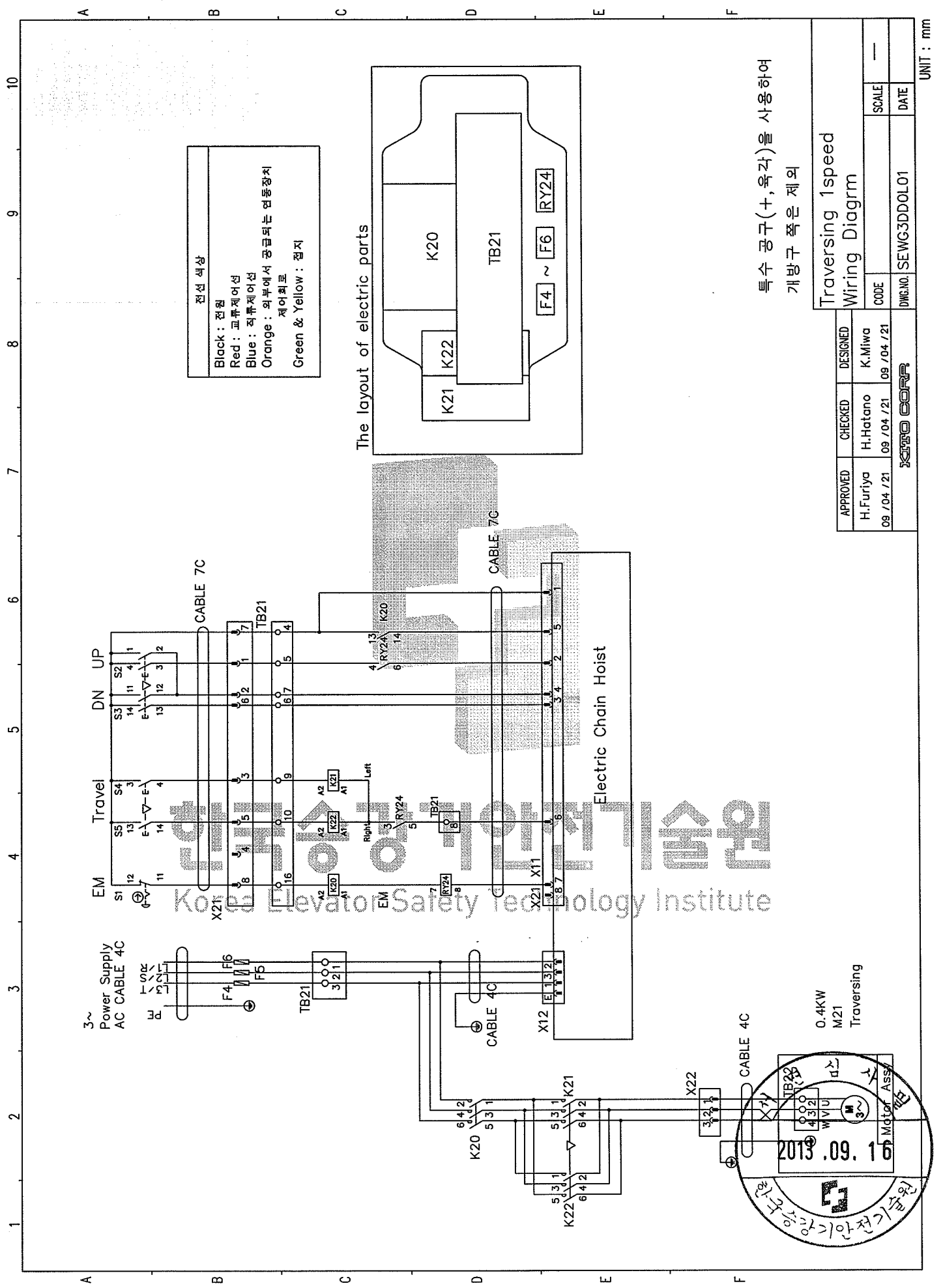
APPROVED	CHECKED	DESIGNED
H.Furiya	H.Hatano	K.Miwa
09 /04 /21	09 /04 /21	09 /04 /21

CODE	SCALE	DATE
—	—	—

UNIT : mm

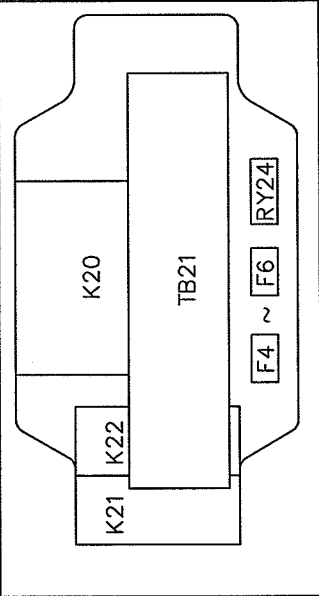
*TP1 is attached only to 380V and 440V.





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 Blue : 직류제어선
 Orange : 외부에서 공급되는 연동장치 제어회로
 Green & Yellow : 접지

The layout of electric parts



특수 공구(+, 육각)을 사용하여
 개방구 쪽은 제외

Traversing 1speed
 Wiring Diagram

APPROVED	CHECKED	DESIGNED
H.Furiya 09 / 04 / 21	H.Hatano 09 / 04 / 21	K.Miwa 09 / 04 / 21

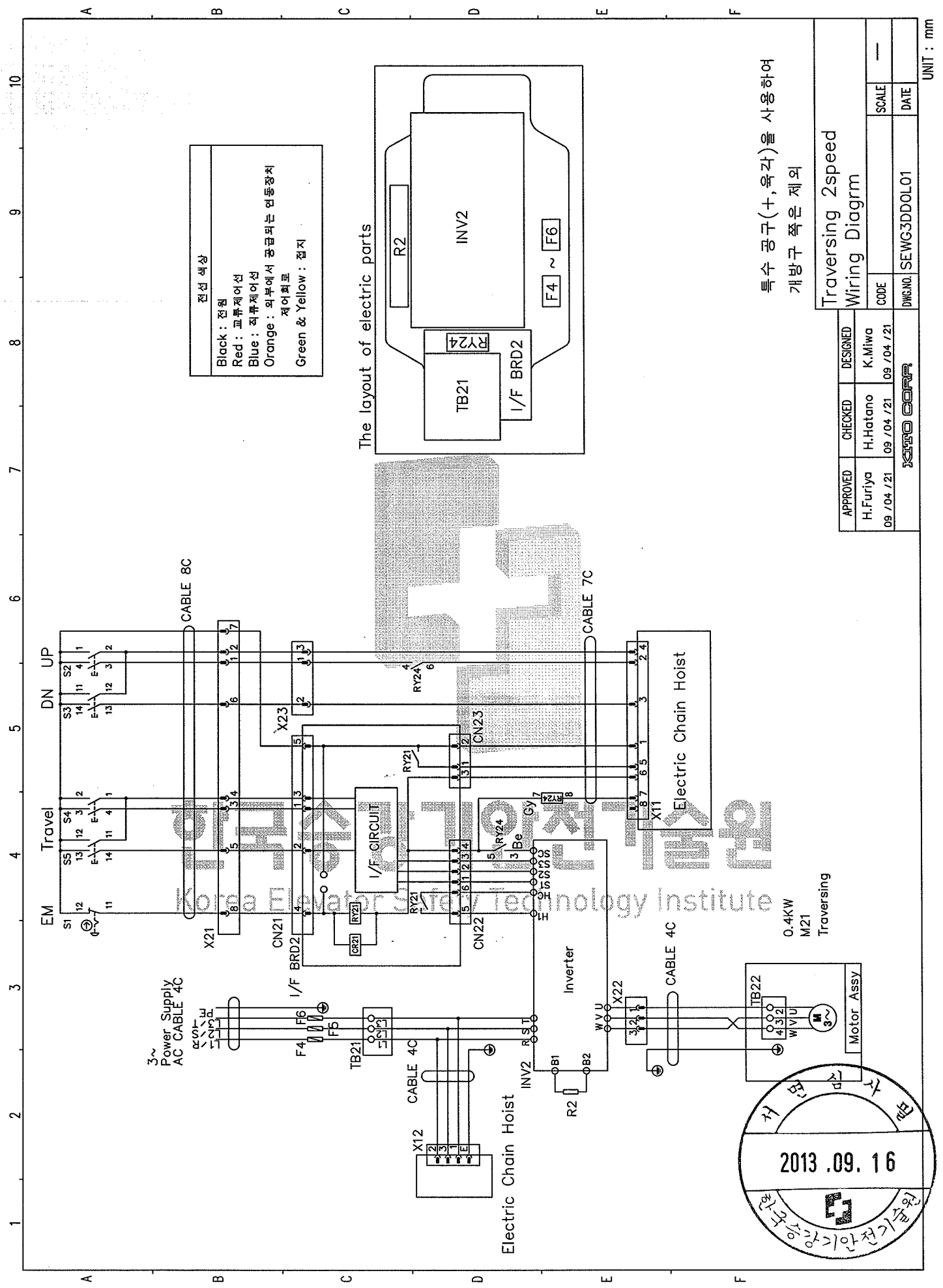
CODE
 DWG.NO. SEWG3DD0L01

UNIT : mm

0.4KW
 M21
 Traversing

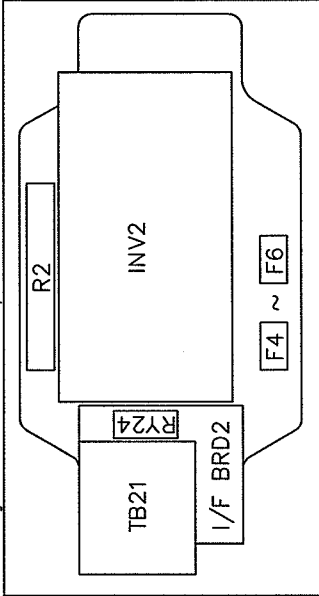
2013.09.16

Motor Ass'y



전선 색상
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 Blue : 직류제어선
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 Green & Yellow : 접지

The layout of electric parts



특수 공구(+, 육각)을 사용하여
 개방구 쪽은 제외

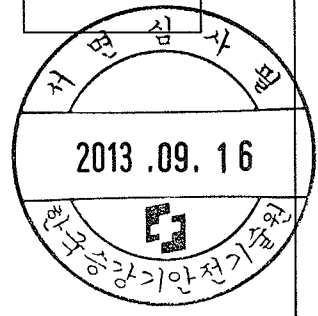
Traversing 2speed
 Wiring Diagram

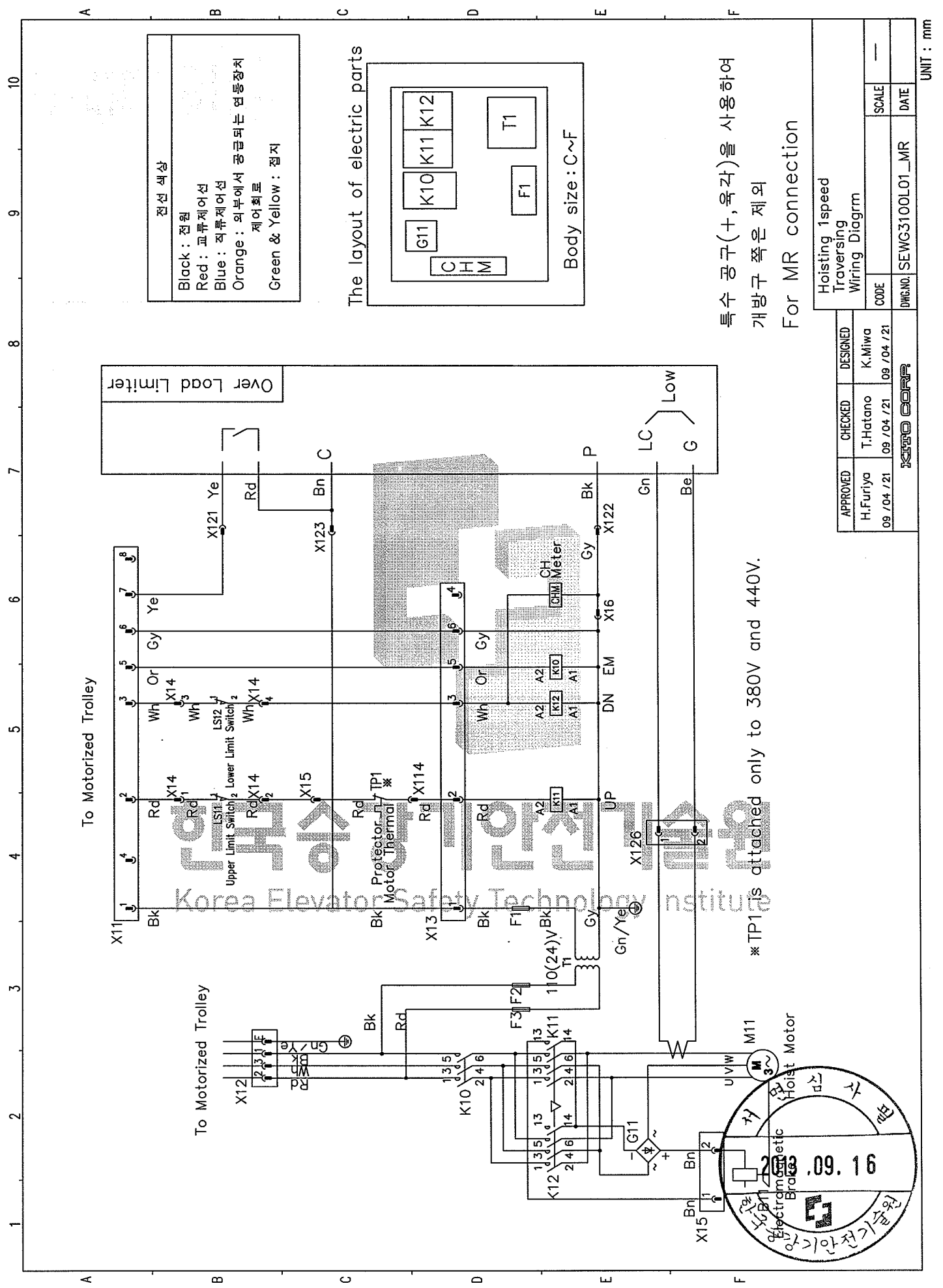
APPROVED	CHECKED	DESIGNED
H.Furiya 09 /04 /21	H.Hatano 09 /04 /21	K.Miwa 09 /04 /21

CODE	SCALE	DATE

DWG.NO. SEWC3DD0L01

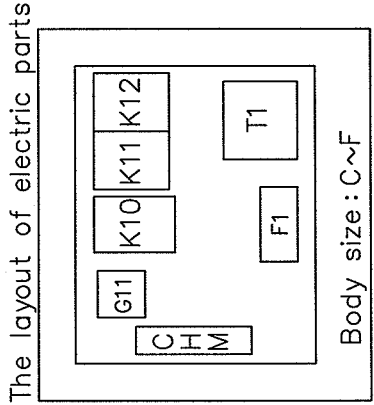
UNIT : mm





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 Red : 교류제어선
 Blue : 직류제어선
 Orange : 외부에서 공급되는 연동장치 제어회로
 Green & Yellow : 접지

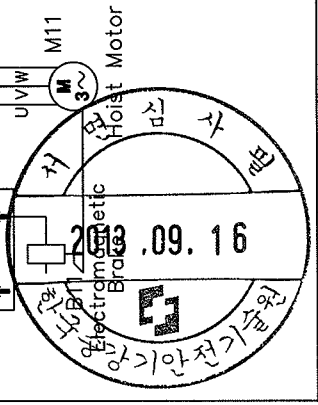


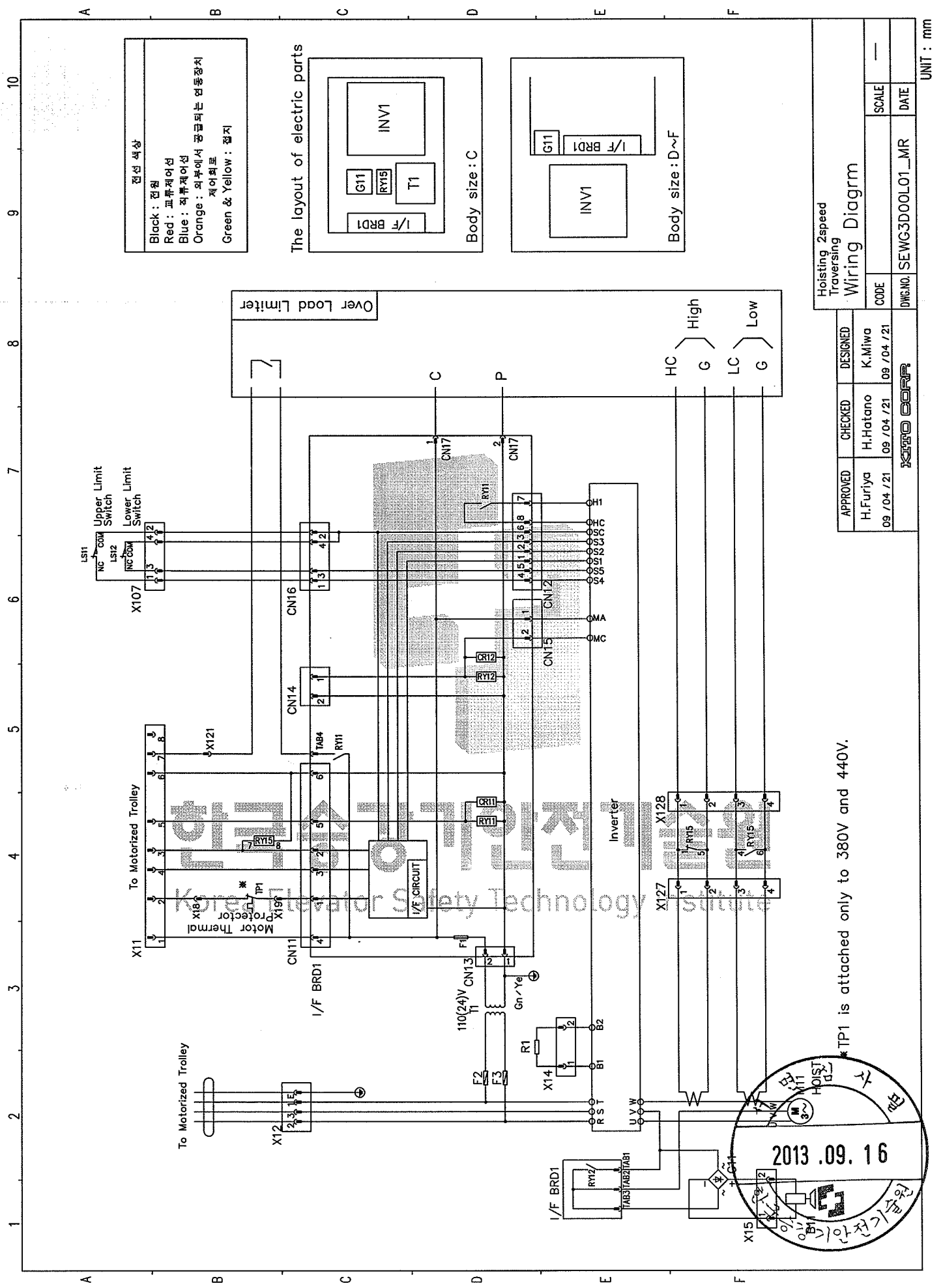
특수 공구(+, 육각)을 사용하여
 개방구 쪽은 제외
 For MR connection

APPROVED		CHECKED	DESIGNED
H.Furiya		T.Hatano	K.Miwa
09 / 04 / 21		09 / 04 / 21	09 / 04 / 21
DWG.NO. SEWG3100L01_MR		SCALE	DATE
		—	—

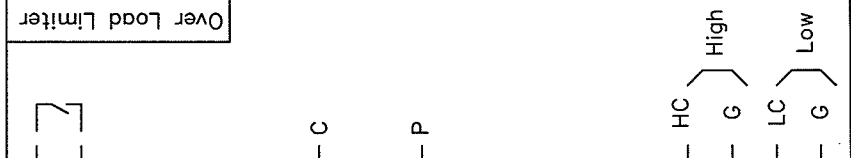
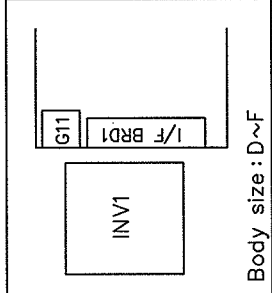
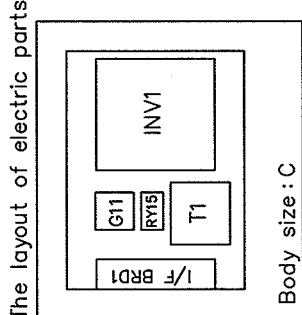
UNIT : mm

※TP1 is attached only to 380V and 440V.





진선 색상
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 Green & Yellow : 접지



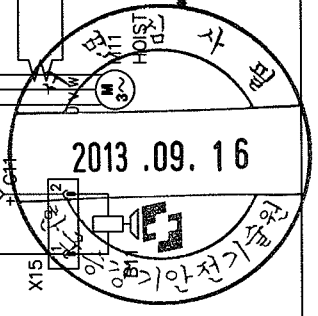
Hoisting 2speed Traversing Wiring Diagram

APPROVED	CHECKED	DESIGNED
H.Furiya	H.Hatano	K.Miwa
09 / 04 / 21	09 / 04 / 21	09 / 04 / 21

CODE
 DWG.NO. SEWC3D00L01_MR

SCALE
 DATE

UNIT : mm



*TP1 is attached only to 380V and 440V.

CABLE 구성도 및 사양 - 권상 용량 0.9kW

CABLE SPECIFICATION FOR ER2M

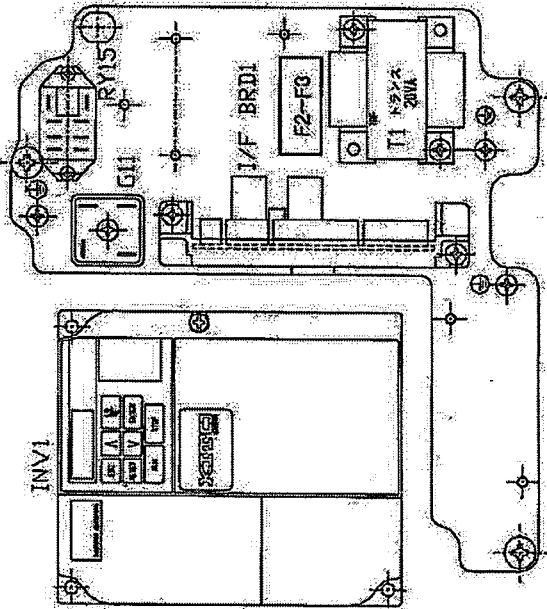
NO	ITEM	TYPE	ER2M10 SIZE
①	Power Line	VCT	3.5sq x 4C
②	Push Button Switch	VCT	1.25sq x 8C
③	Loas Limit	VCT	0.75sq x 8C
④	Power Line for ER	VCT	1.5sq x 4C
⑤	Control Line for ER	VCT	1.25sq x 6C
⑥	Traversing Motor With Earth	VCT	1.25sq x 4C

(3Φ 220(208)V / 380V / 440V 60Hz)

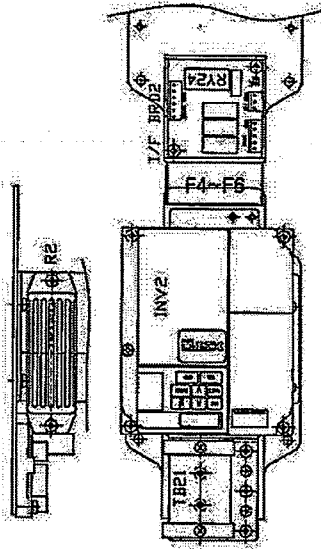


호이스트 CONTROL BOX 배치도

HOISTING CONTROL BOX

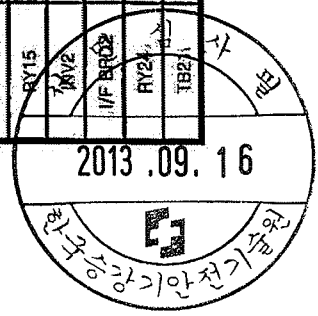


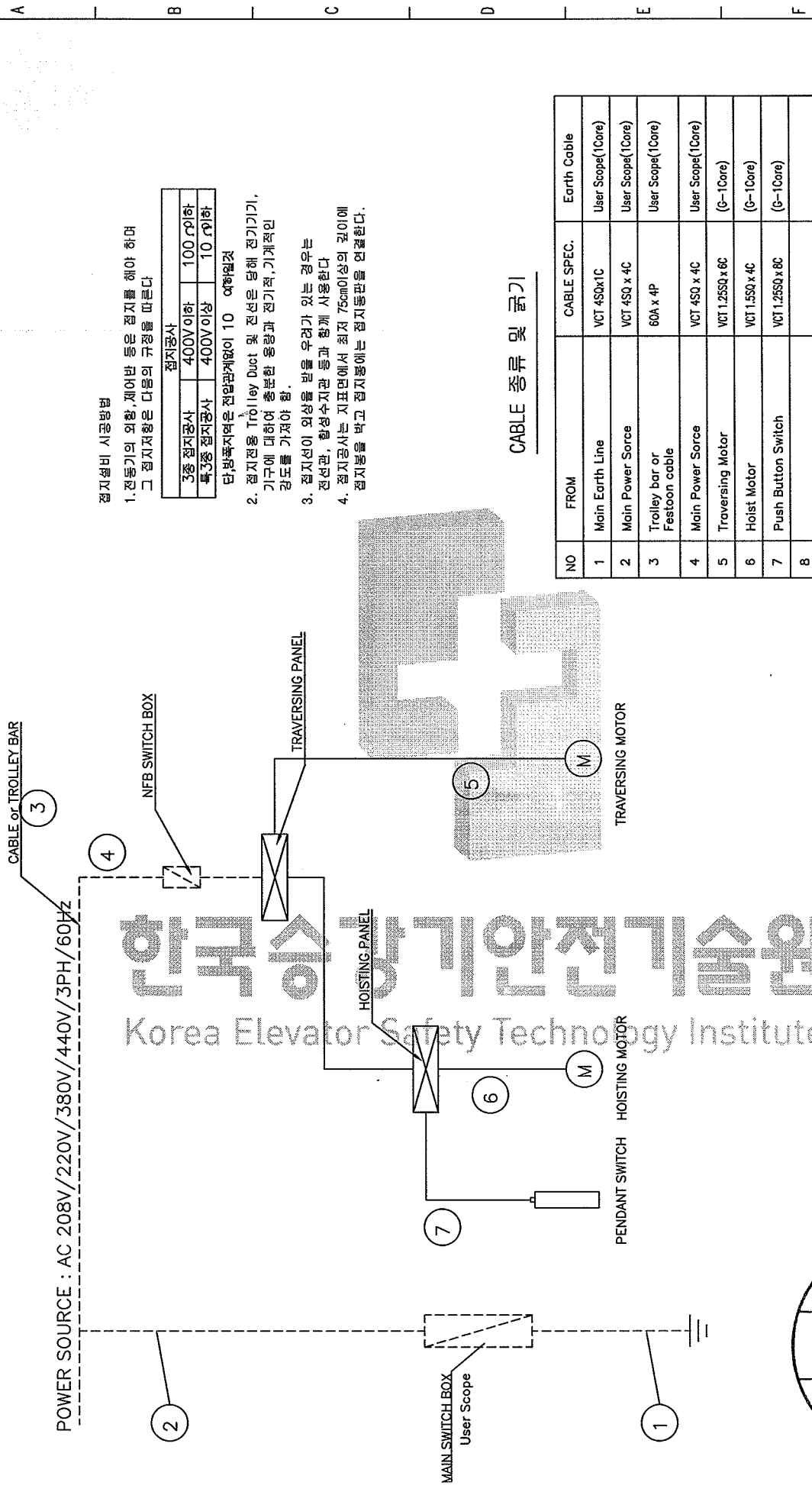
TRaversing CONTROL BOX



ENCLOSURE : HOIST BODY ~ IP55
PUSH BUTTON ~ IP65

MARK	DESCRIPTION	TYPE OF MODEL			QTY	MAKER	REMARKS
		220V	380V	440V			
INV1	INVERTER	V1000	V1000	V1000	1	YASKAWA	UP/DOWN
T1	TRANSFORMER	220V/ 24V(110V) 20VA	380V/ 24V(110V) 20VA	440V/24V(110V) 20VA	1	KITO	CONTROL CIRCUIT
G11	BRIDGE DIODE	S15VB80	S15VB60	S15VB80	1	SHINDENGEN	
I/F BRD1	INTERFACE BOARD	10~15A	10~15A	10~15A	1	KITO	
F2-F3	GLASS FUSE	10A	10A	10A	2	FUJI	
F4-F6	GLASS FUSE	30A	30A	30A	3	FUJI	
RY15	RELAY	110V	110V	110V	1	OMRON	HIGH/LOW
INV2	INVERTER	V1000	V1000	V1000	1	YASKAWA	RIGHT/LEFT
I/F BRD2	INTERFACE BOARD	10~15A	10~15A	10~15A	1	KITO	
RY21	RELAY	110V	110V	110V	1	OMRON	EMERGENCY STOP
TB21	TERMINAL BOARD 21	10~15A	10~15A	10~15A	1	KITO	





접지선비 시공방법
 1. 전동기의 외함, 제어반 등은 접지를 해야 하며 그 접지 저항은 다음의 규정을 따른다

접지공사	
3중 접지공사	400V이하 100 Ω이하
특3중 접지공사	400V이상 10 Ω이하

단, 방폭지역은 전압관계없이 10 Ω이하일것

2. 접지전용 Trolley Duct 및 전선은 당해 전기기기, 기구에 대하여 충분한 용량과 전기적, 기계적인 강도를 가져야 함.
3. 접지선이 외상을 받을 우려가 있는 경우는 전선관, 합성수지관 등과 함께 사용한다
4. 접지공사는 지표면에서 최저 75cm이상의 높이에 접지봉을 박고 접지봉에는 접지등판을 연결한다.

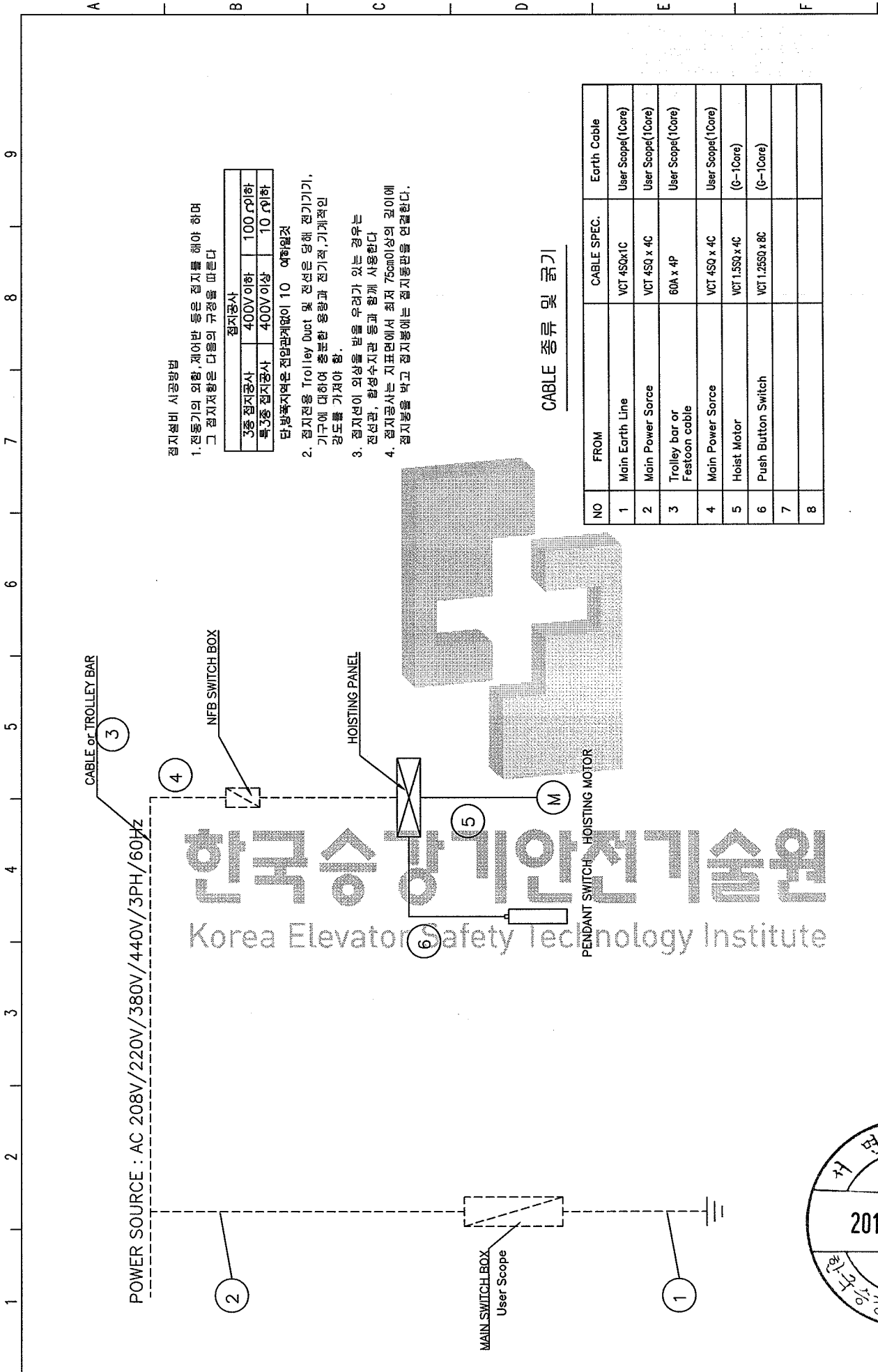
CABLE 종류 및 길이

NO	FROM	CABLE SPEC.	Earth Cable
1	Main Earth Line	VCT 450x1C	User Scope(1Core)
2	Main Power Source	VCT 450 x 4C	User Scope(1Core)
3	Trolley bar or Festoon cable	60A x 4P	User Scope(1Core)
4	Main Power Source	VCT 450 x 4C	User Scope(1Core)
5	Traversing Motor	VCT 1.25SQ x 6C	(0-1Core)
6	Hoist Motor	VCT 1.55Q x 4C	(0-1Core)
7	Push Button Switch	VCT 1.25SQ x 8C	(0-1Core)
8			



REV.	QTY	CONTENTS	DATE	DRAWN	APPROVED	CHECKED	DESIGNED	DRAWN	TITLE
									1t MOTORIZED-4점식 케이블 구성도 및 접지계통도
									MDL. 942513
									DWG. NO. 3NNU942513
									SCALE NOT
									REV. 0

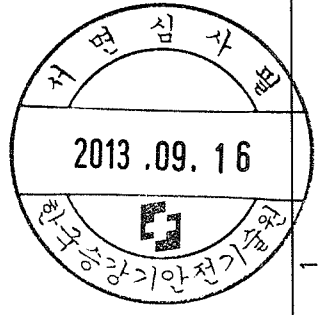
UNIT : mm



CABLE 종류 및 굵기

NO	FROM	CABLE SPEC.	Earth Cable
1	Main Earth Line	VCT 450x1C	User Scope(1Core)
2	Main Power Source	VCT 450 x 4C	User Scope(1Core)
3	Trolley bar or Festoon cable	60A x 4P	User Scope(1Core)
4	Main Power Source	VCT 450 x 4C	User Scope(1Core)
5	Hoist Motor	VCT 1.550 x 4C	(0-1Core)
6	Push Button Switch	VCT 1.2500 x 8C	(0-1Core)
7			
8			

REV.	QTY	CONTENTS	DATE	DRAWN	APPROVED	CHECKED	DESIGNED	DRAWN	TITLE
									1t MOTORIZED-2점식 케이블 구성도 및 접지계통도
									MDL. 942513
									DWG. NO. 3NNU942514
									SCALE NOT
									REV. 0



허용 최대 SPAN 적용표 (I-BEAM, H-BEAM)

PROJECT NAME : KML-SHER2-010
 RATED LOAD : 1 ton
 DESCRIPTION : LIFT(max) 30 m

NO.	I-BEAM-SIZE (B*H*t1/t2)	Ix cm ⁴	Iy cm ⁴	Zx cm ³	Zy cm ³	A cm ²	Wb kg/m	Wh ton	Wg ton	Mg1 ton	L cm	L1 cm	L2 cm	b(최약) cm	E kg/cm ²	ψ	Σσ1	Σσ2	Σσ3	Σσ4	결과			
																					δ1	L/800	δ2	L1/500
I - BEAM																								
1	200x100x7/10t	2170	138	217	27.7	33.06	26	0.14	0.122	0.049	470	190	30	45.8	2100000	1.11	1.1	0.823	1.205	0.843	1.237	0.577	0.588	0.3508
2	250x125x7.5/12.5t	5180	337	414	53.9	48.79	38.3	0.14	0.268	0.103	700	270	30	45.8	2100000	1.11	1.1	0.723	0.982	0.876	1.102	0.859	0.875	0.5062
H - BEAM																								
3	200x200x8/12t	4720	1600	472	160	63.53	29.9	0.14	0.324	0.125	650	250	30	45.8	2100000	1.11	1.1	0.544	0.730	0.290	0.358	0.775	0.813	0.4328
4	300x150x6.5/9t	7210	508	481	67.7	40.8	32	0.14	0.256	0.102	800	320	30	45.8	2100000	1.11	1.1	0.717	1.028	0.905	1.205	0.916	1.000	0.6397
5	350x175x7/11t	13500	984	771	112	62.91	49.4	0.14	0.519	0.198	1050	400	30	45.8	2100000	1.11	1.1	0.876	0.863	0.991	1.104	1.246	1.313	0.7348
6	400x200x8/13t	23500	1740	1170	174	83.37	65.4	0.14	0.85	0.327	1300	500	30	45.8	2100000	1.11	1.1	0.629	0.775	0.998	1.099	1.550	1.625	0.9029
7	450x200x9/14t	32900	1870	1460	187	95.43	74.9	0.14	1.086	0.449	1450	600	30	45.8	2100000	1.11	1.1	0.622	0.834	1.087	1.503	1.672	1.813	1.1941

X 축의 단면 2차모멘트
 Ix X 축의 단면 2차모멘트
 Iy Y 축의 단면계수
 Zx Y 축의 단면계수
 Zy BEAM의 단면적
 A BEAM의 단위중량
 Wb HOIST 자중
 Wh GIRDER 자중
 Wg 켈레베 GIRDER 자중
 Mg1 SPAN-PITCH내 LENGTH
 L SPAN-PITCH내 LENGTH
 L1 HOOK APPROACH
 L2 WHEEL BASE OF HOIST
 b(최약) 종단성계수
 E φ = M
 ψ φ = F

1. PITCH내 작업시 풍하중을 고려 하용응력에 115% 적용
 2. 작업시 켈레베 작업시 풍하중을 고려 하용응력에 115% 적용
 3. 휴지시 PITCH내 켈레베 하용응력에 130% 적용
 4. 휴지시 켈레베 하용응력에 130% 적용
 5. 정하중계수

Σσ1 =(PITCH내 계산응력) < 1.279 TON/CM² 이하일 경우 "O.K" (SS400, 응진효율 80% 적용, 풍하중 115% 적용)
 Σσ2 =(켈레베 계산응력) < 1.600 TON/CM² 이하일 경우 "O.K" (SS400, 풍하중 115% 적용)
 Σσ3 =(PITCH내 계산응력) < 1.447 TON/CM² 이하일 경우 "O.K" (SS400, 풍하중 130% 적용)
 Σσ4 =(켈레베 계산응력) < 1.808 TON/CM² 이하일 경우 "O.K" (SS400, 풍하중 130% 적용)
 δ1 = (PITCH내 처짐량)
 δ2 = (켈레베 처짐량)

1279 KG/CM²
 1600 KG/CM²
 1447 KG/CM²
 1808 KG/CM²

< L / 800 이하일 경우 "O.K"
 < L1 / 500 이하일 경우 "O.K"

IN DOOR or OUT DOOR

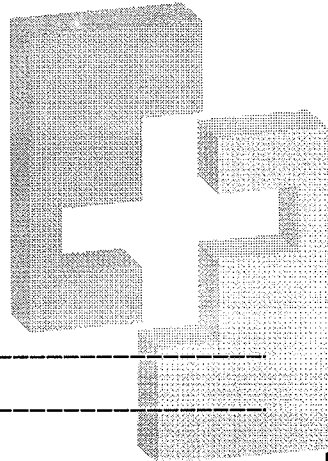
AREA CLASSIFICATION:



1. I-BEAM 최대허용 가능 SPAN 계산

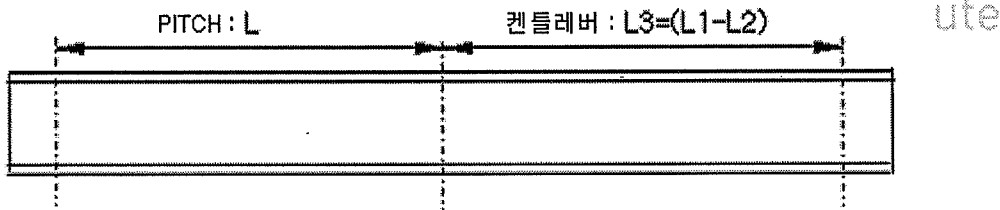
1. SPECIFICATION

.정격하중	-----	Q =	1	ton
.SPAN (PITCH)	-----	L=	470	cm
.켄틸리버	-----	L1=	190	cm
.TROLLEY WHEEL BASE	-----	B=	45.8	cm
.WEIGHT OF HOIST	-----	Wh=	0.14	ton
.PITCH내의 GIRDER 무게	-----	Wg=	0.122	ton
.켄틸리버의 GIRDER 무게	-----	Wg1=	0.049	ton
.I-BEAM SIZE	-----		200x100x7/10t	
		Ix =	2170	cm ⁴
		Iy =	138	cm ⁴
		Zx =	217	cm ³
		Zy =	27.7	cm ³
		A =	33.06	cm ²
		Wb =	26	kg/m
.HOOK APPROACH	-----	L2=	30	cm
탄성계수	-----	E=	2100000	kg/cm ²
동하중 계수(작업계수)	-----	M(Φ)=	1.11	
정하중 계수(충격계수)	-----	F(Ψ)=	1.10	



한국승강기안전기술원

1. DESIGN



2. I-BEAM에 작용하는 하중

$$P = Q + Wh = 1 + 0.14 = 1.14 \text{ ton}$$

3. 수직하중에 의한 BENDING MOMENT

1) PITCH 지점내 BENDING MOMENT

$$Mh1 = F \times M \times P \times (L-B/2)^2 / (4 \times L)$$

$$= 1.11 \times 1.1 \times 1.14 \times (470 - 45.8/2)^2 / (4 \times 470)$$



$$= 148.0 \quad \text{ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$Mh2 = F \times M \times P \times L3$$

$$= 1.11 \times 1.1 \times 1.14 \times (190-30) = 222.71 \quad \text{ton.cm}$$

4. 자중에 의한 BENDING MOMENT(I-BEAM)

1) PITCH 지점내 BENDING MOMENT

$$Mt1 = F \times Wg \times L / 8 = 1.11 \times 0.122 \times 470 / 8 = 7.956 \quad \text{ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$Mt2 = F \times Wg1 \times L1 / 2 = 1.11 \times 0.049 \times 190 / 2 = 5.17 \quad \text{ton.cm}$$

5. 수직하중에 의한 최대 BENDING MOMENT

$$Mmax1 = Mh1 + Mt1 = 156 \quad \text{ton.cm}$$

$$Mmax2 = Mh2 + Mt2 = 227.9 \quad \text{ton.cm}$$

6. 풍하중에 의한 수평하중의 BENDING MOMENT

(1) GIRDER에 의한 수직 풍하중

$$\text{작업시 } V=16\text{m/s}, q = 19.9 \text{ kg/m}^2 \quad h(\text{최고양정}) = 30 \text{ M}$$

$$\text{작업시 } q = 8.5 \times \sqrt{h} = 8.5 \times \sqrt{42} = 19.9 \text{ kg/m}^2$$

$$H(\text{GIRDER높이}) = 0.20 \text{ m}$$

$$\text{PITCH내 풍하중} = F \times L \times H \times q \times 1.6 = 1.11 \times 4.7 \times 0.2 \times 19.9 \times 1.6 = 33 \text{ kg}$$

$$\text{캔틀레버 풍하중} = F \times L1 \times H \times q \times 1.3 = 1.11 \times 1.9 \times 0.2 \times 19.9 \times 1.3 = 11 \text{ kg}$$

$$M_{FGG} = \frac{0.033 \times 470}{8} + \frac{0.011 \times 190}{2} = 0.894 \quad \text{ton.cm}$$

$$M_{FG1} = \frac{0.011 \times 190}{2} = 1.045 \quad \text{ton.cm}$$

(2) HOIST에 의한 풍하중

* 작업시

$$\text{풍하중} = F \times HB \times HH \times q \times 1.2 = 1.11 \times 0.65 \times 1 \times 19.9 \times 1.2 = 17 \text{ kg}$$

*PITCH내 풍하중

$$M_{FHG} = 17 \times 470 / 4 = 1997.5 \text{ kg.cm} = 1.9975 \text{ ton.cm}$$

*캔틀레버 풍하중

$$M_{FH1} = 17 \times 190 = 3230 \text{ kg.cm} = 3.23 \text{ ton.cm}$$

7. COMBINED MOMENT

*PITCH내

$$M_{HCG} = M_{FGG} + M_{FHG} = 0.894 + 1.9975 = 2.892 \text{ ton.cm}$$



*컨틀레버

$$M_{HC1} = M_{FG1} + M_{FH1} = 1.045 + 3.23 = 4.275 \text{ ton.cm}$$

8. BENDING STRESS

A. VERTICAL BENDING STRESS

1. PITCH 내

$$\sigma v1 = M_{max1} / Z_x = 156 / 217 = 0.719 \text{ ton/cm}^2$$

2. 컨틀레버

$$\sigma v2 = M_{max2} / Z_x = 227.9 / 217 = 1.050 \text{ ton/cm}^2$$

B. 수평 최대 응력

1. PITCH 내

$$\sigma v3 = M_{HCG} / Z_y = 2.8915 / 27.7 = 0.104 \text{ ton/cm}^2$$

2. 컨틀레버

$$\sigma v4 = M_{HC1} / Z_y = 4.275 / 27.7 = 0.154 \text{ ton/cm}^2$$

9. 합성 응력

$$\Sigma \sigma 1 = \sigma v1 + \sigma v3 = 0.823$$

0.823	<	1.279	ton/cm ²	O.K
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$$\Sigma \sigma 2 = \sigma v2 + \sigma v4 = 1.205$$

1.205	<	1.6	ton/cm ²	O.K
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용접효율 : 80% 응력 1391x80% x 115% = 1.279 ton/cm²
 컨틀레버는 용접부 없음 1391*작업시1.15(풍하중포함) = 1.600 ton/cm²

10. 휴지시 풍하중에 의한 BENDING MOMENT

휴지시 V=45m/s , q = 158.0 kg/m² h(최고양정) = 30 m
 휴지시 q = M x ⁴√h = 67.5 x ⁴√30 = 158.0 kg/m²
 M = V² / 30 = 67.5

*휴지시 풍하중에 의한 수평하중

PITCH내 풍하중 = F x L x H x q x 1.6 = 1.11 x 4.7 x 0.2 x 158 x 1.6 = 264 kg

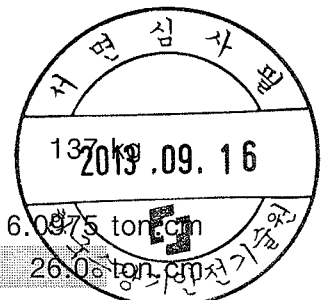
컨틀레버에 대한 풍하중 = F x L₁ x H x q x 1.3 = 1.11 x 1.9 x 0.2 x 158 x 1.3 = 87 kg

$$MM_{G1} = \frac{0.264 \times 470}{8} + \frac{0.087 \times 190}{2} = 7.245 \text{ ton.cm}$$

$$MM_1 = \frac{0.087 \times 190}{2} = 8.265 \text{ ton.cm}$$

HOIST에 대한 풍하중 = F x H_B x H x q x 1.2 = 1.11 x 0.65 x 1 x 158 x 1.2 =

*PITCH내 MM_{H0} = 137 x 470 / 4 = 64390 KG.CM = 16.0875 ton.cm
 *컨틀레버 MM_{H1} = 137 x 190 = 26030 KG.CM = 6.5075 ton.cm



* COMBINED MOMENT

$$MM_2 = MM_{G1} + MM_{H0} = 7.245 + 16.0975 = 23.3425 \text{ ton.cm}$$

$$MM_4 = MM_1 + MM_{H1} = 8.265 + 26 = 34.27 \text{ ton.cm}$$

* BENDING STRESS

$$\Sigma\sigma_3 = MM_2 / Zy = 23.3425 / 27.7 = 0.843 \text{ ton/cm}^2 < 1.447 \text{ ton/cm}^2 \text{---O.K}$$

$$\Sigma\sigma_4 = MM_4 / Zy = 34.265 / 27.7 = 1.237 \text{ ton/cm}^2 < 1.808 \text{ ton/cm}^2 \text{---O.K}$$

PITCH내 휴지시 응력 1391 x 80% x 130% = 1447 ton/cm²
 컨트롤레버는 용접부 없음 휴지시 응력 1391 x 130% = 1808 ton/cm²

11. DEFLECTION OF GIRDER

* PITCH 내

1) DUE TO RATED & TROLLEY LOAD

$$.D1 = \frac{(Q+Q2) \times (L)^3}{48 \times E \times I_x} = 0.5411 \text{ cm}$$

2) DUE TO DEAD LOAD

$$.D2 = \frac{5 \times W_g \times L^3}{384 \times E \times I_x} = 0.0362 \text{ cm}$$

3) TOTAL DEFLECTION

$$.s1 = D1 + D2 = 0.577 \text{ cm}$$

$$\text{RATIO : } D3/L = 1 / 814 < 800 \text{ --- O.K}$$

* 컨트롤레버

1) DUE TO RATED & TROLLEY LOAD

$$.D1 = \frac{(Q+Q2) \times (L1-L2)^3}{3 \times E \times I_x} = 0.3416 \text{ cm}$$

2) DUE TO DEAD LOAD

$$.D2 = \frac{W_{g1} \times L1^3}{8 \times E \times I_x} = 0.0092 \text{ cm}$$

3) TOTAL DEFLECTION

$$.s2 = D1 + D2 = 0.3508 \text{ cm}$$

$$\text{RATIO : } D3/L = 1 / 542 < 500 \text{ --- O.K}$$

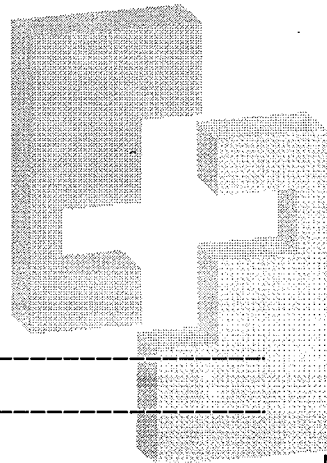
한국승강기안전기술원
 Korea Elevator Safety Technology Institute



2. I-BEAM 최대허용 가능 SPAN 계산

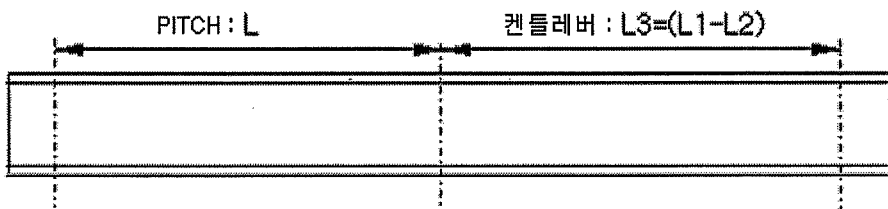
1. SPECIFICATION

.정격하중	-----	Q =	1	ton
.SPAN (PITCH)	-----	L=	700	cm
.켄틸리버	-----	L1=	270	cm
.TROLLEY WHEEL BASE	-----	B=	45.8	cm
.WEIGHT OF HOIST	-----	Wh=	0.14	ton
.PITCH내의 GIRDER 무게	-----	Wg=	0.268	ton
.켄틸리버의 GIRDER 무게	-----	Wg1=	0.103	ton
.I-BEAM SIZE	-----	250x125x7.5/12.5t		
		Ix =	5180	cm ⁴
		Iy =	337	cm ⁴
		Zx =	414	cm ³
		Zy =	53.9	cm ³
		A =	48.79	cm ²
		Wb =	38.3	kg/m
.HOOK APPROACH	-----	L2=	30	cm
탄성계수	-----	E=	2100000	kg/cm ²
동하중 계수(작업계수)	-----	M(Φ)=	1.11	
정하중 계수(충격계수)	-----	F(Ψ)=	1.10	



1. DESIGN

한국승강기안전기술원



2. I-BEAM에 작용하는 하중

$$P = Q + Wh = 1 + 0.14 = 1.14 \text{ ton}$$

3. 수직하중에 의한 BENDING MOMENT

1) PITCH 지점내 BENDING MOMENT

$$Mh1 = F \times M \times P \times (L-B/2)^2 / (4 \times L)$$

$$= 1.11 \times 1.1 \times 1.14 \times (700 - 45.8/2)^2 / (4 \times 700)$$



$$= 227.9 \quad \text{ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$Mh2 = F \times M \times P \times L3$$

$$= 1.11 \times 1.1 \times 1.14 \times (270-30) = 334.07 \quad \text{ton.cm}$$

4. 자중에 의한 BENDING MOMENT(I-BEAM)

1) PITCH 지점내 BENDING MOMENT

$$Mt1 = F \times Wg \times L / 8 = 1.11 \times 0.268 \times 700 / 8 = 26.03 \quad \text{ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$Mt2 = F \times Wg1 \times L1 / 2 = 1.11 \times 0.103 \times 270 / 2 = 15.43 \quad \text{ton.cm}$$

5. 수직하중에 의한 최대 BENDING MOMENT

$$Mmax1 = Mh1 + Mt1 = 253.9 \quad \text{ton.cm}$$

$$Mmax2 = Mh2 + Mt2 = 349.5 \quad \text{ton.cm}$$

6. 풍하중에 의한 수평하중의 BENDING MOMENT

(1) GIRDER에 의한 수직 풍하중

작업시 $V=16\text{m/s}$, $q = 19.9 \text{ kg/m}^2$ $h(\text{최고양정}) = 30 \text{ M}$

작업시 $q = 8.5 \times \sqrt{h} = 8.5 \times \sqrt{42} = 19.9 \text{ kg/m}^2$

$H(\text{GIRDER높이}) = 0.25 \text{ m}$

PITCH내 풍하중 = $F \times L \times H \times q \times 1.7 = 1.11 \times 7 \times 0.25 \times 19.9 \times 1.7 = 66 \text{ kg}$

캔틀레버 풍하중 = $F \times L1 \times H \times q \times 1.4 = 1.11 \times 2.7 \times 0.25 \times 19.9 \times 1.4 = 21 \text{ kg}$

$$M_{FGG} = \frac{0.066 \times 700}{8} + \frac{0.021 \times 270}{2} = 2.94 \quad \text{ton.cm}$$

$$M_{FG1} = \frac{0.021 \times 270}{2} = 2.835 \quad \text{ton.cm}$$

(2) HOIST에 의한 풍하중

* 작업시

$HH = 1.0 \text{ m}$ $HB = 0.65 \text{ m}$
 풍하중 = $F \times HB \times HH \times q \times 1.2 = 1.11 \times 0.65 \times 1 \times 19.9 \times 1.2 = 17 \text{ kg}$

*PITCH내 풍하중

$M_{FHG} = 17 \times 700 / 4 = 2975 \text{ kg.cm} = 2.975 \text{ ton.cm}$

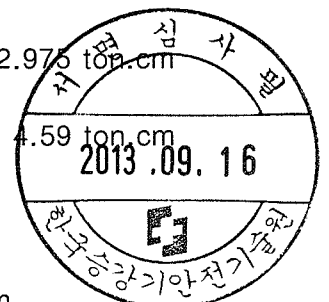
*캔틀레버 풍하중

$M_{FH1} = 17 \times 270 = 4590 \text{ kg.cm} = 4.59 \text{ ton.cm}$

7. COMBINED MOMENT

*PITCH내

$M_{HCG} = M_{FGG} + M_{FHG} = 2.94 + 2.975 = 5.915 \text{ ton.cm}$



*컨틀레버

$$M_{HC1} = M_{FG1} + M_{FH1} = 2.835 + 4.59 = 7.425 \text{ ton.cm}$$

8. BENDING STRESS

A. VERTICAL BENDING STRESS

1. PITCH 내

$$\sigma v1 = M_{max1} / Z_x = 253.9 / 414 = 0.613 \text{ ton/cm}^2$$

2. 컨틀레버

$$\sigma v2 = M_{max2} / Z_x = 349.5 / 414 = 0.844 \text{ ton/cm}^2$$

B. 수평 최대 응력

1. PITCH 내

$$\sigma v3 = M_{HCG} / Z_y = 5.915 / 53.9 = 0.110 \text{ ton/cm}^2$$

2. 컨틀레버

$$\sigma v4 = M_{HC1} / Z_y = 7.425 / 53.9 = 0.138 \text{ ton/cm}^2$$

9. 합성 응력

$$\Sigma \sigma 1 = \sigma v1 + \sigma v3 = 0.723$$

0.723	<	1.279	ton/cm ²	O.K
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$$\Sigma \sigma 2 = \sigma v2 + \sigma v4 = 0.982$$

0.982	<	1.6	ton/cm ²	O.K
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용접효율 : 80% 응력 1391x80% x 115% = 1.279 ton/cm²
 컨틀레버는 용접부 없음 1391*작업시 1.15(풍하중포함) = 1.600 ton/cm²

10. 휴지시 풍하중에 의한 BENDING MOMENT

휴지시 V=45m/s , q = 158.0 kg/m² h(최고양정) = 30 m
 휴지시 q = M x ⁴√h = 67.5 x ⁴√30 = 158.0 kg/m²
 M = V² / 30 = 67.5

*휴지시 풍하중에 의한 수평하중

PITCH내 풍하중 = F x L x H x q x 1.7 = 1.11 x 7 x 0.25 x 158 x 1.7 = 522 kg

컨틀레버에 대한 풍하중 = F x L₁ x H x q x 1.4 = 1.11 x 2.7 x 0.25 x 158 x 1.4 = 166 kg

$$MM_{G1} = \frac{0.522 \times 700}{8} + \frac{0.166 \times 270}{2} = 23.265 \text{ ton.cm}$$

$$MM_1 = \frac{0.166 \times 270}{2} = 22.41 \text{ ton.cm}$$

HOIST에 대한 풍하중 = F x H_B x H_H x q x 1.2 = 1.11 x 0.65 x 1 x 158 x 1.2 =

*PITCH내 MM_{H0} = 137 x 700 / 4 = 95900 KG.CM = 23.975 ton.cm

*컨틀레버 MM_{H1} = 137 x 270 = 36990 KG.CM = 37.0 ton.cm



* COMBINED MOMENT

$$MM_2 = MM_{G1} + MM_{H0} = 23.265 + 23.975 = 47.24 \text{ ton.cm}$$

$$MM_4 = MM_1 + MM_{H1} = 22.41 + 37 = 59.41 \text{ ton.cm}$$

* BENDING STRESS

$$\Sigma\sigma_3 = MM_2 / Zy = 47.24 / 53.9 = 0.876 \text{ ton/cm}^2 < 1.447 \text{ ton/cm}^2 \text{---O.K}$$

$$\Sigma\sigma_4 = MM_4 / Zy = 59.41 / 53.9 = 1.102 \text{ ton/cm}^2 < 1.808 \text{ ton/cm}^2 \text{---O.K}$$

PITCH내 휴지시 응력 $1391 \times 80\% \times 130\% = 1447 \text{ ton/cm}^2$

컨트롤레버는 용접부 없음 휴지시 응력 $1391 \times 130\% = 1808 \text{ ton/cm}^2$

11. DEFLECTION OF GIRDER

* PITCH 내

1) DUE TO RATED & TROLLEY LOAD

$$.D1 = \frac{(Q+Q2) \times (L)^3}{48 \times E \times I_x} = 0.7489 \text{ cm}$$

2) DUE TO DEAD LOAD

$$.D2 = \frac{5 \times Wg \times L^3}{384 \times E \times I_x} = 0.1100 \text{ cm}$$

3) TOTAL DEFLECTION

$$.s1 = D1 + D2 = 0.859 \text{ cm}$$

$$\text{RATIO : } D3/L = 1/815 < 800 \text{ --- O.K}$$

* 컨트롤레버

1) DUE TO RATED & TROLLEY LOAD

$$.D1 = \frac{(Q+Q2) \times (L1-L2)^3}{3 \times E \times I_x} = 0.4829 \text{ cm}$$

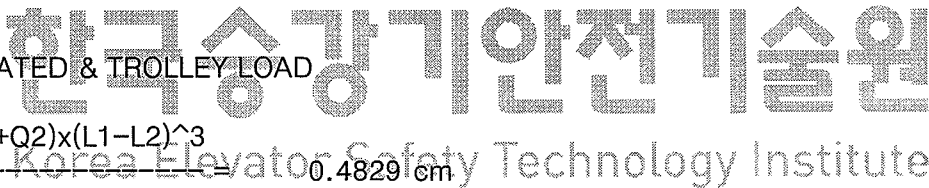
2) DUE TO DEAD LOAD

$$.D2 = \frac{Wg1 \times L1^3}{8 \times E \times I_x} = 0.0233 \text{ cm}$$

3) TOTAL DEFLECTION

$$.s2 = D1 + D2 = 0.5062 \text{ cm}$$

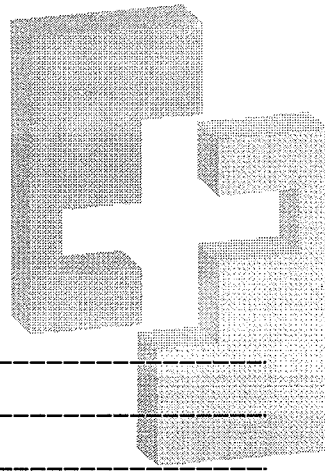
$$\text{RATIO : } D3/L = 1/533 < 500 \text{ --- O.K}$$



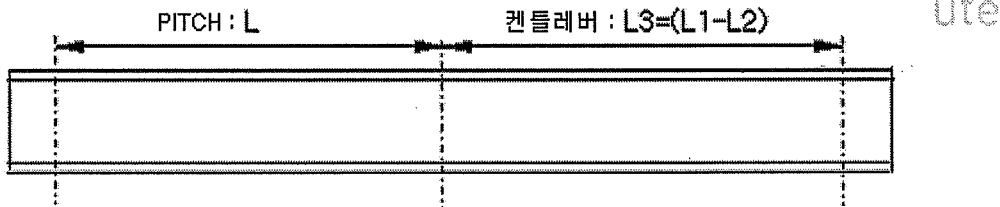
3. H-BEAM 최대허용 가능 SPAN 계산

1. SPECIFICATION

.정격하중	-----	Q =	1	ton
.SPAN (PITCH)	-----	L=	650	cm
.켄틸리버	-----	L1=	250	cm
.TROLLEY WHEEL BASE	-----	B=	45.8	cm
.WEIGHT OF HOIST	-----	Wh=	0.14	ton
.PITCH내의 GIRDER 무게	-----	Wg=	0.324	ton
.켄틸리버의 GIRDER 무게	-----	Wg1=	0.125	ton
.H-BEAM SIZE	-----		200x200x8/12t	
		Ix =	4720	cm ⁴
		Iy =	1600	cm ⁴
		Zx =	472	cm ³
		Zy =	160	cm ³
		A =	63.53	cm ²
		Wb =	49.9	kg/m
.HOOK APPROACH	-----	L2=	30	cm
탄성계수	-----	E=	2100000	kg/cm ²
동하중 계수(작업계수)	-----	M(Φ)=	1.11	
정하중 계수(충격계수)	-----	F(Ψ)=	1.10	



1. DESIGN



2. H-BEAM에 작용하는 하중

$$P = Q + Wh = 1 + 0.14 = 1.14 \text{ ton}$$

3. 수직하중에 의한 BENDING MOMENT

1) PITCH 지점내 BENDING MOMENT

$$M_{h1} = F \times M \times P \times (L-B/2)^2 / (4 \times L)$$

$$= 1.11 \times 1.1 \times 1.14 \times (650-45.8/2)^2 / (4 \times 650)$$



$$= 210.5 \quad \text{ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$Mh2 = F \times M \times P \times L3$$

$$= 1.11 \times 1.1 \times 1.14 \times (250 - 30) = 306.23 \quad \text{ton.cm}$$

4. 자중에 의한 BENDING MOMENT(I-BEAM)

1) PITCH 지점내 BENDING MOMENT

$$Mt1 = F \times Wg \times L / 8 = 1.11 \times 0.324 \times 650 / 8 = 29.221 \quad \text{ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$Mt2 = F \times Wg1 \times L1 / 2 = 1.11 \times 0.125 \times 250 / 2 = 17.34 \quad \text{ton.cm}$$

5. 수직하중에 의한 최대 BENDING MOMENT

$$Mmax1 = Mh1 + Mt1 = 239.8 \quad \text{ton.cm}$$

$$Mmax2 = Mh2 + Mt2 = 323.6 \quad \text{ton.cm}$$

6. 풍하중에 의한 수평하중의 BENDING MOMENT

(1) GIRDER에 의한 수직 풍하중

작업시 $V=16\text{m/s}$, $q = 19.9 \text{ kg/m}^2$ $h(\text{최고양정}) = 30 \text{ m}$

작업시 $q = 8.5 \times \sqrt{h} = 8.5 \times \sqrt{42} = 19.9 \text{ kg/m}^2$

$H(\text{GIRDER높이}) = 0.30 \text{ m}$

PITCH내 풍하중 = $F \times L \times H \times q \times 1.7 = 1.11 \times 6.5 \times 0.3 \times 19.9 \times 1.7 = 73 \text{ kg}$

캔틀레버 풍하중 = $F \times L1 \times H \times q \times 1.4 = 1.11 \times 2.5 \times 0.3 \times 19.9 \times 1.4 = 23 \text{ kg}$

$$M_{FGG} = \frac{0.073 \times 650}{8} + \frac{0.023 \times 250}{2} = 3.056 \quad \text{ton.cm}$$

$$M_{FG1} = \frac{0.023 \times 250}{2} = 2.875 \quad \text{ton.cm}$$

(2) HOIST에 의한 풍하중

* 작업시

$HH = 1.0 \text{ m}$ $HB = 0.65 \text{ m}$
 풍하중 = $F \times HB \times HH \times q \times 1.2 = 1.11 \times 0.65 \times 1 \times 19.9 \times 1.2 = 17 \text{ kg}$

*PITCH내 풍하중

$$M_{FHG} = 17 \times 650 / 4 = 2762.5 \text{ kg.cm} = 2.7625 \text{ ton.cm}$$

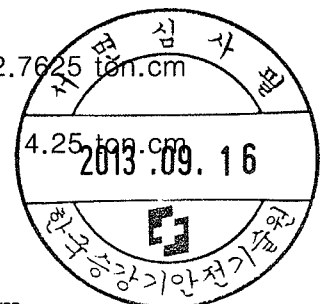
*캔틀레버 풍하중

$$M_{FH1} = 17 \times 250 = 4250 \text{ kg.cm} = 4.25 \text{ ton.cm}$$

7. COMBINED MOMENT

*PITCH내

$$M_{HCG} = M_{FGG} + M_{FHG} = 3.056 + 2.7625 = 5.819 \text{ ton.cm}$$



*컨트레버

$$M_{HC1} = M_{FG1} + M_{FH1} = 2.875 + 4.25 = 7.125 \text{ ton.cm}$$

8. BENDING STRESS

A. VERTICAL BENDING STRESS

1. PITCH 내

$$\sigma v1 = M_{max1} / Z_x = 239.8 / 472 = 0.508 \text{ ton/cm}^2$$

2. 컨트레버

$$\sigma v2 = M_{max2} / Z_x = 323.6 / 472 = 0.686 \text{ ton/cm}^2$$

B. 수평 최대 응력

1. PITCH 내

$$\sigma v3 = M_{HCG} / Z_y = 5.8185 / 160 = 0.036 \text{ ton/cm}^2$$

2. 컨트레버

$$\sigma v4 = M_{HC1} / Z_y = 7.125 / 160 = 0.045 \text{ ton/cm}^2$$

9. 합성 응력

$$\Sigma \sigma 1 = \sigma v1 + \sigma v3 = 0.544$$

0.544	<	1.279	ton/cm ²	O.K
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$$\Sigma \sigma 2 = \sigma v2 + \sigma v4 = 0.730$$

0.730	<	1.6	ton/cm ²	O.K
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용접효율 : 80% 응력 1391 x 80% x 115% = 1.279 ton/cm²
 컨트레버는 용접부 없음 1391 * 작업시 1.15(풍하중포함) = 1.600 ton/cm²

10. 휴지시 풍하중에 의한 BENDING MOMENT

휴지시 V=45m/s , q = 158.0 kg/m² h(최고양정) = 30 m
 휴지시 q = M x ⁴√h = 67.5 x ⁴√30 = 158.0 kg/m²
 M = V² / 30 = 67.5

*휴지시 풍하중에 의한 수평하중

PITCH내 풍하중 = F x L x H x q x 1.7 = 1.11 x 6.5 x 0.3 x 158 x 1.7 = 581 kg

컨트레버에 대한 풍하중 = F x L1 x H x q x 1.4 = 1.11 x 2.5 x 0.3 x 158 x 1.4 = 184 kg

$$MM_{G1} = \frac{0.581 \times 650}{8} + \frac{0.184 \times 250}{2} = 24.20625 \text{ ton.cm}$$

$$MM_1 = \frac{0.184 \times 250}{2} = 23 \text{ ton.cm}$$

HOIST에 대한 풍하중 = F x HB x HH x q x 1.2 = 1.11 x 0.65 x 1 x 158 x 1.2 = 137 kg

*PITCH내 MM_{H0} = 137 x 650 / 4 = 89050 KG.CM = 22,2625 ton.cm

*컨트레버 MM_{H1} = 137 x 250 = 34250 KG.CM = 8,5625 ton.cm

* COMBINED MOMENT



$$MM_2 = MM_{G1} + MM_{H0} = 24.20625 + 22.2625 = 46.46875 \text{ ton.cm}$$

$$MM_4 = MM_1 + MM_{H1} = 23 + 34.3 = 57.30 \text{ ton.cm}$$

* BENDING STRESS

$$\Sigma\sigma_3 = MM_2 / Z_y = 46.46875 / 160 = 0.290 \text{ ton/cm}^2 < 1.447 \text{ ton/cm}^2 \text{---O.K}$$

$$\Sigma\sigma_4 = MM_4 / Z_y = 57.3 / 160 = 0.358 \text{ ton/cm}^2 < 1.808 \text{ ton/cm}^2 \text{---O.K}$$

PITCH내 휴지시 응력 1391 x 80% x 130% =

1447 ton/cm²

컨트롤레버는 용접부 없음 휴지시 응력 1391 x 130% =

1808 ton/cm²

11. DEFLECTION OF GIRDER

* PITCH 내

1) DUE TO RATED & TROLLEY LOAD

$$.D1 = \frac{(Q+Q2) \times (L)^3}{48 \times E \times I_x} = 0.6580 \text{ cm}$$

2) DUE TO DEAD LOAD

$$.D2 = \frac{5 \times W_g \times L^3}{384 \times E \times I_x} = 0.1169 \text{ cm}$$

3) TOTAL DEFLECTION

$$.s1 = D1 + D2 = 0.775 \text{ cm}$$

$$\text{RATIO : } D3/L = 1/839 < 800 \text{---O.K}$$

* 컨트롤레버

1) DUE TO RATED & TROLLEY LOAD

$$.D1 = \frac{(Q+Q2) \times (L1-L2)^3}{3 \times E \times I_x} = 0.4082 \text{ cm}$$

2) DUE TO DEAD LOAD

$$.D2 = \frac{W_g \times L1^3}{8 \times E \times I_x} = 0.0246 \text{ cm}$$

3) TOTAL DEFLECTION

$$.s2 = D1 + D2 = 0.4328 \text{ cm}$$

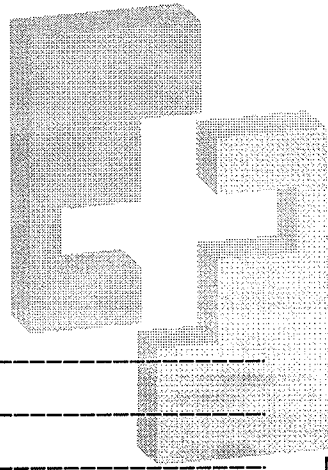
$$\text{RATIO : } D3/L = 1/578 < 500 \text{---O.K}$$



4. H-BEAM 최대허용 가능 SPAN 계산

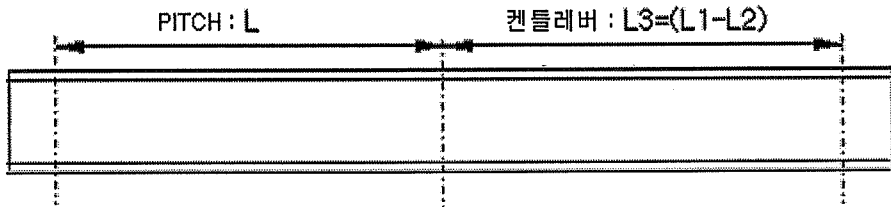
1. SPECIFICATION

.정격하중	-----	Q =	1	ton
.SPAN (PITCH)	-----	L=	800	cm
.켄틸리버	-----	L1=	320	cm
.TROLLEY WHEEL BASE	-----	B=	45.8	cm
.WEIGHT OF HOIST	-----	Wh=	0.14	ton
.PITCH내의 GIRDER 무게	-----	Wg=	0.256	ton
.켄틸리버의 GIRDER 무게	-----	Wg1=	0.102	ton
.H-BEAM SIZE	-----		300x150x6.5/9t	
		Ix =	7210	cm ⁴
		Iy =	508	cm ⁴
		Zx =	481	cm ³
		Zy =	67.7	cm ³
		A =	40.8	cm ²
		Wb =	32	kg/m
.HOOK APPROACH	-----	L2=	30	cm
탄성계수	-----	E=	2100000	kg/cm ²
동하중 계수(작업계수)	-----	M(Φ)=	1.11	
정하중 계수(충격계수)	-----	F(Ψ)=	1.10	



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1. DESIGN



2. H-BEAM에 작용하는 하중

$$P = Q + Wh = 1 + 0.14 = 1.14 \text{ ton}$$

3. 수직하중에 의한 BENDING MOMENT

1) PITCH 지점내 BENDING MOMENT

$$Mh1 = F \times M \times P \times (L-B/2)^2 / (4 \times L)$$

$$= 1.11 \times 1.1 \times 1.14 \times (800 - 45.8/2)^2 / (4 \times 800)$$



$$= 262.7 \text{ ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$Mh2 = F \times M \times P \times L3$$

$$= 1.11 \times 1.1 \times 1.14 \times (320 - 30) = 403.66 \text{ ton.cm}$$

4. 자중에 의한 BENDING MOMENT(I-BEAM)

1) PITCH 지점내 BENDING MOMENT

$$Mt1 = F \times Wg \times L / 8 = 1.11 \times 0.256 \times 800 / 8 = 28.416 \text{ ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$Mt2 = F \times Wg1 \times L1 / 2 = 1.11 \times 0.102 \times 320 / 2 = 18.12 \text{ ton.cm}$$

5. 수직하중에 의한 최대 BENDING MOMENT

$$Mmax1 = Mh1 + Mt1 = 291.1 \text{ ton.cm}$$

$$Mmax2 = Mh2 + Mt2 = 421.8 \text{ ton.cm}$$

6. 풍하중에 의한 수평하중의 BENDING MOMENT

(1) GIRDER에 의한 수직 풍하중

작업시 $V=16\text{m/s}$, $q = 19.9 \text{ kg/m}^2$ $h(\text{최고양정}) = 30 \text{ m}$

작업시 $q = 8.5 \times \sqrt{h} = 8.5 \times \sqrt{42} = 19.9 \text{ kg/m}^2$

$H(\text{GIRDER높이}) = 0.30 \text{ m}$

PITCH내 풍하중 = $F \times L \times H \times q \times 1.7 = 1.11 \times 8 \times 0.3 \times 19.9 \times 1.7 = 90 \text{ kg}$

캔틀레버 풍하중 = $F \times L1 \times H \times q \times 1.4 = 1.11 \times 3.2 \times 0.3 \times 19.9 \times 1.4 = 30 \text{ kg}$

$$M_{FGG} = \frac{0.09 \times 800}{8} + \frac{0.03 \times 320}{2} = 4.2 \text{ ton.cm}$$

$$M_{FG1} = \frac{0.03 \times 320}{2} = 4.8 \text{ ton.cm}$$

(2) HOIST에 의한 풍하중

* 작업시

$HH = 1.0 \text{ m}$ $HB = 0.65 \text{ m}$
 풍하중 = $F \times HB \times HH \times q \times 1.2 = 1.11 \times 0.65 \times 1 \times 19.9 \times 1.2 = 17 \text{ kg}$

*PITCH내 풍하중

$M_{FHG} = 17 \times 800 / 4 = 3400 \text{ kg.cm} =$

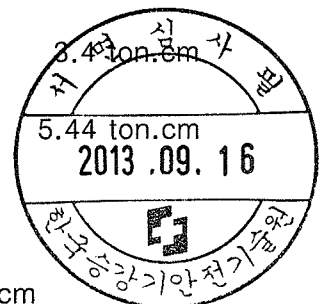
*캔틀레버 풍하중

$M_{FH1} = 17 \times 320 = 5440 \text{ kg.cm} =$

7. COMBINED MOMENT

*PITCH내

$M_{HCG} = M_{FGG} + M_{FHG} = 4.2 + 3.4 = 7.600 \text{ ton.cm}$



*컨트레버

$$M_{HC1} = M_{FG1} + M_{FH1} = 4.8 + 5.44 = 10.240 \text{ ton.cm}$$

8. BENDING STRESS

A. VERTICAL BENDING STRESS

1. PITCH 내

$$\sigma v1 = M_{max1} / Zx = 291.1 / 481 = 0.605 \text{ ton/cm}^2$$

2. 컨트레버

$$\sigma v2 = M_{max2} / Zx = 421.8 / 481 = 0.877 \text{ ton/cm}^2$$

B. 수평 최대 응력

1. PITCH 내

$$\sigma v3 = M_{HCG} / Zy = 7.6 / 67.7 = 0.112 \text{ ton/cm}^2$$

2. 컨트레버

$$\sigma v4 = M_{HC1} / Zy = 10.24 / 67.7 = 0.151 \text{ ton/cm}^2$$

9. 합성 응력

$$\Sigma \sigma 1 = \sigma v1 + \sigma v3 = 0.717$$

0.717	<	1.279	ton/cm ²	O.K
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$$\Sigma \sigma 2 = \sigma v2 + \sigma v4 = 1.028$$

1.028	<	1.6	ton/cm ²	O.K
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용접효율 : 80% 응력 1391 x 80% x 115% = 1.279 ton/cm²
 컨트레버는 용접부 없음 1391 * 작업시 1.15 (풍하중포함) = 1.600 ton/cm²

10. 휴지시 풍하중에 의한 BENDING MOMENT

휴지시 V=45m/s, q = 158.0 kg/m² h(최고양정) = 30 m
 휴지시 q = M x ⁴√h = 67.5 x ⁴√30 = 158.0 kg/m²
 M = V² / 30 = 67.5

*휴지시 풍하중에 의한 수평하중

PITCH내 풍하중 = F x L x H x q x 1.7 = 1.11 x 8 x 0.3 x 158 x 1.7 = 716 kg

컨트레버에 대한 풍하중 = F x L1 x H x q x 1.4 = 1.11 x 3.2 x 0.3 x 158 x 1.4 = 236 kg

$$MM_{G1} = \frac{0.716 \times 800}{8} + \frac{0.236 \times 320}{2} = 33.84 \text{ ton.cm}$$

$$MM_1 = \frac{0.236 \times 320}{2} = 37.76 \text{ ton.cm}$$

HOIST에 대한 풍하중 = F x HB x HH x q x 1.2 = 1.11 x 0.65 x 1 x 158 x 1.2 =

*PITCH내 MM_{H0} = 137 x 800 / 4 = 109600 KG.CM =

*컨트레버 MM_{H1} = 137 x 320 = 43840 KG.CM =

* COMBINED MOMENT



$$MM_2 = MM_{G1} + MM_{H0} = 33.84 + 27.4 = 61.24 \text{ ton.cm}$$

$$MM_4 = MM_1 + MM_{H1} = 37.76 + 43.8 = 81.56 \text{ ton.cm}$$

* BENDING STRESS

$$\Sigma\sigma_3 = MM_2 / Z_y = 61.24 / 67.7 = 0.905 \text{ ton/cm}^2 < 1.447 \text{ ton/cm}^2 \text{---O.K.}$$

$$\Sigma\sigma_4 = MM_4 / Z_y = 81.56 / 67.7 = 1.205 \text{ ton/cm}^2 < 1.808 \text{ ton/cm}^2 \text{---O.K.}$$

PITCH내 휴지시 응력 1391 x 80% x 130% = 1447 ton/cm²

컨트롤레버는 용접부 없음 휴지시 응력 1391 x 130% = 1808 ton/cm²

11. DEFLECTION OF GIRDER

* PITCH 내

1) DUE TO RATED & TROLLEY LOAD

$$.D1 = \frac{(Q+Q2) \times (L)^3}{48 \times E \times I_x} = 0.8031 \text{ cm}$$

2) DUE TO DEAD LOAD

$$.D2 = \frac{5 \times W_g \times L^3}{384 \times E \times I_x} = 0.1127 \text{ cm}$$

3) TOTAL DEFLECTION

$$.s1 = D1 + D2 = 0.916 \text{ cm}$$

$$\text{RATIO : } D3/L = 1/874 < 800 \text{---O.K.}$$

* 컨트롤레버

1) DUE TO RATED & TROLLEY LOAD

$$.D1 = \frac{(Q+Q2) \times (L1-L2)^3}{3 \times E \times I_x} = 0.6121 \text{ cm}$$

2) DUE TO DEAD LOAD

$$.D2 = \frac{W_g \times L1^3}{8 \times E \times I_x} = 0.0276 \text{ cm}$$

3) TOTAL DEFLECTION

$$.s2 = D1 + D2 = 0.6397 \text{ cm}$$

$$\text{RATIO : } D3/L = 1/500 < 500 \text{---O.K.}$$

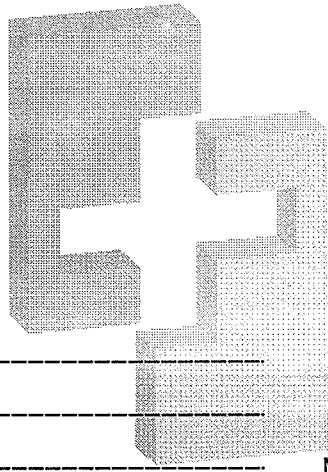
한국승강기안전기술원
Korea Elevator Safety Technology Institute



5. H-BEAM 최대허용 가능 SPAN 계산

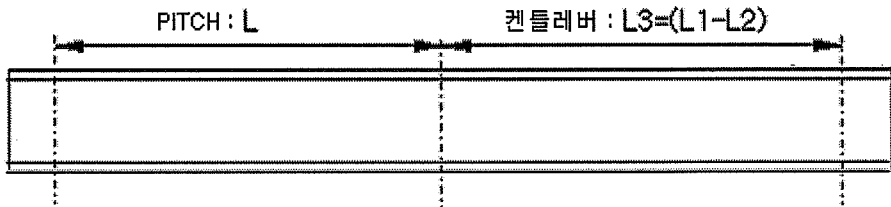
1. SPECIFICATION

.정격하중	-----	Q =	1	ton
.SPAN (PITCH)	-----	L=	1050	cm
.켄틸리버	-----	L1=	400	cm
.TROLLEY WHEEL BASE	-----	B=	45.8	cm
.WEIGHT OF HOIST	-----	Wh=	0.14	ton
.PITCH내의 GIRDER 무게	-----	Wg=	0.519	ton
.켄틸리버의 GIRDER 무게	-----	Wg1=	0.198	ton
.H-BEAM SIZE	-----		350x175x7/11t	
		Ix =	13500	cm ⁴
		Iy =	984	cm ⁴
		Zx =	771	cm ³
		Zy =	112	cm ³
		A =	62.91	cm ²
		Wb =	49.4	kg/m
.HOOK APPROACH	-----	L2=	30	cm
탄성계수	-----	E=	2100000	kg/cm ²
동하중 계수(작업계수)	-----	M(Φ)=	1.11	
정하중 계수(충격계수)	-----	F(Ψ)=	1.10	



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1. DESIGN



2. H-BEAM에 작용하는 하중

$$P = Q + Wh = 1 + 0.14 = 1.14 \text{ ton}$$

3. 수직하중에 의한 BENDING MOMENT

1) PITCH 지점내 BENDING MOMENT

$$Mh1 = F \times M \times P \times (L-B/2)^2 / (4 \times L)$$

$$= 1.11 \times 1.1 \times 1.14 \times (1050 - 45.8/2)^2 / (4 \times 1050)$$



$$= 349.6 \text{ ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$Mh2 = F \times M \times P \times L3$$

$$= 1.11 \times 1.1 \times 1.14 \times (400-30) = 515.02 \text{ ton.cm}$$

4. 자중에 의한 BENDING MOMENT(I-BEAM)

1) PITCH 지점내 BENDING MOMENT

$$Mt1 = F \times Wg \times L / 8 = 1.11 \times 0.519 \times 1050 / 8 = 75.612 \text{ ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$Mt2 = F \times Wg1 \times L1 / 2 = 1.11 \times 0.198 \times 400 / 2 = 43.96 \text{ ton.cm}$$

5. 수직하중에 의한 최대 BENDING MOMENT

$$Mmax1 = Mh1 + Mt1 = 425.2 \text{ ton.cm}$$

$$Mmax2 = Mh2 + Mt2 = 559 \text{ ton.cm}$$

6. 풍하중에 의한 수평하중의 BENDING MOMENT

(1) GIRDER에 의한 수직 풍하중

작업시 $V=16\text{m/s}$, $q = 19.9 \text{ kg/m}^2$ $h(\text{최고양정}) = 30 \text{ m}$

작업시 $q = 8.5 \times \sqrt{h} = 8.5 \times \sqrt{42} = 19.9 \text{ kg/m}^2$

$H(\text{GIRDER높이}) = 0.35 \text{ m}$

PITCH내 풍하중 = $F \times L \times H \times q \times 1.7 = 1.11 \times 10.5 \times 0.35 \times 19.9 \times 1.7 = 138 \text{ kg}$

캔틀레버 풍하중 = $F \times L1 \times H \times q \times 1.4 = 1.11 \times 4 \times 0.35 \times 19.9 \times 1.4 = 43 \text{ kg}$

$$M_{FGG} = \frac{0.138 \times 1050}{8} + \frac{0.043 \times 400}{2} = 9.513 \text{ ton.cm}$$

$$M_{FG1} = \frac{0.043 \times 400}{2} = 8.6 \text{ ton.cm}$$

(2) HOIST에 의한 풍하중

* 작업시

$HH = 1.0 \text{ m}$ $HB = 0.65 \text{ m}$

풍하중 = $F \times HB \times HH \times q \times 1.2 = 1.11 \times 0.65 \times 1 \times 19.9 \times 1.2 = 17 \text{ kg}$

*PITCH내 풍하중

$M_{FHG} = 17 \times 1050 / 4 = 4462.5 \text{ kg.cm} = 4.4625 \text{ ton.cm}$

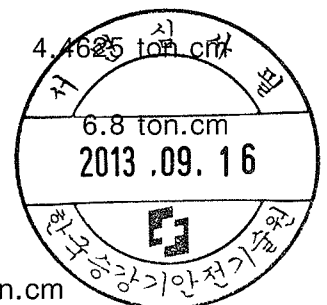
*캔틀레버 풍하중

$M_{FH1} = 17 \times 400 = 6800 \text{ kg.cm} = 6.8 \text{ ton.cm}$

7. COMBINED MOMENT

*PITCH내

$M_{HCG} = M_{FGG} + M_{FHG} = 9.513 + 4.4625 = 13.976 \text{ ton.cm}$



*켄틀레버

$$M_{HC1} = M_{FG1} + M_{FH1} = 8.6 + 6.8 = 15.400 \text{ ton.cm}$$

8. BENDING STRESS

A. VERTICAL BENDING STRESS

1. PITCH 내

$$\sigma v1 = M_{max1} / Z_x = 425.2 / 771 = 0.551 \text{ ton/cm}^2$$

2. 켄틀레버

$$\sigma v2 = M_{max2} / Z_x = 559 / 771 = 0.725 \text{ ton/cm}^2$$

B. 수평 최대 응력

1. PITCH 내

$$\sigma v3 = M_{HCG} / Z_y = 13.9755 / 112 = 0.125 \text{ ton/cm}^2$$

2. 켄틀레버

$$\sigma v4 = M_{HC1} / Z_y = 15.4 / 112 = 0.138 \text{ ton/cm}^2$$

9. 합성 응력

$$\Sigma \sigma 1 = \sigma v1 + \sigma v3 = 0.676$$

0.676	<	1.279	ton/cm ²	O.K
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$$\Sigma \sigma 2 = \sigma v2 + \sigma v4 = 0.863$$

0.863	<	1.6	ton/cm ²	O.K
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용접효율 : 80% 응력 1391x80% x 115% = 1.279 ton/cm²
 켄틀레버는 용접부 없음 1391*작업시 1.15(풍하중포함) = 1.600 ton/cm²

10. 휴지시 풍하중에 의한 BENDING MOMENT

휴지시 V=45m/s , q = 158.0 kg/m² h(최고양정) = 30 m
 휴지시 q = M x ⁴√h = 67.5 x ⁴√30 = 158.0 kg/m²
 M = V² / 30 = 67.5

*휴지시 풍하중에 의한 수평하중

PITCH내 풍하중 = F x L x H x q x 1.7 = 1.11 x 10.5 x 0.35 x 158 x 1.7 = 1096 kg

켄틀레버에 대한 풍하중 = F x L x H x q x 1.4 = 1.11 x 4 x 0.35 x 158 x 1.4 = 344 kg

$$MM_{G1} = \frac{1.096 \times 1050}{8} - \frac{0.344 \times 400}{2} = 75.05 \text{ ton.cm}$$

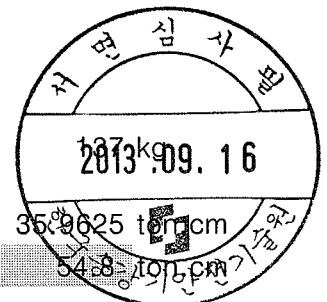
$$MM_1 = \frac{0.344 \times 400}{2} = 68.8 \text{ ton.cm}$$

HOIST에 대한 풍하중 = F x H B x H H x q x 1.2 = 1.11 x 0.65 x 1 x 158 x 1.2 =

*PITCH내 MM_{H0} = 137 x 1050 / 4 = 143850 KG.CM =

*켄틀레버 MM_{H1} = 137 x 400 = 54800 KG.CM =

* COMBINED MOMENT



$$MM_2 = MM_{G1} + MM_{H0} = 75.05 + 35.9625 = 111.0125 \text{ ton.cm}$$

$$MM_4 = MM_1 + MM_{H1} = 68.8 + 54.8 = 123.60 \text{ ton.cm}$$

*** BENDING STRESS**

$$\Sigma\sigma_3 = MM_2 / Z_y = 111.0125 / 112 = 0.991 \text{ ton/cm}^2 < 1.447 \text{ ton/cm}^2 \text{---O.K}$$

$$\Sigma\sigma_4 = MM_4 / Z_y = 123.6 / 112 = 1.104 \text{ ton/cm}^2 < 1.808 \text{ ton/cm}^2 \text{---O.K}$$

PITCH내 휴지시 응력 1391 x 80% x 130% = 1447 ton/cm²
 켄틀레버는 용접부 없음 휴지시 응력 1391 x 130% = 1808 ton/cm²

11. DEFLECTION OF GIRDER

*** PITCH 내**

1) DUE TO RATED & TROLLEY LOAD

$$.D1 = \frac{(Q+Q2)x(L)^3}{48xEx Ix} = 0.9698 \text{ cm}$$

2) DUE TO DEAD LOAD

$$.D2 = \frac{5 x Wgx L^3}{384 x E x Ix} = 0.2759 \text{ cm}$$

3) TOTAL DEFLECTION

$$.s1 = D1 + D2 = 1.246 \text{ cm}$$

$$\text{RATIO : } D3/L = 1 / 843 < 800 \text{---O.K}$$

*** 켄틀레버**

1) DUE TO RATED & TROLLEY LOAD

$$.D1 = \frac{(Q+Q2)x(L1-L2)^3}{3xEx Ix} = 0.6789 \text{ cm}$$

2) DUE TO DEAD LOAD

$$.D2 = \frac{Wg1x L1^3}{8 x E x Ix} = 0.0559 \text{ cm}$$

3) TOTAL DEFLECTION

$$.s2 = D1 + D2 = 0.7348 \text{ cm}$$

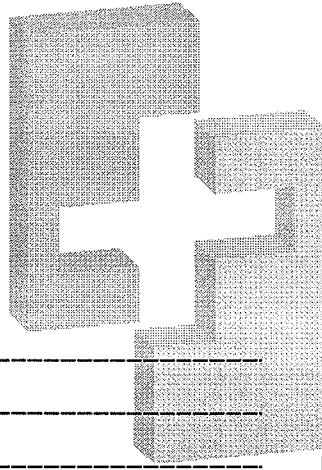
$$\text{RATIO : } D3/L = 1 / 544 < 500 \text{---O.K}$$



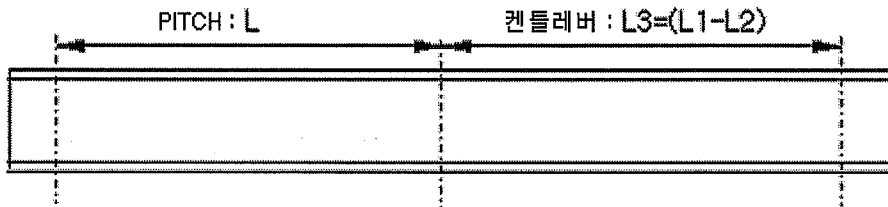
6. H-BEAM 최대허용 가능 SPAN 계산

1. SPECIFICATION

.정격하중	-----	Q =	1	ton
.SPAN (PITCH)	-----	L=	1300	cm
.켤리리버	-----	L1=	500	cm
.TROLLEY WHEEL BASE	-----	B=	45.8	cm
.WEIGHT OF HOIST	-----	Wh=	0.14	ton
.PITCH내의 GIRDER 무게	-----	Wg=	0.85	ton
.켤리리버의 GIRDER 무게	-----	Wg1=	0.327	ton
.H-BEAM SIZE	-----		400x200x8/13t	
		Ix =	23500	cm ⁴
		Iy =	1740	cm ⁴
		Zx =	1170	cm ³
		Zy =	174	cm ³
		A =	83.37	cm ²
		Wb =	65.4	kg/m
.HOOK APPROACH	-----	L2=	30	cm
탄성계수	-----	E=	2100000	kg/cm ²
동하중 계수(작업계수)	-----	M(Φ)=	1.11	
정하중 계수(충격계수)	-----	F(Ψ)=	1.10	



1. DESIGN



2. H-BEAM에 작용하는 하중

$$P = Q + Wh = 1 + 0.14 = 1.14 \text{ ton}$$

3. 수직하중에 의한 BENDING MOMENT

1) PITCH 지점내 BENDING MOMENT

$$M_{h1} = F \times M \times P \times (L-B/2)^2 / (4 \times L)$$

$$= 1.11 \times 1.1 \times 1.14 \times (1300 - 45.8/2)^2 / (4 \times 1300)$$



$$= 436.6 \text{ ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$Mh2 = F \times M \times P \times L3$$

$$= 1.11 \times 1.1 \times 1.14 \times (500-30) = 654.21 \text{ ton.cm}$$

4. 자중에 의한 BENDING MOMENT(I-BEAM)

1) PITCH 지점내 BENDING MOMENT

$$Mt1 = F \times Wg \times L / 8 = 1.11 \times 0.85 \times 1300 / 8 = 153.319 \text{ ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$Mt2 = F \times Wg1 \times L1 / 2 = 1.11 \times 0.327 \times 500 / 2 = 90.74 \text{ ton.cm}$$

5. 수직하중에 의한 최대 BENDING MOMENT

$$Mmax1 = Mh1 + Mt1 = 589.9 \text{ ton.cm}$$

$$Mmax2 = Mh2 + Mt2 = 745 \text{ ton.cm}$$

6. 풍하중에 의한 수평하중의 BENDING MOMENT

(1) GIRDER에 의한 수직 풍하중

$$\text{작업시 } V=16\text{m/s}, q = 19.9 \text{ kg/m}^2 \quad h(\text{최고양정}) = 30 \text{ m}$$

$$\text{작업시 } q = 8.5 \times \sqrt[4]{h} = 8.5 \times \sqrt[4]{42} = 19.9 \text{ kg/m}^2$$

$$H(\text{GIRDER높이}) = 0.40 \text{ m}$$

$$\text{PITCH내 풍하중} = F \times L \times H \times q \times 1.7 = 1.11 \times 13 \times 0.4 \times 19.9 \times 1.7 = 195 \text{ kg}$$

$$\text{캔틀레버 풍하중} = F \times L1 \times H \times q \times 1.4 = 1.11 \times 5 \times 0.4 \times 19.9 \times 1.4 = 62 \text{ kg}$$

$$M_{FGG} = \frac{0.195 \times 1300}{8} - \frac{0.062 \times 500}{2} = 16.188 \text{ ton.cm}$$

$$M_{FG1} = \frac{0.062 \times 500}{2} = 15.5 \text{ ton.cm}$$

(2) HOIST에 의한 풍하중

* 작업시

$$\text{풍하중} = F \times HB \times HH \times q \times 1.2 = 1.11 \times 0.65 \times 1 \times 19.9 \times 1.2 = 17 \text{ kg}$$

*PITCH내 풍하중

$$M_{FHG} = 17 \times 1300 / 4 = 5525 \text{ kg.cm} = 5.525 \text{ ton.cm}$$

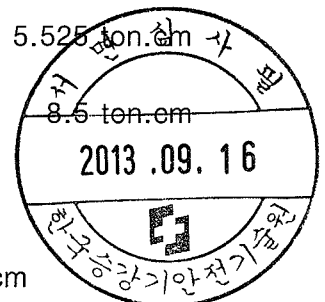
*캔틀레버 풍하중

$$M_{FH1} = 17 \times 500 = 8500 \text{ kg.cm} = 8.5 \text{ ton.cm}$$

7. COMBINED MOMENT

*PITCH내

$$M_{HCG} = M_{FGG} + M_{FHG} = 16.188 + 5.525 = 21.713 \text{ ton.cm}$$



*켄틀레버

$$M_{HC1} = M_{FG1} + M_{FH1} = 15.5 + 8.5 = 24.000 \text{ ton.cm}$$

8. BENDING STRESS

A. VERTICAL BENDING STRESS

1. PITCH 내

$$\sigma v1 = M_{max1} / Zx = 589.9 / 1170 = 0.504 \text{ ton/cm}^2$$

2. 켄틀레버

$$\sigma v2 = M_{max2} / Zx = 745 / 1170 = 0.637 \text{ ton/cm}^2$$

B. 수평 최대 응력

1. PITCH 내

$$\sigma v3 = M_{HCG} / Zy = 21.713 / 174 = 0.125 \text{ ton/cm}^2$$

2. 켄틀레버

$$\sigma v4 = M_{HC1} / Zy = 24 / 174 = 0.138 \text{ ton/cm}^2$$

9. 합성 응력

$$\Sigma \sigma 1 = \sigma v1 + \sigma v3 = 0.629$$

0.629	<	1.279	ton/cm ²	O.K
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$$\Sigma \sigma 2 = \sigma v2 + \sigma v4 = 0.775$$

0.775	<	1.6	ton/cm ²	O.K
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용접효율 : 80% 응력 1391x80% x 115% = 1.279 ton/cm²
 켄틀레버는 용접부 없음 1391*작업시 1.15(풍하중포함) = 1.600 ton/cm²

10. 휴지시 풍하중에 의한 BENDING MOMENT

휴지시 V=45m/s, q = 158.0 kg/m² h(최고양정) = 30 m
 휴지시 q = M x ⁴√h = 67.5 x ⁴√30 = 158.0 kg/m²
 M = V² / 30 = 67.5

*휴지시 풍하중에 의한 수평하중

PITCH내 풍하중 = F x L x H x q x 1.7 = 1.11 x 13 x 0.4 x 158 x 1.7 = 1550 kg

켄틀레버에 대한 풍하중 = F x L1 x H x q x 1.4 = 1.11 x 5 x 0.4 x 158 x 1.4 = 491 kg

$$MM_{G1} = \frac{1.55 \times 1300}{8} - \frac{0.491 \times 500}{2} = 129.125 \text{ ton.cm}$$

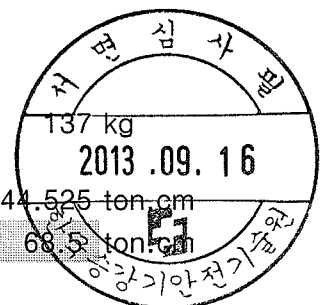
$$MM_1 = \frac{0.491 \times 500}{2} = 122.75 \text{ ton.cm}$$

HOIST에 대한 풍하중 = F x HB x HH x q x 1.2 = 1.11 x 0.65 x 1 x 158 x 1.2 =

*PITCH내 MM_{H0} = 137 x 1300 / 4 = 178100 KG.CM =

*켄틀레버 MM_{H1} = 137 x 500 = 68500 KG.CM =

* COMBINED MOMENT



$$MM_2 = MM_{G1} + MM_{H0} = 129.125 + 44.525 = 173.65 \text{ ton.cm}$$

$$MM_4 = MM_1 + MM_{H1} = 122.75 + 68.5 = 191.25 \text{ ton.cm}$$

* BENDING STRESS

$$\Sigma\sigma_3 = MM_2 / Z_y = 173.65 / 174 = 0.998 \text{ ton/cm}^2 < 1.447 \text{ ton/cm}^2 \text{---O.K}$$

$$\Sigma\sigma_4 = MM_4 / Z_y = 191.25 / 174 = 1.099 \text{ ton/cm}^2 < 1.808 \text{ ton/cm}^2 \text{---O.K}$$

PITCH내 휴지시 응력 1391 x 80% x 130% = 1447 ton/cm²

컨트롤레버는 용접부 없음 휴지시 응력 1391 x 130% = 1808 ton/cm²

11. DEFLECTION OF GIRDER

* PITCH 내

1) DUE TO RATED & TROLLEY LOAD

$$.D1 = \frac{(Q+Q2) \times (L)^3}{48 \times E \times I_x} = 1.0573 \text{ cm}$$

2) DUE TO DEAD LOAD

$$.D2 = \frac{5 \times W_g \times L^3}{384 \times E \times I_x} = 0.4927 \text{ cm}$$

3) TOTAL DEFLECTION

$$.s1 = D1 + D2 = 1.550 \text{ cm}$$

$$\text{RATIO : } D3/L = 1/839 < 800 \text{---O.K}$$

* 컨트롤레버

1) DUE TO RATED & TROLLEY LOAD

$$.D1 = \frac{(Q+Q2) \times (L1-L2)^3}{3 \times E \times I_x} = 0.7994 \text{ cm}$$

2) DUE TO DEAD LOAD

$$.D2 = \frac{W_g \times L1^3}{8 \times E \times I_x} = 0.1035 \text{ cm}$$

3) TOTAL DEFLECTION

$$.s2 = D1 + D2 = 0.9029 \text{ cm}$$

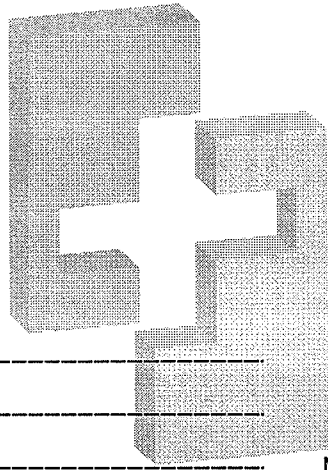
$$\text{RATIO : } D3/L = 1/554 < 500 \text{---O.K}$$



7. H-BEAM 최대허용 가능 SPAN 계산

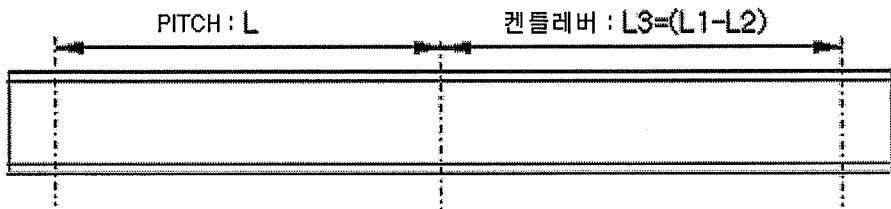
1. SPECIFICATION

.정격하중	-----	Q =	1	ton
.SPAN (PITCH)	-----	L=	1450	cm
.켄틸리버	-----	L1=	600	cm
.TROLLEY WHEEL BASE	-----	B=	45.8	cm
.WEIGHT OF HOIST	-----	Wh=	0.14	ton
.PITCH내의 GIRDER 무게	-----	Wg=	1.086	ton
.켄틸리버의 GIRDER 무게	-----	Wg1=	0.449	ton
.H-BEAM SIZE	-----		450x200x9/14t	
		Ix =	32900	cm ⁴
		Iy =	1870	cm ⁴
		Zx =	1460	cm ³
		Zy =	187	cm ³
		A =	95.43	cm ²
		Wb =	74.9	kg/m
.HOOK APPROACH	-----	L2=	30	cm
탄성계수	-----	E=	2100000	kg/cm ²
동하중 계수(작업계수)	-----	M(Φ)=	1.11	
정하중 계수(충격계수)	-----	F(Ψ)=	1.10	



한국승강기안전기술원

1. DESIGN



2. H-BEAM에 작용하는 하중

$$P = Q + Wh = 1 + 0.14 = 1.14 \text{ ton}$$

3. 수직하중에 의한 BENDING MOMENT

1) PITCH 지점내 BENDING MOMENT

$$Mh1 = F \times M \times P \times (L-B/2)^2 / (4 \times L)$$

$$= 1.11 \times 1.1 \times 1.14 \times (1450 - 45.8/2)^2 / (4 \times 1450)$$



$$= 488.8 \quad \text{ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$Mh2 = F \times M \times P \times L3$$

$$= 1.11 \times 1.1 \times 1.14 \times (600 - 30) = 793.41 \quad \text{ton.cm}$$

4. 자중에 의한 BENDING MOMENT(I-BEAM)

1) PITCH 지점내 BENDING MOMENT

$$Mt1 = F \times Wg \times L / 8 = 1.11 \times 1.086 \times 1450 / 8 = 218.49 \quad \text{ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$Mt2 = F \times Wg1 \times L1 / 2 = 1.11 \times 0.449 \times 600 / 2 = 149.52 \quad \text{ton.cm}$$

5. 수직하중에 의한 최대 BENDING MOMENT

$$Mmax1 = Mh1 + Mt1 = 707.3 \quad \text{ton.cm}$$

$$Mmax2 = Mh2 + Mt2 = 942.9 \quad \text{ton.cm}$$

6. 풍하중에 의한 수평하중의 BENDING MOMENT

(1) GIRDER에 의한 수직 풍하중

$$\text{작업시 } V=16\text{m/s}, q = 19.9 \text{ kg/m}^2 \quad h(\text{최고양정}) = 30 \text{ m}$$

$$\text{작업시 } q = 8.5 \times \sqrt{h} = 8.5 \times \sqrt{42} = 19.9 \text{ kg/m}^2$$

$$H(\text{GIRDER높이}) = 0.45 \text{ m}$$

$$\text{PITCH내 풍하중} = F \times L \times H \times q \times 1.7 = 1.11 \times 14.5 \times 0.45 \times 19.9 \times 1.7 = 245 \text{ kg}$$

$$\text{캔틀레버 풍하중} = F \times L1 \times H \times q \times 1.4 = 1.11 \times 6 \times 0.45 \times 19.9 \times 1.4 = 83 \text{ kg}$$

$$M_{FGG} = \frac{0.245 \times 1450}{8} + \frac{0.083 \times 600}{2} = 19.506 \quad \text{ton.cm}$$

$$M_{FG1} = \frac{0.083 \times 600}{2} = 24.9 \quad \text{ton.cm}$$

(2) HOIST에 의한 풍하중

* 작업시

$$\text{풍하중} = F \times HB \times HH \times q \times 1.2 = 1.11 \times 0.65 \times 1 \times 19.9 \times 1.2 = 17 \text{ kg}$$

*PITCH내 풍하중

$$M_{FHG} = 17 \times 1450 / 4 = 6162.5 \text{ kg.cm} = 6.1625 \text{ ton.cm}$$

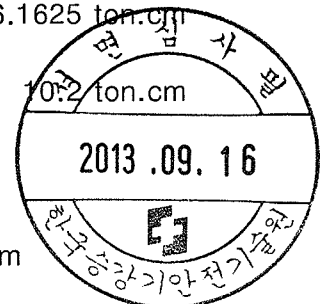
*캔틀레버 풍하중

$$M_{FH1} = 17 \times 600 = 10200 \text{ kg.cm} = 10.2 \text{ ton.cm}$$

7. COMBINED MOMENT

*PITCH내

$$M_{HCG} = M_{FGG} + M_{FHG} = 19.506 + 6.1625 = 25.669 \text{ ton.cm}$$



*컨트레버

$$M_{HC1} = M_{FG1} + M_{FH1} = 24.9 + 10.2 = 35.100 \text{ ton.cm}$$

8. BENDING STRESS

A. VERTICAL BENDING STRESS

1. PITCH 내

$$\sigma_1 = M_{max1} / Z_x = 707.3 / 1460 = 0.484 \text{ ton/cm}^2$$

2. 컨트레버

$$\sigma_2 = M_{max2} / Z_x = 942.9 / 1460 = 0.646 \text{ ton/cm}^2$$

B. 수평 최대 응력

1. PITCH 내

$$\sigma_3 = M_{HCG} / Z_y = 25.6685 / 187 = 0.137 \text{ ton/cm}^2$$

2. 컨트레버

$$\sigma_4 = M_{HC1} / Z_y = 35.1 / 187 = 0.188 \text{ ton/cm}^2$$

9. 합성 응력

$$\Sigma \sigma_1 = \sigma_1 + \sigma_3 = 0.622$$

0.622	<	1.279	ton/cm ²	O.K
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$$\Sigma \sigma_2 = \sigma_2 + \sigma_4 = 0.834$$

0.834	<	1.6	ton/cm ²	O.K
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용접효율 : 80% 응력 1391 x 80% x 115% = 1.279 ton/cm²
 컨트레버는 용접부 없음 1391 * 작업시 1.15 (풍하중포함) = 1.600 ton/cm²

10. 휴지시 풍하중에 의한 BENDING MOMENT

휴지시 V=45m/s, q = 158.0 kg/m² h(최고양정) = 30 m
 휴지시 q = M x ⁴√h = 67.5 x ⁴√30 = 158.0 kg/m²
 M = V² / 30 = 67.5

*휴지시 풍하중에 의한 수평하중

PITCH내 풍하중 = F x L x H x q x 1.7 = 1.11 x 14.5 x 0.45 x 158 x 1.7 = 1945 kg

컨트레버에 대한 풍하중 = F x L₁ x H x q x 1.4 = 1.11 x 6 x 0.45 x 158 x 1.4 = 663 kg

$$MM_{G1} = \frac{1.945 \times 1450}{8} - \frac{0.663 \times 600}{2} = 153.63125 \text{ ton.cm}$$

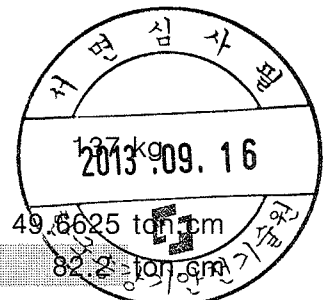
$$MM_1 = \frac{0.663 \times 600}{2} = 198.9 \text{ ton.cm}$$

HOIST에 대한 풍하중 = F x H_B x H x q x 1.2 = 1.11 x 0.65 x 1 x 158 x 1.2 =

*PITCH내 MM_{H0} = 137 x 1450 / 4 = 198650 KG.CM =

*컨트레버 MM_{H1} = 137 x 600 = 82200 KG.CM =

* COMBINED MOMENT



$$MM_2 = MM_{G1} + MM_{H0} = 153.63125 + 49.6625 = 203.29375 \text{ ton.cm}$$

$$MM_4 = MM_1 + MM_{H1} = 198.9 + 82.2 = 281.10 \text{ ton.cm}$$

*** BENDING STRESS**

$$\Sigma\sigma_3 = MM_2 / Z_y = 203.29375 / 187 = 1.087 \text{ ton/cm}^2 < 1.447 \text{ ton/cm}^2 \text{---O.K}$$

$$\Sigma\sigma_4 = MM_4 / Z_y = 281.1 / 187 = 1.503 \text{ ton/cm}^2 < 1.808 \text{ ton/cm}^2 \text{---O.K}$$

PITCH내 휴지시 응력 1391 x 80% x 130% =

1447 ton/cm²

컨트롤레버는 용접부 없음 휴지시 응력 1391 x 130% =

1808 ton/cm²

11. DEFLECTION OF GIRDER

*** PITCH 내**

1) DUE TO RATED & TROLLEY LOAD

$$.D1 = \frac{(Q+Q2) \times (L)^3}{48 \times E \times I_x} = 1.0480 \text{ cm}$$

2) DUE TO DEAD LOAD

$$.D2 = \frac{5 \times W_g \times L^3}{384 \times E \times I_x} = 0.6240 \text{ cm}$$

3) TOTAL DEFLECTION

$$.s1 = D1 + D2 = 1.672 \text{ cm}$$

$$\text{RATIO : } D3/L = 1/867 < 800 \text{---O.K}$$

*** 컨트롤레버**

1) DUE TO RATED & TROLLEY LOAD

$$.D1 = \frac{(Q+Q2) \times (L1-L2)^3}{3 \times E \times I_x} = 1.0186 \text{ cm}$$

2) DUE TO DEAD LOAD

$$.D2 = \frac{W_g \times L1^3}{8 \times E \times I_x} = 0.1755 \text{ cm}$$

3) TOTAL DEFLECTION

$$.s2 = D1 + D2 = 1.1941 \text{ cm}$$

$$\text{RATIO : } D3/L = 1/502 < 500 \text{---O.K}$$



SUSPENSION MONORAIL 볼트 및 용접강도계산

1. 볼트로 고정시의 강도계산

* 허용 최대 SPAN 및 하중은 최악의 조건으로 계산한다.
(허용 최대 스팬 14.5 M, , HOIST자중 : 140 KG)

.Q = RATED LOAD= 1000 KG
.Q1=HOIST SELF WEIGHT= 140 KG
.Q2=I-BEAM WEIGHT = 1086 KG

BOLT 재질 :	H.T.B 9.8 이상 사용할것
BOLT 허용전단응력 :	2100 KG/CM ²

M12 일때 do= 1.0106 CM n= 4.6 . n = 유효산수(산수*80%너트1종너트사용)
M14 일때 do= 1.1835 CM n= 4.4
M16 일때 do= 1.3835 CM n= 5.2
M18 일때 do= 1.5294 CM n= 4.8

$$P = 1.14*(Q+Q1) + 1.1* Q2/2$$

$$= 1.14*(1000+140) + 1.1* 1086/2$$

$$= 1897 \quad \text{KG} \quad (\sigma_a = 2100 \text{ KG/CM}^2)$$

1)인장(전단)강도 : $\sigma = P/A ; \sigma = \frac{4 \times P}{\pi \times do^2 \times Z} \times Z = \frac{4 \times P}{\pi \times do^2 \times \sigma}$

M12 일때 $Z = \frac{4 \times P}{\pi \times do^2 \times \sigma} = \frac{4 \times 1897}{\pi \times 1.0106^2 \times 2100} = 1.13 \text{ 개}$

M14 일때 $Z = \frac{4 \times P}{\pi \times do^2 \times \sigma} = \frac{4 \times 1897}{\pi \times 1.1835^2 \times 2100} = 0.82 \text{ 개}$

M16 일때 $Z = \frac{4 \times P}{\pi \times do^2 \times \sigma} = \frac{4 \times 1897}{\pi \times 1.3835^2 \times 2100} = 0.6 \text{ 개}$

M18 일때 $Z = \frac{4 \times P}{\pi \times do^2 \times \sigma} = \frac{4 \times 1897}{\pi \times 1.5294^2 \times 2100} = 0.49 \text{ 개}$

2)접촉 면압 강도 ($\sigma_a = 400 \text{ KG/CM}^2$)

$$\sigma = \frac{4 \times P}{\pi \times (d^2 - do^2) \times Z \times n} \quad Z = \frac{4 \times P}{\pi \times (d^2 - do^2) \times \sigma \times n}$$

M12 일때 $Z = \frac{4 \times 1897}{\pi \times (1.2^2 - 1.0106^2) \times 400 \times 4.57} = 3.16 \text{ 개}$

M14 일때 $Z = \frac{4 \times 1897}{\pi \times (1.4^2 - 1.1835^2) \times 400 \times 4.4} = 2.45 \text{ 개}$

M16 일때 $Z = \frac{4 \times 1897}{\pi \times (1.6^2 - 1.3835^2) \times 400 \times 5.2} = 1.8 \text{ 개}$

M18 일때 $Z = \frac{4 \times 1897}{\pi \times (1.8^2 - 1.5294^2) \times 400 \times 4.8} = 1.4 \text{ 개}$



- . do = 골경, P=브라켓 한 개에 작용하는 하중, Z = 볼트수
- . n = 유효산수(산수*80%), 너트1종너트사용

3)따라서 다음과 같이 적용한다

H.T.B M12,M14,M16,M18일때 : 브라켓트당 4개 이상 사용한다.

2. I-빔을 용접시공으로 고정시의 강도계산

* 허용 최대 SPAN 및 하중은 최악의 조건으로 계산한다.
(허용 최대 스팬 14.5 M, , HOIST자중 : 140 KG)

- .Q = RATED LOAD= 1000 KG (h : 용접두께
- .Q1=HOIST SELF WEIGHT= 140 KG (L : 용접길이)
- .Q2=I-BEAM WEIGHT = 1086 KG

1) 용접이음부 인장강도 계산 (용접두께 : 45° 용접부위)

$$\sigma = \frac{1.414 \times P}{h \times L} ; (\sigma_a = 1200 \text{ KG/CM}^2) \quad L = \frac{1.414 \times P}{h \times \sigma}$$

(1) h 가 5일때

$$L = \frac{1.414 \times 1897}{0.5 \times 1200} = 4.47 \text{ CM} = 44.7 \text{ mm}$$

(2) h 가 6일때

$$L = \frac{1.414 \times 1897}{0.6 \times 1200} = 3.73 \text{ CM} = 37.3 \text{ mm}$$

(3) h 가 7일때

$$L = \frac{1.414 \times 1897}{0.7 \times 1200} = 3.19 \text{ CM} = 31.9 \text{ mm}$$

2) 적용

h = 5일때, 한 브라켓트당 용접길이 L = 60mm 이상 용접한다.

h = 6일때, 한 브라켓트당 용접길이 L = 50mm 이상 용접한다.

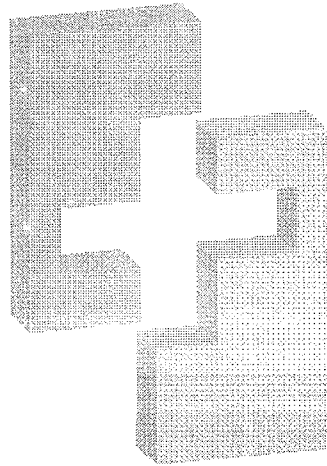
h = 7일때, 한 브라켓트당 용접길이 L = 40mm 이상 용접한다.

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6. FOR REFERENCE

- 1) LOAD CHAIN 시험성적서
- 2) MOTOR DATA SHEET
- 3) HOIST 사용설명서(operation manual)



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Date: 2009/04/14

Certificate of Compliance

We certify that the ER2 protection degrees conform to the IP rating as follows:

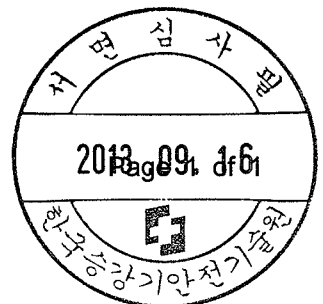
Hoist body - IP55 based on JIS C 4034-5, "Rotating electrical machines – Part5: Classification of degrees of protection provided by enclosures of rotating electrical machines (IP code)".

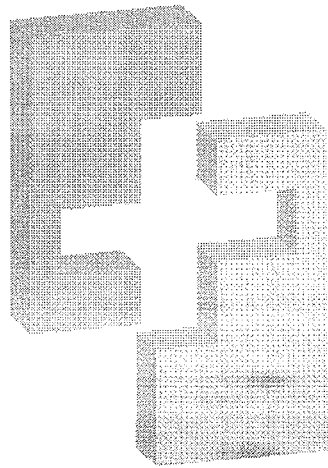
Push button - IP65 based on JIS C 0920, "Tests to prove protection against ingress of water and degrees of protection against ingress of solid objects for electrical equipment".

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Technical Control Group





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Motor Test Report for Electric Chain Hoist

Motor type : Three phase squirrel cage type induction motor.

Manufacturer : Yasukawa Electric Mfg. Co.

Production No. : -

Rating

Model	Output	Pole	Intermittent Rating	Voltage	Frequency
IBQ	0.9kW	4P	60%ED	220V	60Hz

Full load characteristics

Voltage	Frequency	220V	60Hz
Load	%	100	
Current	A	4.7	
Speed	rpm	1660	

Insulation class **E** Korea Elevator Safety Technology Institute

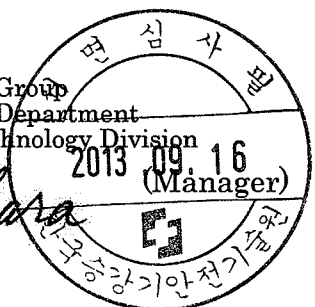
The above characteristics are obtained from calculation where the motor is assembled with an electric chain hoist and the hoist is subjected to full load



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Quality Assurance Department
Development & Technology Division

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Messrs. _____

Motor Test Report for Electric Chain Hoist

Motor type : Three phase squirrel cage type induction motor.

Manufacturer : Yasukawa Electric Mfg. Co.

Production No. : -

Rating

Model	Output	Pole	Intermittent Rating	Voltage	Frequency
IBQ	0.9kW	4P	40/20%ED	220V	Speed Control by Inverter

Full load characteristics

Voltage	Frequency	220V	Speed Control by Inverter
Load	%	100	
Current	A	5.7	
Speed	rpm	~	

Insulation class E

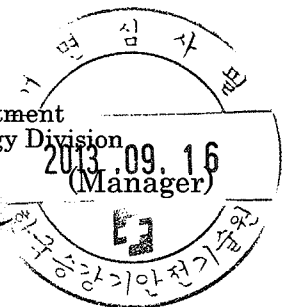
The above characteristics are obtained from calculation where the motor is assembled with an electric chain hoist and the hoist is subjected to full load



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Certificate No.: MM070011c

Date of Issue: 2008/03/21

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Motor Test Report for Electric Chain Hoist

Motor type : Three phase squirrel cage type induction motor.

Manufacturer : Yasukawa Electric Mfg. Co.

Production No. :

Rating

Model	Output	Pole	Intermittent Rating	Voltage	Frequency
IBQ	0.9kW	4P	60%ED	380 - 440V	60Hz

Full load characteristics

Voltage Frequency	380 - 440V 60Hz	
Load	%	100
Current	A	2.6
Speed	rpm	1640

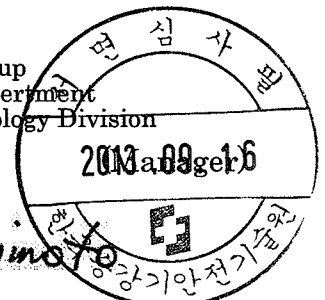
Insulation class B

The above characteristics are obtained from calculation where the motor is assembled with an electric chain hoist and the hoist is subjected to full load

 **KITO** CORP.

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R. Kishimoto

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Motor Test Report for Electric Chain Hoist

Motor type : Three phase squirrel cage type induction motor.

Manufacturer : Yasukawa Electric Mfg. Co.

Production No. :

Rating

Model	Output	Pole	Intermittent Rating	Voltage	Frequency
IBQ	0.9kW	4P	60%ED	380 - 440V	Speed Control by Inverter

Full load characteristics

Voltage Frequency	380 - 440V	Speed Control by Inverter
Load %	100	
Current A	3.6	
Speed rpm	~	

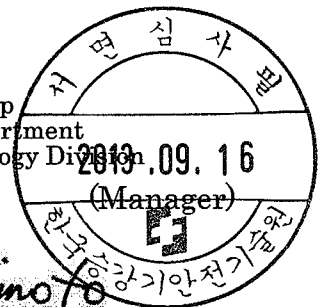
Insulation class B

The above characteristics are obtained from calculation where the motor is assembled with an electric chain hoist and the hoist is subjected to full load



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Motor Test Report for Electric Trolley

Motor type : Three phase squirrel cage type induction motor.

Manufacturer : Yasukawa Electric Mfg. Co.

Production No. : -

Rating

Model	Output	Pole	Intermittent Rating	Voltage	Frequency
IBQ-T	0.4kW	4P	40%ED	220V	60Hz

Full load characteristics

Voltage Frequency	220V 60Hz	
Load %	100	
Current A	3.0	
Speed rpm	1685	

Insulation class E

The above characteristics are obtained from calculation where the motor is assembled with an electric trolley and the trolley is subjected to full load



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Motor Test Report for Electric Trolley

Motor type : Three phase squirrel cage type induction motor.

Manufacturer : Yasukawa Electric Mfg. Co.

Production No. : -

Rating

Model	Output	Pole	Intermittent Rating	Voltage	Frequency
IBQ-T	0.4kW	4P	27/13%ED	220V	Speed Control by Inverter

Full load characteristics

Voltage	Frequency	220V	Speed Control by Inverter
Load	%	100	
Current	A	3.0	
Speed	rpm	~	

Insulation class E

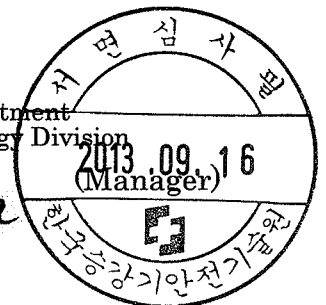
The above characteristics are obtained from calculation where the motor is assembled with an electric trolley and the trolley is subjected to full load



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Motor Test Report for End Carriage

Motor type : Three phase squirrel cage type induction motor.

Manufacturer : Yasukawa Electric Mfg. Co.

Production No. :

Rating

Model	Output	Pole	Intermittent Rating	Voltage	Frequency
IBQ-T	0.4kW	4P	40%ED	380 - 440V	60Hz

Full load characteristics

Voltage	Frequency	380 - 440V	60Hz
Load	%	100	
Current	A	2.2	
Speed	rpm	1670	

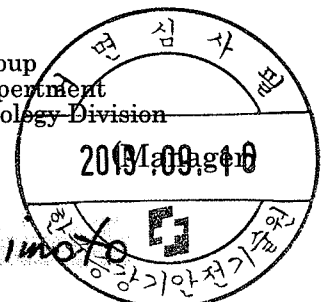
Insulation class B

The above characteristics are obtained from calculation where the motor is assembled with an electric chain hoist and the hoist is subjected to full load



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Motor Test Report for End Carriage

Motor type : Three phase squirrel cage type induction motor.

Manufacturer : Yasukawa Electric Mfg. Co.

Production No. :

Rating

Model	Output	Pole	Intermittent Rating	Voltage	Frequency
IBQ-T	0.4kW	4P	40%ED	380 - 440V	Speed Control by Inverter

Full load characteristics

Voltage	Frequency	220 - 230V	Speed Control by Inverter
Load	%	100	
Current	A	2.5	
Speed	rpm	~	

Insulation class B

The above characteristics are obtained from calculation where the motor is assembled with an electric chain hoist and the hoist is subjected to full load



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